

# **Fatigue and Fracture**

**Static Strength and Fracture  
Stress Concentration Factors**



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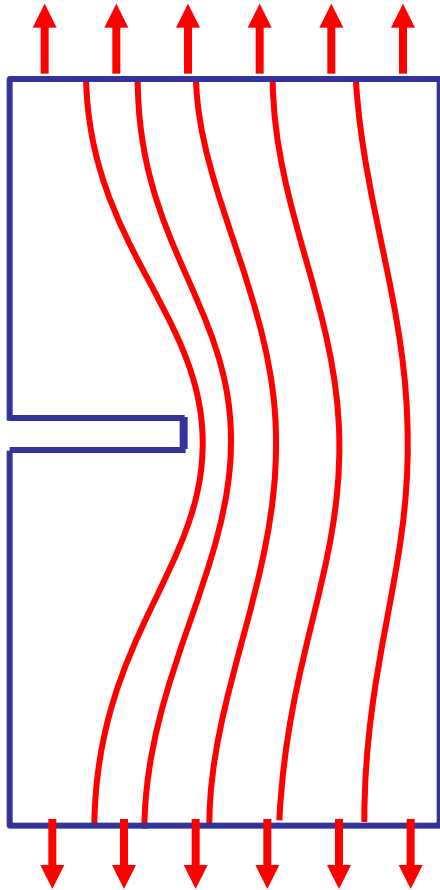


# Static Strength and Fracture

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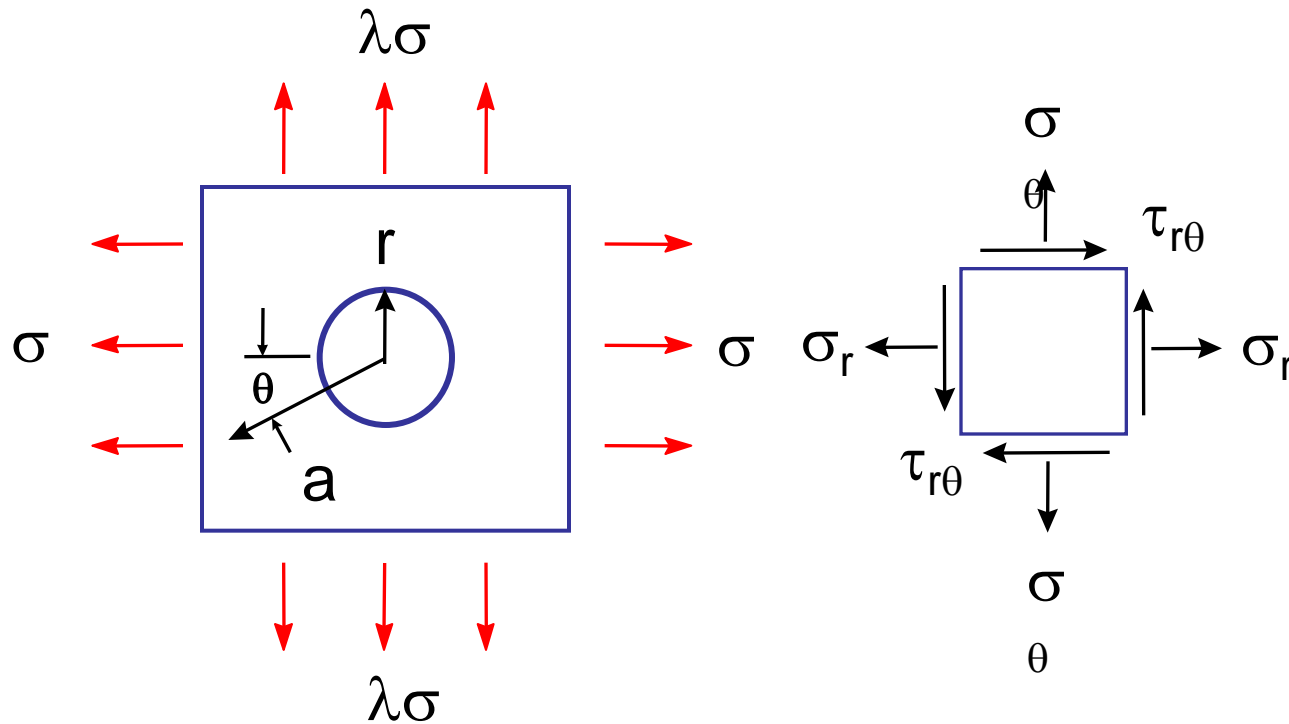
- **Stress Concentration Factors**
- Fracture Mechanics
- Approximate Stress Intensity Factors
- Ductile vs. Brittle Fracture

