

Probabilistic Aspects of Fatigue

Introduction



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering

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Contact Information

Darrell Socie
Mechanical Engineering
1206 West Green
Urbana, Illinois 61801

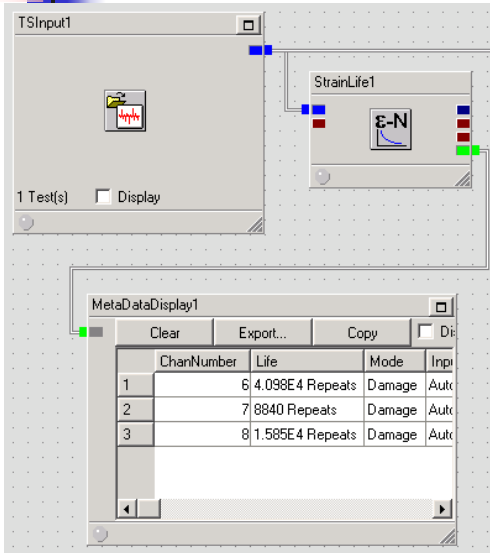
Office: 3015 Mechanical Engineering Laboratory

dsocie@uiuc.edu

Tel: 217 333 7630

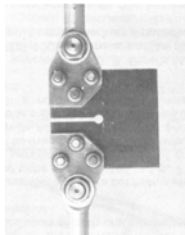
Fax: 217 333 5634

Fatigue Calculations

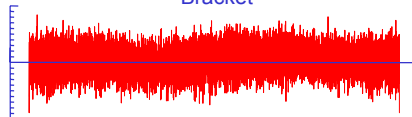


Who really believes these numbers ?

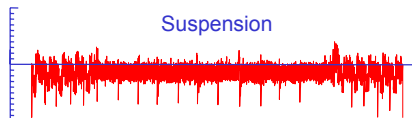
SAE Specimen



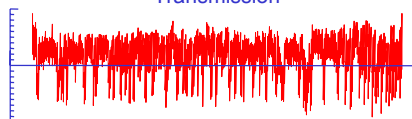
Bracket



Suspension



Transmission

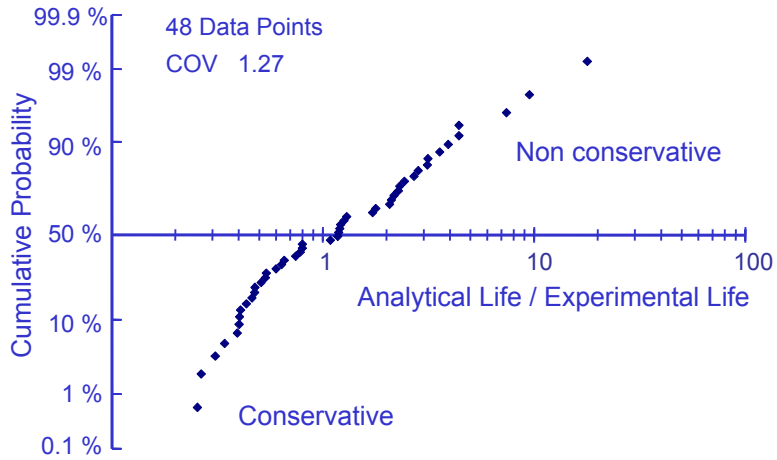


Fatigue Under Complex Loading: Analysis and Experiments, SAE AE6, 1977

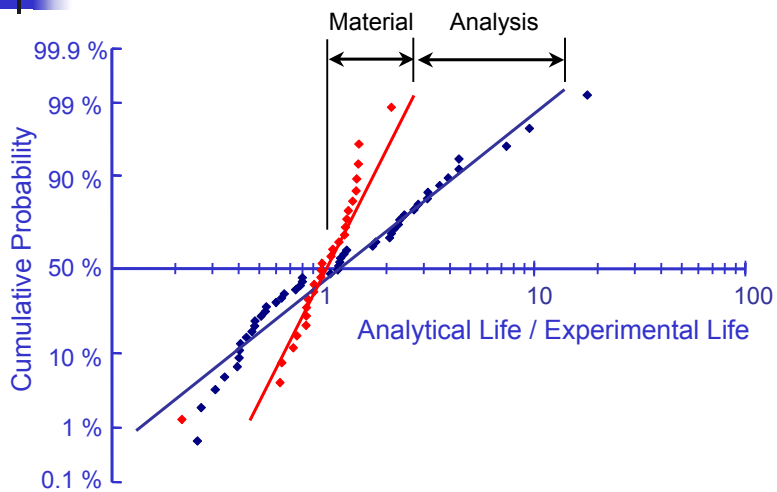


Analysis Results

Strain-Life analysis of all test data



Material Variability



Strain-Life back calculation of specimen lives



Probabilistic Models

- Probabilistic models are no better than the underlying deterministic models
- They require more work to implement
- Why use them?



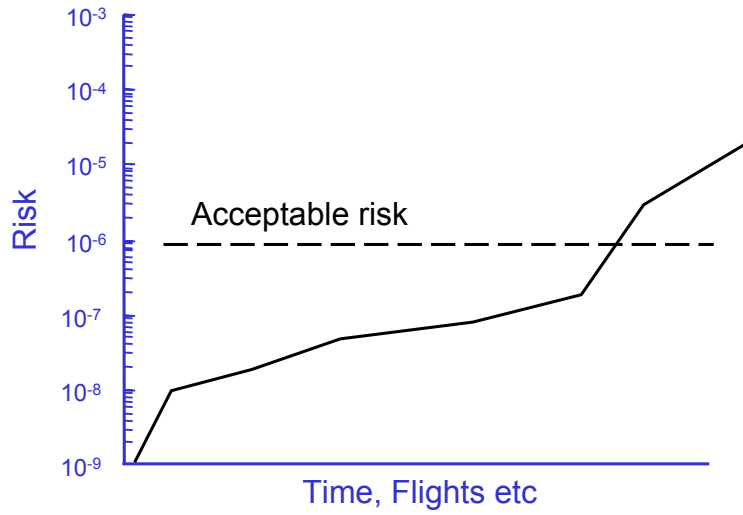
Quality and Cost

- Taguchi
 - Identify factors that influence performance
 - Robust design – reduce sensitivity to noise
 - Assess economic impact of variation

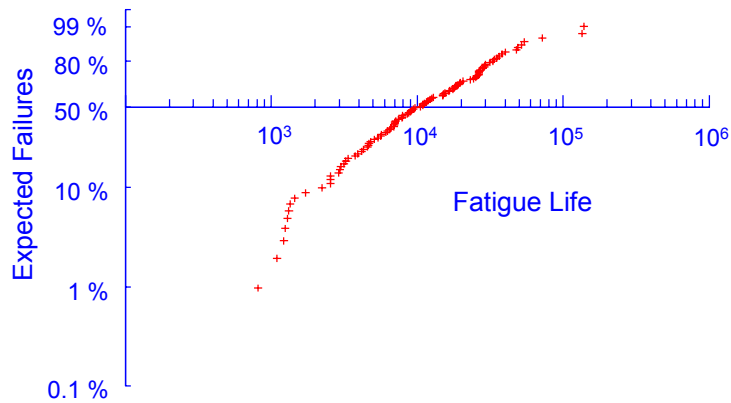
- Risk / Reliability
 - What is the increased risk from reduced testing ?



Risk

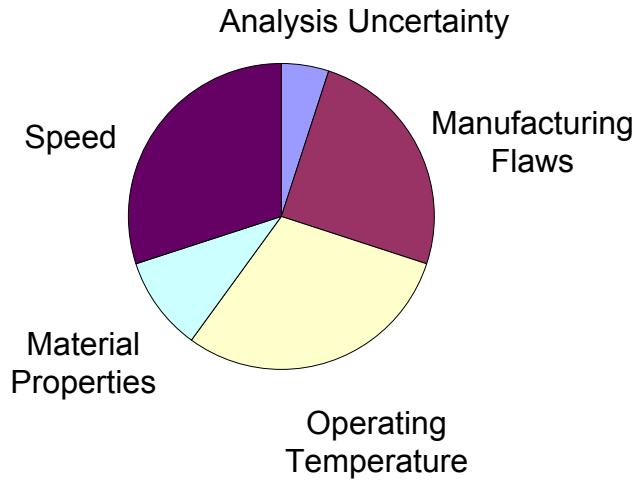


Reliability

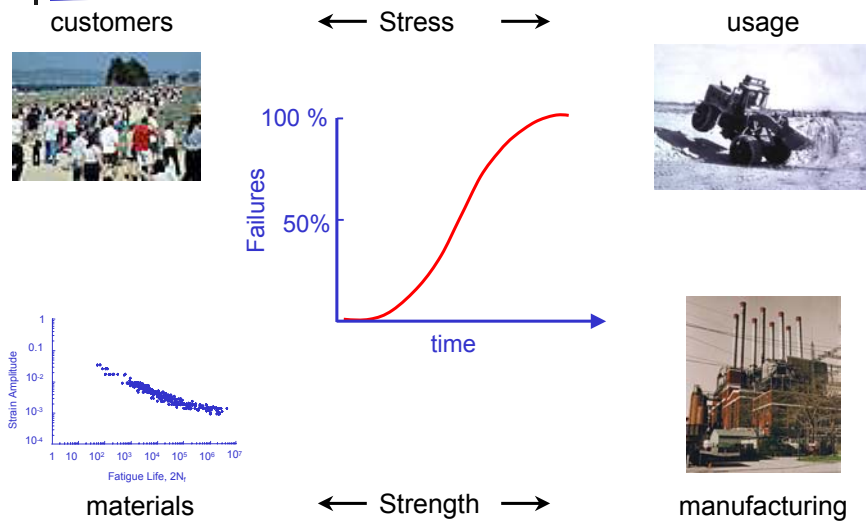




Risk Contribution Factors



Uncertainty and Variability





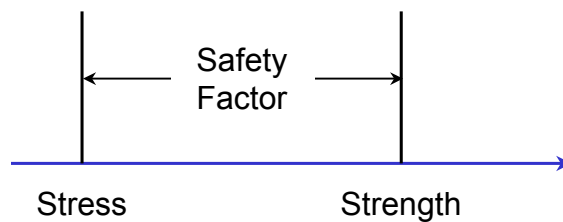
Deterministic versus Random

Deterministic – from past measurements the future position of a satellite can be predicted with reasonable accuracy

Random – from past measurements the future position of a car can only be described in terms of probability and statistical averages



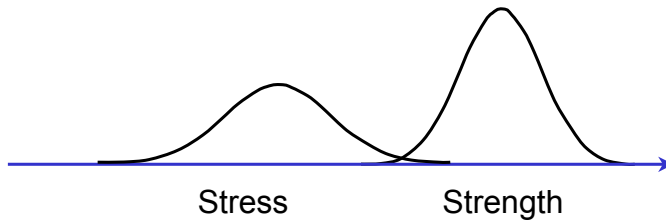
Deterministic Design



Variability and uncertainty is accommodated by introducing safety factors. Larger safety factors are better, but how much better and at what cost?



Probabilistic Design



$$\text{Reliability} = 1 - P(\text{Stress} > \text{Strength})$$



3 σ Approach

3 σ contains 99.87% of the data

$$P(s < S) = 2.3 \cdot 10^{-3}$$

If we use 3 σ on both stress and strength

$$P(\text{failure}) = P(\Sigma \geq s \cap s \leq S) = 5.3 \cdot 10^{-6} \approx 4.5 \sigma$$

The probability of the part with the lowest strength having the highest stress is very small

For 3 variables, each at 3 σ :

$$P(\text{failure}) = 1.2 \cdot 10^{-8} \approx 5.7 \sigma$$



Benefits

- Reduces conservatism (cost) compared to assuming the “worst case” for every design variable
- Quantifies life drivers – what are the most important variables and how well are they known or controlled ?
- Quantifies risk



Probabilistic Aspects of Fatigue

- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- Analysis Methods
- Characterizing Variability
- Case Studies
- FatigueCalculator.com
- [GlyphWorks](http://GlyphWorks.com)

Probabilistic Aspects of Fatigue

Basic Probability and Statistics



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Deterministic versus Random

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Random Variables

- Discrete - fixed number of outcomes
 - Colors
- Continuous - may have any value in the sample space
 - Strength



Descriptive Statistics

- Mean or Expected Value
- Variance / Standard Deviation
- Coefficient of Variation
- Skewness
- Kurtosis
- Correlation Coefficient



Mean or Expected Value

Central tendency of the data

$$\text{Mean} = \mu_x = \bar{x} = E(X) = \frac{\sum_{i=1}^N x_i}{N}$$



Variance / Standard Deviation

Dispersion of the data

$$\text{Var}(X) = \frac{\sum_{i=1}^N (x_i - \bar{x})^2}{N}$$

Standard deviation

$$\sigma_x = \sqrt{\text{Var}(X)}$$



Coefficient of Variation

$$\text{COV} = \frac{\sigma_x}{\mu_x}$$

Useful to compare different dispersions

$\mu = 10$	$\mu = 100$
$\sigma = 1$	$\sigma = 10$
$\text{COV} = 0.1$	$\text{COV} = 0.1$



Skewness

Skewness is a measure of the asymmetry of the data around the sample mean. If skewness is negative, the data are spread out more to the left of the mean than to the right. If skewness is positive, the data are spread out more to the right. The skewness of the normal distribution (or any perfectly symmetric distribution) is zero.

$$\text{Skewness}(X) = \frac{\sum_{i=1}^N (x_i - \bar{x})^3}{N\sigma^3}$$



Kurtosis

Kurtosis is a measure of how outlier-prone a distribution is. The kurtosis of the normal distribution is 3. Distributions that are more outlier-prone than the normal distribution have kurtosis greater than 3; distributions that are less outlier-prone have kurtosis less than 3.

$$\text{Kurtosis}(X) = \frac{\sum_{i=1}^N (x_i - \bar{x})^4}{N\sigma^4}$$



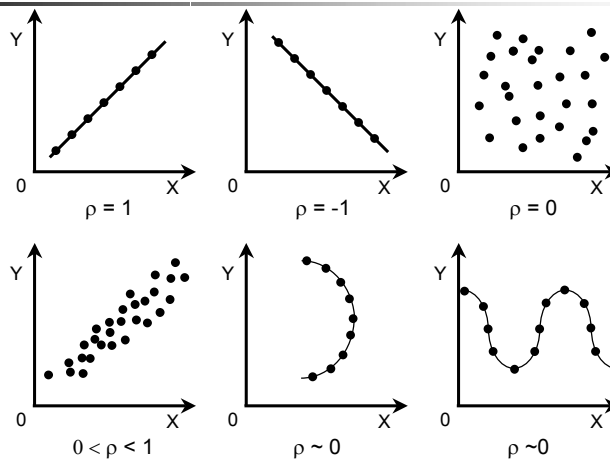
Covariance

A measure of the linear association between random variables

$$\sigma_{xy} = \text{COV}(X, Y) = E[(X - \mu_X)(Y - \mu_Y)]$$



Correlation Coefficient



$$\rho = \frac{E[(X - \mu_X)(Y - \mu_Y)]}{\sigma_X \sigma_Y}$$



Probability

- Basic probability
- Conditional probability
- Reliability



Basic Probability

The probability of event A occurring:

$$0 \leq P(A) \leq 1$$

$$P(A) = 1 \text{ certain}$$

$$P(A) = 0 \text{ impossible}$$



Conditional Basic Probability

$$P(A \cup B) = P(A) + P(B) - P(A \cap B)$$

A or B

union

A and B

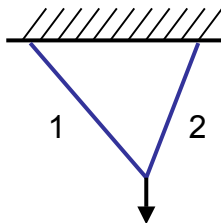
intersection

$$P(B \text{ given } A \text{ has occurred}) \quad P(B|A) = \frac{P(A \cap B)}{P(A)}$$

$$P(A|B) = \frac{P(A \cap B)}{P(B)}$$



Independent Events



If A and B are unrelated

$$P(B|A) = P(B)$$

$$P(A|B) = P(A)$$

$$P(A \cap B) = P(A) \cdot P(B)$$

Suppose the probability of bar 1 failing is 0.03
and the probability of bar 2 failing is 0.04.

What is the probability of the structure failing?



Failure Probability

Bar 1 or bar 2 fails

$$P(A \cup B) = P(A) + P(B) - P(A \cap B)$$

$$P(A) = 0.03$$

$$P(B) = 0.04$$

$$P(A \cap B) = P(A) \cdot P(B) = 0.0012$$

$$P(\text{failure}) = 0.03 + 0.04 - 0.0012 = 0.0688$$



Reliability

$P(\text{no failure}) = \text{Reliability}$

Define $P(\bar{A})$ probability of not A

$$P(\bar{A}) = P(1 - A)$$

$$\text{Reliability} = P(\bar{A} \cap \bar{B})$$

$$P(\bar{A} \cap \bar{B}) = P(\bar{A}) \cdot P(\bar{B})$$

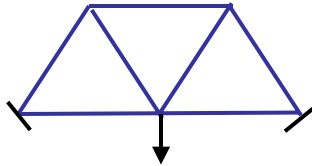
For the 2 bar structure

$$P(\bar{A} \cap \bar{B}) = 0.97 \cdot 0.96 = 0.9312$$

$$P(\text{failure}) = 1 - \text{Reliability} = 0.0688$$



Structural Reliability



Collapse occurs if any member fails

$$P(\bar{A}) = P(\bar{A}_1 \cap \bar{A}_2 \cap \bar{A}_3 \cap \bar{A}_4 \cap \dots \cap \bar{A}_n)$$

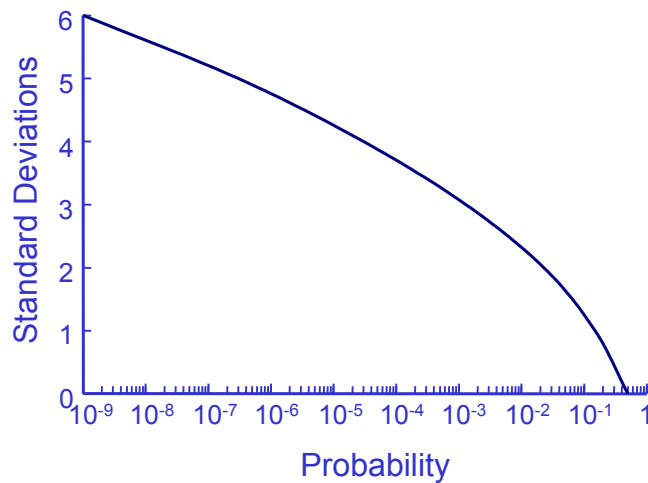
$$P(\bar{A}) = P(\bar{A}_1) \cdot P(\bar{A}_2) \cdot P(\bar{A}_3) \cdot P(\bar{A}_4) \cdot \dots \cdot P(\bar{A}_n)$$

Suppose $P(A) = 0.01$ for each link

$$P(\bar{A}) = (0.99)^7 = 0.932$$



Reliability





6 Sigma

6 sigma is 1 in a billion (0.999999999 Reliability)

Suppose a structure has 1000 bolted joints:

$$P(\bar{A}) = (0.999999999)^{1000} = 0.999999$$

1 in a million

3 sigma is (0.99865 Reliability)

$$P(\bar{A}) = (0.99865)^{1000} = 0.26$$

74 % failures



Statistical Distributions

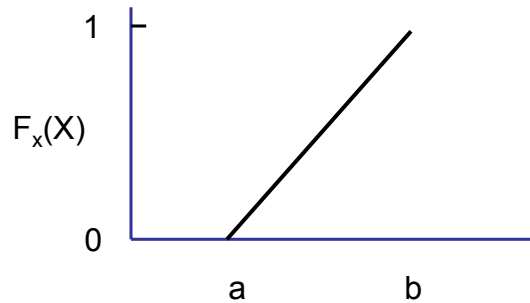
- Uniform
- Normal
- LogNormal
- Gumble
- Weibull

A useful on-line reference:

<http://www.itl.nist.gov/div898/handbook/>



Cumulative Distribution Function



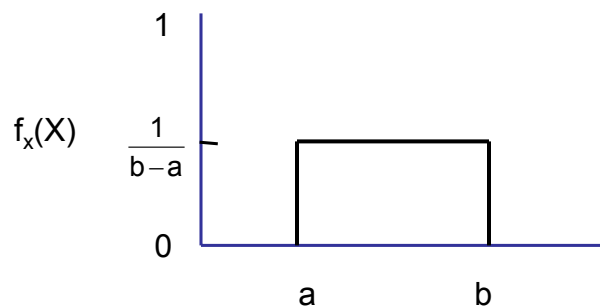
$$F_x(X) = P(X \leq x)$$

$$F_x(-\infty) = 0$$

$$F_x(\infty) = 1$$



Probability Density Function

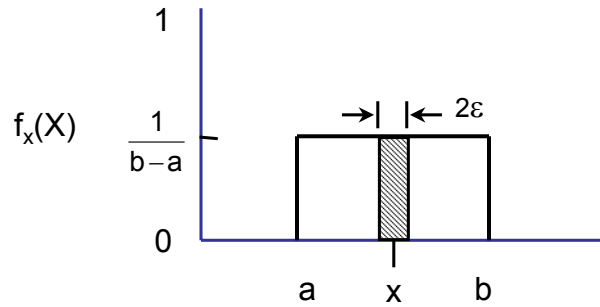


$$P(a < X \leq b) = \int_a^b f_x(X) dx$$

$$F_x(X) = \int_{-\infty}^X f_x(\xi) d\xi$$



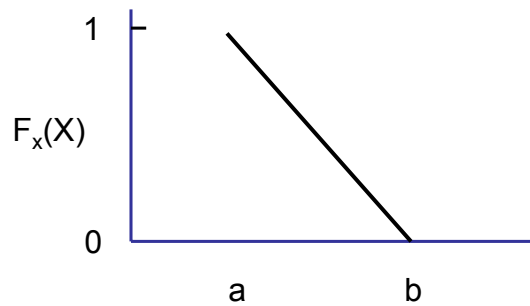
Probability



$$P(x - \varepsilon < X \leq x + \varepsilon) = \int_{x - \varepsilon}^{x + \varepsilon} f_x(X) dx$$



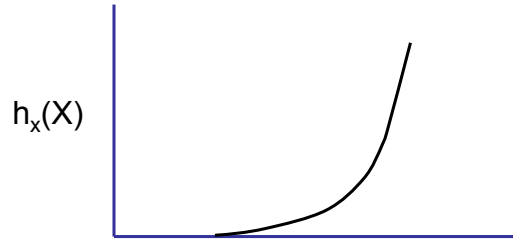
Survival Function



$$S_x(X) = 1 - F_x(X)$$



Hazard Function

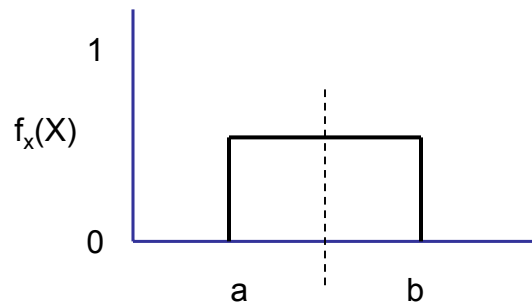


$$h_x(X) = \frac{f_x(X)}{1 - F_x(X)}$$

Instantaneous failure rate



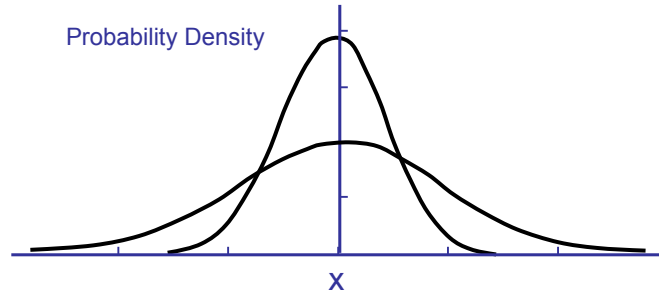
Mean or Expected Value



$$\mu_x = E(X) = \int_{-\infty}^{\infty} x f_x(X) dx$$



Variance



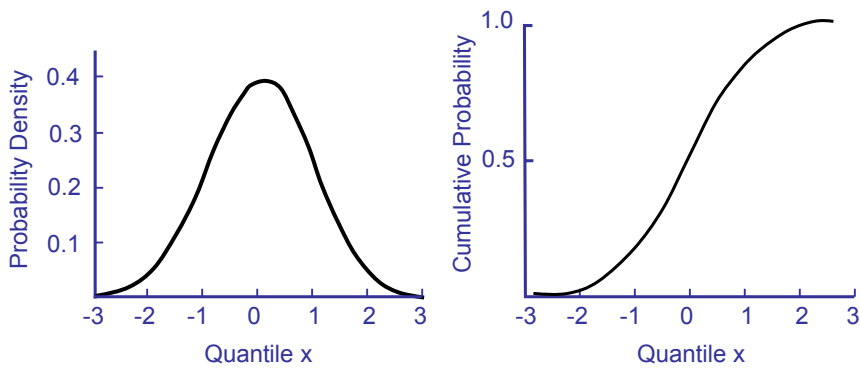
$$\text{Var}(X) = \int_{-\infty}^{\infty} (X - \mu_x)^2 f_x(X) dx$$

$$\text{Var}(X) = E(X^2) - \mu_x^2$$

$$\sigma_x = \sqrt{\text{Var}(X)}$$



Normal Distribution

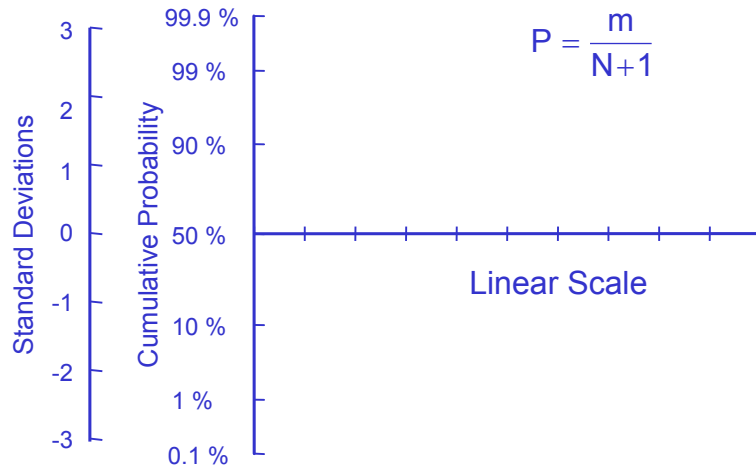


$$f_x(x) = \frac{1}{\sigma\sqrt{2\pi}} \exp\left[\frac{-(x-\mu)^2}{2\sigma^2}\right]$$

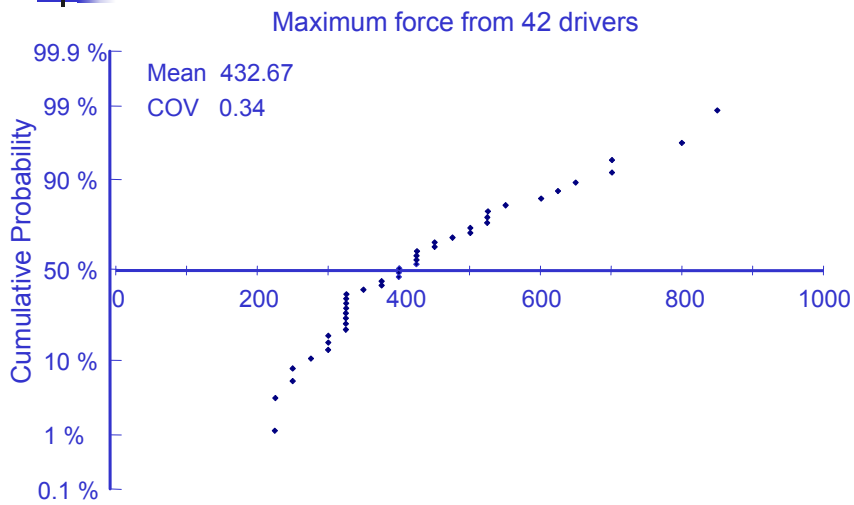
tables



Normal Probability Plot

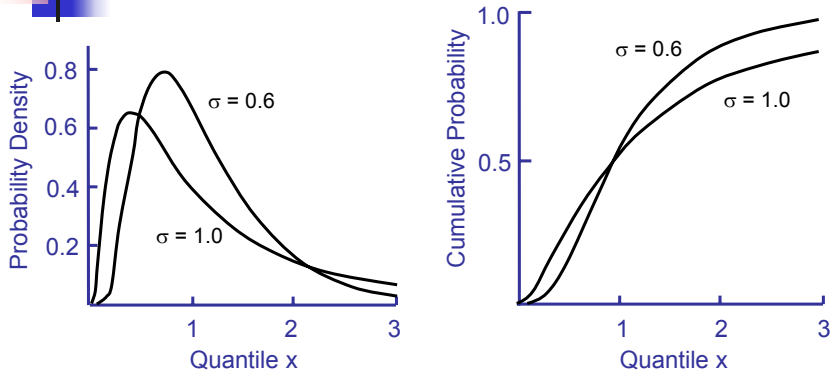


Normal Plot





LogNormal Distribution



$$m = \exp(\mu)$$

$$f_x(x) = \frac{1}{x\sigma\sqrt{2\pi}} \exp\left[-\frac{\left(\ln\left(\frac{x}{m}\right)\right)^2}{2\sigma^2}\right]$$

tables



Useful Relationships

$$\mu_{\ln x} = \ln\left(\frac{\mu_x}{\sqrt{1 + \text{COV}_x^2}}\right)$$

$$\sigma_{\ln x}^2 = \ln(1 + \text{COV}_x^2)$$

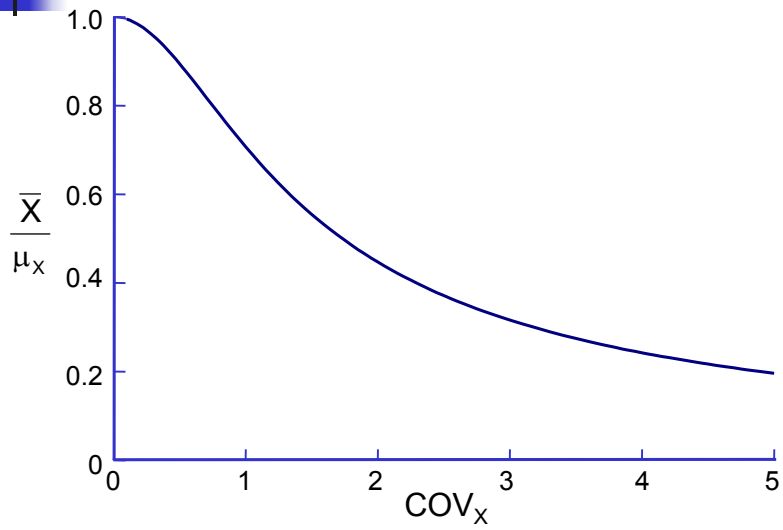
$$\text{COV}_x = \sqrt{\exp(\sigma_{\ln x}^2) - 1}$$

$$\mu_x = \exp(\mu_{\ln x} + 0.5\sigma_{\ln x}^2)$$

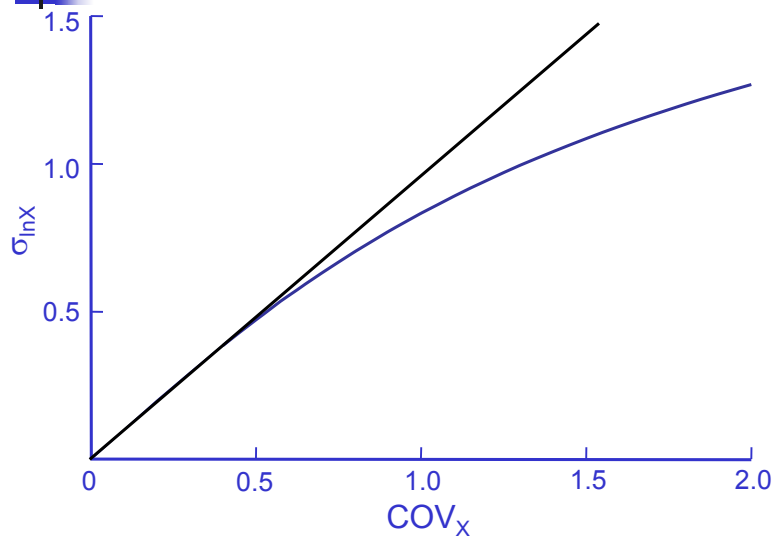
$$\bar{X}_x = \exp(\mu_{\ln x}) = \frac{\mu_x}{\sqrt{1 + \text{COV}_x^2}}$$



Median / Mean ratio

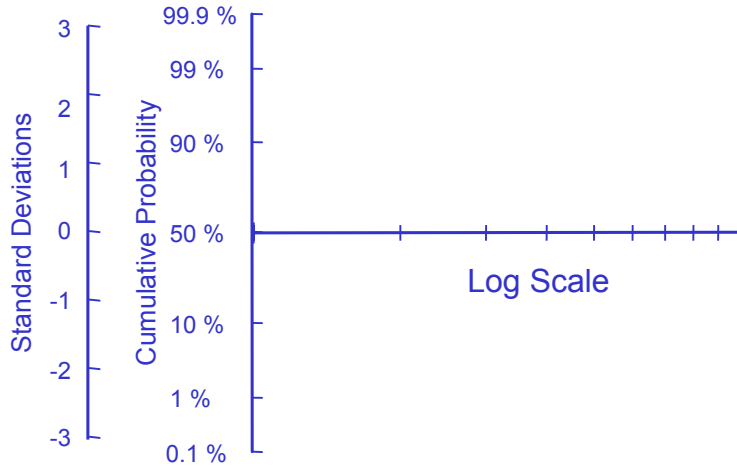


$\sigma_{\ln X} - COV_x$

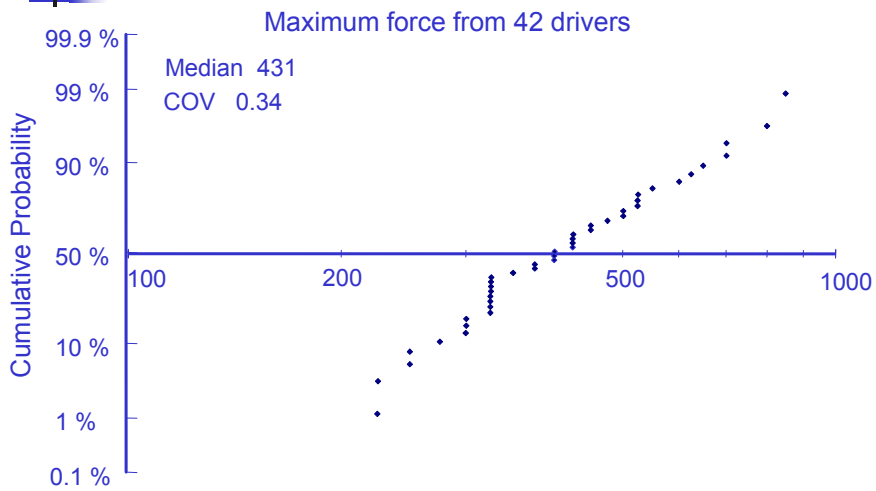




LogNormal Probability Plot

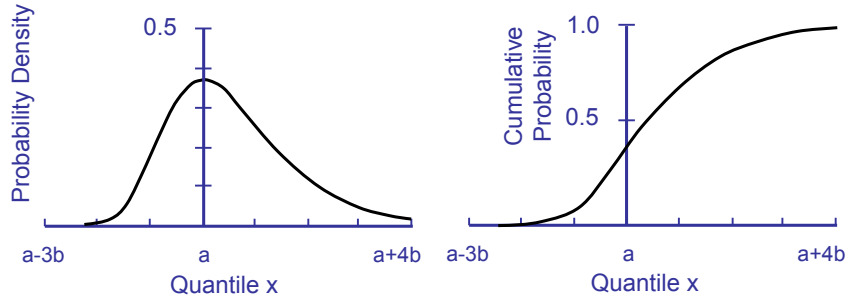


LogNormal Plot





Gumbel Extreme Value

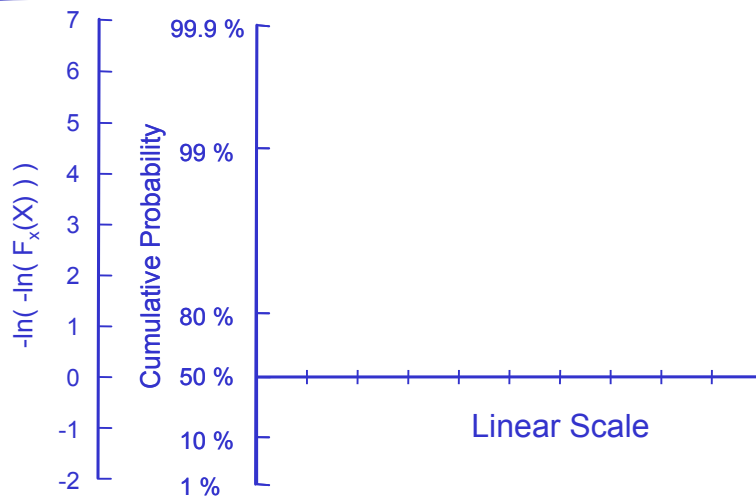


$$f_x(x) = \frac{1}{b} \exp\left[\frac{-(x-a)}{b}\right] \exp\left\{-\exp\left[\frac{-(x-a)}{b}\right]\right\}$$

$$F_x(x) = \exp\left\{-\exp\left[\frac{-(x-a)}{b}\right]\right\}$$

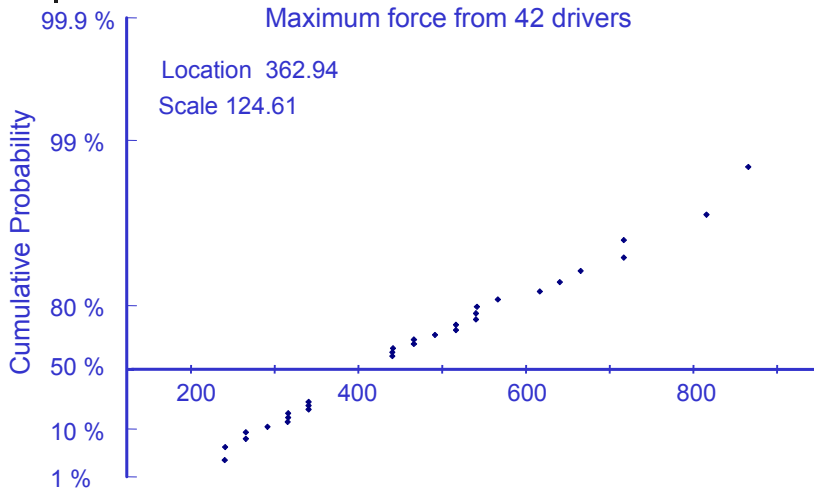


Gumbel Probability Plot

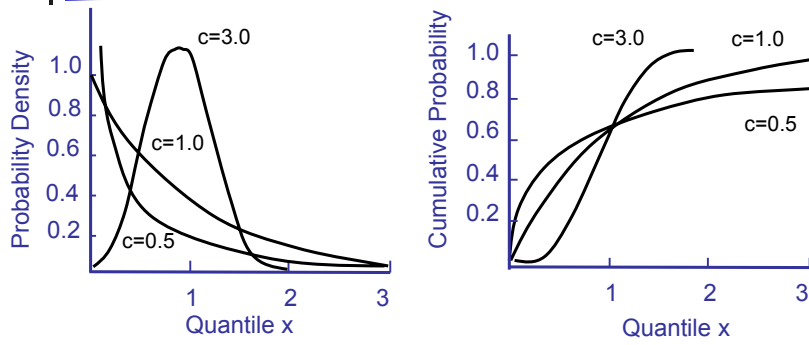




Gumble Plot



Weibull

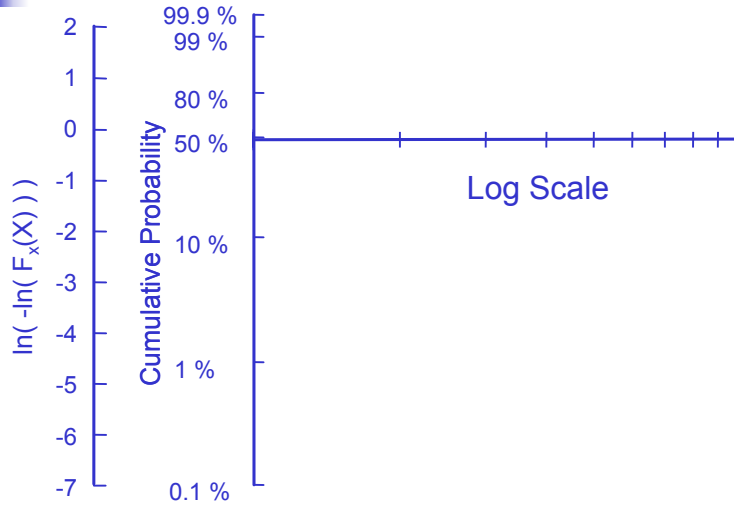


$$f_x(x) = \left(\frac{cx^{c-1}}{b^c} \right) \exp \left[- \left(\frac{x}{b} \right)^c \right]$$

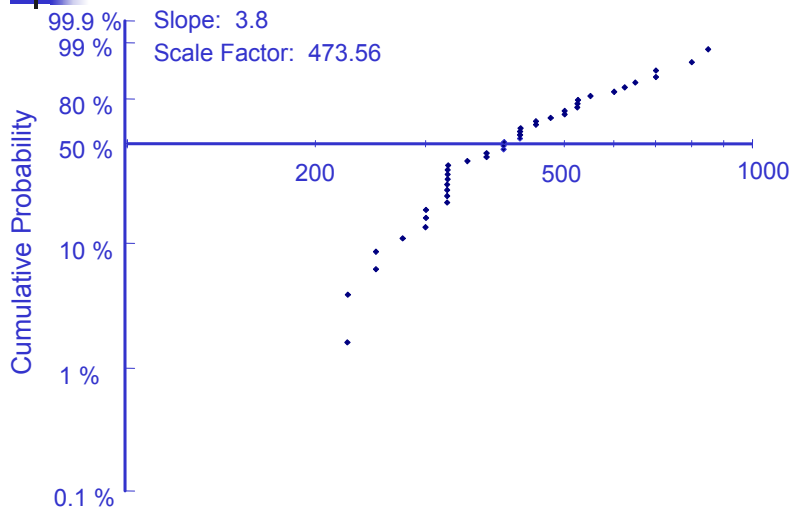
$$F_x(x) = 1 - \exp \left[- \left(\frac{x}{b} \right)^c \right]$$



Weibull Probability Plot



Weibull Plot





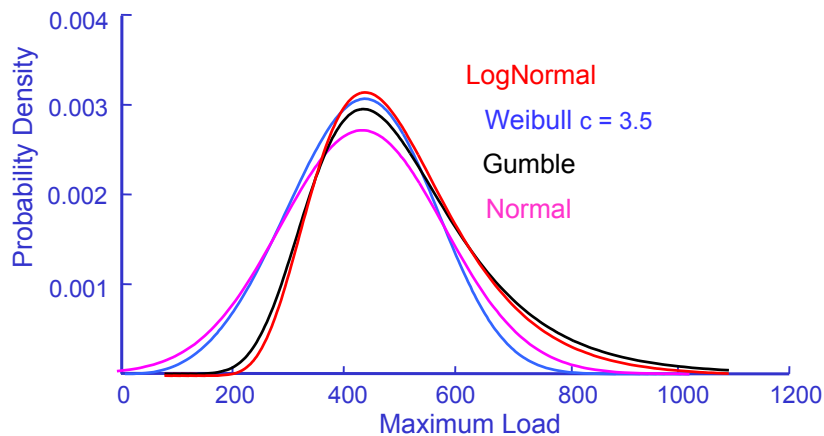
99.9 % Probability

Estimated maximum force from the distributions

Normal	893
LogNormal	1125
Gumbel	1223
Weibull	785



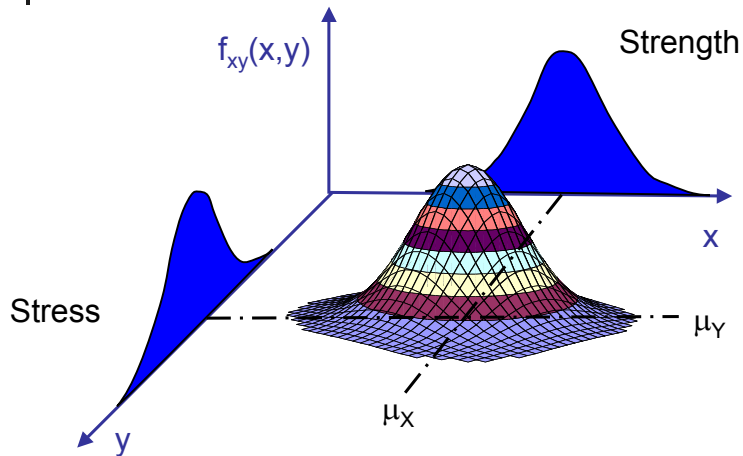
Comparison



All distributions are approximately normal around the median



Joint Probability Density



$$\text{Reliability} = 1 - P(\text{Stress} > \text{Strength})$$



Joint Probability Density

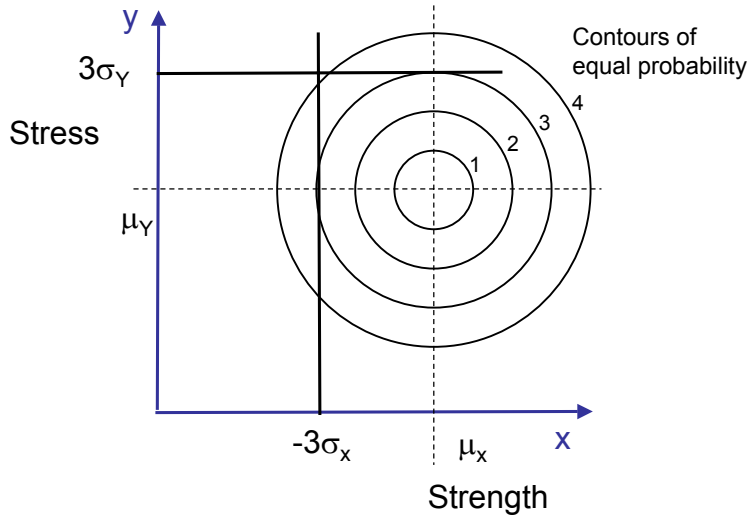
Normal distributions

$$f_{xy}(x,y) = \frac{1}{2\pi\sigma_x\sigma_y\sqrt{1-\rho^2}} \exp\left[\frac{-1}{2(1-\rho^2)}\left\{\left(\frac{x-\mu_x}{\sigma_x}\right)^2 - 2\rho\left(\frac{x-\mu_x}{\sigma_x}\right)\left(\frac{y-\mu_y}{\sigma_y}\right) + \left(\frac{y-\mu_y}{\sigma_y}\right)^2\right\}\right]$$

ρ correlation coefficient



2D - Joint Probability Density



Acceptable Risk

- Safety
 - $P(\text{failure}) \sim 10^{-6} \rightarrow 10^{-9}$
- Economic
 - $P(\text{failure}) \sim 10^{-2} \rightarrow 10^{-4}$



Extreme Values

For most durability problems, we are not interested in the “large extremes” of stress or strength. Failure is much more likely to come from moderately high stresses combined with moderately low strengths.



Key Points

- Basic Nomenclature
 - Random Variables
 - Statistical Distribution
 - Cumulative distribution function
 - Mean
 - Variance
 - Probability
 - Marginal
 - Conditional
 - Joint

Probabilistic Aspects of Fatigue

Statistical Techniques



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering

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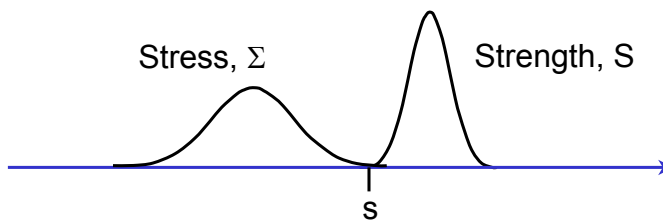


Statistical Techniques

- Normal Distributions
- LogNormal Distributions
- Monte Carlo
- Sampling
- Distribution Fitting



Failure Probability



Let Σ be the stress and S the fatigue strength

Given the distributions of Σ and S find the probability of failure

$$P(\Sigma \geq s \cap s \leq S)$$



Normal Variables

Linear Response Function

$$Z = a_0 + \sum_{i=1}^n a_i X_i$$

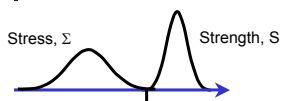
$$X_i \sim N(\mu_i, C_i)$$

$$\mu_z = a_0 + \sum_{i=1}^n a_i \mu_i$$

$$\sigma_z = \sqrt{\sum_{i=1}^n a_i^2 \sigma_i^2}$$



Calculations



$$S \sim N(200, 0.1) \quad \sigma_S = 20$$

$$\Sigma \sim N(100, 0.2) \quad \sigma_\Sigma = 20$$

Safety factor of 2

Let Z be a random variable:

$$Z = S - \Sigma$$

$$\mu_z = \mu_S - \mu_\Sigma$$

$$\mu_z = 200 - 100 = 100$$

$$\sigma_z = \sqrt{\sigma_S^2 + \sigma_\Sigma^2}$$

$$\sigma_z = \sqrt{20^2 + 20^2} = 28.2$$



Failure Probability

$$Z = S - \Sigma$$

Failure will occur whenever $Z \leq 0$

$$Z = \mu_z - z \sigma_z = 0$$

$$z = \frac{\mu_z}{\sigma_z} = \frac{100}{28.2}$$

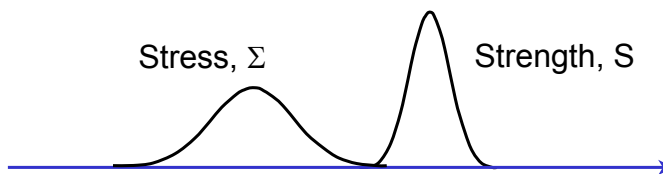
$z = 3.54$ standard deviations

$$P(\text{failure}) = 2 \times 10^{-4}$$

For this case only, a safety factor of 2 means a probability of failure of 2×10^{-4} . Other situations will require different safety factors to achieve the same reliability.



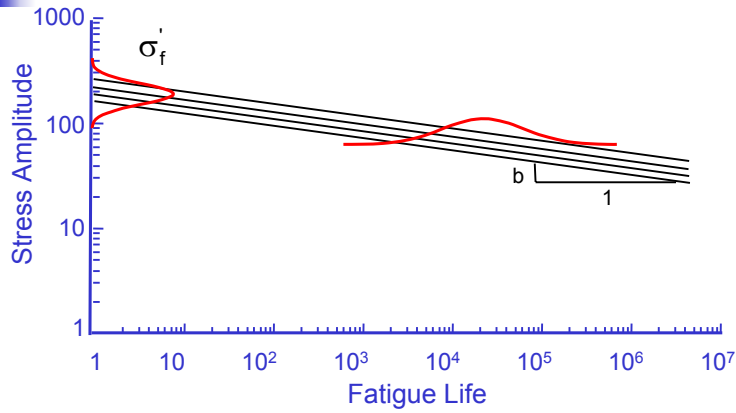
Failure Distribution



What is the expected distribution in fatigue lives?



Fatigue Data



$$\sigma_f' = \frac{\Delta S}{2(2N_f)^b} \qquad 2N_f = \left(\frac{\Delta S}{2\sigma_f'} \right)^{\frac{1}{b}}$$




LogNormal Variables

$$Z = a_o \prod_{i=1}^n X_i^{a_i}$$

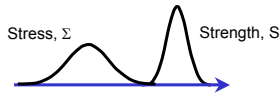
a_i 's are constant and $X_i \sim \text{LN}(x_i, C_i)$

$$\text{median } \bar{Z} = a_o \prod_{i=1}^n \bar{X}_i^{a_i}$$

$$\text{COV } C_Z = \sqrt{\prod_{i=1}^n (1 + C_{X_i}^2)^{a_i^2}} - 1$$



Calculations



$$\sigma_f' \sim \text{LN}(1000, 0.1) \quad \sigma = 100$$

$$\frac{\Delta S}{2} \sim \text{LN}(250, 0.2) \quad \sigma = 50$$

$$b = -0.125$$

$$2N_f = \left(\frac{\Delta S}{2\sigma_f'} \right)^{\frac{1}{b}}$$

$$Z = 2N_f = \left(\frac{\Delta S}{2} \right)^{-8} \sigma_f'^8$$

$$\bar{Z} = 2N_f = \left(\frac{\Delta \bar{S}}{2} \right)^{-8} \bar{\sigma}_f'^8$$

$$\text{COV}_Z = \sqrt{\left(1 + \text{COV}_{\Delta S}^2\right)^{-8^2} \left(1 + \text{COV}_{\sigma_f'}^2\right)^{8^2} - 1}$$



Results

	$\Delta S/2$	σ_f'	$2N_f$	Percentile	Life
μ_x	250	1000	355,368	99.9	17,706,069
COV_x	0.2	0.1	4.72	99	4,566,613
				95	1,363,200
$\mu_{\ln x}$	5.50	6.90	11.21	90	715,589
X	245	995	73,676	50	73,676
σ_x	50	100	1,676,831	10	7,586
$\sigma_{\ln x}$	0.198	0.100	1.774	5	3,982
				1	1,189
b =	-0.125			0.1	307



Monte Carlo Methods

$$\frac{K_f \Delta S}{2} = \sqrt{E \left(\frac{\sigma_f'^2}{E} (2N_f)^{2b} + \sigma_f' \varepsilon_f' (2N_f)^{b+c} \right)}$$

Given random variables for K_f , ΔS , σ_f' and ε_f'
Find the distribution of $2N_f$

$$Z = 2N_f = ?$$



Simple Example

Probability of rolling a 3 on a die



Uniform discrete distribution



Computer Simulation

1. Generate random numbers between 1 and 6, all integers
2. Count the number of 3's

Let $X_i = 1$ if 3
0 otherwise

$$P(3) = \frac{1}{n} \sum_{i=1}^n X_i$$



EXCEL

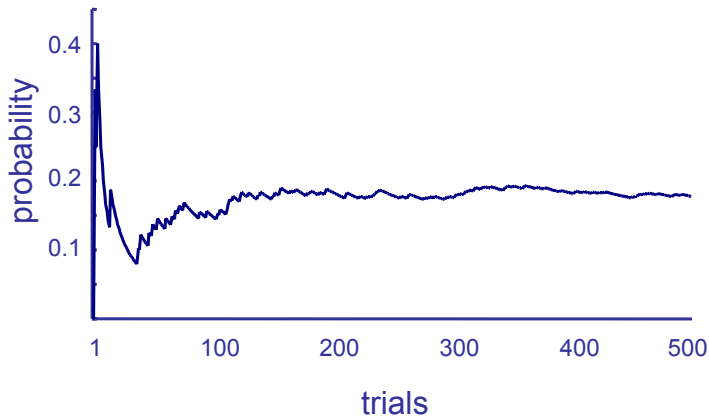
=ROUNDUP(6 * RAND() , 0)

=IF(A1 = 3 , 1 , 0)

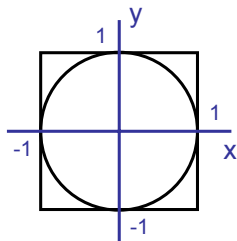
=SUM(\$B\$1:B1)/ROW(B1)

5	0	0
3	1	0.5
4	0	0.333333
4	0	0.25
5	0	0.2
6	0	0.166667
1	0	0.142857
3	1	0.25
3	1	0.333333
6	0	0.3

Results



Evaluate π



P(inside circle)

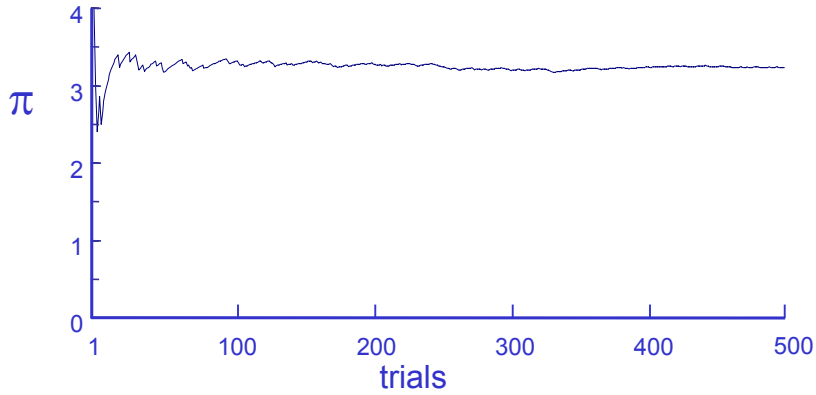
$$P = \frac{\pi r^2}{4}$$

$$\pi = 4 P$$

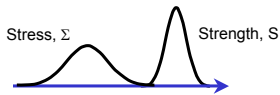
$$x = 2 * \text{RAND}() - 1$$

$$y = 2 * \text{RAND}() - 1$$

$$\text{IF}(x^2 + y^2 < 1, 1, 0)$$



Monte Carlo Simulation



$$2N_f = \left(\frac{\Delta S}{2\sigma_f'} \right)^{\frac{1}{b}}$$

$$\sigma_f' \sim \text{LN}(1000, 0.1) \quad \sigma = 100$$

$$\frac{\Delta S}{2} \sim \text{LN}(250, 0.2) \quad \sigma = 50$$

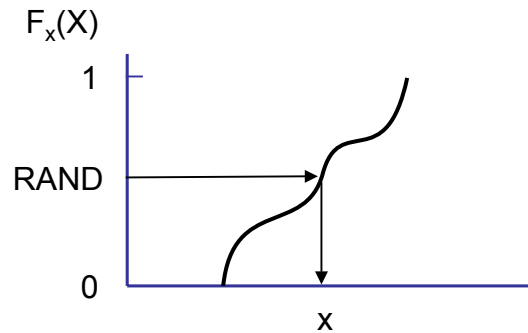
$$b = -0.125$$

Randomly choose values of S and σ_f' from their distributions

Repeat many times



Generating Distributions



Randomly choose a value between 0 and 1

$$x = F_x^{-1}(\text{RAND})$$



Generating Distributions in EXCEL

Normal

$$=\text{NORMINV}(\text{RAND}(), \mu, \sigma)$$

Log Normal

$$=\text{LOGINV}(\text{RAND}(), \ln \mu, \ln \sigma)$$

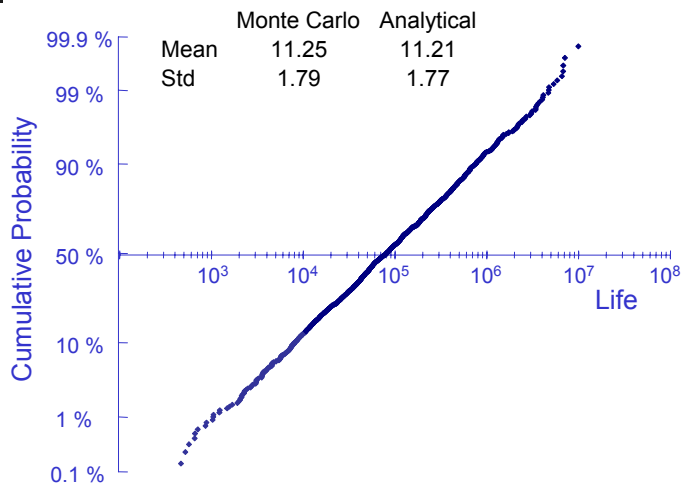


EXCEL

σ_f	$\frac{\Delta S}{2}$	$2N_f$
893	204	134,677
1102	301	32,180
852	285	6,355
963	173	929,249
1050	283	35,565
1080	265	77,057
965	313	8,227
1073	213	420,456
1052	226	224,000
954	322	5,878
965	240	68,671
993	207	277,192
1191	368	11,967
831	210	59,473



Simulation Results





Summary

Simulation is relatively straightforward and simple

Obtaining the necessary input data is difficult



Nomenclature

Population: the totality of the observations

Sample: a subset of the population

Population mean: μ_x

Population variance: σ_x^2

Sample mean: \bar{X}

Sample variance: s^2

$$E(\bar{X}) = \mu_x$$

$$E(s^2) = \sigma_x^2$$



Sample Mean and Variance

Mean
$$\bar{X} = \frac{\sum_1^n x}{n}$$

in EXCEL =AVERAGE(A1:A_n)

Variance
$$s^2 = \frac{\sum_1^n (x - \bar{X})^2}{n-1}$$

in EXCEL =STDEV(A1:A_n)



Confidence Intervals

$$E(\bar{X}) = \mu_x$$

What is the probability that a sample \bar{X} is greater than μ_x ? 50%

$$P(L \leq \mu_x \leq U) = 1 - \alpha$$

There is a $1 - \alpha$ chance of selecting a sample in the interval between L and U that contains the true mean of the population



Translation

90% confidence

If we sampled a population many times to estimate the mean, 90% of the time the true population mean would lie between the computed upper and lower limit.