# Stress Concentration



“Load flow” lines

# Circular Notch





# Stresses

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$$\frac{\sigma_r}{\sigma} = \frac{1+\lambda}{2} \left( 1 - \left( \frac{r}{a} \right)^2 \right) + \frac{1-\lambda}{2} \left( 1 + 3 \left( \frac{r}{a} \right)^4 - 4 \left( \frac{r}{a} \right)^2 \right) \cos 2\theta$$

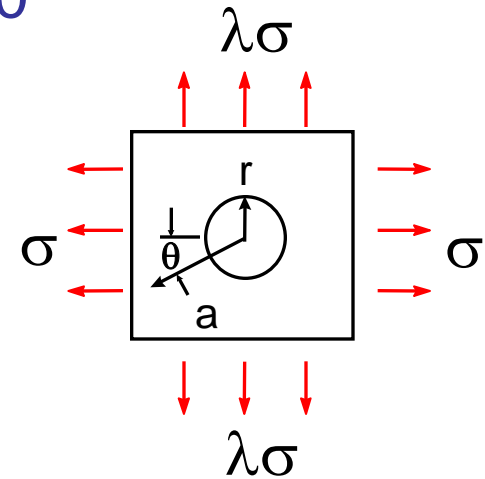
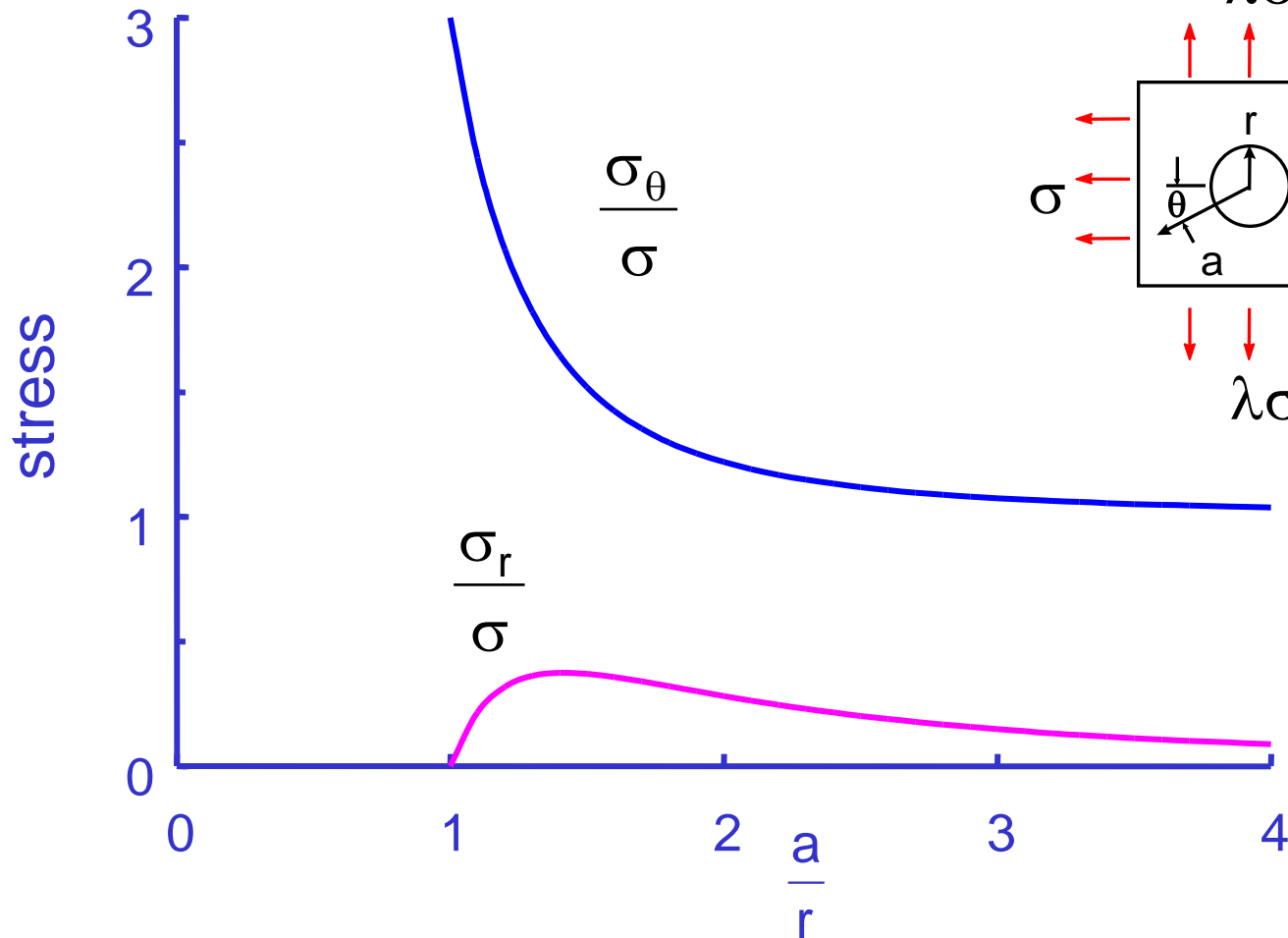
$$\frac{\sigma_\theta}{\sigma} = \frac{1+\lambda}{2} \left( 1 + \left( \frac{r}{a} \right)^2 \right) - \frac{1-\lambda}{2} \left( 1 + 3 \left( \frac{r}{a} \right)^4 \right) \cos 2\theta$$