Confidence Interval - mean

For a normal distribution:

Lower limit of μ

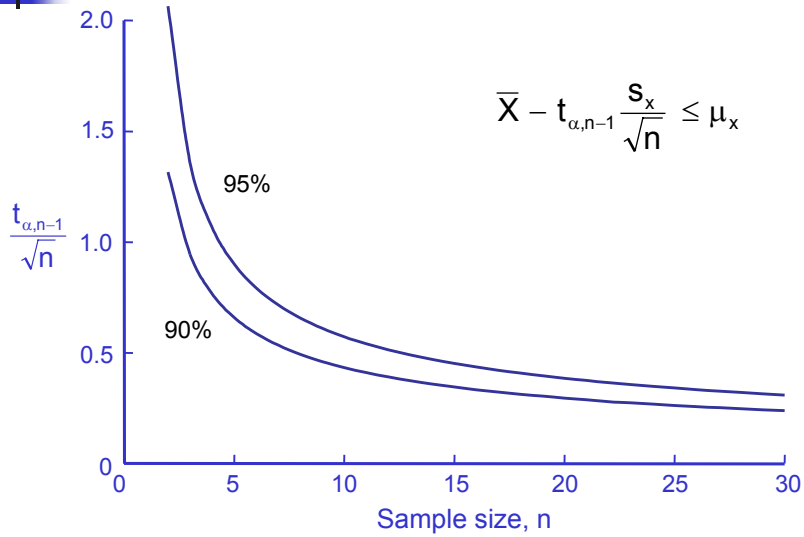
$$\bar{X} - t_{\alpha, n-1} \frac{s_x}{\sqrt{n}} \leq \mu_x$$

Upper limit of μ

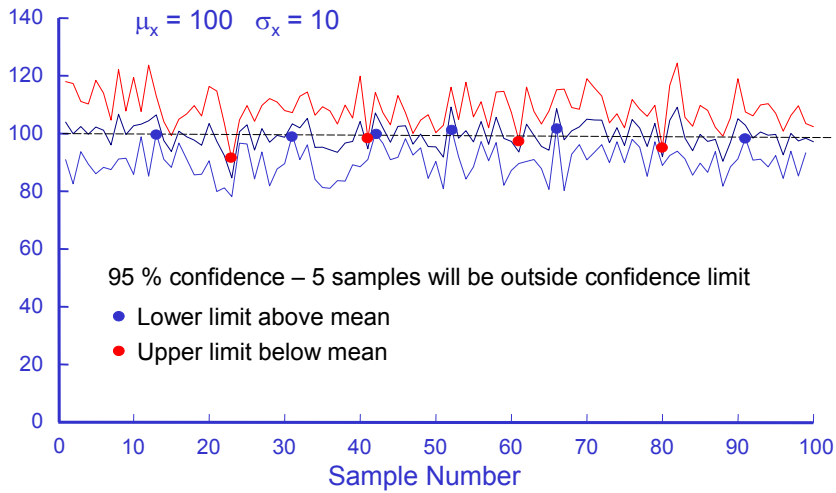
$$\bar{X} + t_{\alpha, n-1} \frac{s_x}{\sqrt{n}} \geq \mu_x$$



Sample Size



5 Samples from Normal Distribution





Confidence Interval - variance

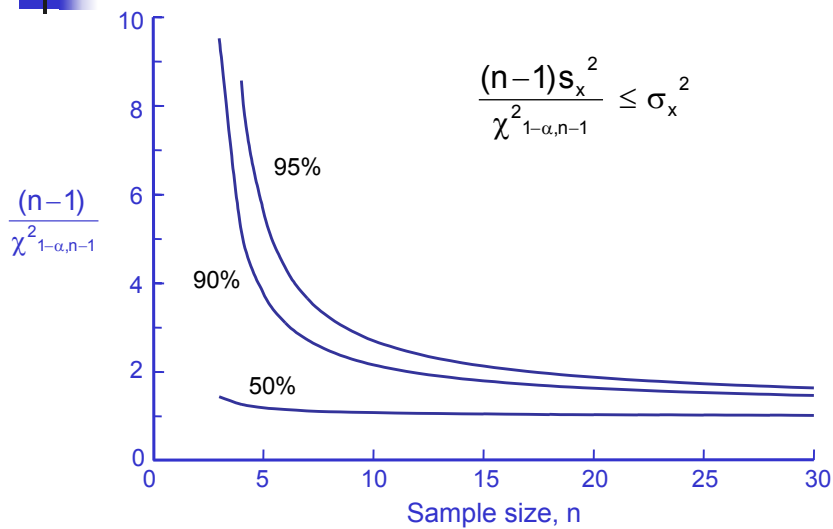
For a normal distribution:

Upper limit of σ

$$\frac{(n-1)s_x^2}{\chi^2_{1-\alpha, n-1}} \leq \sigma_x^2$$

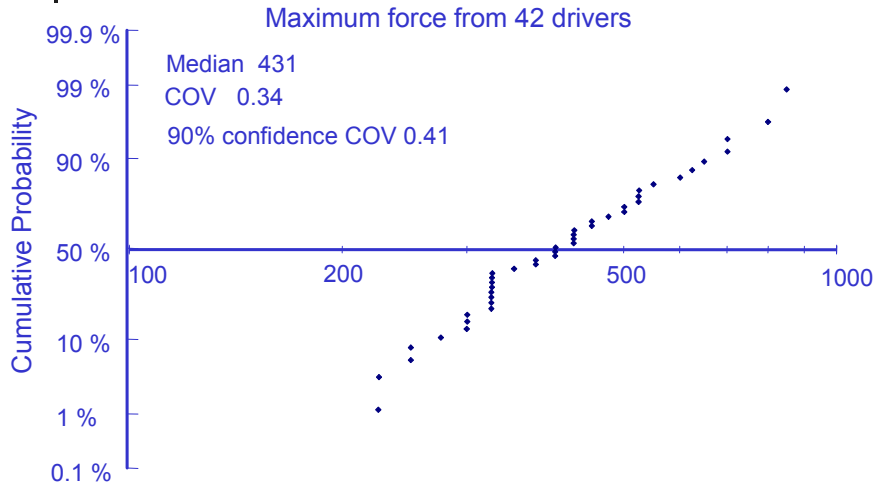


Sample Size

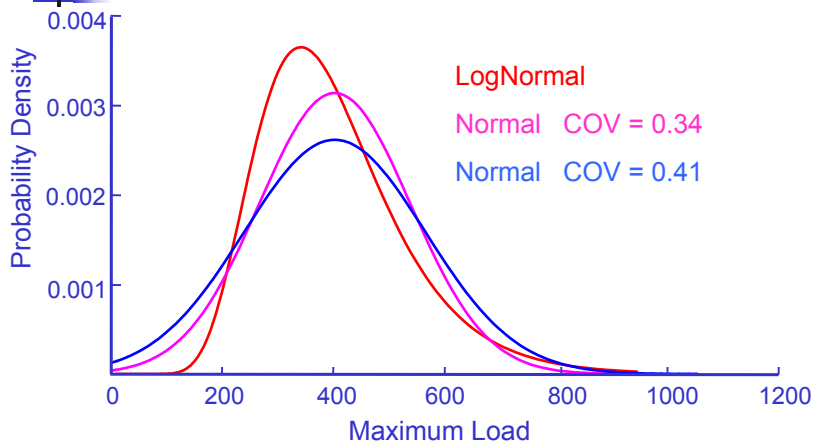




Maximum Load Data



Maximum Load Data

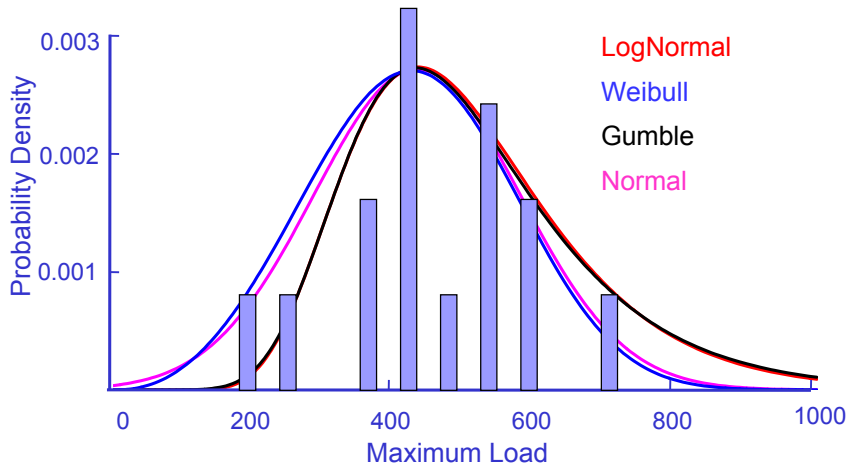


Uncertainty in Variance is just as important,
perhaps more important than the choice of the distribution

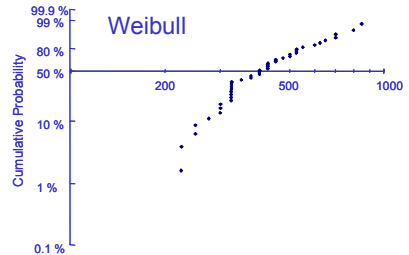
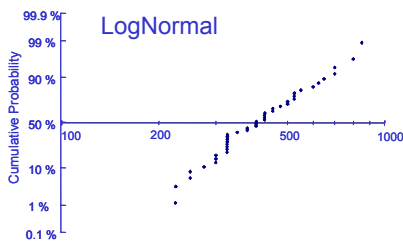
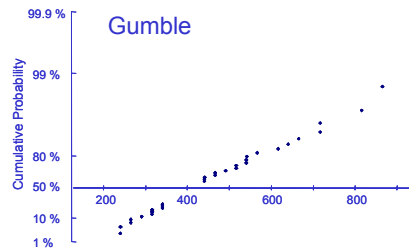
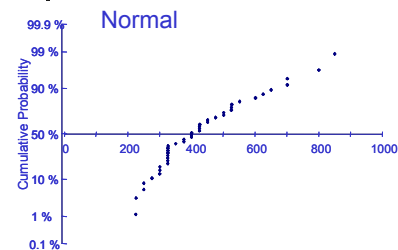


Choose the "Best" Distribution

15 samples from a Normal Distribution



Goodness of Fit Tests





Analytical Tests

Chi-square

any univariate distribution

Snedecor, George W. and Cochran, William G. (1989), *Statistical Methods, Eighth Edition*, Iowa State University Press.

Kolmogorov-Smirnov

tends to be more sensitive near the center of the distribution

Chakravarti, Laha, and Roy, (1967). *Handbook of Methods of Applied Statistics, Volume I*, John Wiley and Sons, pp. 392-394.

Anderson-Darling

gives more weight to the tails

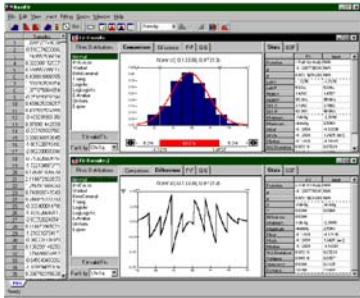
Stephens, M. A. (1974). *EDF Statistics for Goodness of Fit and Some Comparisons*, Journal of the American Statistical Association, Vol. 69, pp. 730-737.



Distributions

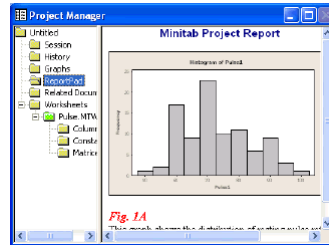
- Normal
 - Strength
 - Dimensions
- LogNormal
 - Fatigue Lives
 - Large variance in properties or loads
- Gumble
 - Maximums in a population
- Weibull
 - Fatigue Lives

Statistics Software



BestFit

www.palisade.com



Minitab

www.minitab.com

Central Limit Theorem

If $X_1, X_2, X_3, \dots, X_n$ is a random sample from the population, with sample mean \bar{X} , then the limiting form of

$$Z = \frac{\bar{X} - \mu_X}{\sigma / \sqrt{n}}$$

as $n \rightarrow \infty$ is the standard normal distribution



Translation

When there are many variables affecting the outcome,
The final result will be normally distributed even if the
individual variable distributions are not.



Example

Probability of rolling a die

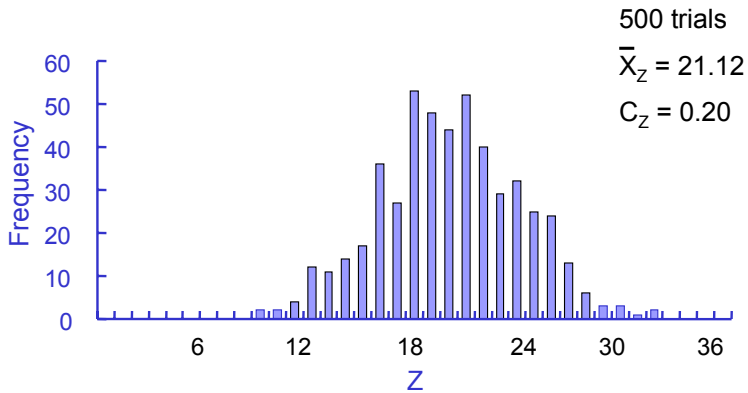


Uniform discrete distribution

Let Z be the summation of six dice

$$Z = X_1 + X_2 + X_3 + X_4 + X_5 + X_6$$

Results



Central limit theorem states that the result should be normal for large n

Central Limit Theorem

Sums: $Z = X_1 \pm X_2 \pm X_3 \pm X_4 \pm \dots X_n$

$Z \rightarrow$ Normal as n increases

Products: $Z = X_1 \cdot X_2 \cdot X_3 \cdot X_4 \cdot \dots X_n$

$Z \rightarrow$ LogNormal as n increases

Normal and LogNormal distributions are often employed for analysis even though the underlying population distribution is unknown.



Key Points

- All variables are random and can be characterized by a statistical distribution with a mean and variance.
- The final result will be normally distributed even if the individual variable distributions are not.

Probabilistic Aspects of Fatigue

Analysis Methods



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering



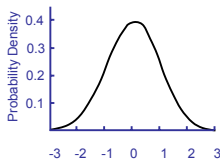
Probabilistic Aspects of Fatigue

- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- **Analysis Methods**
- Characterizing Variability
- Case Studies
- FatigueCalculator.com
- GlyphWorks

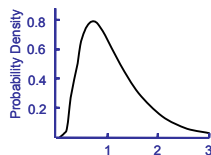


Reliability Analysis

Stressing Variables



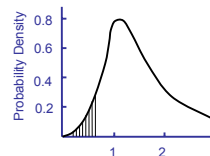
Strength Variables



Analysis

?

P(Failure)





Probabilistic Analysis Methods

- Monte Carlo
 - Simple
 - Hypercube sampling
 - Importance sampling
- Analytical
 - First order reliability method FORM
 - Second order reliability method SORM



Limit States

- Limit States
 - Equilibrium
 - Strength
 - Deformation
 - Wear
 - Functional



Limit State Problems

Response function

$$Z(X) = Z(Q, L, b, h)$$

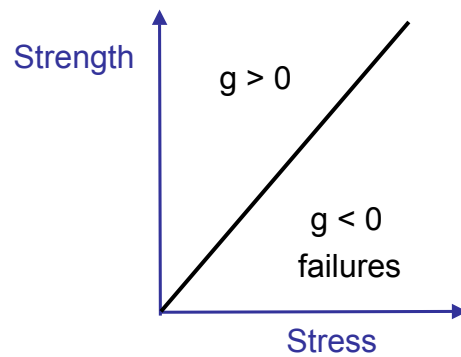
Limit State function

$$g = Z(X) - Z_o = 0$$

Same as the failure function



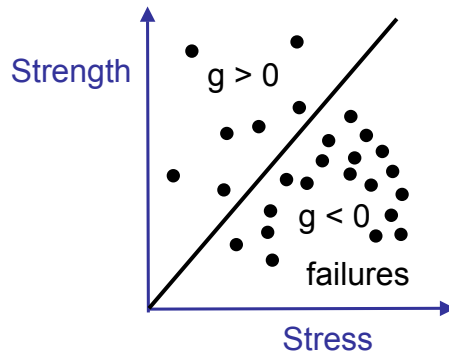
Limit States



$$g = \text{Strength} - \text{Stress}$$



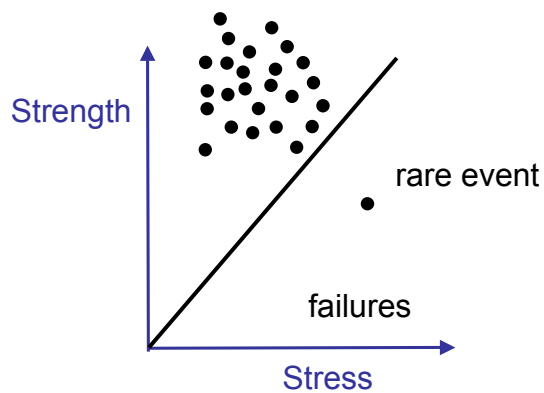
Monte Carlo



Monte Carlo is simple but what if each of these calculations required a separate FEM model?



Low Probabilities of Failure



Need about 10^5 simulations for $P(\text{Failure}) = 10^{-4}$

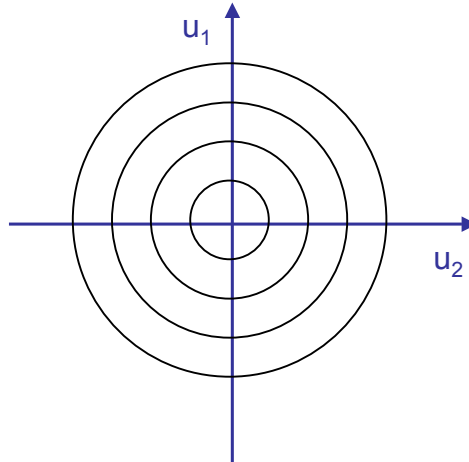


Transform Variables

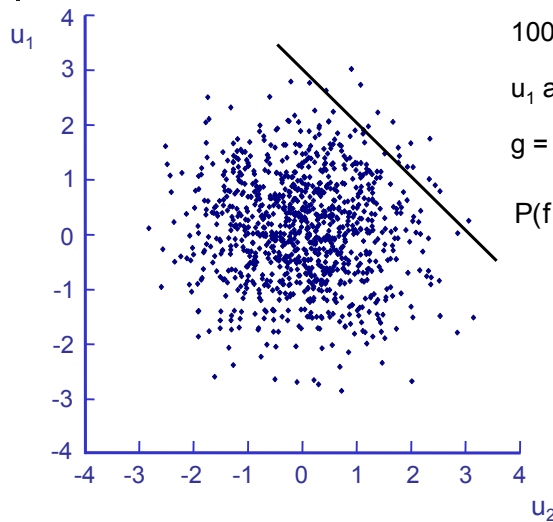
$$u_1 = \frac{X - \mu_x}{\sigma_x}$$

$$\mu_{u_1} = 0$$

$$\sigma_{u_1} = 1$$



Standard Monte Carlo



1000 trials

u_1 and u_2 normally distributed

$$g = 3 - (u_1 + u_2)$$

$$P(f) = \frac{N_f}{N} = 0.018$$



Multiple Simulations

1	25
2	15
3	19
4	21
5	16
6	22
7	16
8	12
9	15
10	11
11	19
12	15
13	20
14	17
15	9
16	23
17	15
18	20
19	14
20	21
21	16
22	18
23	22
24	13
25	20

1000 trials

25 simulations

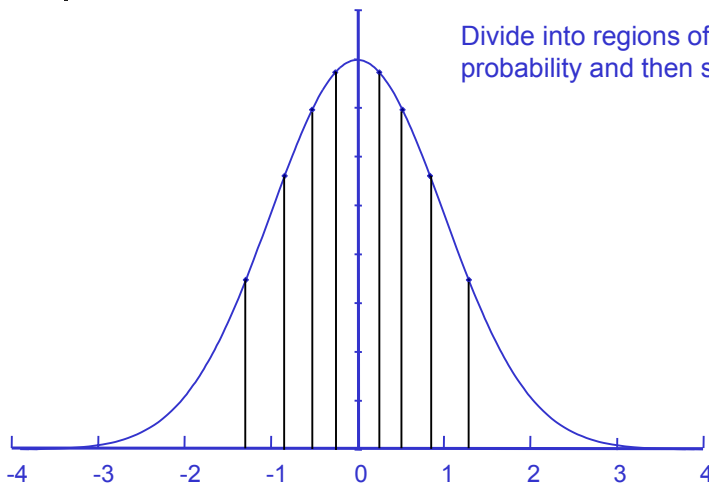
25,000 calculations

$\mu = 0.017$

$\sigma = 0.0040$



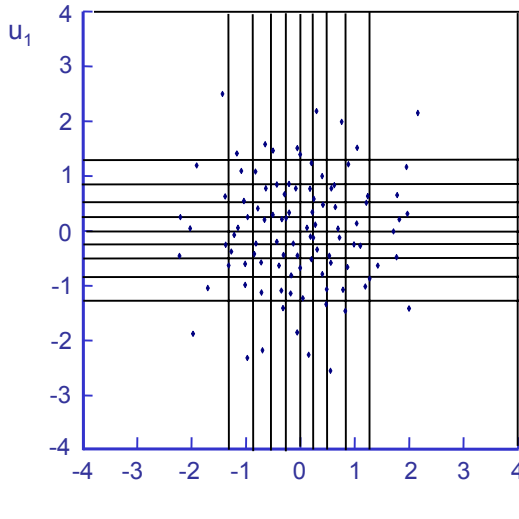
Stratified Sampling Methods



Divide into regions of constant probability and then sample by region



Stratified Sample



Each box should have one sample drawn at random from the underlying pdf

100 trials

25 simulations

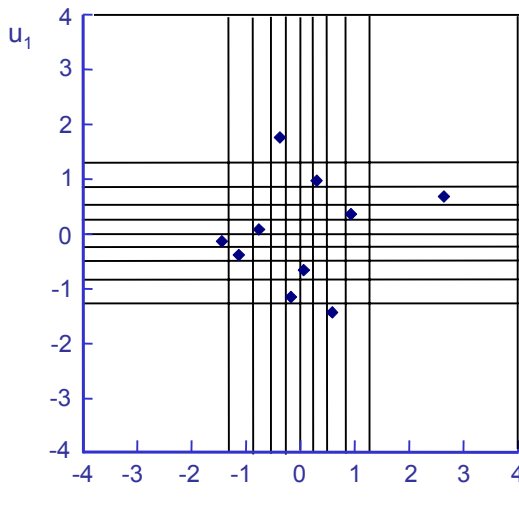
2,500 calculations

$\mu = 0.02$

$\sigma = 0.0093$



Latin Hypercube Sample



10 trials

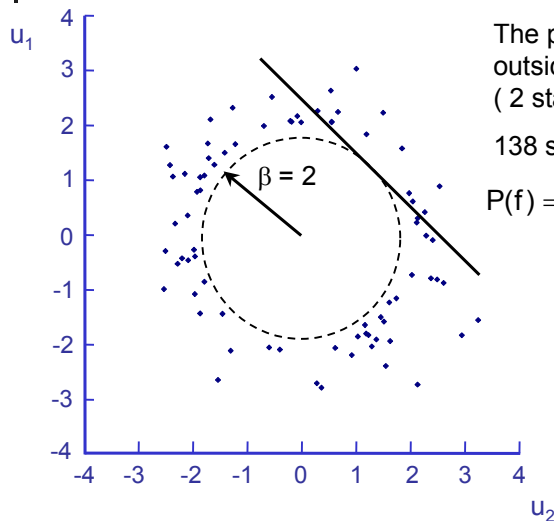
25 simulations

250 calculations

$\mu = 0.017$

$\sigma = 0.038$

Importance Sampling (continued)



The probability of a sample outside the circle is $P_s = 0.14$ (2 standard deviations)

138 simulations

$$P(f) = \frac{N_f}{N_s} P_s = 0.0183$$

Analytical Methods Strategy

- Develop a response function
- Transform the set of N variables to a set of uncorrelated u variables with $\mu=0$ and $\sigma=1$
- Locate the most likely failure point
- Approximate the integral of the joint probability distribution over the failure region



Analytical Methods Outline

- Response Surface
- Transform Variables
- Limit State Concept
- Most Probable Point
- Probability Integration
 - FORM
 - SORM



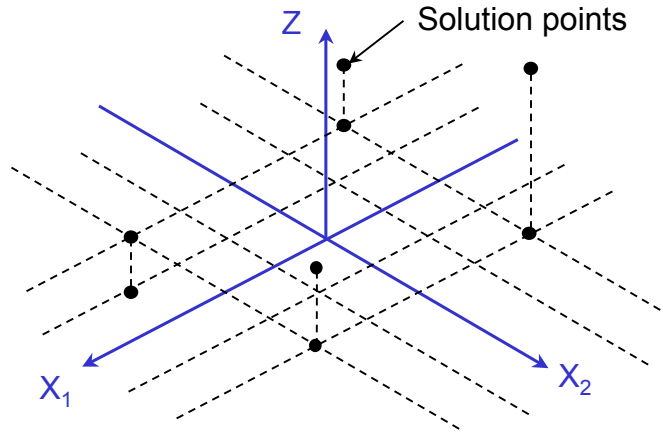
Response Surface Methodology

Response surface methodology is a well established collection of mathematical and statistical techniques for applications where the response of interest is influenced by several variables.

$$Z(X) = f(x_1, x_2, x_3, \dots, x_n) + \varepsilon$$



Response Surface



Fit a surface through the points



Mathematical Representation

$$Z(X) = A_0 + A_1x_1 + A_2x_2 + A_3x_3 + \dots \quad \text{Linear}$$

$$B_1x_1^2 + B_2x_2^2 + B_3x_3^2 + \dots \quad \text{Incomplete Quadratic}$$

$$C_1x_1x_2 + C_2x_1x_3 + C_3x_2x_3 + \dots \quad \text{Complete Quadratic}$$

	Solutions
Linear	$N + 1$
Incomplete Quadratic	$2N + 1$
Complete Quadratic	$N(N + 1)/2$



Evaluation of Response Surface

Suppose stress is affected by speed and temperature

Factorial Design

		temperature	
		high	low
speed	high	X	X
	low	X	X

4 deterministic solutions needed
 2^n solutions



Evaluation of Response Surface

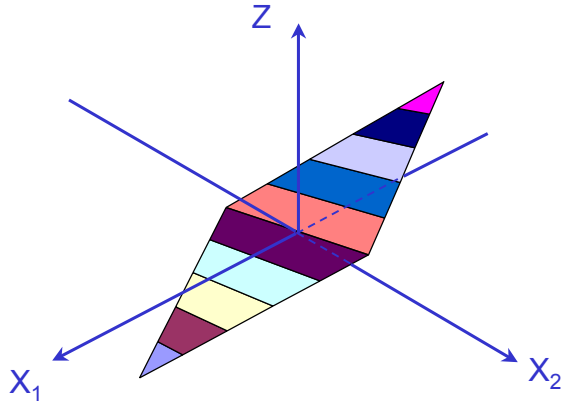
Factorial Design

		temperature		
		high		low
speed	high	X	X	X
		X	X	X
	low	X	X	X

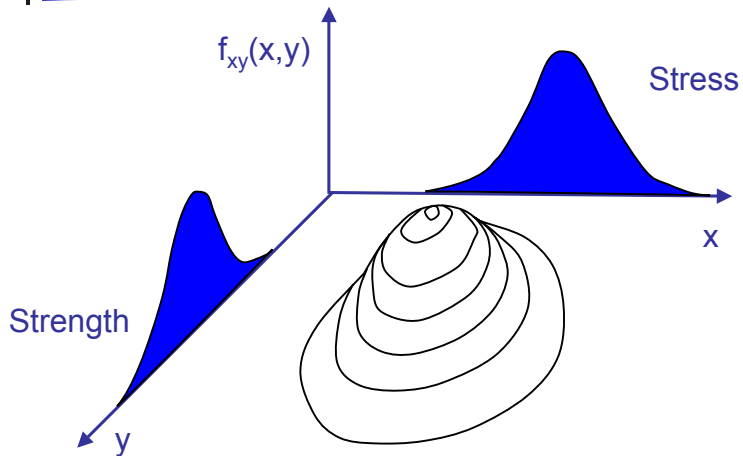
9 deterministic solutions needed
 3^n solutions



Graphical Representation

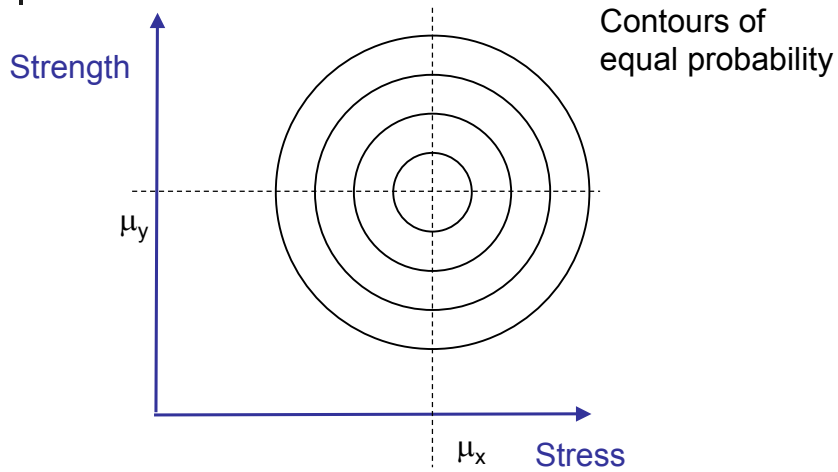


Joint Probability Density



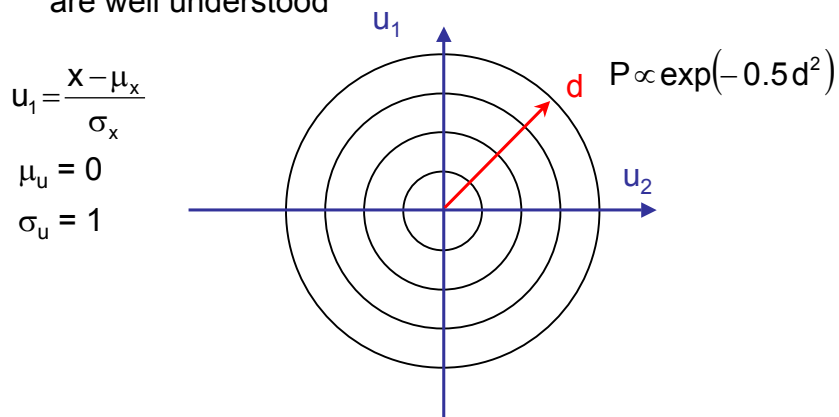


2D - Joint Probability Density



Transform Variables

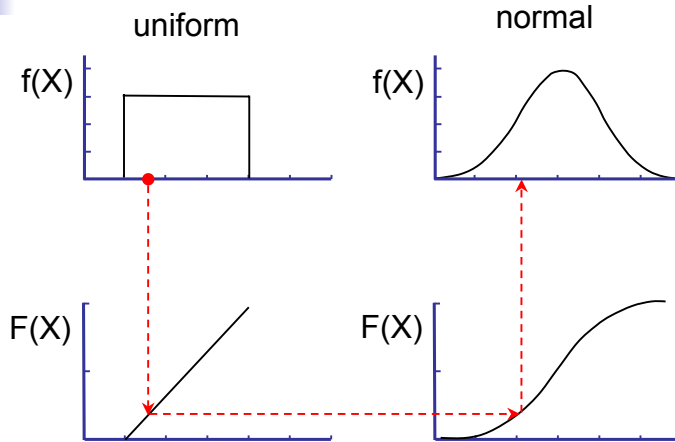
The mathematics and behavior of normal distributions are well understood



Any distribution can be mapped into a normal distribution



X to u mapping



Limit States

Limit states define probability problems

For example:

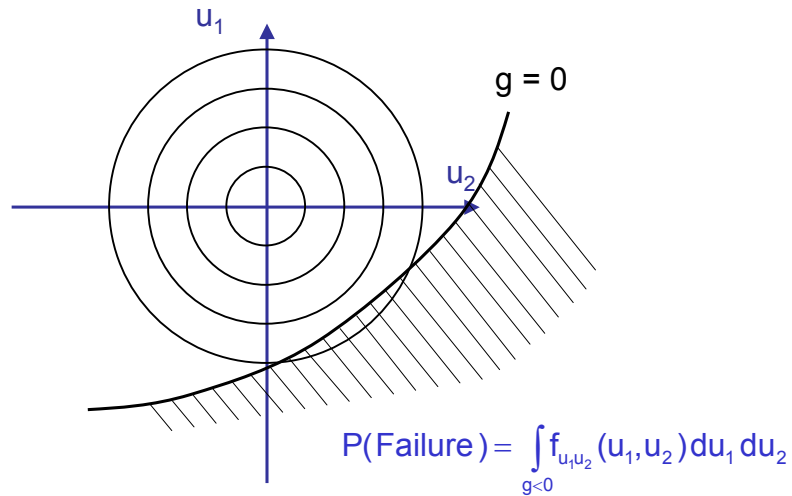
$$g = \text{Strength}(x_1) - \text{Stress}(x_2)$$

$$\text{Prob}(g \leq 0) = \text{Probability of failure} = P_f$$

Analysis focuses on $g = 0$

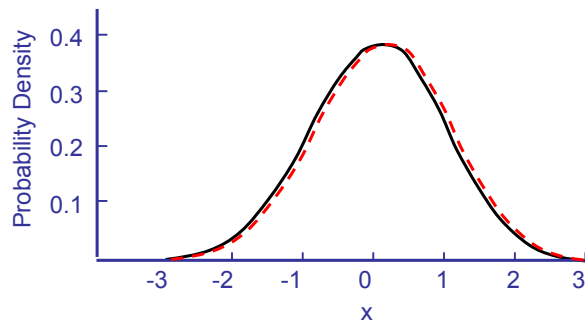


Failure Probability



Thought Experiment

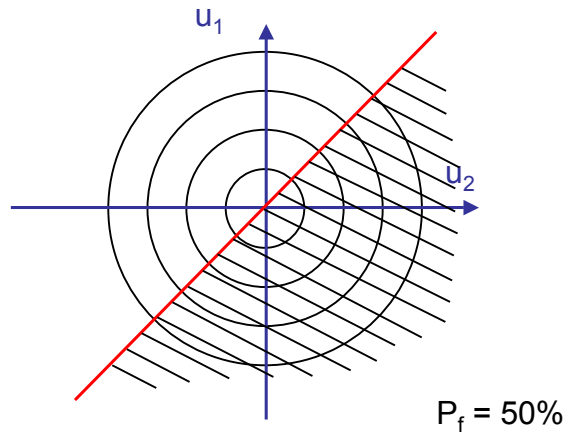
Suppose stress and strength had the same distribution



What is the probability of failure?

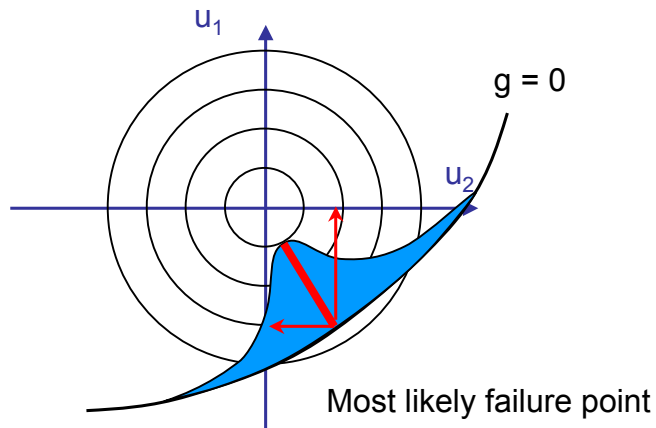
g function

$$g = \text{Strength}(x_1) - \text{Stress}(x_2)$$



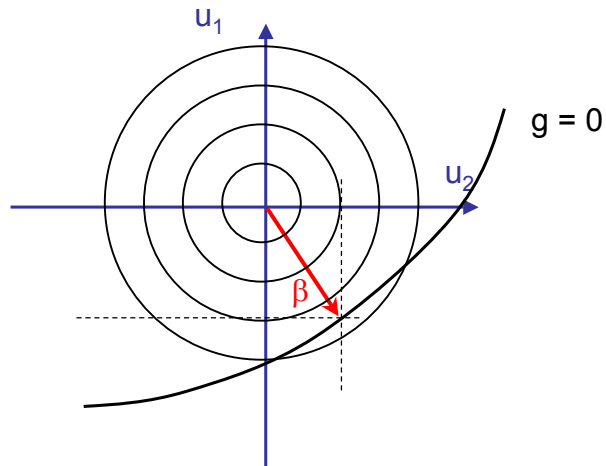
Probable Combinations of u_1 and u_2

g functions are not straight lines in N dimensional space





Most Probable Point



Probability is related to the minimum distance point β



Safety Index β

Sometimes called reliability index

β is a number measured in standard deviations

Unlike safety factors, failure probability is directly related to the safety index

$$P_f = \Phi(-\beta)$$



Determining the MPP

Requires an efficient numerical search to find the tangent point of a hypersphere (β -sphere) and the limit state function in \mathbf{u} space



Probability Integration

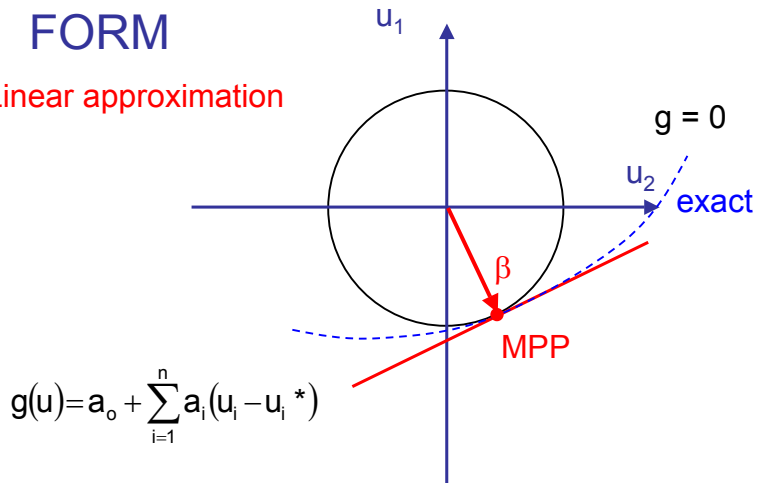
- FORM
- SORM



First Order Reliability Model

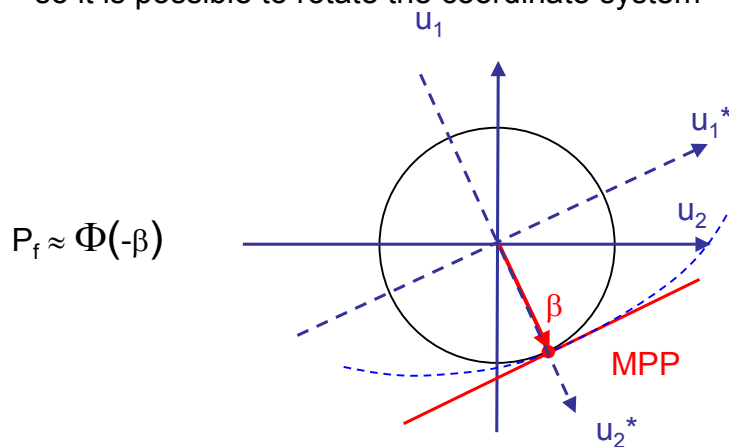
FORM

Linear approximation



FORM

The joint probability density is rotationally symmetric so it is possible to rotate the coordinate system

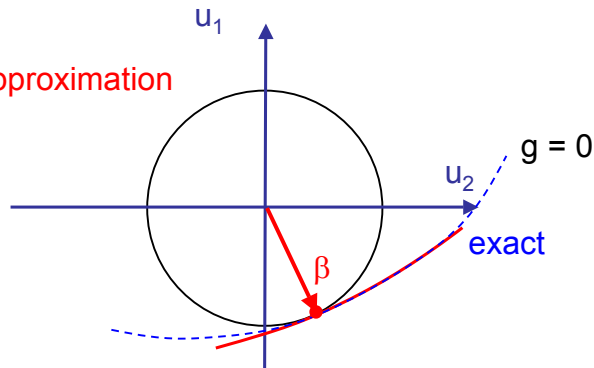




Second Order Reliability Model

SORM

Quadratic approximation



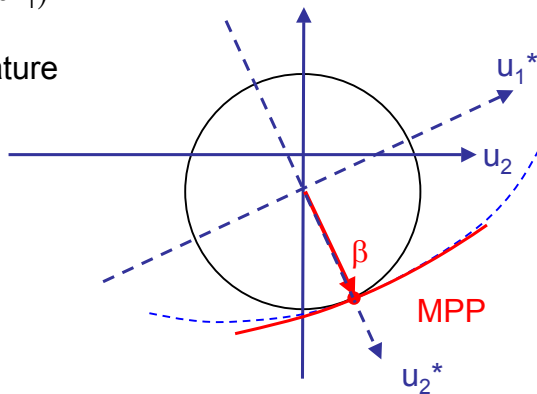
$$g(u) = a_0 + \sum_{i=1}^n a_i (u_i - u_i^*) + \sum_{i=1}^n b_i (u_i - u_i^*)^2 + \sum_{i=1}^n \sum_{j=1}^n c_{ij} (u_i - u_i^*) (u_j - u_j^*)$$



SORM

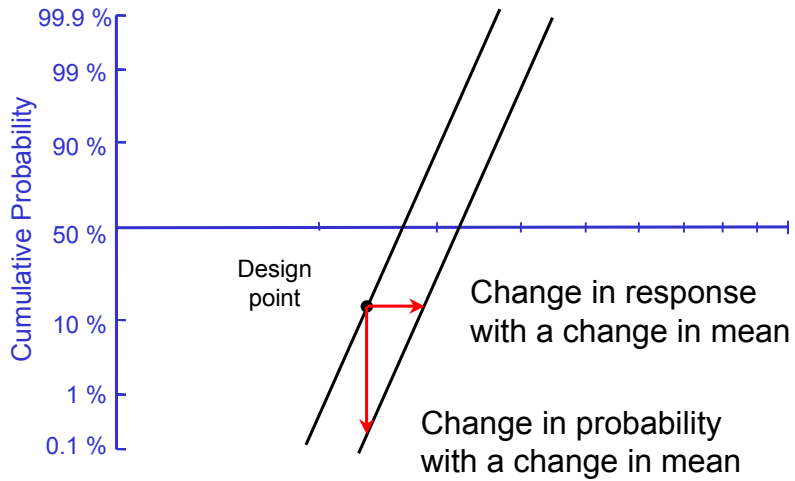
$$P_f = \Phi(-\beta) \prod_{i=1}^n (1 - \beta \kappa_i)$$

κ_i surface curvature

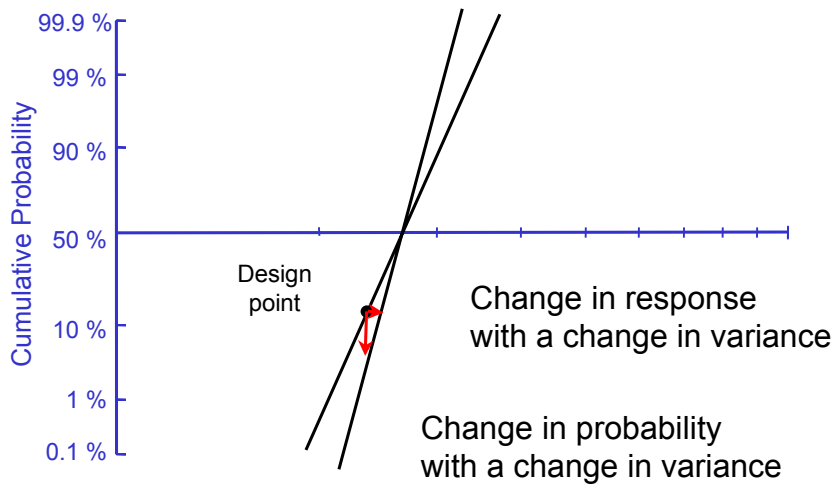




Sensitivity Factors



Sensitivity Factors





Deterministic Sensitivity Factors

$$\frac{\partial Z(X)}{\partial X_i}$$

Change in response mean with respect to a change in the input variable mean

Frequently normalized by the means to compare relative importance of variables

$$\frac{\partial Z(X) / \bar{Z}}{\partial X_i / \bar{X}}$$



Probabilistic Sensitivity Coefficient

Standard deviation sensitivity

$$S_{\sigma_i} = \frac{\partial P / P}{\partial \sigma_i / \sigma_i}$$

Mean deviation sensitivity

$$S_{\mu_i} = \frac{\partial P / P}{\partial \mu_i / \mu_i}$$



Probabilistic Sensitivity Factors

$$\alpha_i \propto \left(\frac{\partial Z(X)}{\partial X_i} \right) \sigma_i \quad \sum \alpha_i^2 = 1$$

- α_i - probabilistic sensitivity
- $Z(X)$ - response
- X_i - input variable
- σ_i - standard deviation of X_i



Software

- General Purpose
- Durability

Probabilistic Aspects of Fatigue

Variability



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering

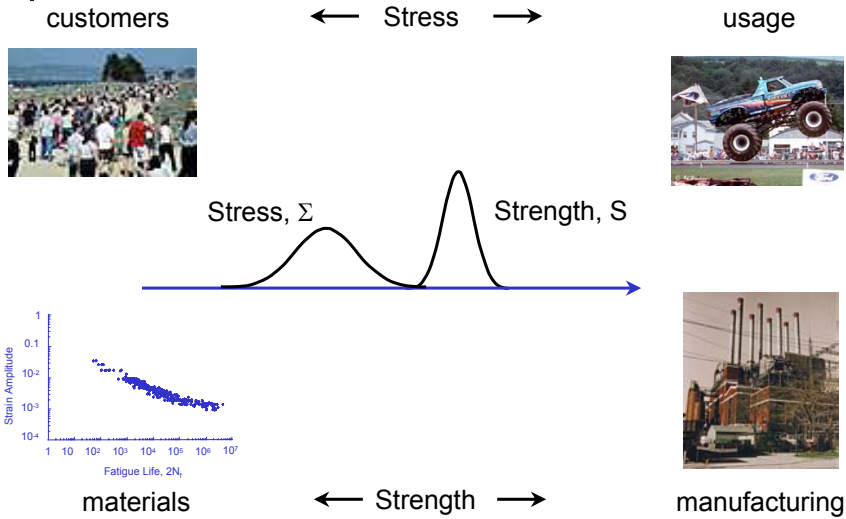
© 2003-2005 Darrell Socie, All Rights Reserved



Probabilistic Aspects of Fatigue

- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- Analysis Methods
- **Characterizing Variability**
- Case Studies
- FatigueCalculator.com
- [GlyphWorks](http://GlyphWorks.com)

Sources of Variability



Variability and Uncertainty

Variability: Every apple on a tree has a different mass.

Uncertainty: The variety of the apple is unknown.

Variability: Fracture toughness of a material

Uncertainty: The correct stress intensity factor solution



Sources of Variability

- Stress Variables
 - Loading
 - Customer Usage
 - Environment
- Strength Variables
 - Material
 - Processing
 - Manufacturing Tolerance
 - Environment



Sources of Uncertainty

- Statistical Uncertainty
 - Incomplete data (small sample sizes)
- Modeling Error
 - Analysis assumptions
- Human Error
 - Calculation errors
 - Judgment errors

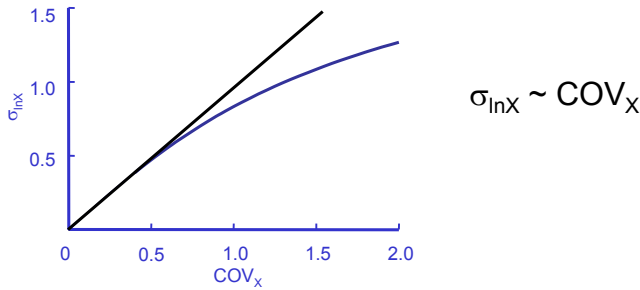


Modeling Variability

Central Limit Theorem:

$$\text{Products: } Z = X_1 \cdot X_2 \cdot X_3 \cdot X_4 \cdot \dots \cdot X_n$$

$Z \rightarrow \text{LogNormal}$ as n increases



COV_x is a good measure of variability



COV and LogNormal Distributions

COV_x	Standard Deviation, $\ln x$		
	1	2	3
	68.3%	95.4%	99.7%
0.05	1.05	1.11	1.16
0.1	1.10	1.23	1.33
0.25	1.28	1.66	2.04
0.5	1.60	2.64	3.92
1	2.30	5.53	11.1

99.7% of the data is within a factor of ± 1.33 of the mean for a $\text{COV} = 0.1$



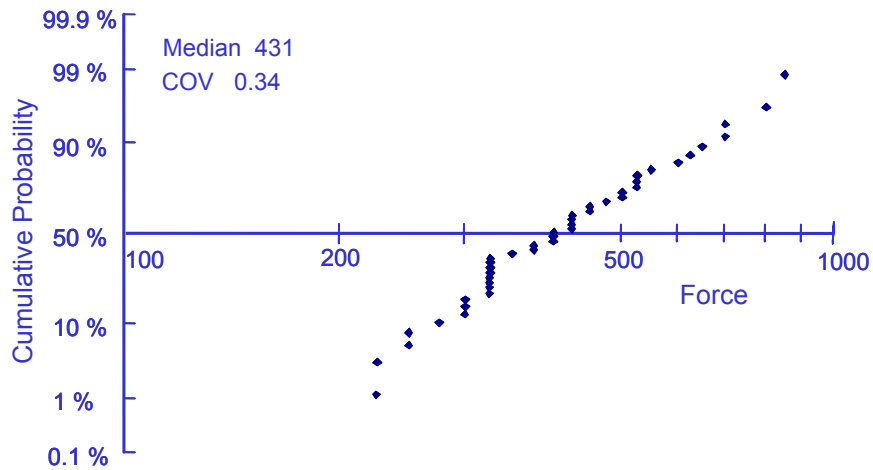
Variability in Service Loading

- Quantifying Loading Variability
 - Maximum Load
 - Load Range
 - Equivalent Stress



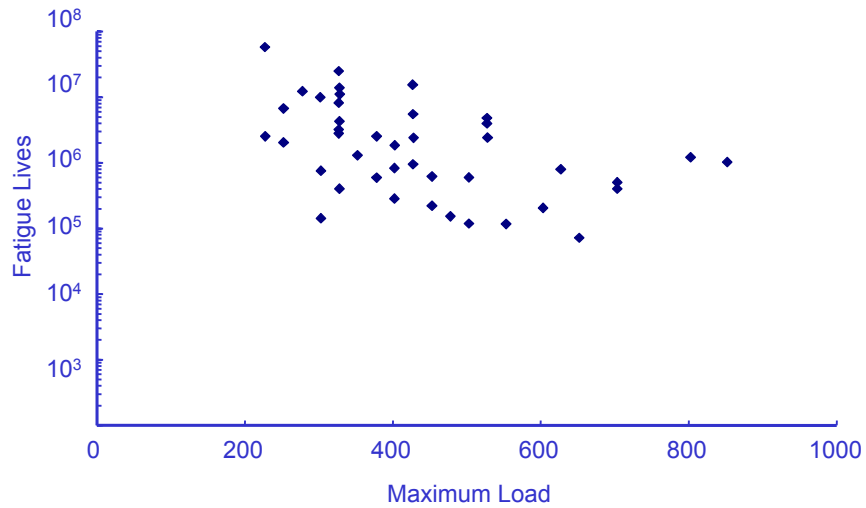
Maximum Force

Maximum force from 42 automobile drivers

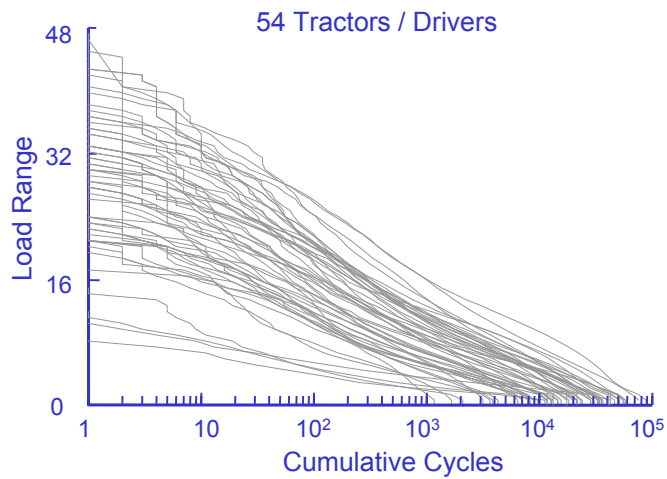




Maximum Load Correlation

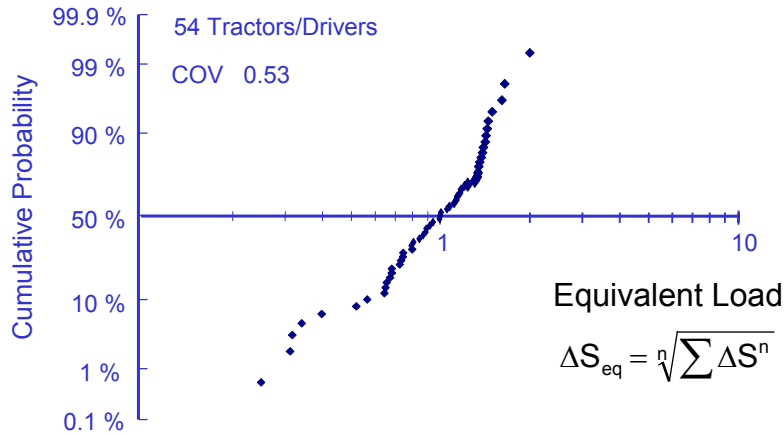


Loading Variability

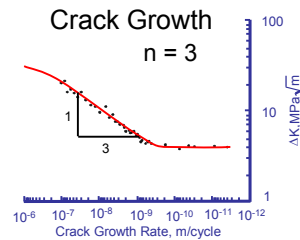
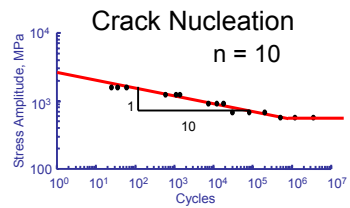
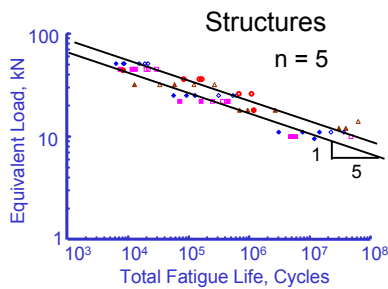




Variability in Loading



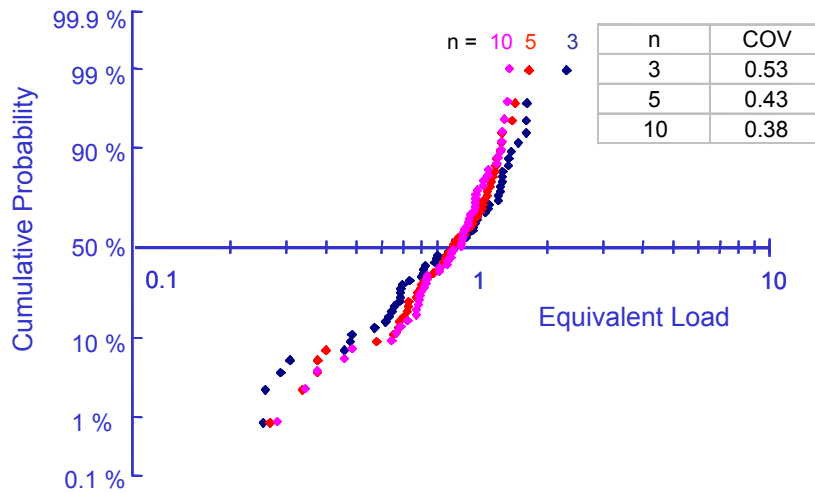
Mechanisms and Slopes



A combination of nucleation and growth



Effect of Slope on Variability

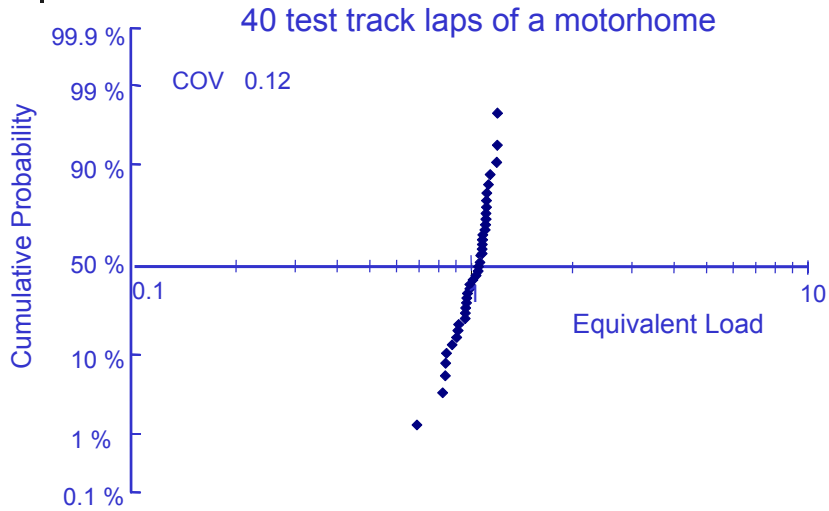


Loading History Variability

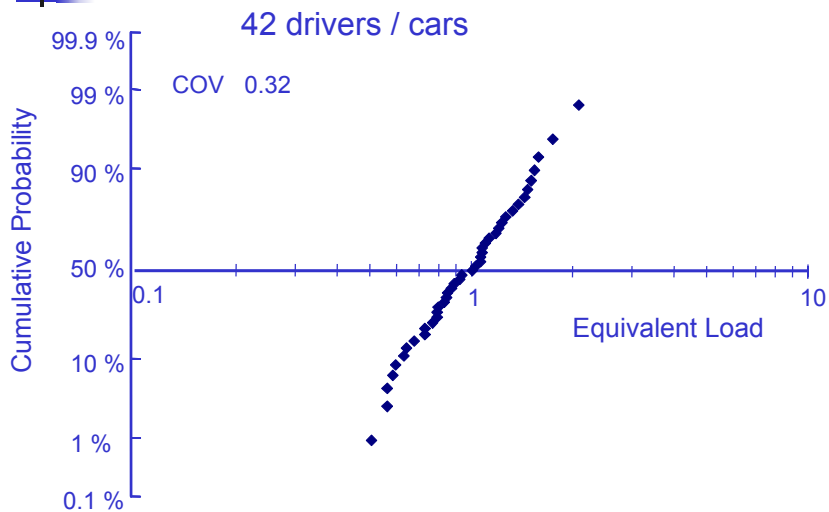
- Test Track
- Customer Service



Test Track Variability



Customer Usage Variability



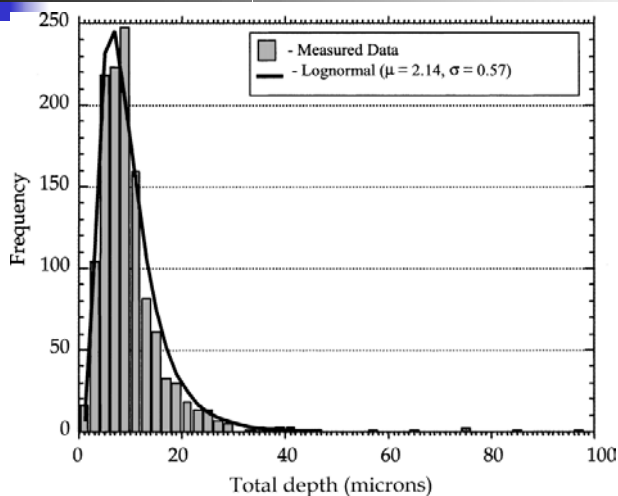


Variability in Environment

- Inclusions
- Pit depth



Inclusions That Initiated Cracks

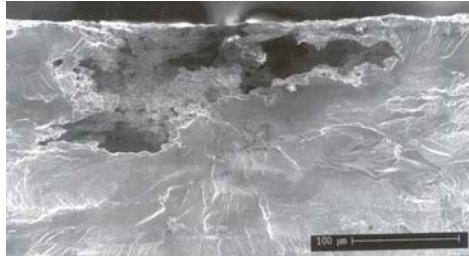
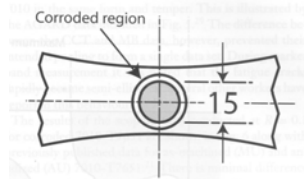


COV = 0.27

Barter, S. A., Sharp, P. K., Holden, G. & Clark, G. "Initiation and early growth of fatigue cracks in an aerospace aluminium alloy", *Fatigue & Fracture of Engineering Materials & Structures* **25** (2), 111-125.



Pits That Initiated Cracks



7010-T7651

Pre-corroded specimens

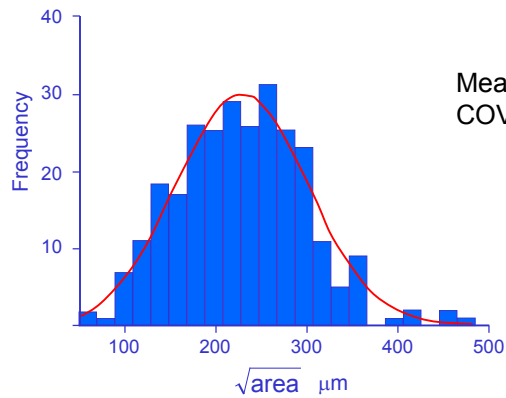
300 specimens

246 failed from pits

Crawford et al. "The EIFS Distribution for Anodized and Pre-corroded 7010-T7651 under Constant Amplitude Loading"
Fatigue and Fracture of Engineering Materials and Structures, Vol. 28, No. 9 2005, 795-808

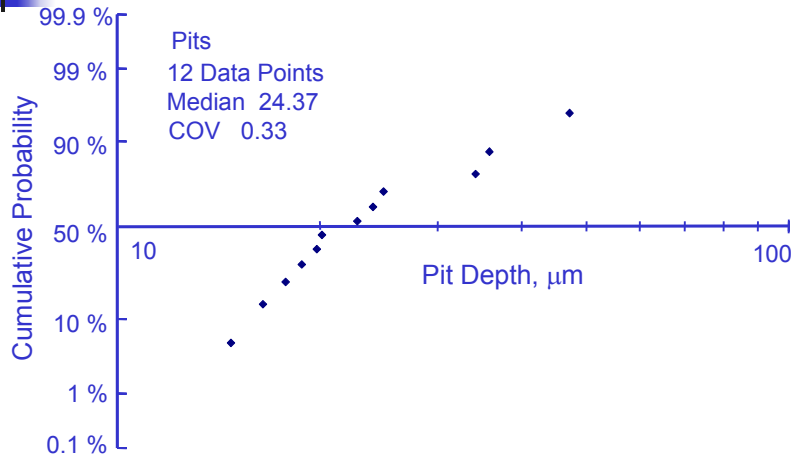


Pit Size Distribution





Pit Depth Variability



Dolly, Lee, Wei, "The Effect of Pitting Corrosion on Fatigue Life"
Fatigue and Fracture of Engineering Materials and Structures, Vol. 23, 2000, 555-560

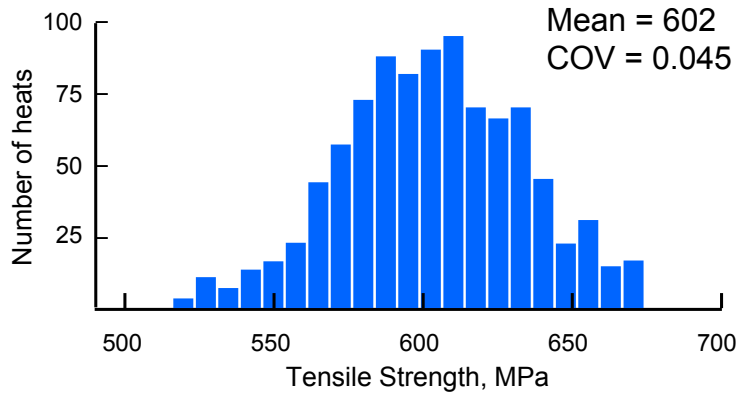


Variability in Materials

- Tensile Strength
- Fracture Toughness
- Fatigue
 - Fatigue Strength
 - Fatigue Life
- Strain-Life
- Crack Growth



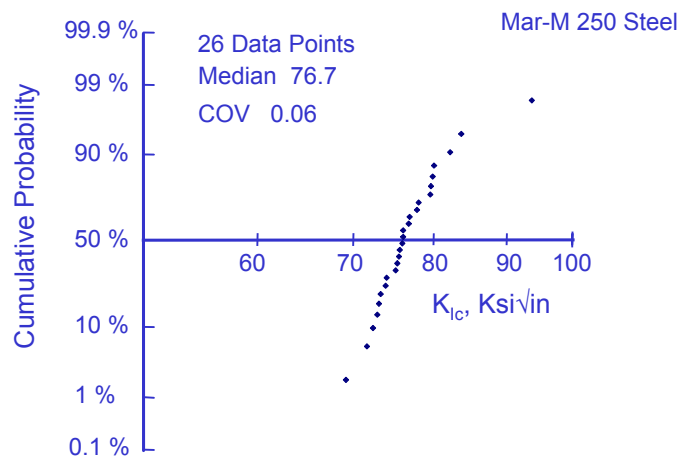
Tensile Strength - 1035 Steel



Metals Handbook, 8th Edition, Vol. 1, p64



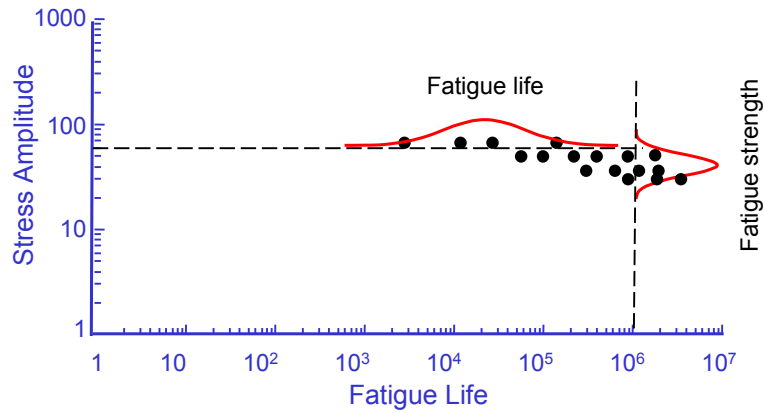
Fracture Toughness



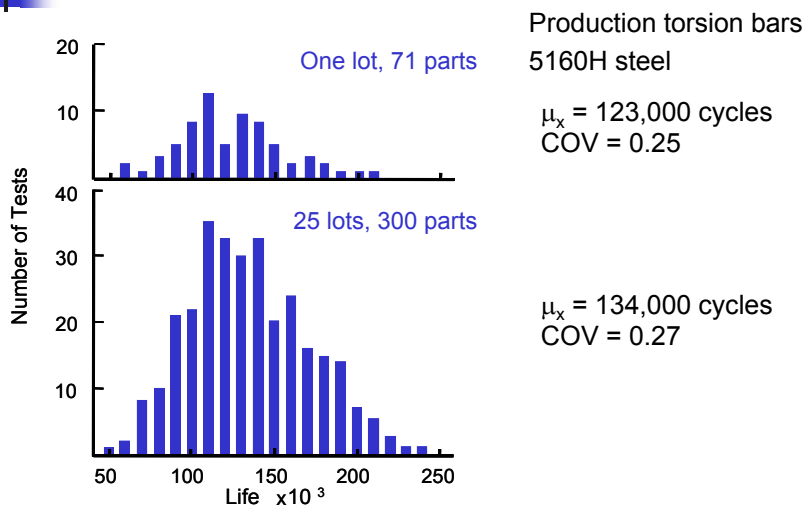
Kies, J.A., Smith, H.L., Romine, H.E. and Bernstein, H, "Fracture Testing of Weldments", ASTM STP 381, 1965, 328-356



Fatigue Variability



Fatigue Life Variability

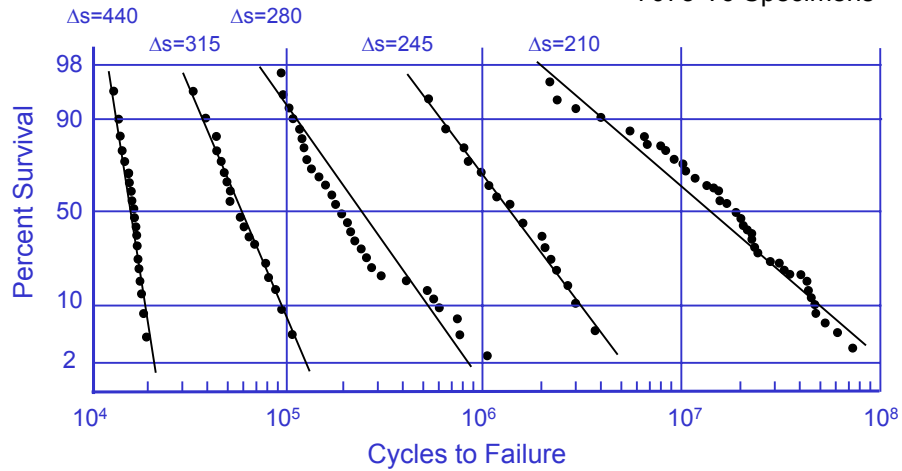


Metals Handbook, 8th Edition, Vol. 1, p219



Statistical Variability of Fatigue Life

7075-T6 Specimens



Sinclair and Dolan, "Effect of Stress Amplitude on the Variability in Fatigue Life of 7075-T6 Aluminum Alloy" Transactions ASME, 1953



COV vs Fatigue Life

ΔS	\bar{X}	COV
440	14,000	0.12
315	25,000	0.38
280	220,000	0.70
245	1,200,000	0.67
210	12,000,000	1.39