$$\frac{\tau_{r\theta}}{\sigma} = -\frac{1-\lambda}{2} \left( 1 - 3 \left( \frac{r}{a} \right)^4 + 2 \left( \frac{r}{a} \right)^2 \right) \sin 2\theta$$

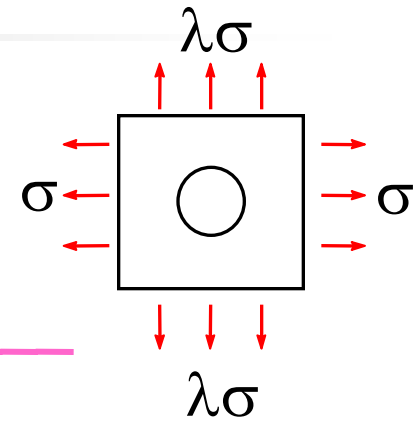
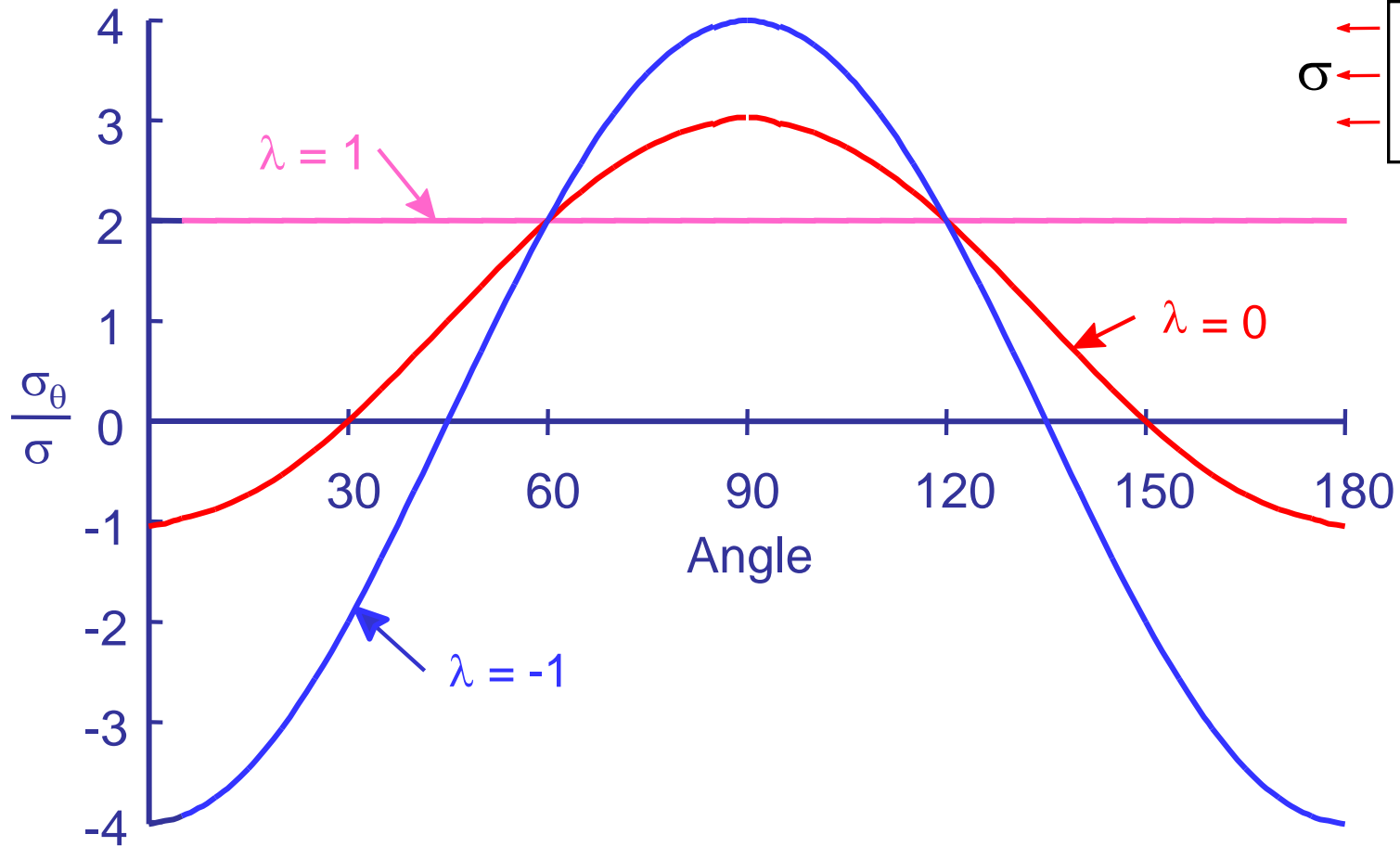
Independent of size, dependant only on r/a