Variability in Fatigue Strength

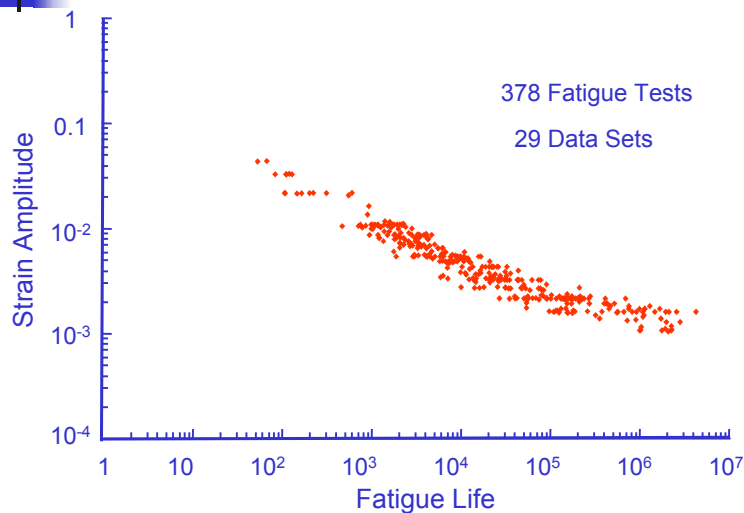
$$\frac{\Delta S}{2} = S'_f (N_f)^b \quad b \approx -0.085$$

$$\text{COV } C = \sqrt{\prod_{i=1}^n (1 + C_{X_i}^2)^{a_i^2} - 1}$$

$$C_{S'_f} = \sqrt{(1 + 1.39^2)^{(-0.085)^2} - 1} = 0.088$$

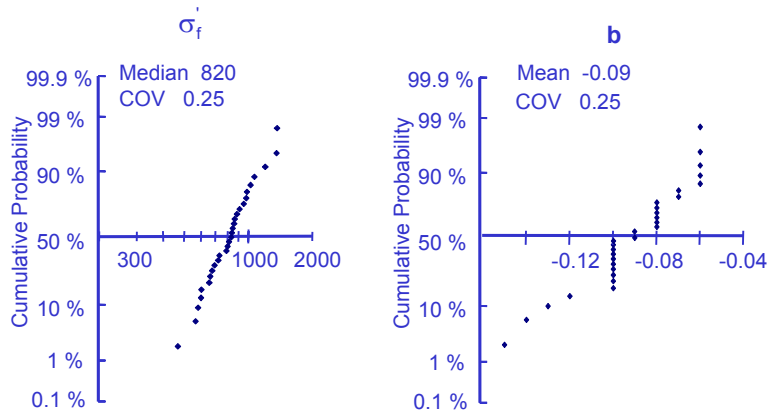


Strain Life Data for 950X Steel

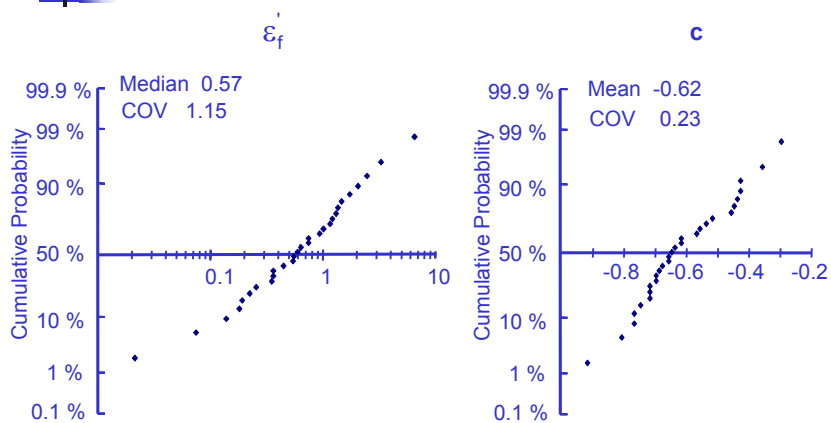




29 Individual Data Sets



29 Individual Data Sets (continued)



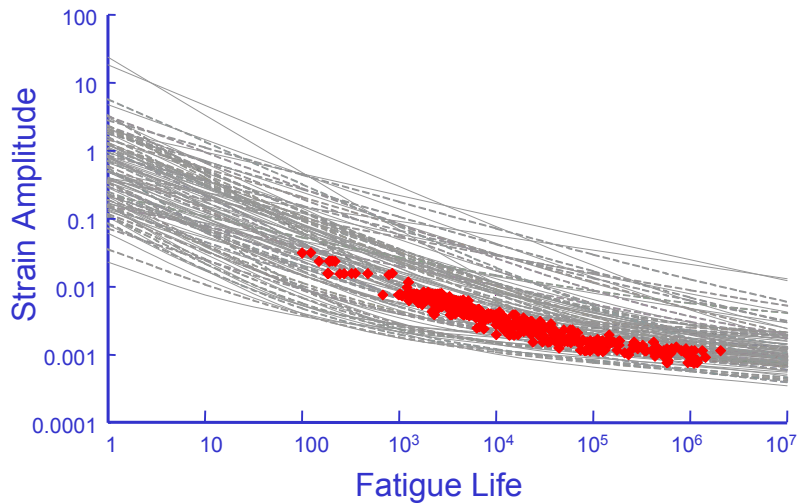


Input Data Simulation

$$\frac{\Delta \varepsilon}{2} = \frac{\sigma'_f(L, \mu_{\sigma_f}, \sigma_{\sigma_f})}{E} (2N_f)^{b(N, \mu_b, \sigma_b)} + \varepsilon'_f(L, \mu_{\varepsilon_f}, \sigma_{\varepsilon_f}) (2N_f)^{c(N, \mu_c, \sigma_c)}$$

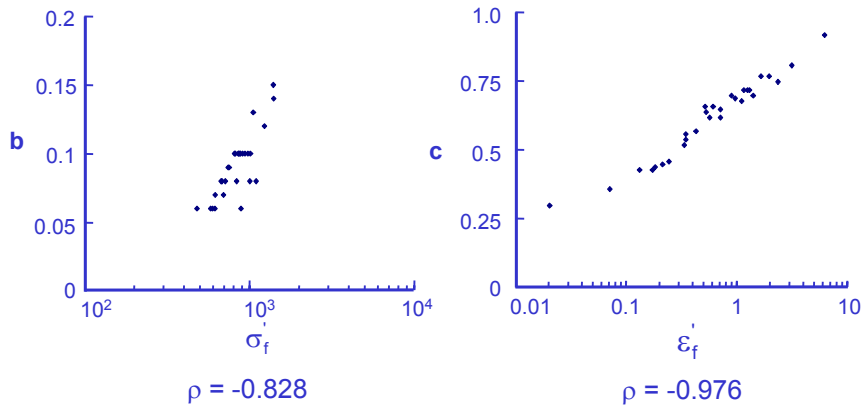


Simulation Results





Correlation



Generating Correlated Data

$$z_1 = \Phi(\text{rand}()) \quad z_1 = N(0,1)$$

$$z_2 = \Phi(\text{rand}())$$

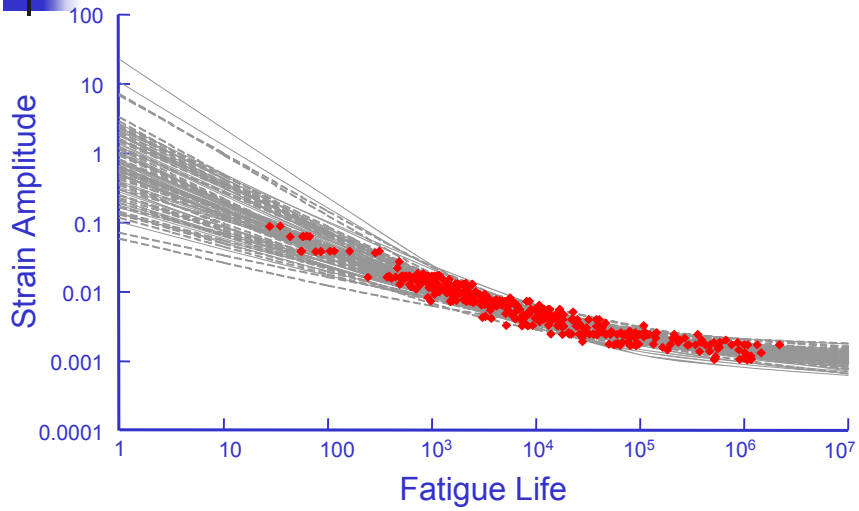
$$z_3 = z_1\rho + z_2\sqrt{1-\rho^2}$$

$$\sigma_f' = \exp(\mu_{\ln\sigma_f'} + \sigma_{\ln\sigma_f'} z_1)$$

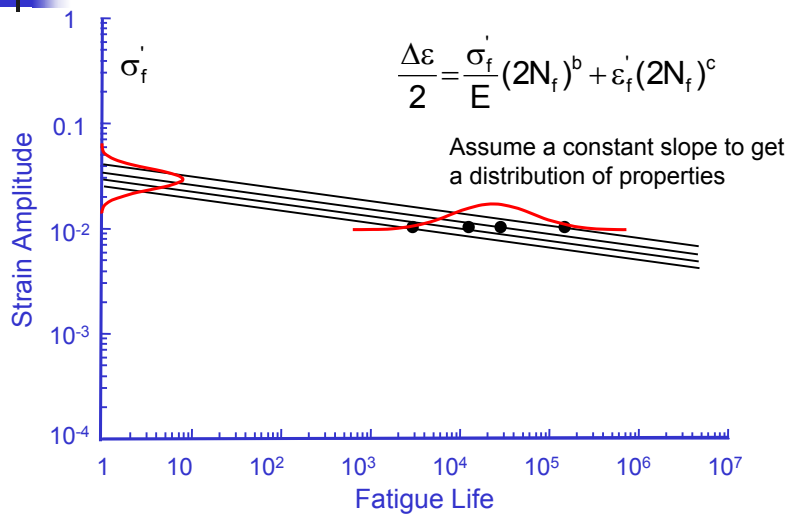
$$b = \mu_b + \sigma_b z_3$$



Correlated Properties

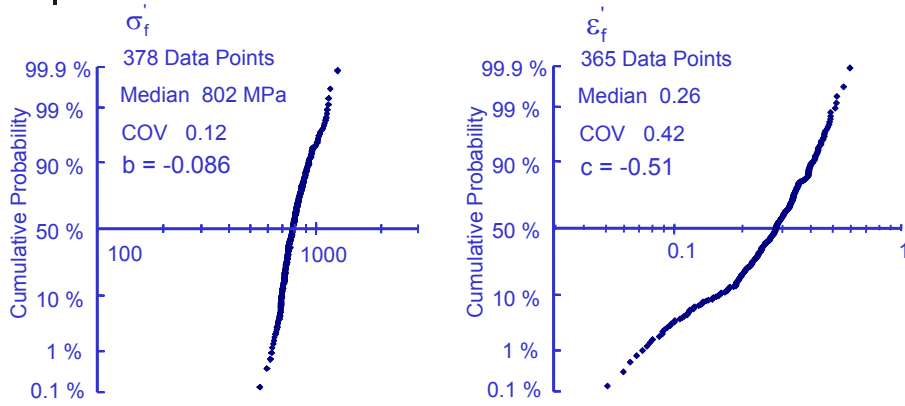


Curve Fitting

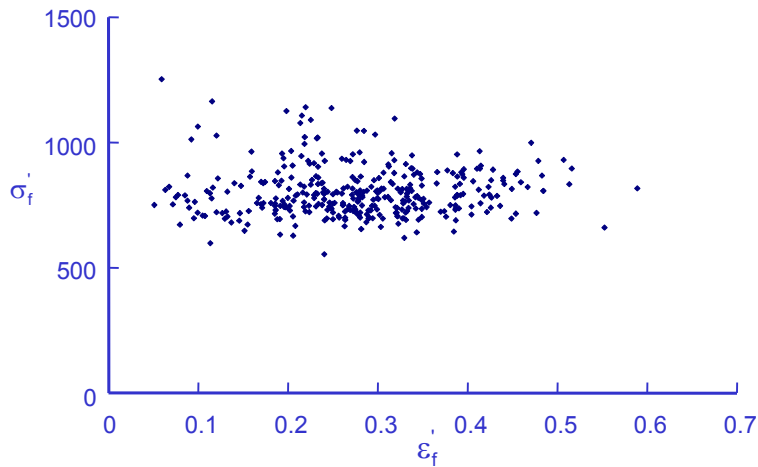




Property Distribution

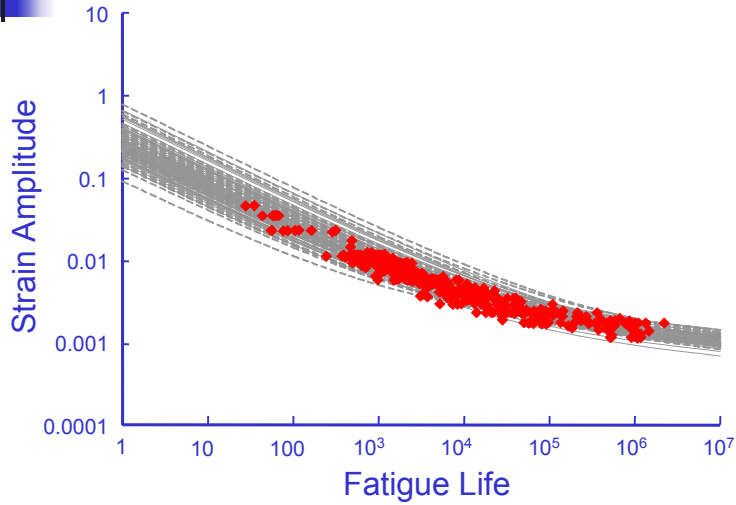


Correlation

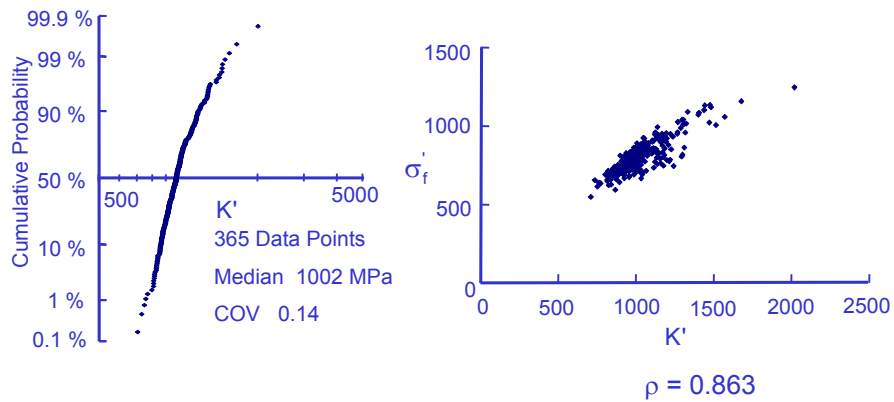




Simulation

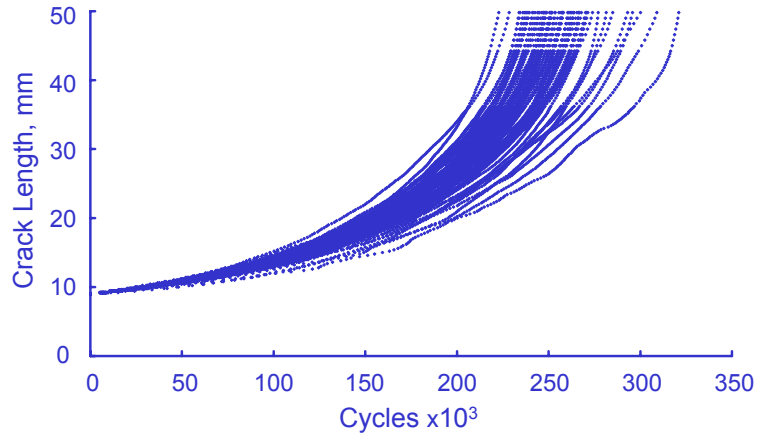


Strength Coefficient





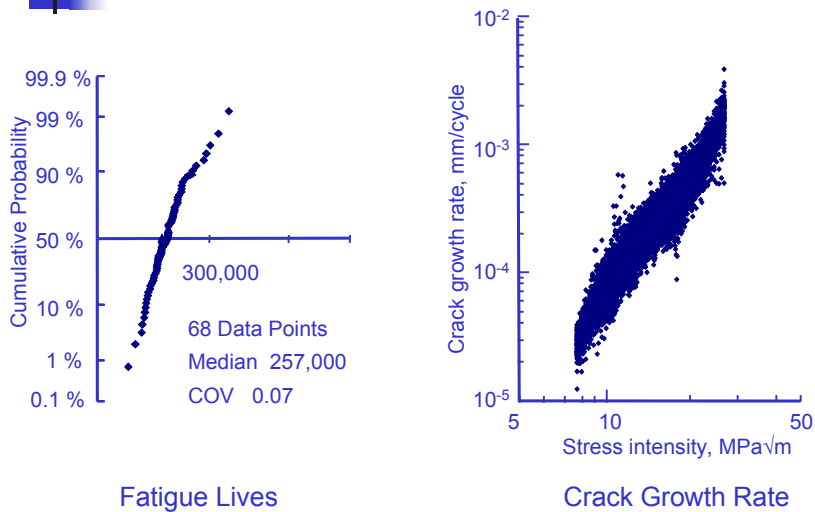
Crack Growth Data



Virkler, Hillberry and Goel, "The Statistical Nature of Fatigue Crack Propagation", Journal of Engineering Materials and Technology, Vol. 101, 1979, 148-153



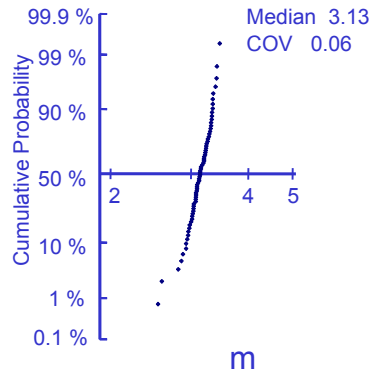
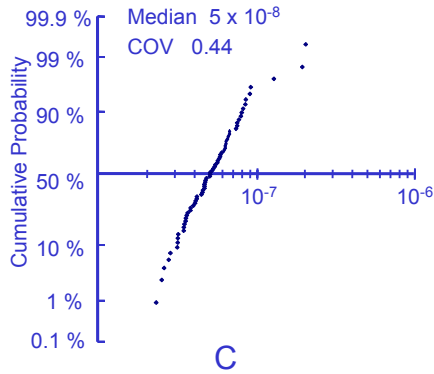
Crack Growth Rate Data





Crack Growth Properties

$$\frac{da}{dN} = C \Delta K^m$$



Beware of Correlated Variables

$$N_f = \frac{a_f^{1-m/2} - a_i^{1-m/2}}{C \Delta S^m \pi^2 (1-m/2)}$$

N_f and C are linearly related and should have the same variability, but

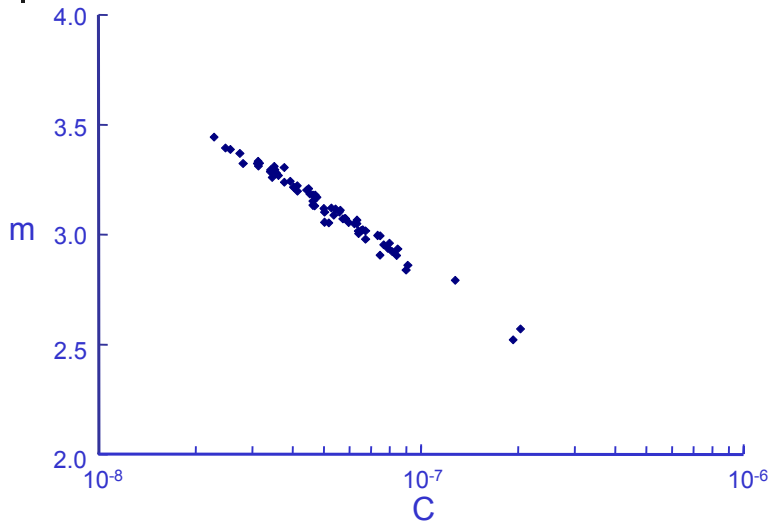
$$COV_{N_f} = 0.07$$

$$COV_C = 0.44$$

because C and m are correlated.

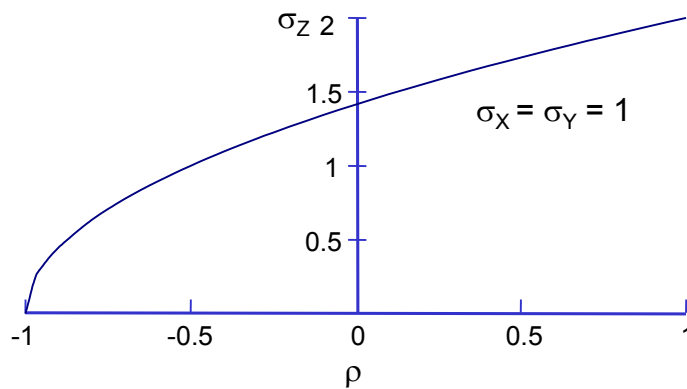


Correlation



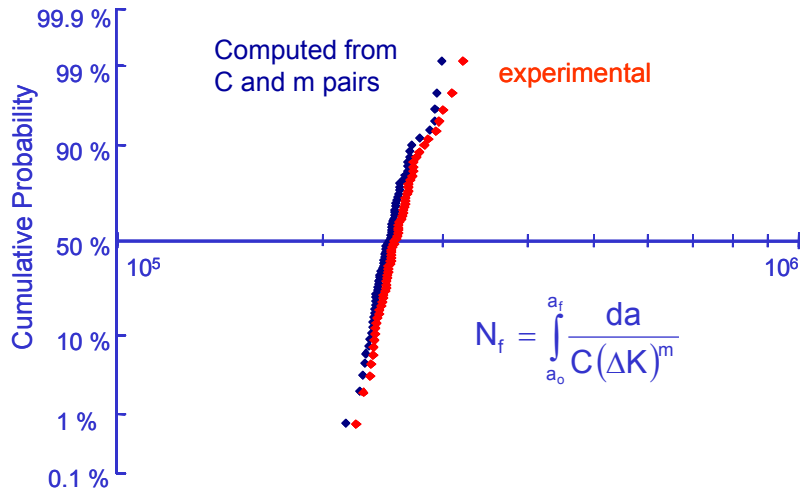
Correlated Variables

$$\sigma_Z = \sqrt{\sigma_X^2 + 2\rho\sigma_X\sigma_Y + \sigma_Y^2}$$





Calculated Lives

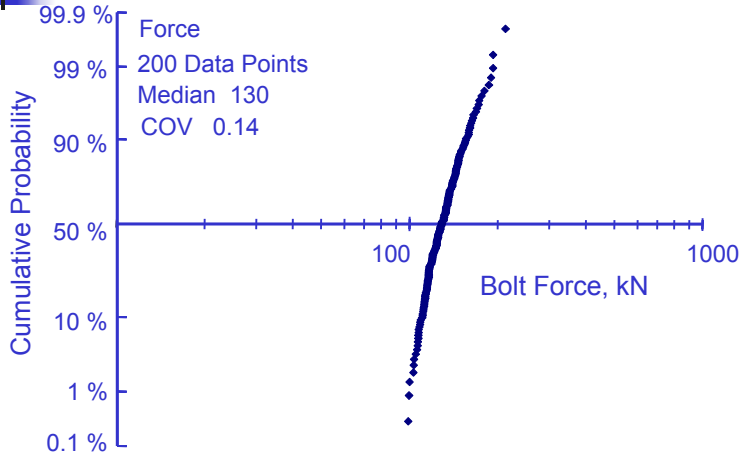


Manufacturing/Processing Variability

- Bolt Forces
- Surface Finish
- Drilled Holes



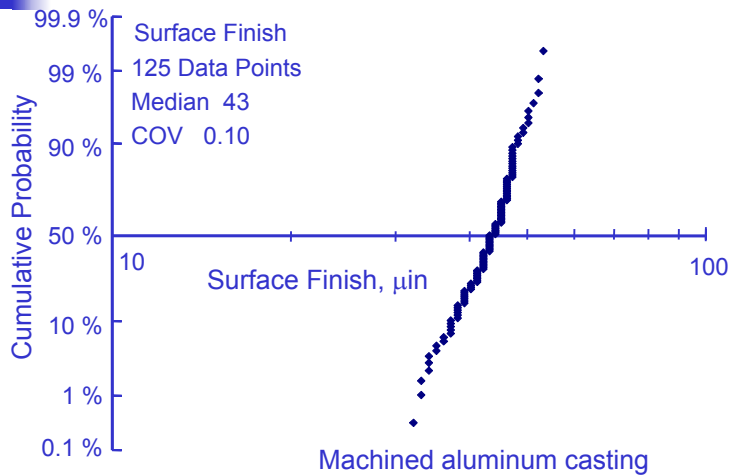
Variability in Bolt Force



Preload force in bolts tightened to 350 Nm



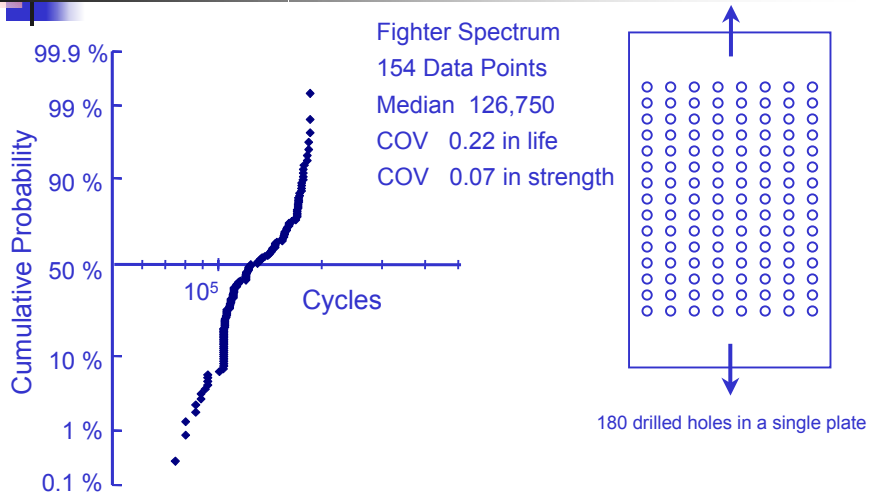
Surface Roughness Variability



Machined aluminum casting



Drilled Holes



From: J.P. Butler and D.A. Rees, "Development of Statistical Fatigue Failure Characteristics of 0.125-inch 2024-T3 Aluminum Under Simulated Flight-by-Flight Loading," ADA-002310 (NTIS no.), July 1974.

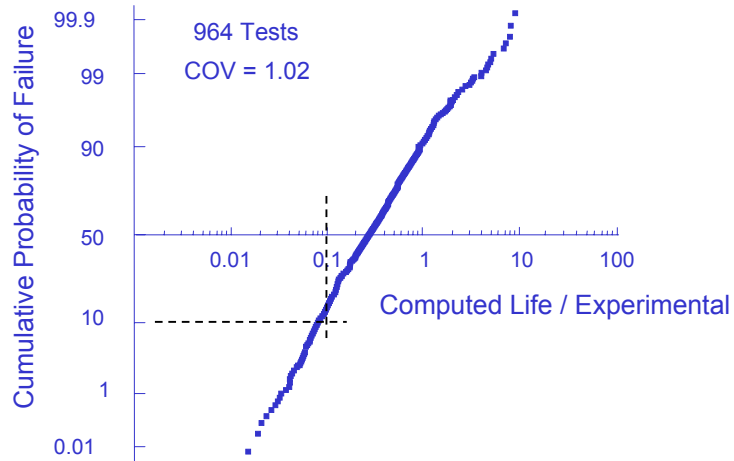


Analysis Uncertainty

- Miners Linear Damage rule
- Strain Life Analysis



Miners Rule

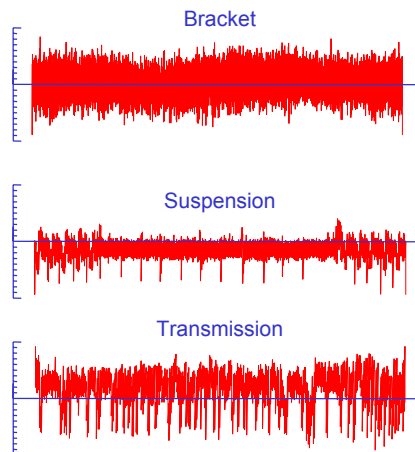
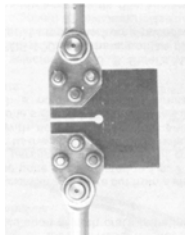


A safety factor of 10 in life would result in a 10% chance of failure

From Erwin Haibach "Betriebsfestigkeit", Springer-Verlag, 2002



SAE Specimen

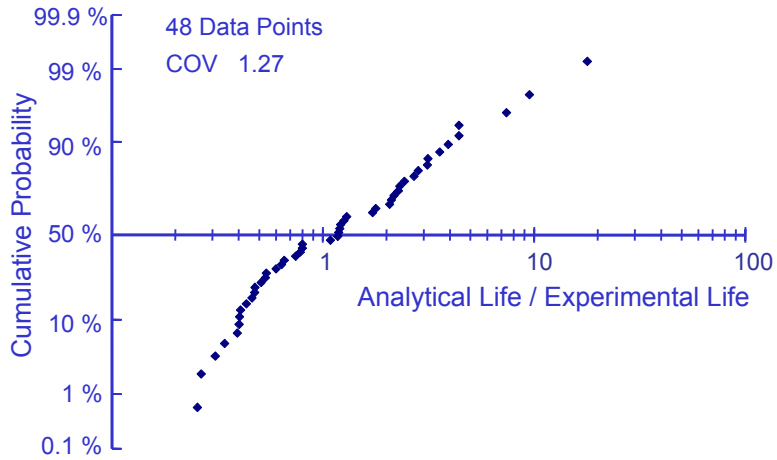


Fatigue Under Complex Loading: Analysis and Experiments, SAE AE6, 1977

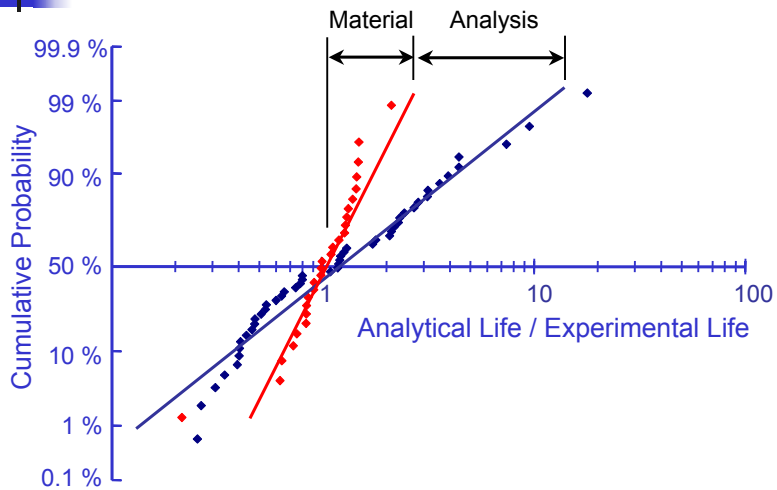


Analysis Results

Strain-Life analysis of all test data



Material Variability



Strain-Life back calculation of specimen lives



Modeling Uncertainty

Analysis Uncertainty $C_U = ?$

The variability in reproducing the original strain life data from the material constants is $C_M \sim 0.44$

$$\text{COV } C = \sqrt{\prod_{i=1}^n (1 + C_{X_i}^2)^{a_i^2} - 1}$$

$$1 + C_U^2 = \frac{1 + C_{N_f}^2}{1 + C_M^2}$$

$$C_U = 1.09$$

90% of the time the analysis is within a factor of 3 !

99% of the time the analysis is within a factor of 10 !



Variability from Multiple Sources

$$\text{COV } C = \sqrt{\prod_{i=1}^n (1 + C_{X_i}^2)^{a_i^2} - 1}$$

Suppose we have 4 variables each with a COV = 0.1

The combined variability is COV = 0.29

Suppose we reduce the variability of one of the variables to 0.05

The combined variability is now COV = 0.27

If all of the COV's are the same, it doesn't do any good to reduce only one of them, you must reduce all of them !



Variability from Multiple Sources

$$\text{COV } C = \sqrt{\prod_{i=1}^n (1 + C_{X_i}^2)^{a_i^2} - 1}$$

Suppose we have 3 variables each with a COV = 0.1 and one with COV = 0.4

The combined variability is COV = 0.65

Suppose we reduce the variability of these variables to 0.05

The combined variability is now COV = 0.60

If one of the COV's is large, it doesn't do any good to reduce the others, you must reduce the largest one !



Variability Summary

	Source	COV
Stress	Service Loading	0.5
	Environment	0.3
Strength	Materials	0.1
	Manufacturing	0.1
	Surface Finish	0.1

Fatigue Lives	1.0
Analysis Uncertainty	1.0

$$\text{Fatigue life} \propto \left(\frac{\text{Strength}}{\text{Stress}} \right)^5$$



Variability and Uncertainty

Variability: Every apple on a tree has a different mass.

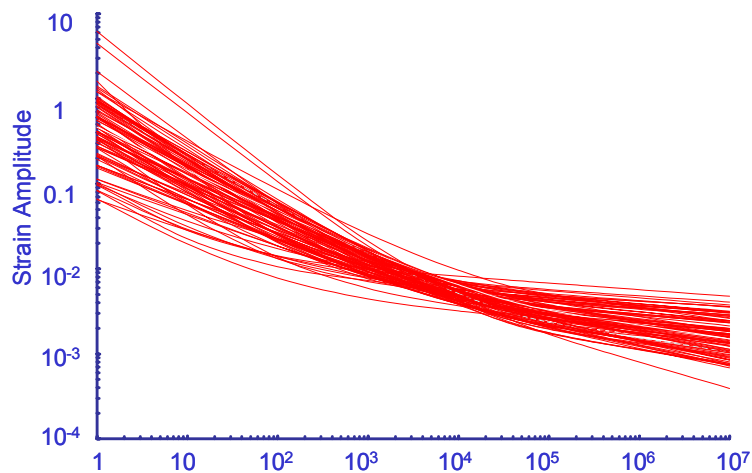
Uncertainty: The variety of the apple is unknown.

Variability: Multiple samples of the same material

Uncertainty: What is the material

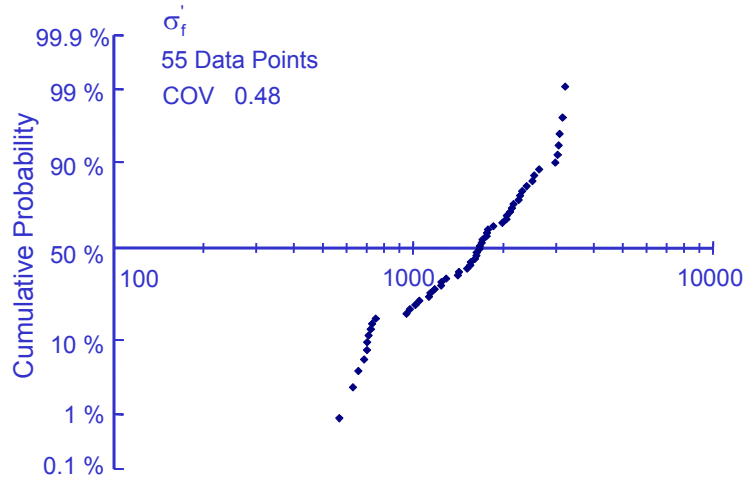


Strain Life Data for 93 Steels

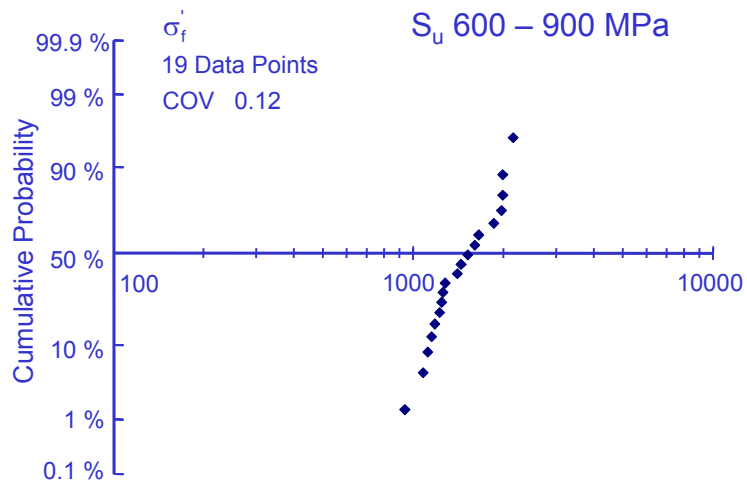




Uncertainty for all Steels



Uncertainty for Structural Steels





Variability and Uncertainty

Fatigue Strength Coefficient

	Variability	Uncertainty	Combined
All Steels	0.12	0.48	0.75
Structural Steel	0.12	0.12	0.24



Quiz

At my last seminar everyone hit a golf ball and we recorded the maximum acceleration.

What is the expected variability ?





Results

	12.9
	7.7
	5.88
	11.1
	15.5
	10.3
	18.1
μ	<u>11.64</u>
σ	4.26
COV	0.37

Probabilistic Aspects of Fatigue

Case Studies



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering



Probabilistic Aspects of Fatigue

- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- Analysis Methods
- Characterizing Variability
- **Case Studies**
- FatigueCalculator.com
- GlyphWorks



Case Studies

- DARWIN
 - Southwest Research
- Bicycle
- Loading Histories



A Software Framework for Probabilistic Fatigue Life Assessment

ASTM Symposium on
Probabilistic Aspects of Life Prediction
Miami Beach, Florida
November 6-7, 2002

R. C. McClung, M. P. Enright, H. R. Millwater*,
G. R. Leverant, and S. J. Hudak, Jr.
Southwest Research

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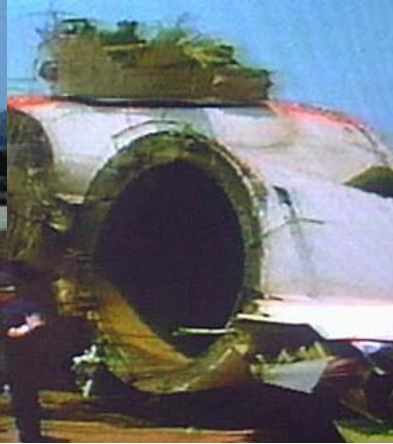
Motivation





UAL Flight 232

July 19, 1989



Turbine Disk Failure

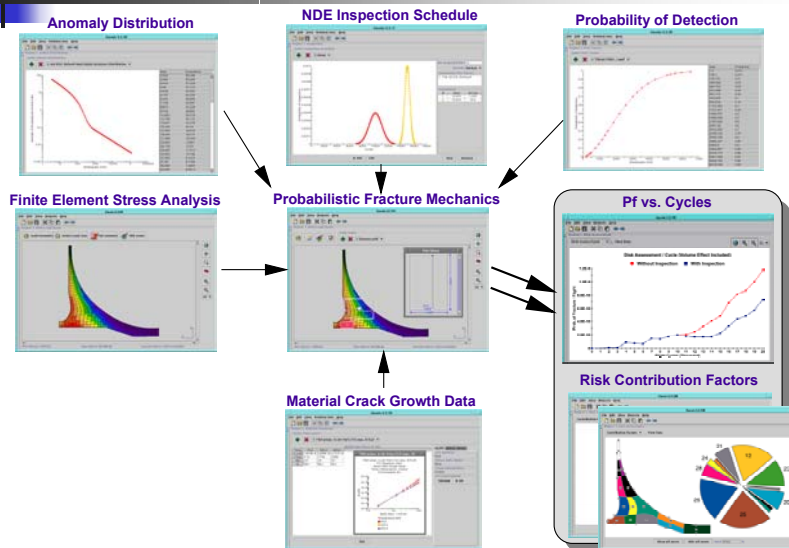
Anomalies in titanium engine disks

- Hard Alpha
- Very rare
- Can cause failure
- Not addressed by safe life methods
- Enhanced life management process
- Requested by FAA
- Developed by engine industry
- Probabilistic damage tolerance methods
- Supplement to safe life approach



SwRI and engine industry developed DARWIN with FAA funding

Probabilistic Damage Tolerance



Zone-Based Risk Assessment

Define zones based on similar stress, inspection, anomaly distribution, lifetime

Total probability of fracture for zone:

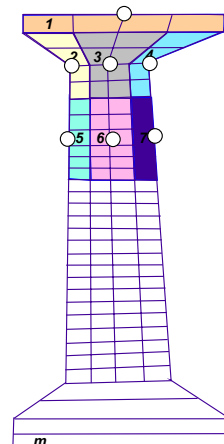
(probability of having a defect) x (POF given a defect)

Defect probability determined by anomaly distribution, zone volume

POF assuming a defect computed with Monte Carlo sampling or advanced methods

POF for disk obtained by summing zone probabilities

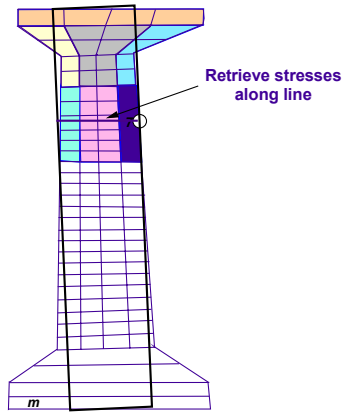
As individual zones become smaller (number of zones increases), risk converges down to "exact" answer



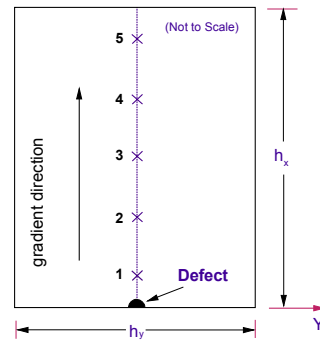


Fracture Mechanics Model of Zone

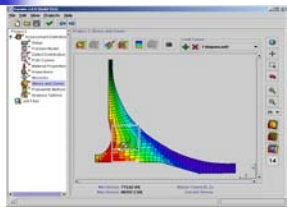
Finite Element Model



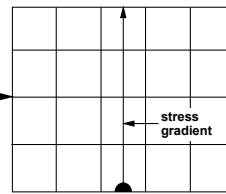
Fracture Mechanics Model



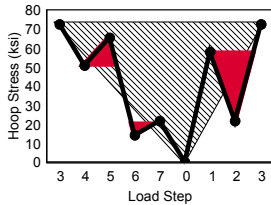
Stress Processing



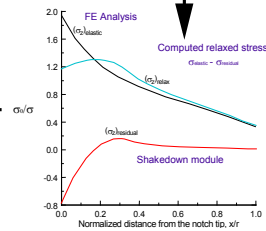
FE Stresses and plate definition



Stress gradient extraction



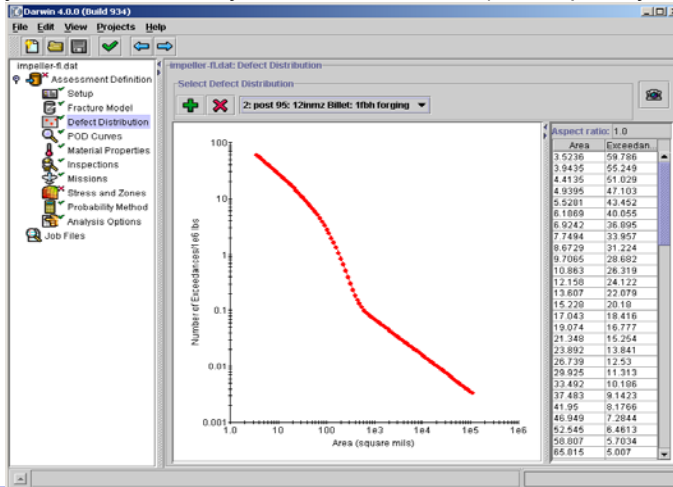
Rainflow stress pairing



Residual stress analysis

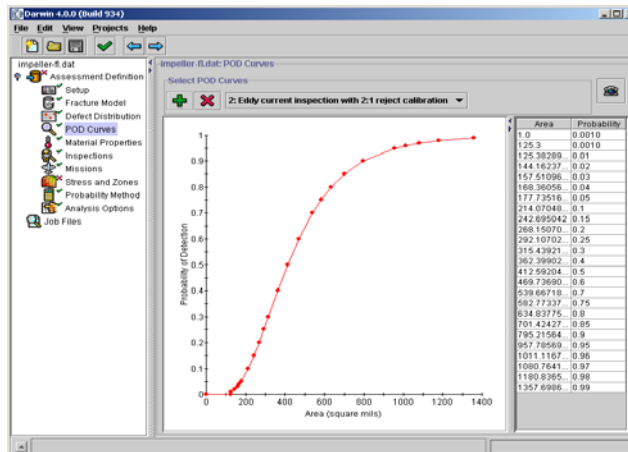
Anomaly Distribution

of anomalies per volume of material as function of defect size
 Library of default anomaly distributions for HA (developed by RISC)



Probability of Detection Curves

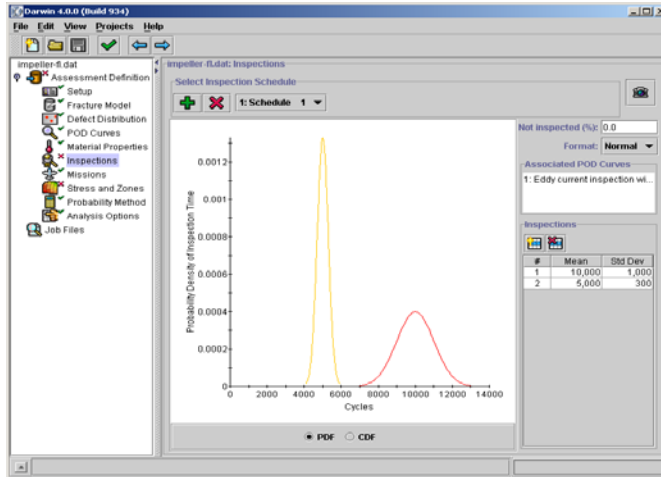
Define probability of NDE flaw detection as function of flaw size
 Can specify different PODs for different zones, schedules
 Built-in POD library or user-defined POD



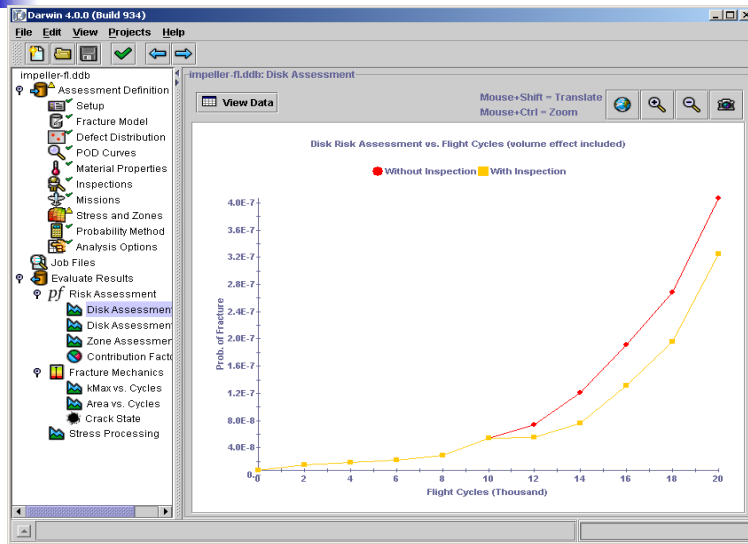


Random Inspection Time

“Opportunity Inspections” during on-condition maintenance
Inspection time modeled with Normal distribution or CDF table



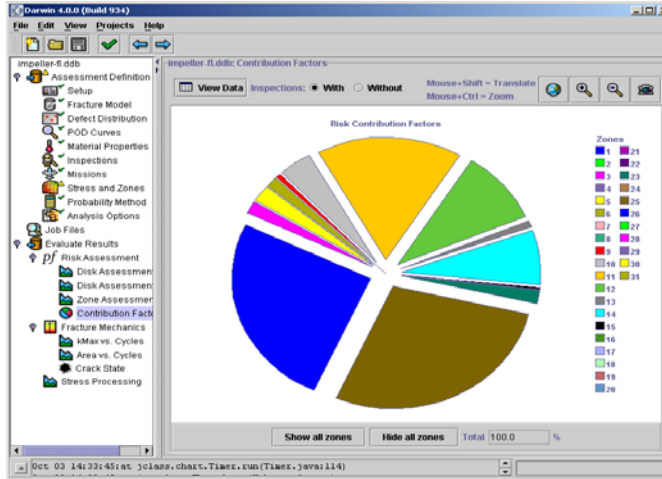
Output: Risk vs. Flight Cycles





Output: Risk Contribution Factors

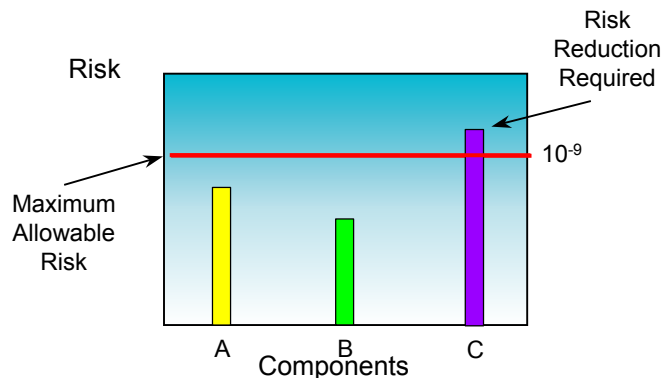
Identify regions of component with highest risk



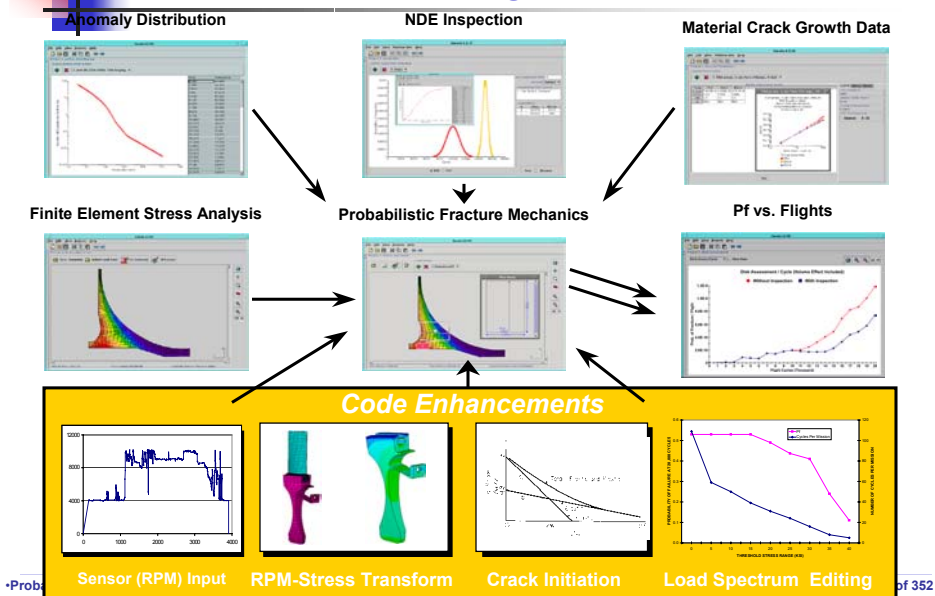
Implementation in Industry

FAA Advisory Circular 33.14 requests risk assessment be performed for all new titanium rotor designs

Designs must pass design target risk for rotors



DARWIN for Prognosis Studies



Three Sources of Variability

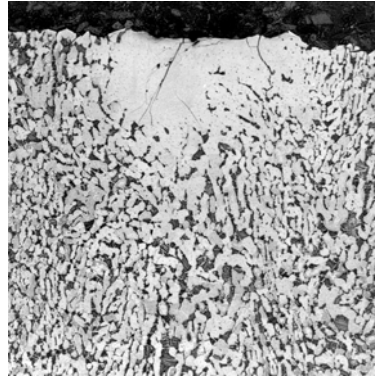
- Anomaly size (initial crack size)
- FCG properties (life scatter)
- Mission histories (stress scatter)



Hard Alpha Defects in Titanium

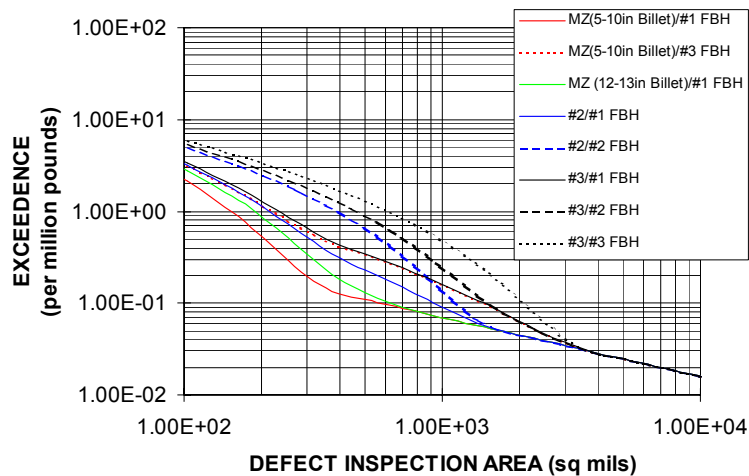
Initial DARWIN focus on Hard Alpha

- Small brittle zone in microstructure
- Alpha phase stabilized by N accidentally introduced during melting
- Cracks initiate quickly
- Extensive industry effort to develop HA distribution



Resulting Anomaly Distributions

Post 1995 Triple Melt/Cold Hearth + Vacuum Arc Remelt



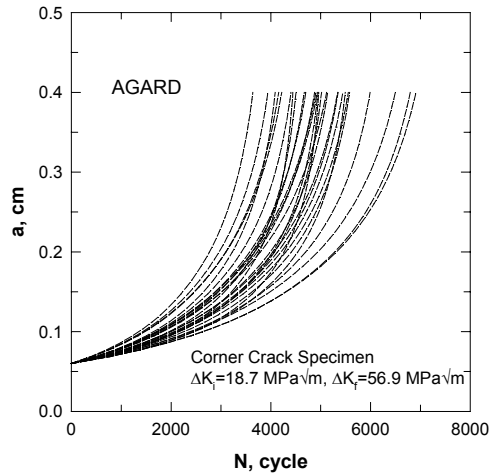


FCG Simulations for AGARD Data

Use individual fits to generate set of a vs. N curves for identical conditions

Characterize resulting scatter in total propagation life

Lognormal distribution appropriate in most cases



Engine Usage Variability

Stress/Speed:

$$\Delta\sigma \propto (\text{RPM})^2$$

Total Cyclic Life (LCF):

$$N_f = N_i + N_p$$

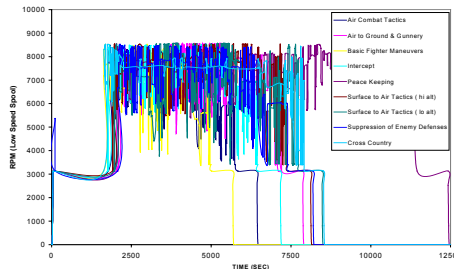
$$N_i \propto \Delta\sigma^{-3-5}$$

$$N_p \propto \Delta\sigma^{-3-4}$$

Life/Speed:

$$N_f \propto (\text{RPM})^6$$

Component life is very sensitive to actual usage

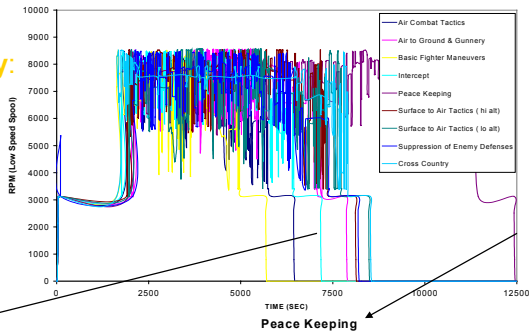




Usage Variability

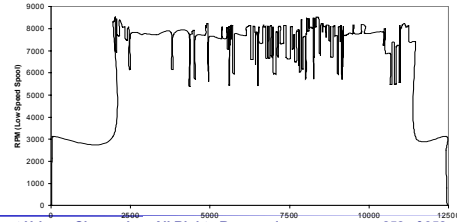
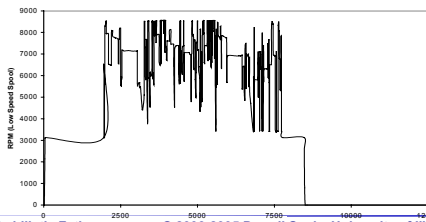
Components of Usage Variability:

- Mission type
- Mission-to-mission variability
- Mission mixing variability

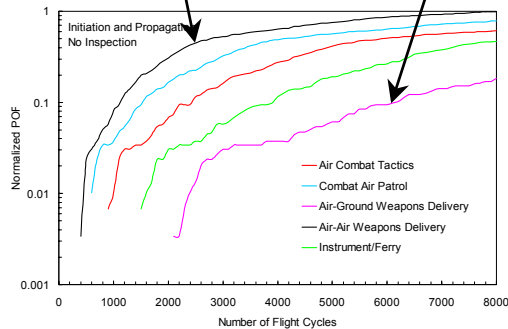
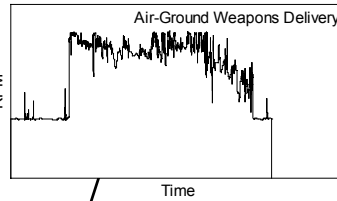
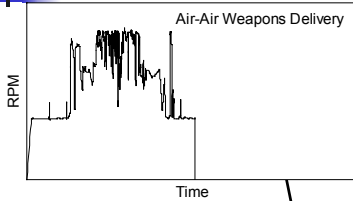


Surface to Air Tactics (Lo alt)

Peace Keeping



Variability in Mission Type



Web Site: www.darwin.swri.org

DARWIN
Design Assessment of Reliability
With INSpection

Winner R&D 100 Award

- The DARWIN computer program predicts the probability of fracture of aircraft turbine rotor disks
- DARWIN risk assessment considers finite element stress analysis, defect growth analysis, and nondestructive inspection simulation
- DARWIN identifies the most likely failure regions and risk reduction associated with single or multiple inspections

DARWIN is sponsored by the
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View a video describing the main features of Darwin. *RealPlayer 8 Basic* is required to view the interactive video segments included in this tutorial. If you don't have the free *RealPlayer 8 Basic*, click [here to download](#)

[DARWIN Overview \(16 MB\)](#) [DARWIN GUI \(20 MB\)](#)

Download a printable Darwin brochure in Adobe PDF format. If you don't have the free Adobe Acrobat reader, click [here to download](#).

Fatigue Design and Reliability in the Automotive Industry

Fatigue Design and Reliability
ESIS Publication 23

J-J. Thomas, G. Perroud, A. Bignonnet and D. Monnet

PSA Peugeot Citroën

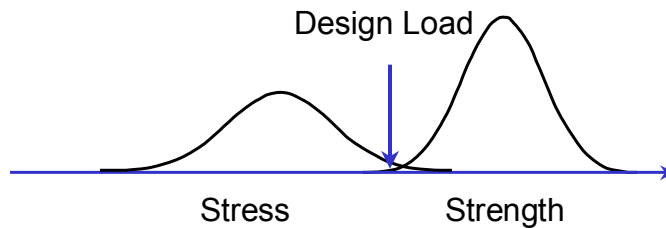


Fatigue Design at PSA

Fatigue design at PSA is done with a probabilistic approach that includes analysis of customer usage, production scatter, definition of the appropriate design loads and an acceptance testing criterion.



Stress-Strength Interference



$$\text{Reliability} = 1 - P(\text{Stress} > \text{Strength})$$

How do you really get these distributions ?

How do you establish a design load ?

How do you validate the design ?



Stress Distribution

Variability in loading has two components, how it is used and how it is driven.

Car Usage

Highway, city, fully loaded, empty etc.

Driving Style

passive, aggressive etc.

The usage of a car is independent of the owners driving style so that the distributions of car usage and driving style can be obtained separately.



Car Usage

Customer surveys

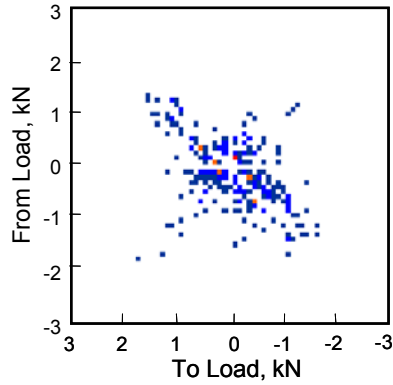
k	l		c_k %	r_{kl} %
1		Unloaded	27	
	1	Highway		10
	2	Good Road		25
	3	Mountain		40
	4	City		25
2		Half Load	58	
	1	Highway		5
	2	Good Road		30
	3	Mountain		30
	4	City		35
3		Fully Loaded	15	
	1	Highway		15
	2	Good Road		25
	3	Mountain		40
	4	City		20

12 Customer Usage Categories



Owner Behavior

Instrumented Vehicles



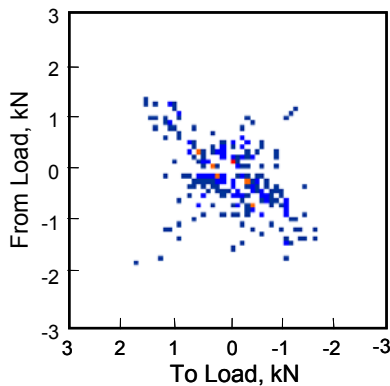
Extensive field testing for each customer usage category produces a large number of histograms.

Let the usage histogram be denoted U_{kl}



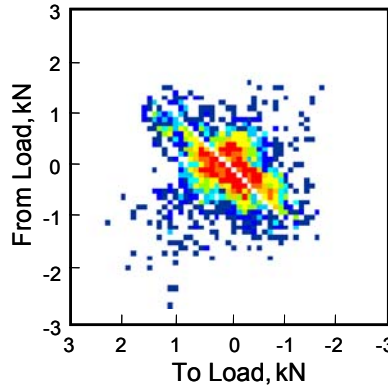
Extrapolation

U_{kl}



Measured data, e.g. 1,000 km

U^*_{kl}



Extrapolated data, 300,000 km



Virtual Customers

Thousands of virtual customers can now be generated by combining customer usage with driving style.

$$[U^i] = N \sum c_k r_{kl} [U_{kl}^*]$$

$[U^i]$ rainflow histogram for an individual car

N kilometers

c_k car loading, %

r_{kl} car usage, %

$[U_{kl}^*]$ distributions of rainflow histograms for different car usage classifications



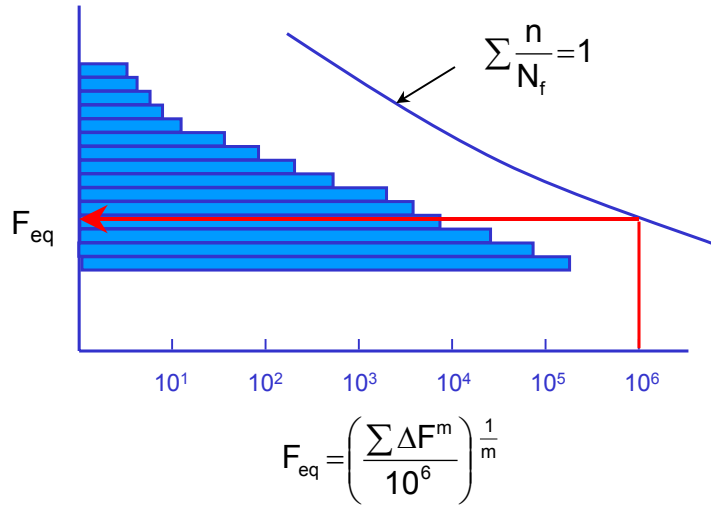
Design Loads

Initial design is done on the basis of a single constant amplitude load, F_{eq}

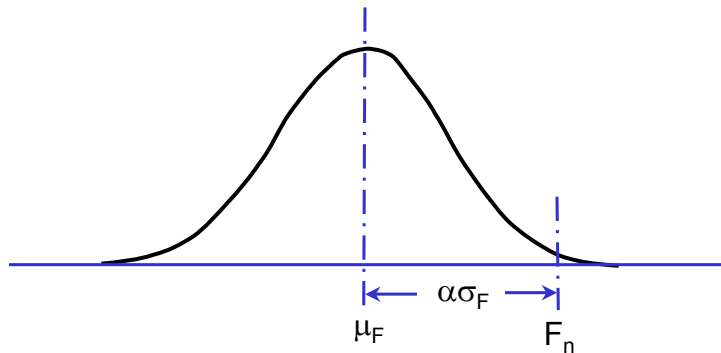
Find a constant amplitude load and number of cycles that will produce the same fatigue damage as the customer operating a car for the design life.