# Stress Distribution

For tension  $\lambda = 0$  and  $\theta = 90$

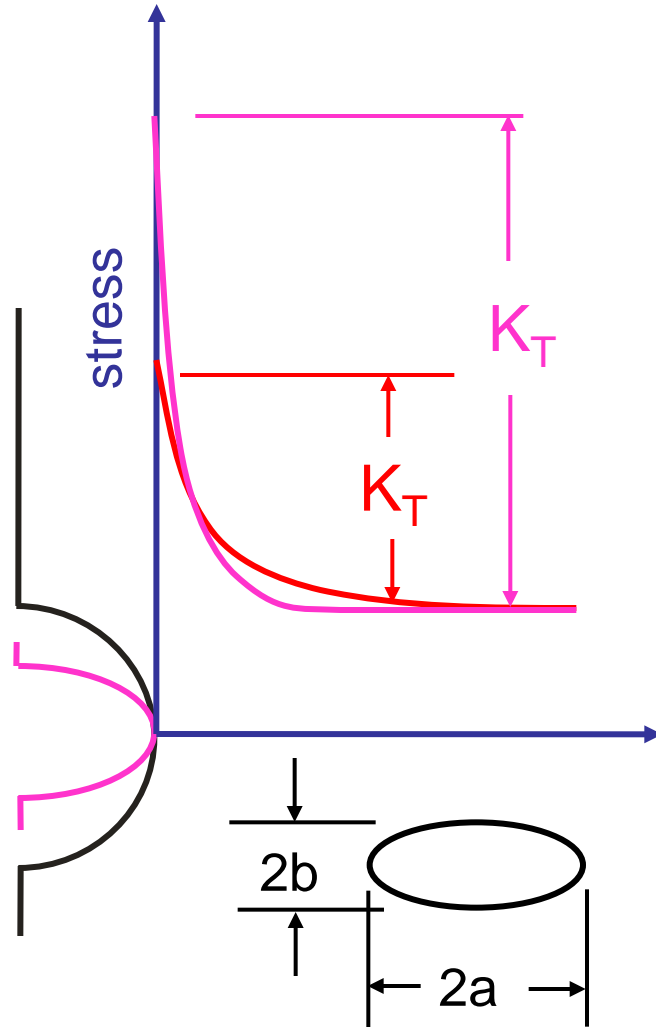


# Stress Ratio Effects



stresses around the circumference of a hole

# Elliptical Notches



$$K_T = 1 + 2\sqrt{\frac{a}{\rho}}$$