Equivalent Fatigue Loading



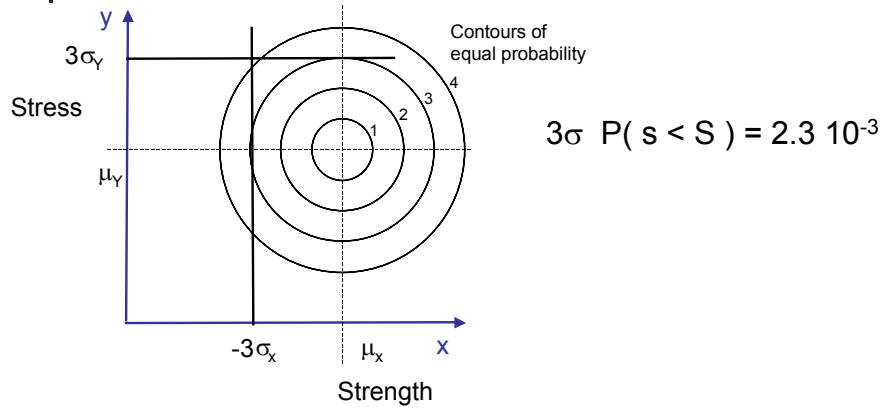
Distribution of F_{eq}



Design Customer: $F_n = \mu_F + \alpha \sigma_F$



Choosing α

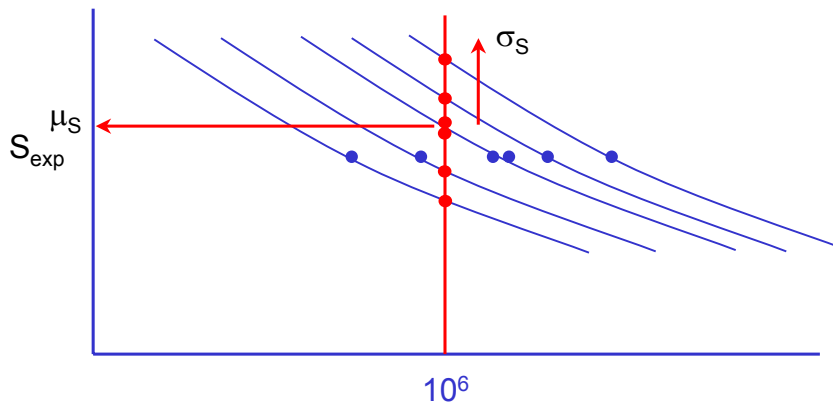


If we use 3σ on both stress and strength

$$P(\text{failure}) = P(\Sigma \geq s \cap s \leq S) = 5.3 \cdot 10^{-6} \approx 4.5\sigma$$

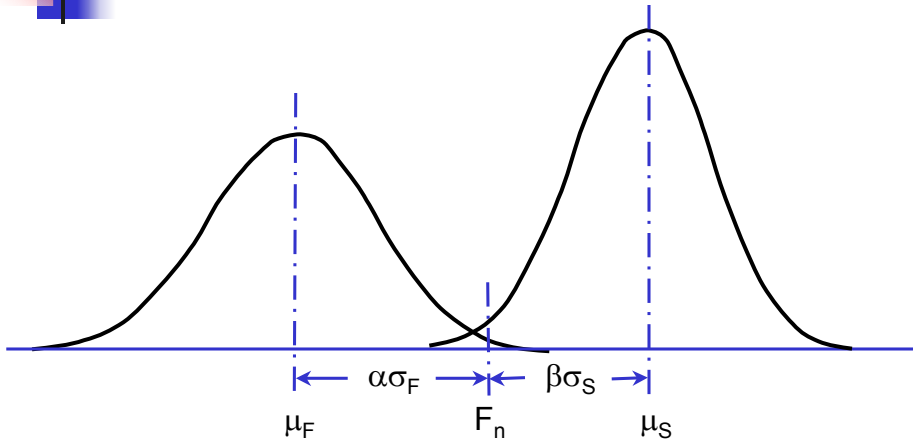


Distribution of Strength





What Strength is Needed?



Some Statistics

$$Z = S - F$$

Suppose we want a probability of failure of 1 in 50,000

$$P_f = 2 \times 10^{-5}$$

$$\Phi^{-1}(P_f) = 4.1 = \frac{\mu_Z}{\sigma_Z} = \frac{\mu_S - \mu_F}{\sqrt{\sigma_S^2 + \sigma_F^2}}$$

$$F_n = \mu_S (1 - \beta \text{COV}_S)$$

$$F_n = \mu_F (1 + \alpha \text{COV}_F)$$



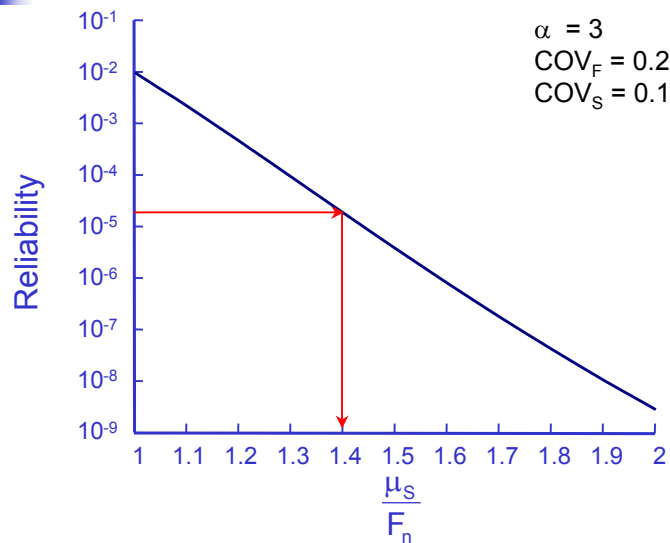
Mean Component Strength

$$\Phi^{-1}(P_f) = 4.1 = \frac{\mu_Z}{\sigma_Z} = \frac{\mu_S - \mu_F}{\sqrt{\sigma_S^2 + \sigma_F^2}}$$

$$\Phi^{-1}(P_f) = \frac{\frac{\mu_S}{F_n} - \frac{1}{1 + \alpha \text{COV}_F}}{\sqrt{\left(\frac{\mu_S}{F_n} \text{COV}_S\right)^2 + \left(\frac{\text{COV}_F}{1 + \alpha \text{COV}_F}\right)^2}}$$

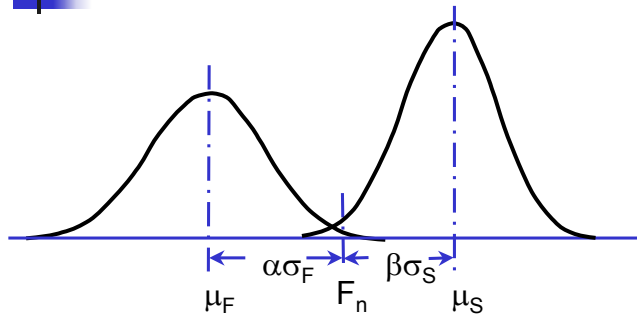


Reliability





Mean Strength



$\mu_S = 1.4 F_n$ for a reliability of 2×10^{-5}

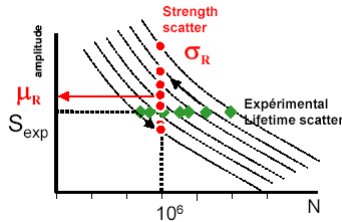
$$\beta = \frac{1 - \frac{F_n}{\mu_S}}{\text{COV}_S} = 2.85$$



Design

- Design calculations are made with loads F_n
 - Large database relates F_n to vehicle parameters so that a new vehicle can be designed from historical measurements
- Fatigue calculations are made with material properties β standard deviations from the mean

Component Testing



Component strength not life !

Test load is frequently higher than F_n to get failures near the design life

Interpreting the Test Data

Component tests are done with small sample sizes

Confidence limits

$$\frac{(N-1)s_x^2}{\chi^2_{1-\alpha, N-1}} \leq \sigma_x^2$$

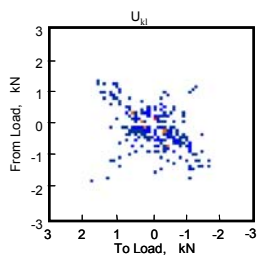
$$\Phi^{-1}(P_f) = \frac{\frac{\mu_S}{F_n} - \frac{1}{1 + \alpha \text{COV}_F}}{\sqrt{\left(\frac{\mu_S}{F_n} \text{COV}_S\right)^2 \frac{N-1}{\chi^2_{1-\alpha, N-1}} + \left(\frac{\text{COV}_F}{1 + \alpha \text{COV}_F}\right)^2}}$$

Validation

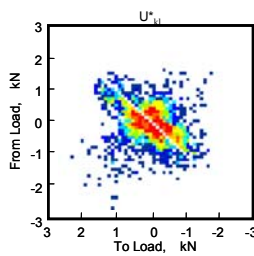


Full scale vehicle simulation done at the end for design final validation

Extrapolation



Measured data, e.g. 1,000 km

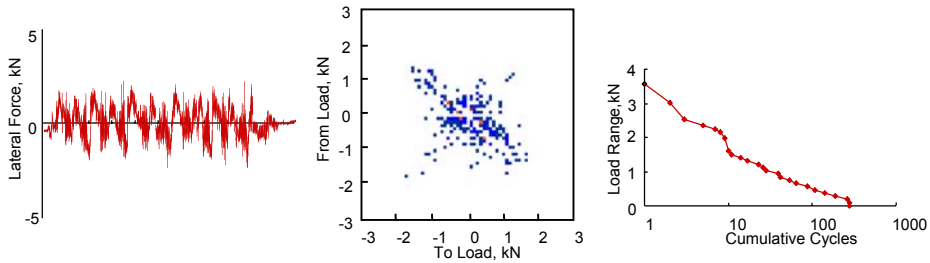


Extrapolated data, 300,000 km

How does this process work?



Service Loading Spectra



Time history

Rainflow
Histogram

Exceedance
Diagram



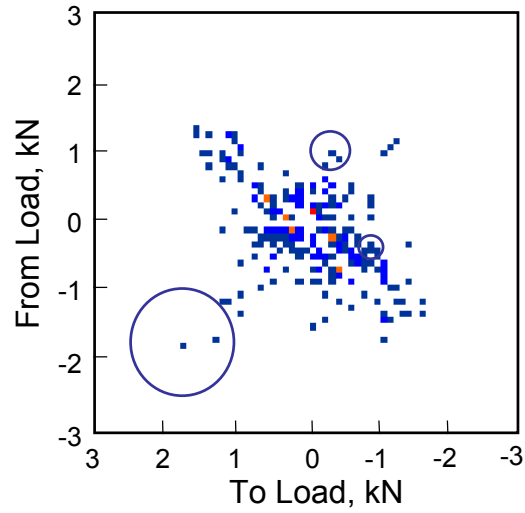
Problem Statement

Given a rainflow histogram for a single user,
extrapolate to longer times

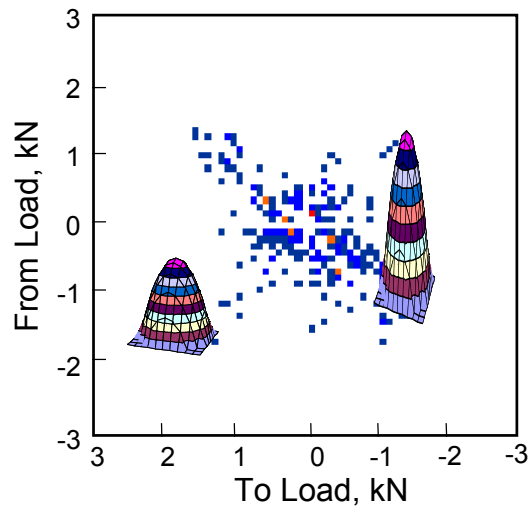
Given rainflow histograms for multiple users,
extrapolate to more users



Probability Density

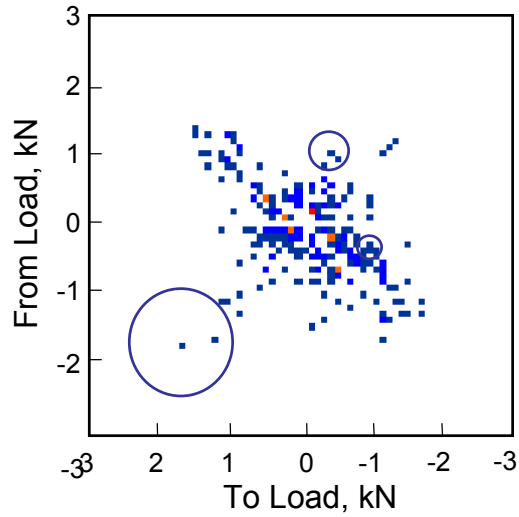


Kernel Smoothing

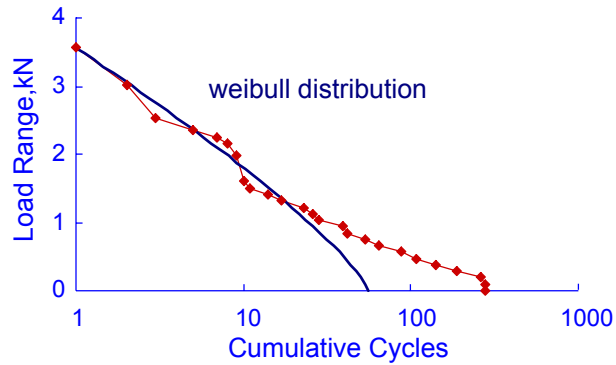




Sparse Data

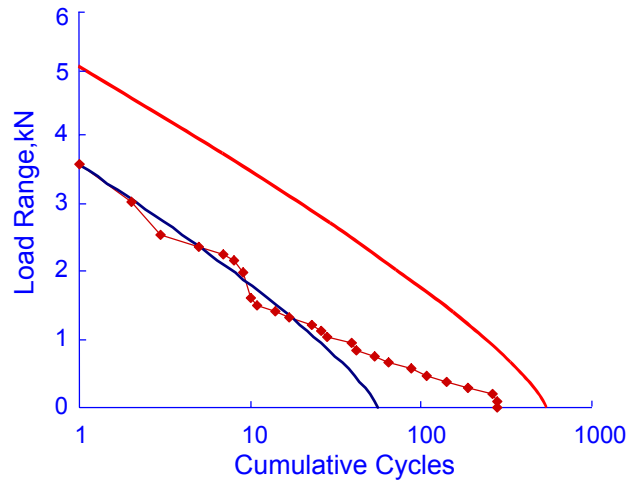


Exceedance Plot of 1 Lap

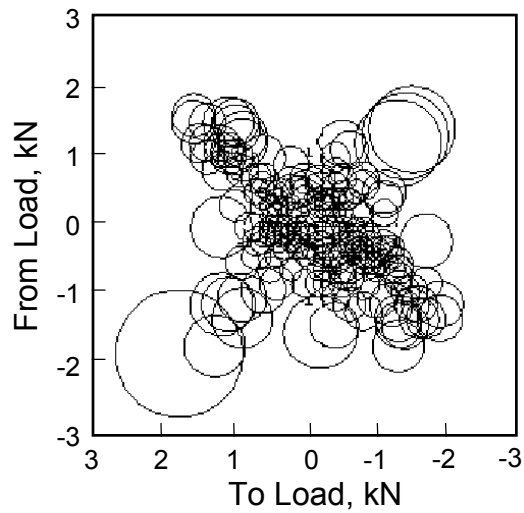




10X Extrapolation

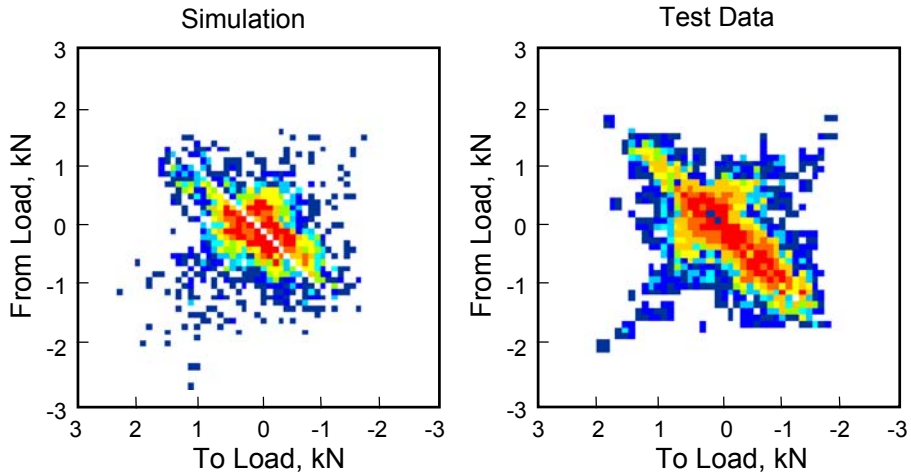


Probability Density

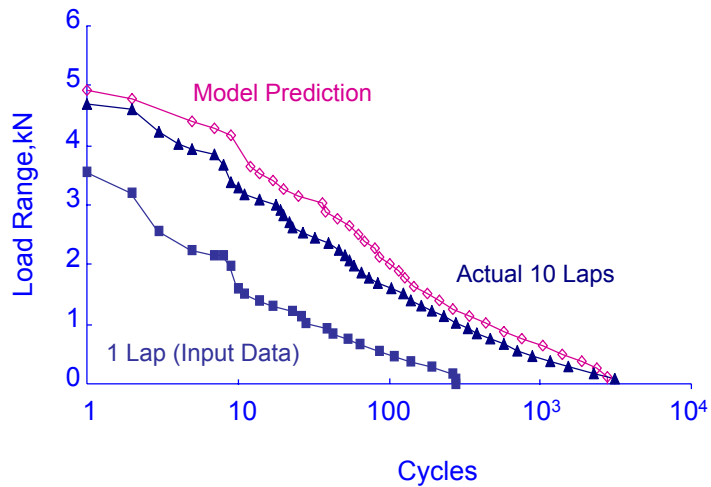




Results



Exceedance Diagram



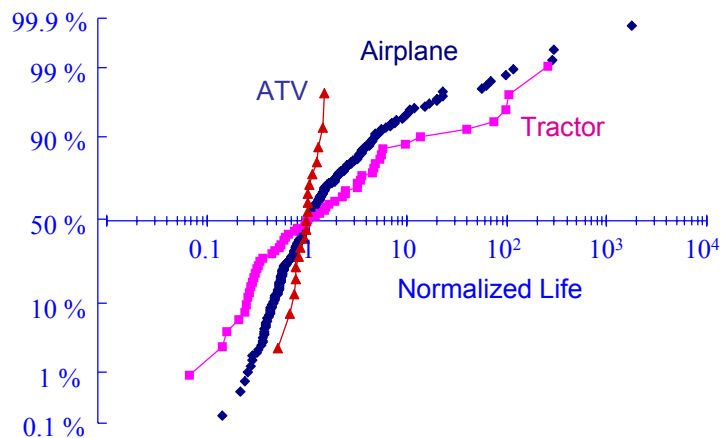


Problem Statement

- Given a rainflow histogram for a single user, extrapolate to longer times
- Given rainflow histograms for multiple users, extrapolate to more users



Extrapolated Data Sets





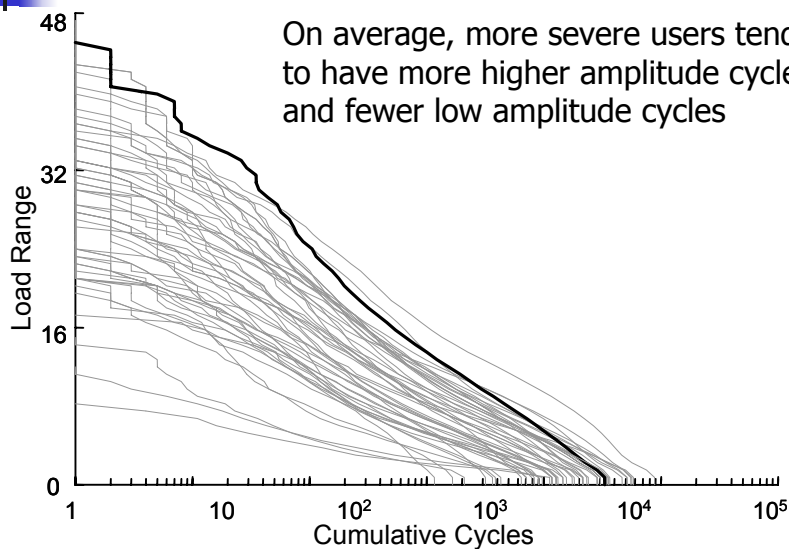
Issues

In the first problem the number of cycles is known but the variability is unknown and must be estimated

In the second problem the variability is known but the number and location of cycles is unknown and must be estimated

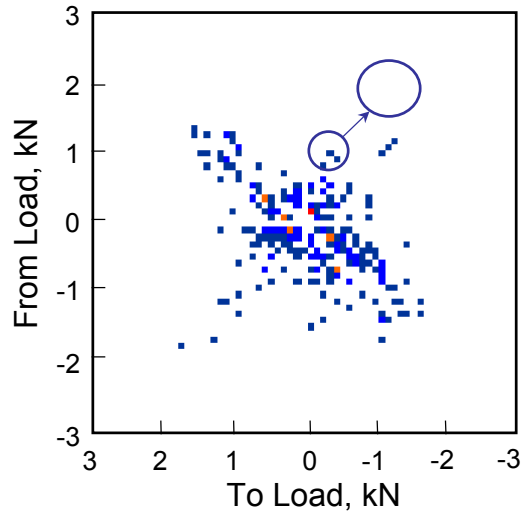


Assumption

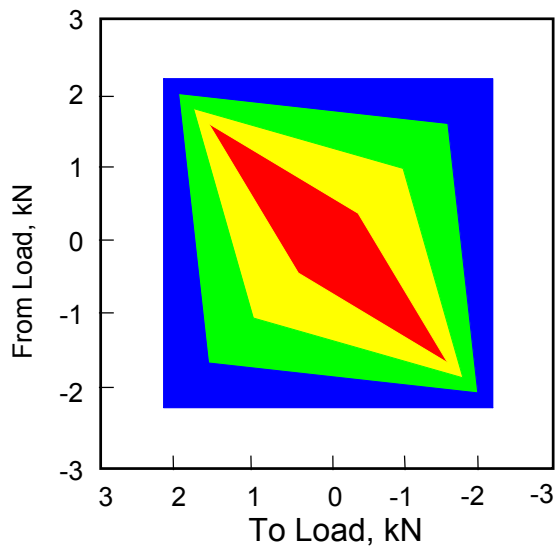




Translation

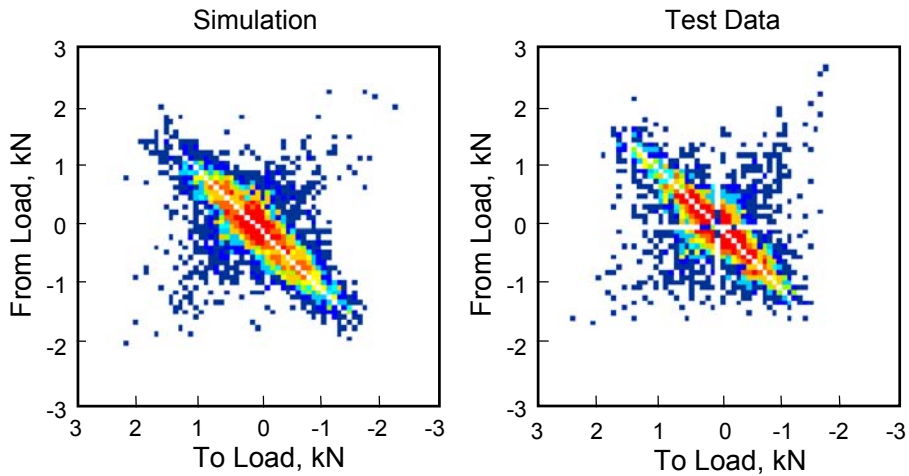


Damage Regions

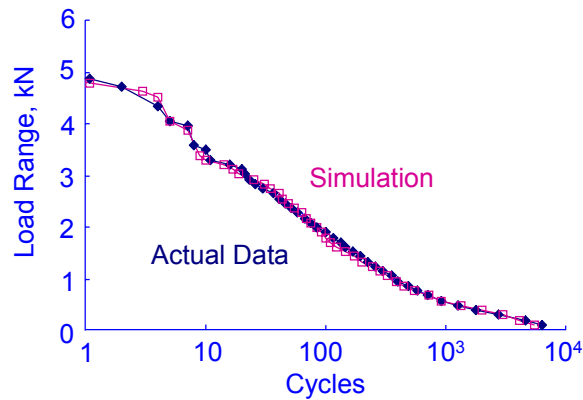




ATV Data - Most Damaging in 19

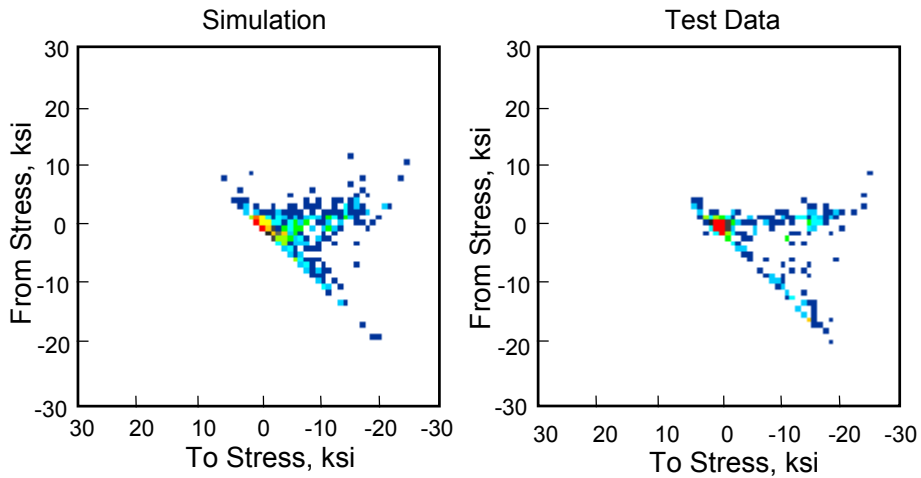


ATV Exceedance

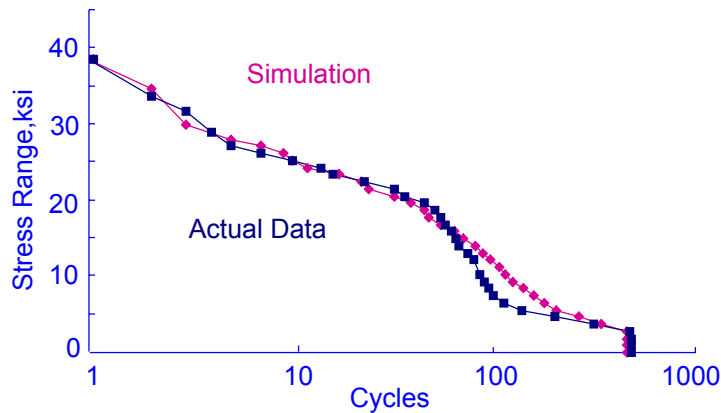




Airplane Data - Most Damaging in 334

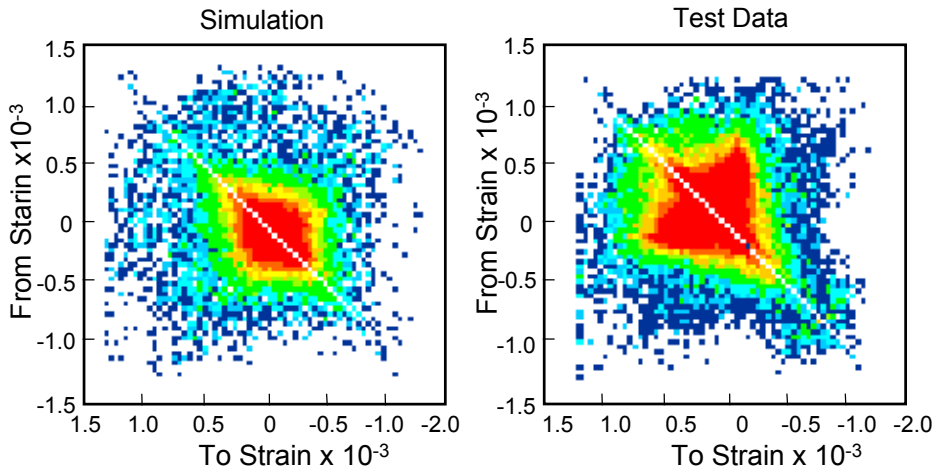


Airplane Exceedance

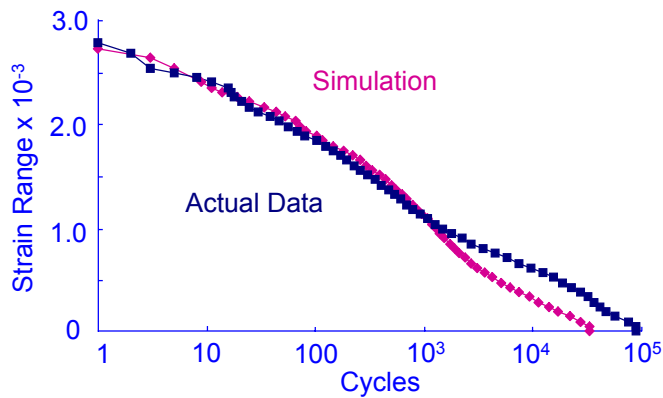




Tractor Data - Most Damaging in 54



Tractor Exceedance



Probabilistic Aspects of Fatigue

www.FatigueCalculator.com



Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering

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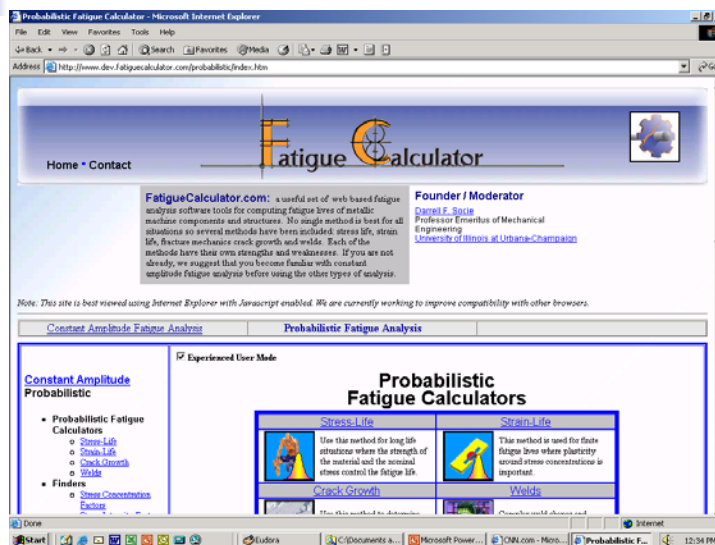
Probabilistic Aspects of Fatigue

- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- Analysis Methods
- Characterizing Variability
- Case Studies
- **FatigueCalculator.com**
- GlyphWorks

www.FatigueCalculator.com



Probabilistic Fatigue Analysis



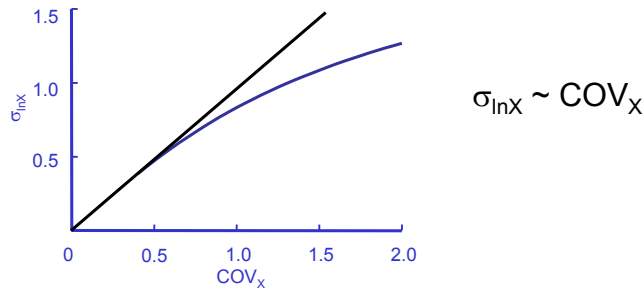


Modeling Variability

Central Limit Theorem:

$$\text{Products: } Z = X_1 \cdot X_2 \cdot X_3 \cdot X_4 \cdot \dots \cdot X_n$$

$Z \rightarrow \text{LogNormal}$ as n increases



COV_X is a good measure of variability



COV and LogNormal Distributions

COV_X	Standard Deviation, $\ln x$		
	1 68.3%	2 95.4%	3 99.7%
0.05	1.05	1.11	1.16
0.1	1.10	1.23	1.33
0.25	1.28	1.66	2.04
0.5	1.60	2.64	3.92
1	2.30	5.53	11.1

99.7% of the data is within a factor of ± 1.33 of the mean for a $\text{COV} = 0.1$

Strain Life Analysis

The screenshot shows the 'Probabilistic Strain-Life Analysis' section of the Fatigue Calculator website. It includes a navigation menu with 'Home' and 'Contact', a logo for 'Fatigue Calculator', and a sidebar with portraits of researchers like Rankine (1820-1872) and Bauschinger (1824-1896). The main content area is titled 'Loading' and contains a table for inputting material properties.

	Value	Units	Distribution Type	Coefficient of Variance	
Maximum	S_{max} or ϵ_{max}	1000	micro-strain	Normal	0.2
Minimum	S_{min} or ϵ_{min}	-1000	micro-strain	Normal	0.2
Range	ΔS or $\Delta \epsilon$		micro-strain	Normal	0.2
Mean	S_m or ϵ_m		micro-strain	Normal	0.2

Material Properties

The screenshot shows the 'Material' section of the Fatigue Calculator website. It features a sidebar with portraits of researchers like Wolfer (1819-1914) and Eyring (1897-1979). The main content area is titled 'Material' and includes a 'Material Property Finder' section with a table for inputting material properties for 'Steel 1020'.

Type	Value	Units	Distribution Type	Coefficient of Variance	Cumulation Coefficient
Fatigue Strength Coefficient	σ'_f	883	MPa	Log-Normal	0.1
Fatigue Strength Exponent	b	-0.118		None	
Fatigue Ductility Coefficient	ϵ'_f	0.16		Log-Normal	0.2
Fatigue Ductility Exponent	c	-0.412		None	
Elastic Modulus	E	206800	MPa	None	
Fatigue Limit	σ'_{FL}		MPa	None	
Cyclic Strength Coefficient	K'	1441	MPa	None	
Cyclic Strain Hardening Exponent	n'	0.283		None	

Stress Concentration

Stress Concentration

Either specify K_t directly or enter K_t and the radius. If you choose to use K_t the distribution specified for K_t will be used. There is no need to specify a distribution for either the radius or ultimate strength. The uncertainty in these measurements is insignificant when compared to the uncertainty in K_t .

Use K_t in analysis? No Yes

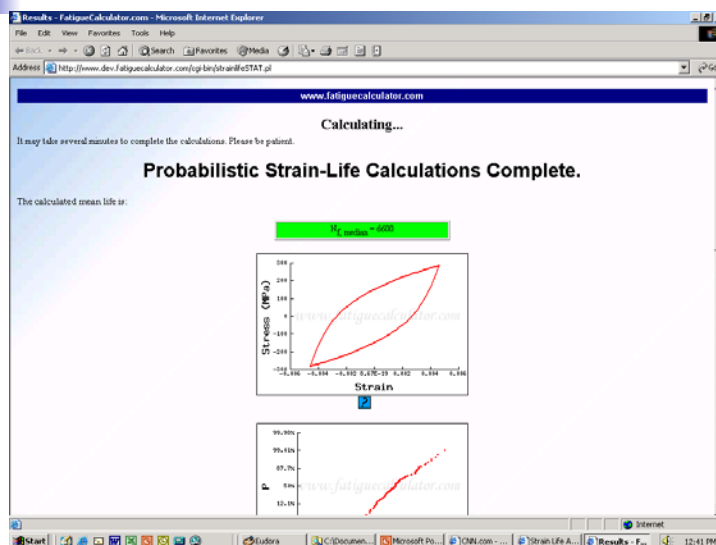
K_t

Radius mm

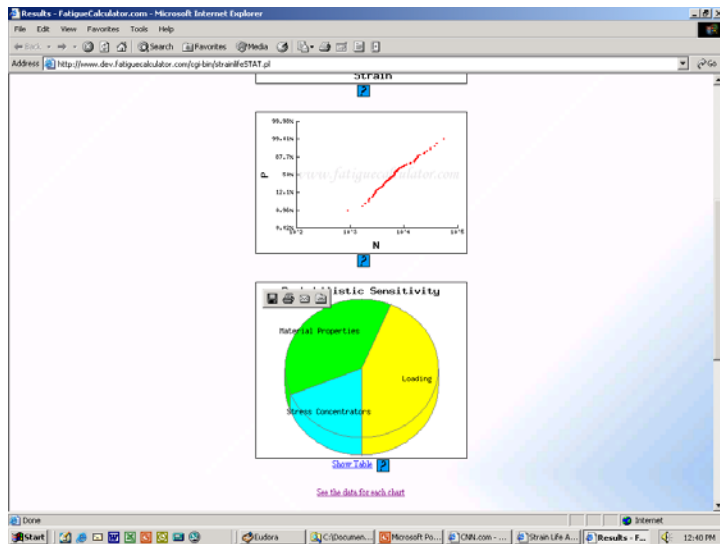
Material MPa

Ultimate Strength MPa

Results



Results (continued)

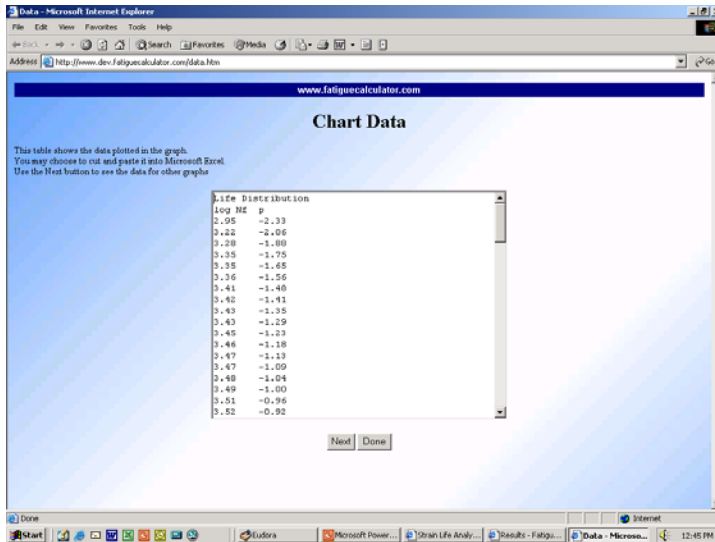


Results (continued)

The screenshot shows the 'Data' page of the FatigueCalculator.com, displaying a table of sensitivity analysis results. The table is titled 'www.fatiguecalculator.com' and has the following structure:

Variable	Mean Life	Deterministic Sensitivity	Probabilistic Sensitivity	Median	COV
Mean Life	6930			6640	0.898
Loading		-3.53	0.72		
$\sigma_{max,N}$ (0.001, 0.2)		-2.22	0.621	9.92*10 ⁻⁴	0.196
$\epsilon_{max,N}$ (-0.001, 0.2)		-1.3	0.364	-0.00103	0.181
Material Properties		-6.99	0.621		
K		0.666	0	1440	0
n'		-0.14	0	0.283	0
E		-1.57	0	2.07*10 ⁵	0
b		-1.04	0	-0.118	0
c		-8.02	0	-0.412	0
σ_{UL} (883, 0.1)		0.944	0.132	883	0.0971
ϵ_{UL} (0.16, 0.2)		2.17	0.607	0.16	0.203
Stress Concentrators		-4.43	0.309		
K N (3, 0.05)		-4.43	0.309	3	0.0463

Results (continued)

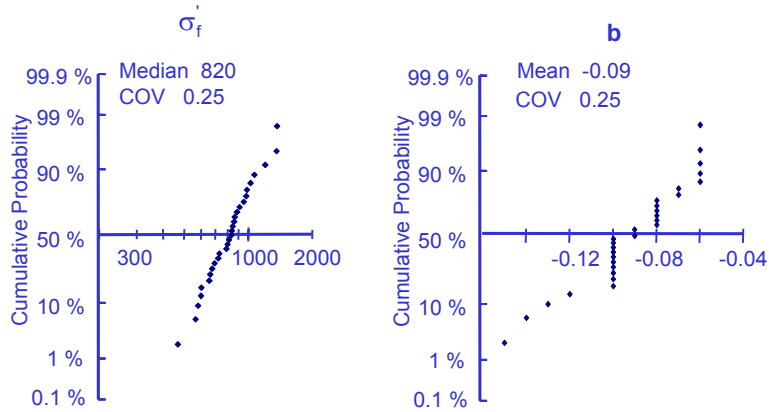


Ten Simulations

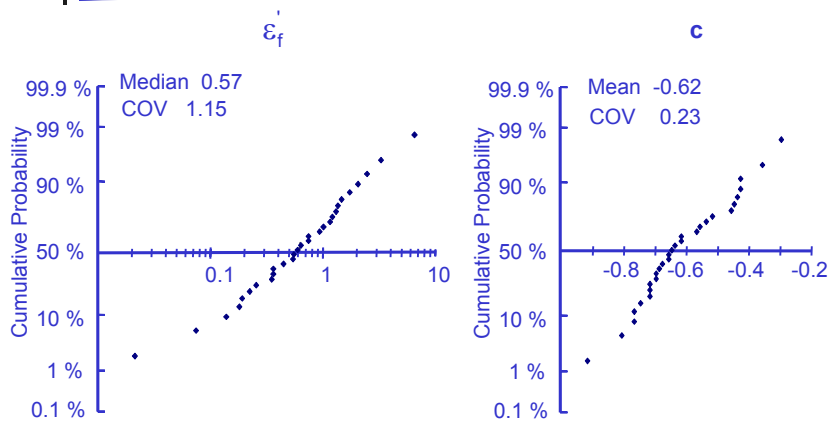
	Life	COV
	6470	0.959
	6930	0.898
	6710	0.688
	6640	0.908
	6580	0.869
	6470	0.959
	7010	0.723
	6690	0.908
	6170	0.791
	6560	0.971
Mean	6623	0.8674
COV	0.038	0.114



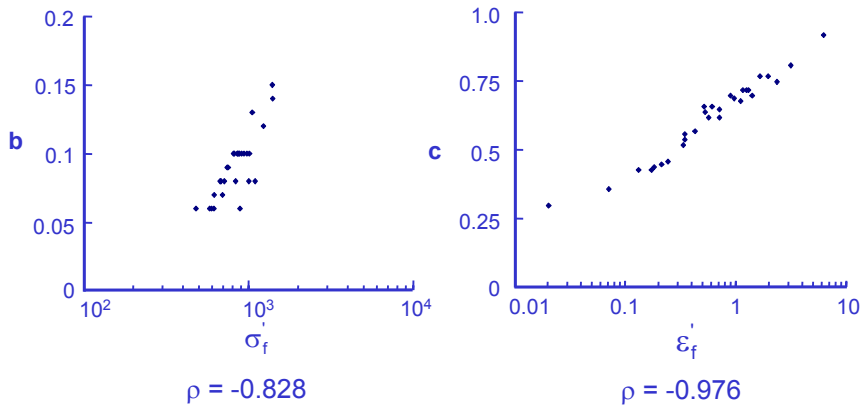
29 Individual Data Sets



29 Individual Data Sets (continued)



Correlation



Correlated Variables

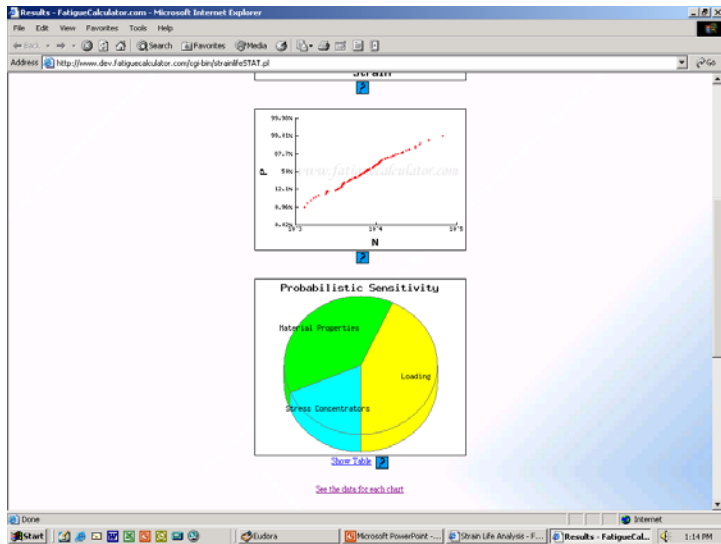
Material

You may use the Material Finder to look up the proper values for your material but you must specify the distribution you wish to use manually.

Property	Value	Units	Distribution Type	Coefficient of Variance	Correlation Coefficient
Fatigue Strength Coefficient	693	MPa	Log-Normal	.25	
Fatigue Strength Exponent	-0.118		Normal	.25	-.83
Fatigue Ductility Coefficient	0.16		Log-Normal	1.15	
Fatigue Ductility Exponent	-0.412		Normal	.23	-.98
Elastic Modulus	206800	MPa	None		
Fatigue Limit		MPa	None		
Cyclic Strength Coefficient	1441	MPa	None		
Cyclic Strain Hardening Exponent	0.283		None		



Results (continued)

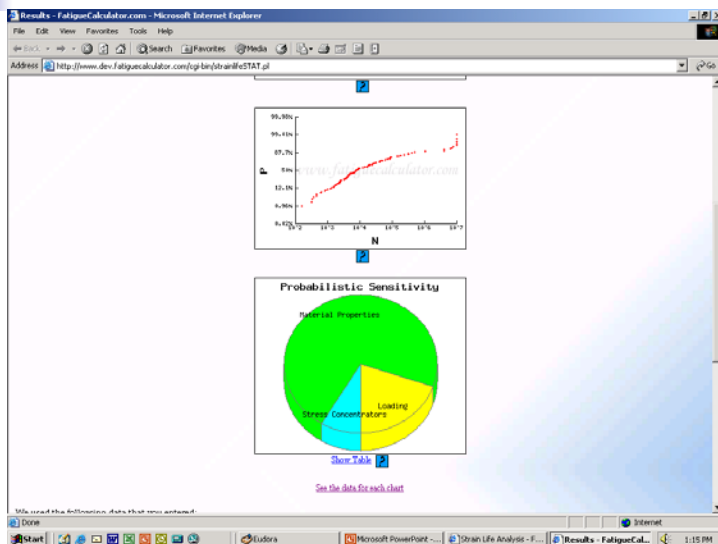


Results (continued)

The screenshot shows a data table from the Fatigue Calculator. The table has five columns: Variable, Deterministic Sensitivity, Probabilistic Sensitivity, Mean Life, and COV. The data is as follows:

Variable	Deterministic Sensitivity	Probabilistic Sensitivity	Mean Life	COV
Mean Life	740		660	0.913
Loading	-3.33	0.71		
σ_{max} N(0.001, 0.2)	-2.22	0.613	9.94×10^4	0.202
σ_{min} N(-0.001, 0.2)	-1.3	0.359	-0.00101	0.219
Material Properties	-6.99	0.633		
K'	0.666	0	1440	0
n'	-0.14	0	0.293	0
E	-1.57	0	2.07×10^5	0
b N(-0.118, 25), COV= .83	-1.04	0.2	-0.119	0.24
c N(-0.412, 23)	-8.02	0.206	-0.41	0.0467
S _L L(882, 25)	0.944	0.325	861	0.263
E _L L(0.16, 1.15)	2.17	0.0344	0.16	0.0106
Stress Concentrations	-4.43	0.305		
K N(3, 0.05)	-4.43	0.305	3	0.0472

Uncorrelated Variables



Results (continued)

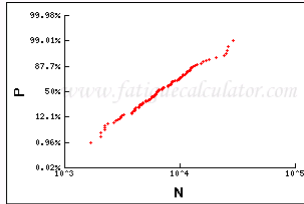
The screenshot shows a detailed data table from the fatigue analysis. The table has five columns: Variable, Deterministic Sensitivity, Probabilistic Sensitivity, Median, and COV. The data is as follows:

Variable	Deterministic Sensitivity	Probabilistic Sensitivity	Median	COV
Mean Life	1.900×10^6		6640	155
Loading	-3.53	0.263		
$\sigma_{max} N(0.001, 0.2)$	-2.22	0.227	9.94×10^4	0.202
$\sigma_{min} N(0.001, 0.2)$	-1.3	0.133	-0.00101	0.219
Material Properties	-6.99	0.958		
K'	0.666	0	140	0
n'	-0.14	0	0.283	0
E	-1.57	0	2.07×10^4	0
$b N(0.118, 25)$	-1.04	0.132	-0.119	0.26
$c N(0.412, 23)$	-0.02	0.941	-0.401	0.25
$S_y U(883, 25)$	0.944	0.12	861	0.262
$\beta U(0.16, 1.15)$	2.17	0.0137	0.16	0.0106
Stress Concentrations	-4.43	0.113		
K N(3, 0.05)	-4.8	0.113	3	0.0472



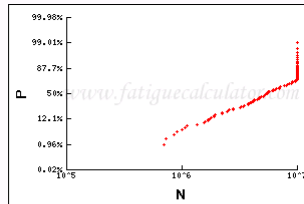
Strain Amplitude

± 1000

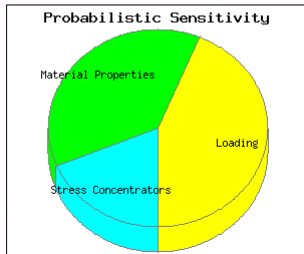


[?](#)

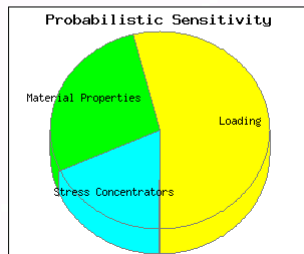
± 250



[?](#)



[Show Table](#) [?](#)



[Show Table](#) [?](#)



Deterministic Analysis

Stress Life Analysis - FatigueCalculator.com - Microsoft Internet Explorer

File Edit View Favorites Tools Help

Address http://www.dev.fatiguecalculator.com/fatreslife.asp.htm

Home • Contact

Fatigue Calculator

Constant Amplitude Stress-Life Analysis

Enter as much data as you know. If it is not enough, you will be asked for more. Fields in red represent absolutely required data to begin calculations. Other data may become necessary as calculation proceeds.

Loading

You may choose to specify either the stresses or the desired life / safety factor and compute the other. If you choose to calculate the stresses, leave this section blank.

Maximum S_{max} or σ_{max} = MPa
 Minimum S_{min} or σ_{min} = MPa
 Or
 Alternating S_a or σ_a = MPa
 Mean S_m or σ_m = MPa

Material

Deterministic Analysis (continued)

Material

Alternating S_a or σ_a = MPa

Mean S_m or σ_m = MPa

Type

Ultimate Strength S_u = MPa

Fatigue Limit S_{FL} = MPa

Modifying Factors

Surface Factor K_{sp} = or

Loading Factor K_L = or

Size Factor K_{size} =

Diameter d = mm

Deterministic Analysis (continued)

Stress Concentration

K_t =

Do K_t in analysis?

Radius = mm

Safety Factor for Infinite Life

Safety Factor =

Mean Stress Definition =

Definition =

Calculate Life / Safety Factor

Deterministic Analysis Results

Results - FatigueCalculator.com - Microsoft Internet Explorer

Address: http://www.dev.fatiguecalculator.com/cgi-bin/stresslife.pl

Stress-Life Calculations Complete.

The calculated safety factor is:

n = 3.2

We used the following data that you entered:

$S_u = 500 \text{ MPa}$
 $K_f = 3$
 $S_{max} = 20 \text{ MPa}$
 $S_{min} = -20 \text{ MPa}$

We calculated the following parameters based on default values and values that you entered:

$\lambda_1 = 1$
 $\lambda_{max} = 1$
 $S_{PT} = 210 \text{ MPa}$
 $\lambda_{SP} = 0.12$
 $S_a = 20 \text{ MPa}$
 $S_m = 0 \text{ MPa}$

Probabilistic Analysis

Stress Life Analysis - FatigueCalculator.com - Microsoft Internet Explorer

Address: http://www.dev.fatiguecalculator.com/stresslife.asp.htm

Home • Contact

Fatigue Calculator

Constant Amplitude Stress-Life Analysis

Enter as much data as you know. If it is not enough, you will be asked for more. Fields in red represent absolutely required data to begin calculations. Other data may become necessary as calculation proceeds.

Loading bottom of page

You may choose to specify either the stresses or the desired life / safety factor and compute the other. If you choose to calculate the stresses, leave this section blank.

Maximum S_{max} or s_{max} = 20 MPa
 Minimum S_{min} or s_{min} = -20 MPa
 Or
 Alternating S_a or s_a = MPa
 Mean S_m or s_m = MPa

Material bottom of page

Probabilistic Analysis (continued)

Material

You may use the Material Finder by clicking on the Material Property Finder icon to look up the proper values for your material or specify values directly. If you do not enter a value for the Fatigue Limit, a default value of $1/2 S_u$ will be assumed.

Material:

Parameter	Value	Units	Distribution Type	Coefficient of Variation	Correlation Coefficient
Ultimate Strength S_u	500	MPa	Normal	0.1	
Fatigue Limit S_{FL}		MPa	None		
Elastic Modulus E		MPa	None		
Intercept S_f		MPa	Log-Normal		
Slope b			None		

If this section is left blank, values will be estimated.

Modifying Factors

Either specify the modifying factor directly or choose a finish from the drop-down box. If you don't know, a default value of 1 will be used.

Probabilistic Analysis (continued)

Modifying Factors

Either specify the modifying factor directly or choose a finish from the drop-down box. If you don't know, a default value of 1 will be used.

Factor	Value or Finish	Distribution Type	Coefficient of Variation
Surface Factor k_{su}	<input type="text" value="1"/> or <input type="text" value="Machined"/>	Normal	0.1
Loading Factor k_L	<input type="text" value="1"/> or <input type="text" value="None"/>		
Size Factor k_{size}	<input type="text" value="1"/>		
Diameter d	<input type="text" value=""/>	mm	

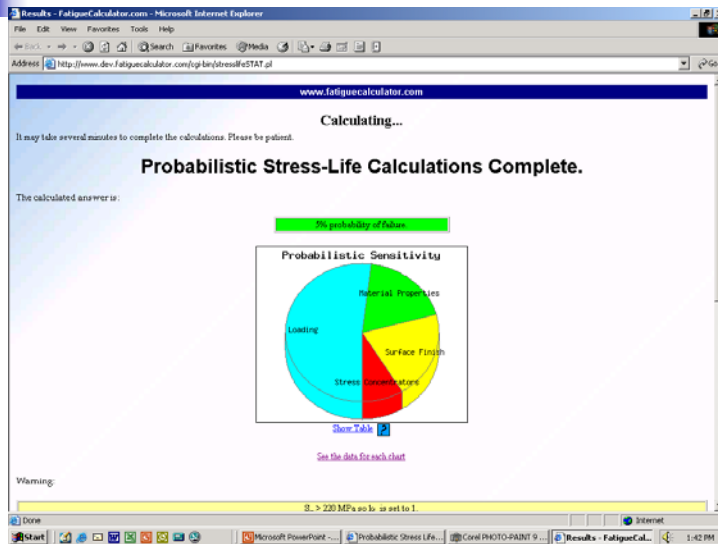
Stress Concentration

Either specify K_f directly or enter K_t and the radius.

Parameter	Value	Distribution Type	Coefficient of Variation
Stress Concentration Factor K_t	3	Normal	0.05
Use K_f in analysis?	No		
Radius	<input type="text" value=""/>	mm	

Buttons: Calculate Life, Clear Form

Probabilistic Analysis Results



Probabilistic Aspects of Fatigue

GlyphWorks

Professor Darrell F. Socie
Department of Mechanical and
Industrial Engineering

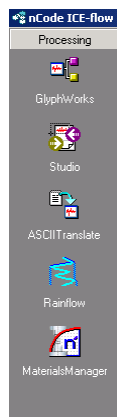


Probabilistic Aspects of Fatigue

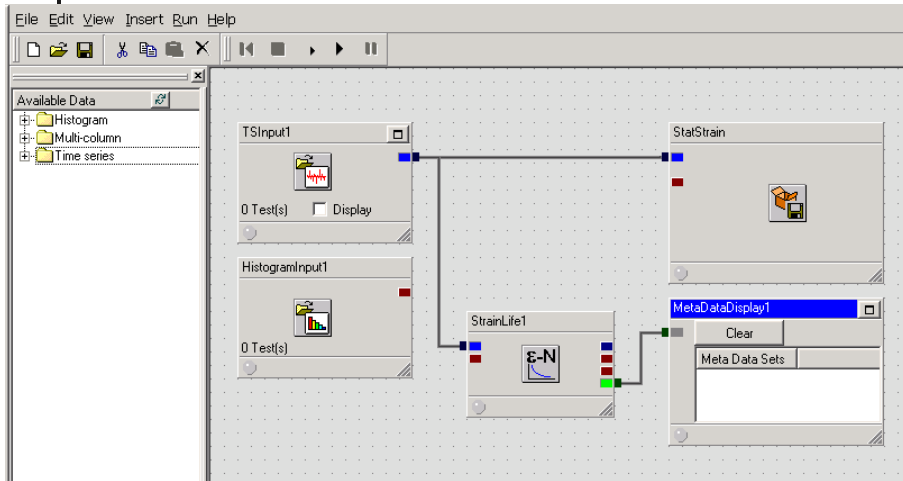
- Introduction
- Basic Probability and Statistics
- Statistical Techniques
- Analysis Methods
- Characterizing Variability
- Case Studies
- FatigueCalculator.com
- **GlyphWorks**



GlyphWorks / Rainflow



StatStrain.flo



StatStrain Glyph

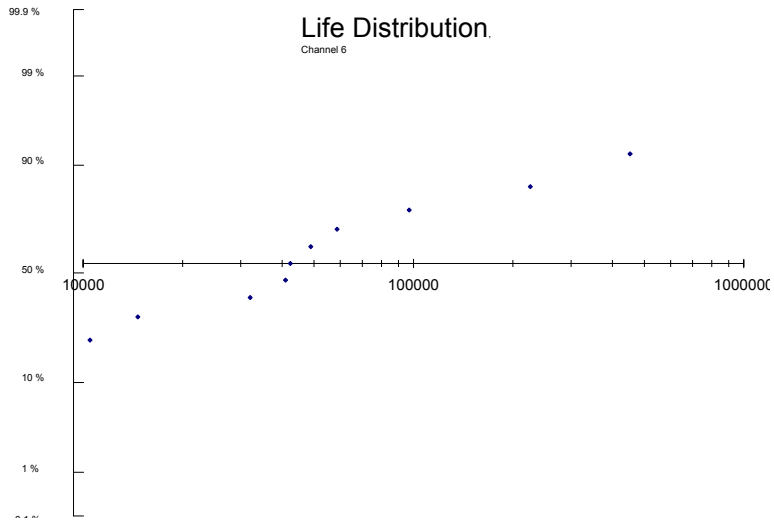
The screenshot shows the 'Statistical Analysis' window for 'Strain Life Analysis'. The window displays a table of parameters and their values. The 'Loading' row is highlighted in yellow. The table includes the following data:

Name	Value	Distribution	Scale Parameter	Correlation Coefficient	Description
Loading					
Scale Factor	100	Normal	25		Scale Factor
Offset	0	Normal	0.1		Offset
Geometry					
K ₁	3	Uniform	.05		Fatigue Concentration Factor
Material Data					
E	200000	None			Young's Modulus
S _{1p}	1000	Log-Normal	0.1		Fatigue strength coefficient
b	-0.1	None			Fatigue strength exponent
e _{1p}	1	None	0.2		Fatigue ductility coefficient
c	-0.5	None			Fatigue ductility exponent
n _p	0.2	None			Cycle: strain hardening exponent
K _p	1200	Log-Normal	0.1		Cycle: strength coefficient
Analysis Properties					
NumCases	10				Number of simulations to run
Damage Sum	1	Log-Norm	5		Uncertainty

At the bottom of the window, there is a path field showing 'c:\ncode_home\StatStrain' and buttons for 'Ok', 'Cancel', 'About', and 'Help'. A checkbox at the bottom left is labeled 'All channels use the same data'.

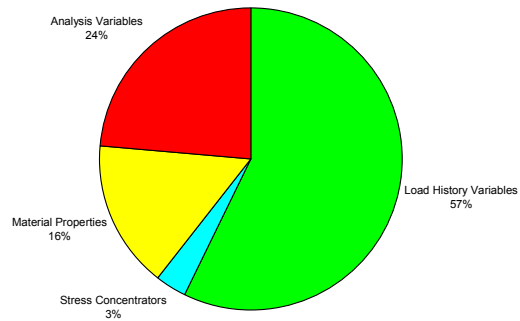


StatOutput.xls



StatOutput.xls (continued)

Probabilistic Sensitivity, Channel 6





StatOutput.xls (continued)

Channel # 6 Total Channels 1

Life Distribution

NumCases	11
Median	45634
COV	1.91
Probability (%)	Life
99	817179
90	223648
50	45634
10	9311
1	2548

Sensitivity

Variable	Probabilistic	Deterministic
Load History Variables	0.89	-5.42
ScaleFactor	0.89	-5.42
Offset	0.00	0.00
Stress Concentrators	0.05	-5.51
Kf	0.05	-5.51
Material Properties	0.25	-14.54
E	0.00	-4.37
Sfp	0.23	3.42
b	0.00	-4.39
efp	0.00	1.42
c	0.00	-9.86
np	0.00	-2.32
Kp	0.10	1.56
Analysis Variables	0.37	1.00
Uncertainty	0.37	1.00



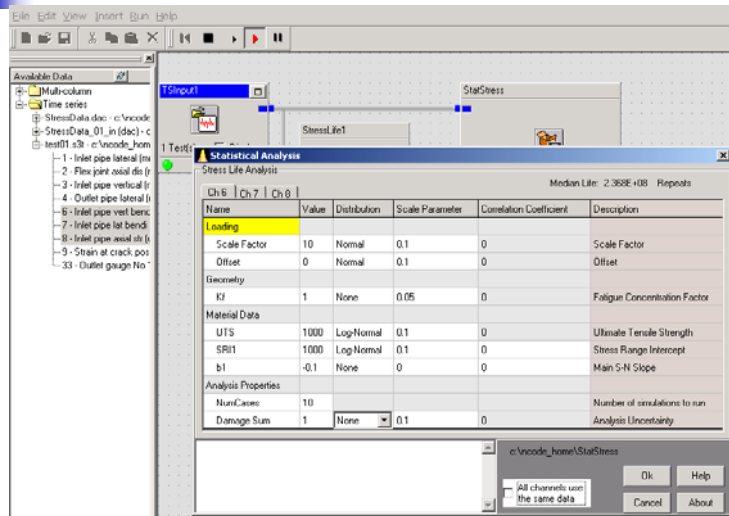
StatOutput.xls (continued)

Inputs

Outputs

Variable	Distribution	Inputs		Outputs	
		Value	Scale Parameter	Value	Scale Parameter
ScaleFactor	Normal	100	0.25	103	0.18
Offset	Normal	0	0.10	-0.01	0.08
Kf	Uniform	3	0.05	3.0	0.04
E	None	208000	0.00	208000	0.00
Sfp	Log-Normal	1000	0.10	1054	0.11
b	None	-0.1	0.00	-0.10	0.00
efp	None	1	0.20	1.0	0.00
c	None	-0.5	0.00	-0.50	0.00
np	None	0.2	0.00	0.20	0.00
Kp	Log-Normal	1200	0.10	1194	0.10
Uncertainty	Log-Normal	1	0.50	0.93	0.48

StatStress.flo

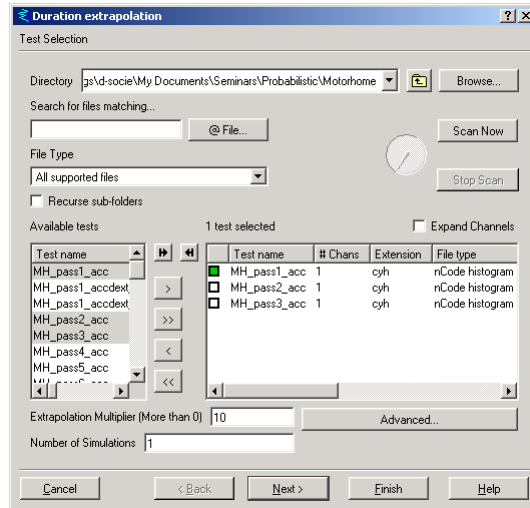


Rainflow Extrapolation

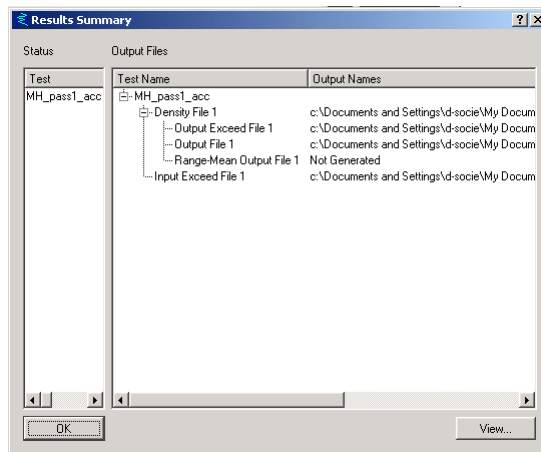




Duration Extrapolation

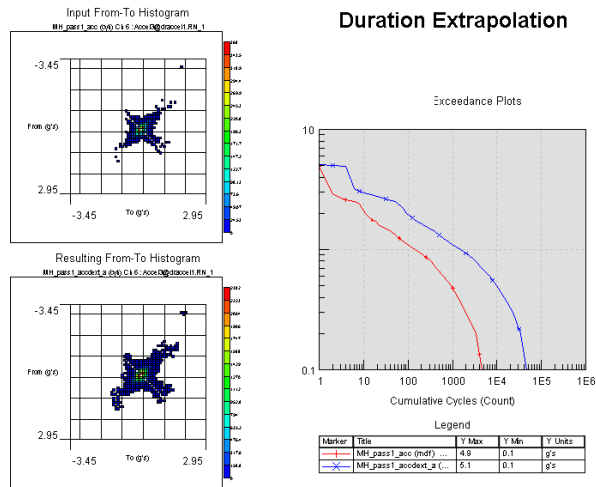


Results Files

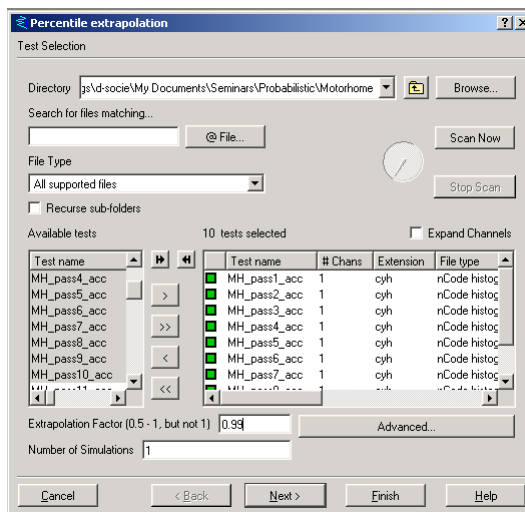




Rainflow Duration

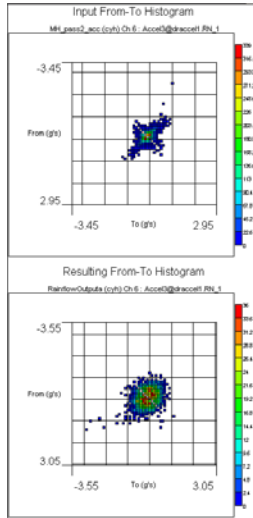


Percentile Extrapolation

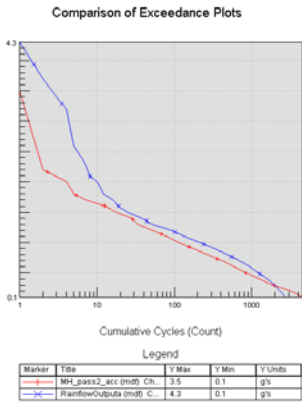




Results



Percentile Extrapolation



Probabilistic Aspects of Fatigue