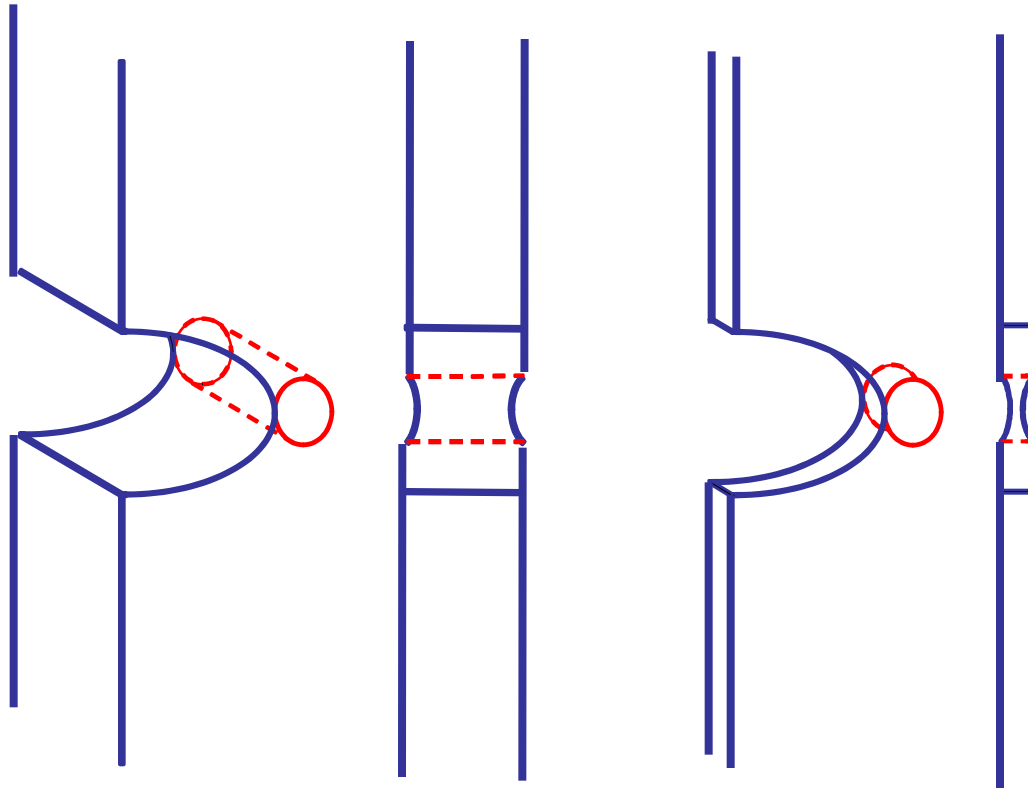
$$\rho = \frac{b^2}{a}$$

Sharp Notch:  
high  $K_T$   
high gradient

Blunt Notch:  
low  $K_T$   
low gradient



# Plane Stress – Plane Strain



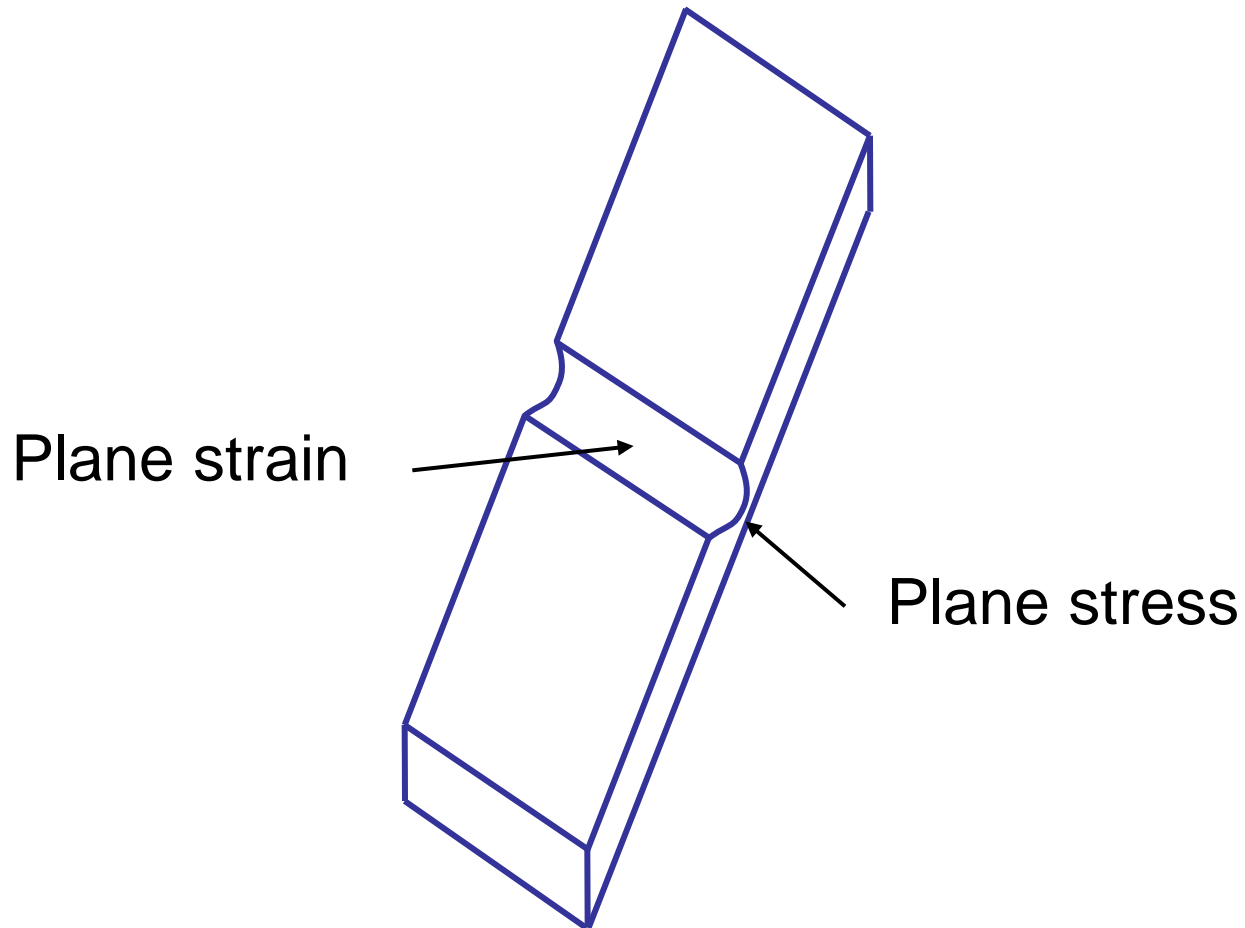
thick plate

thin plate

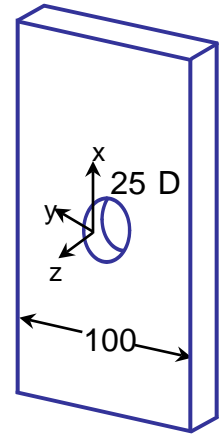
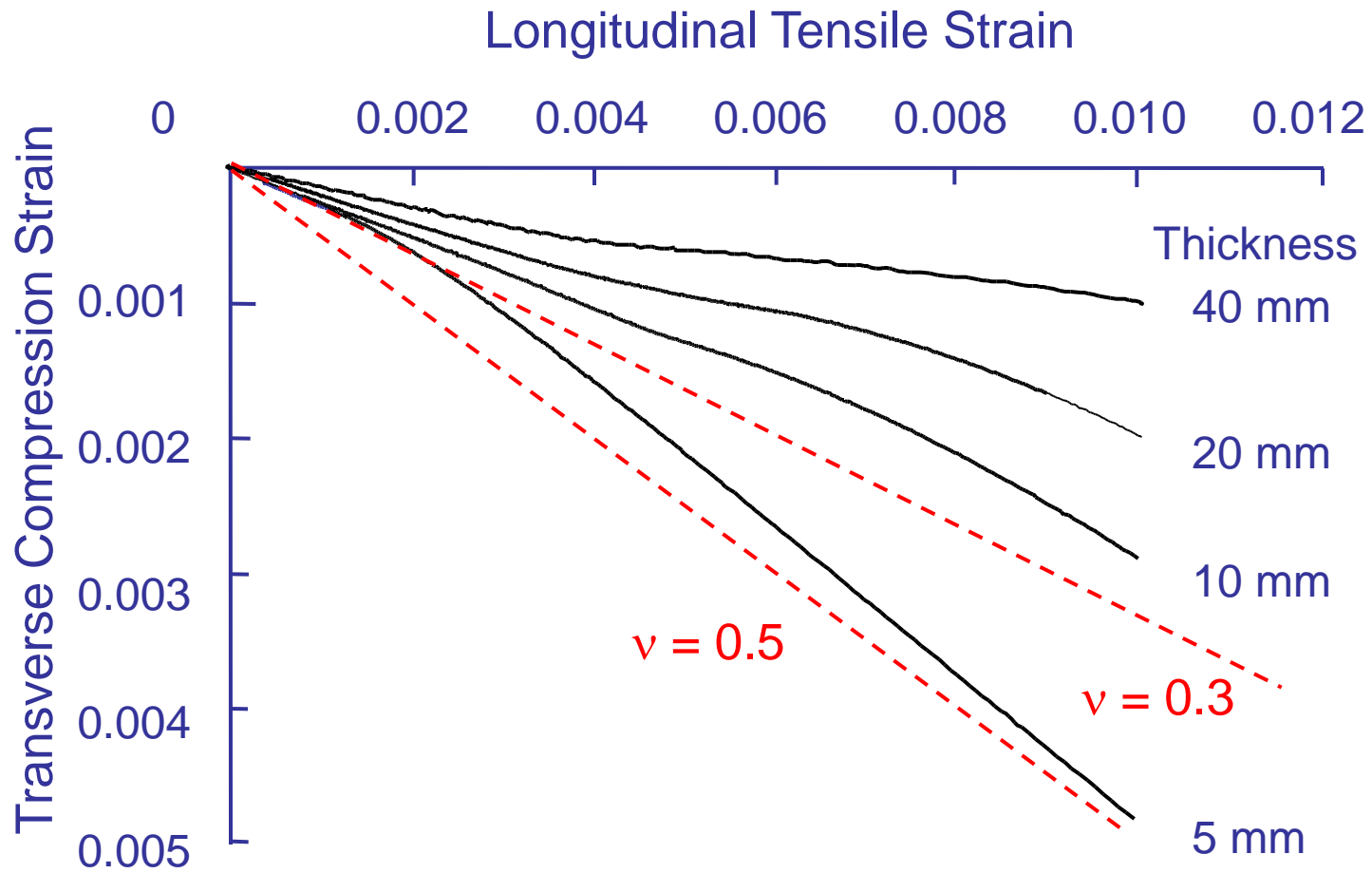
plane strain  $\sigma_z \neq 0$   $\varepsilon_z = 0$

plane stress  $\sigma_z = 0$   $\varepsilon_z \neq 0$

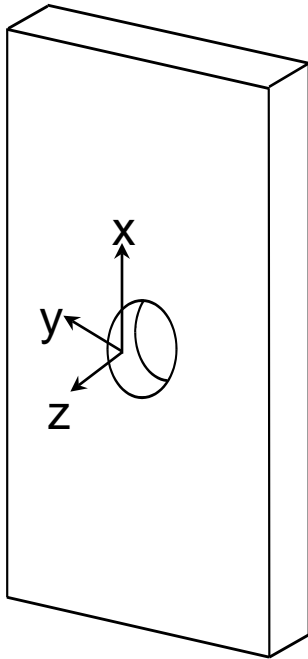
# Thick or Thin ?



# Transverse Strains



# Notch Stresses



t	$\epsilon_x$	$\epsilon_z$	$\sigma_x$	$\sigma_z$
7	0.01	-0.005	63.5	0
15	0.01	-0.003	70.6	14.1
30	0.01	-0.002	73.0	21.8
50	0.01	-0.001	75.1	29.3



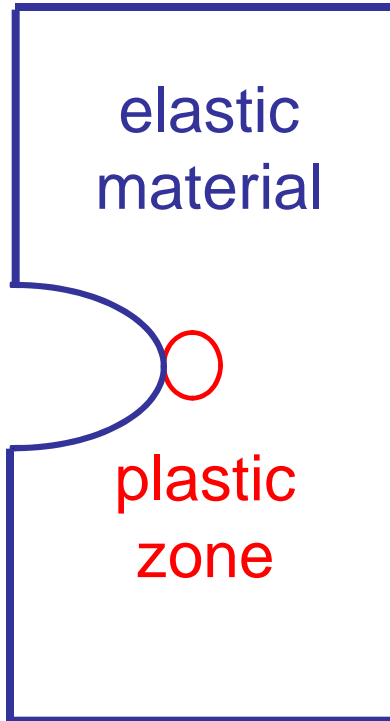
# Fracture Surfaces



# Fracture Surfaces



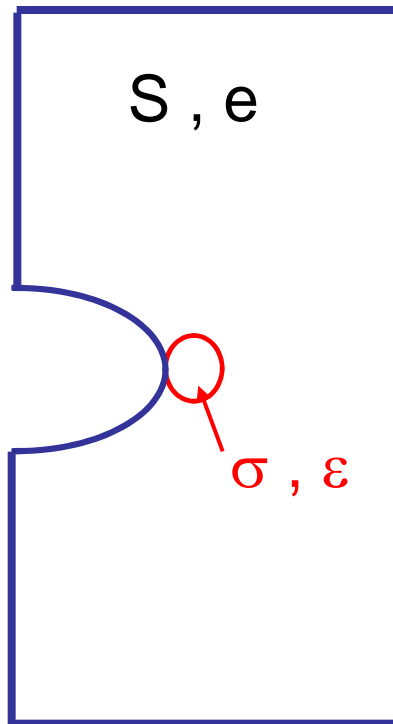
# Stress or Strain Control?



Elastic material surrounding the plastic zone forces the displacements to be compatible, I.e. no gaps form in the structure.

Boundary conditions acting on the plastic zone boundary are displacements. Strains are the first derivative of displacement

# Define $K_\sigma$ and $K_\epsilon$



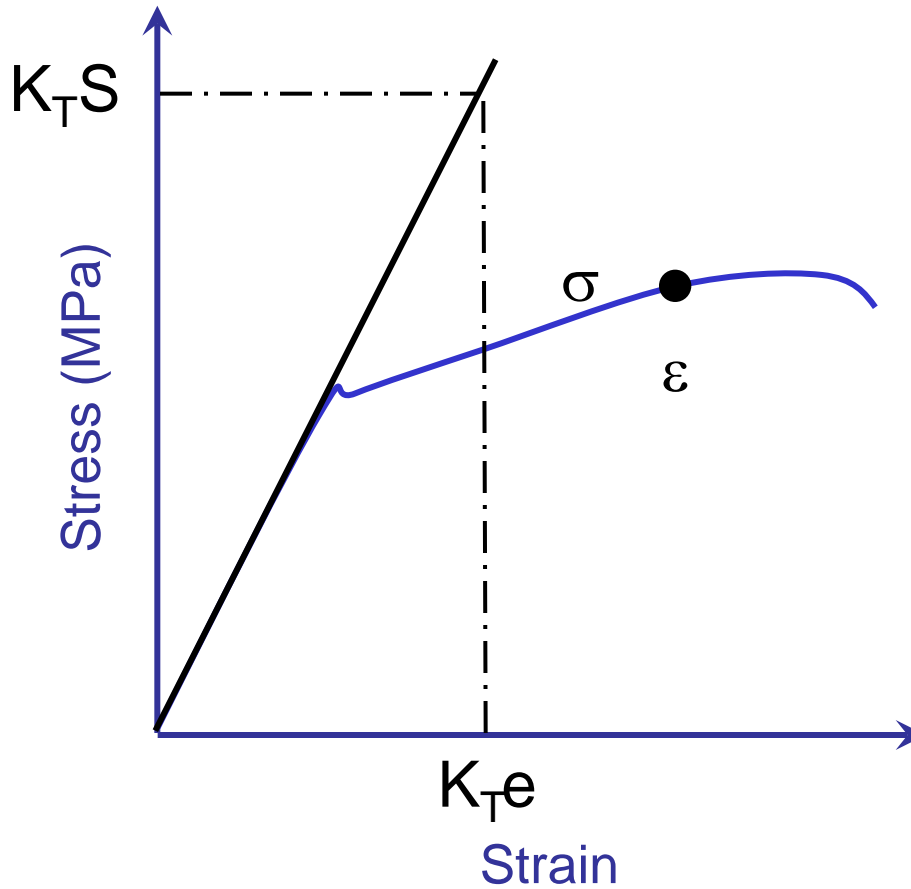
Define: nominal stress,  $S$   
nominal strain,  $e$   
notch stress,  $\sigma$   
notch strain,  $\epsilon$

Stress concentration  $K_\sigma = \frac{\sigma}{S}$

Strain concentration  $K_\epsilon = \frac{\epsilon}{e}$



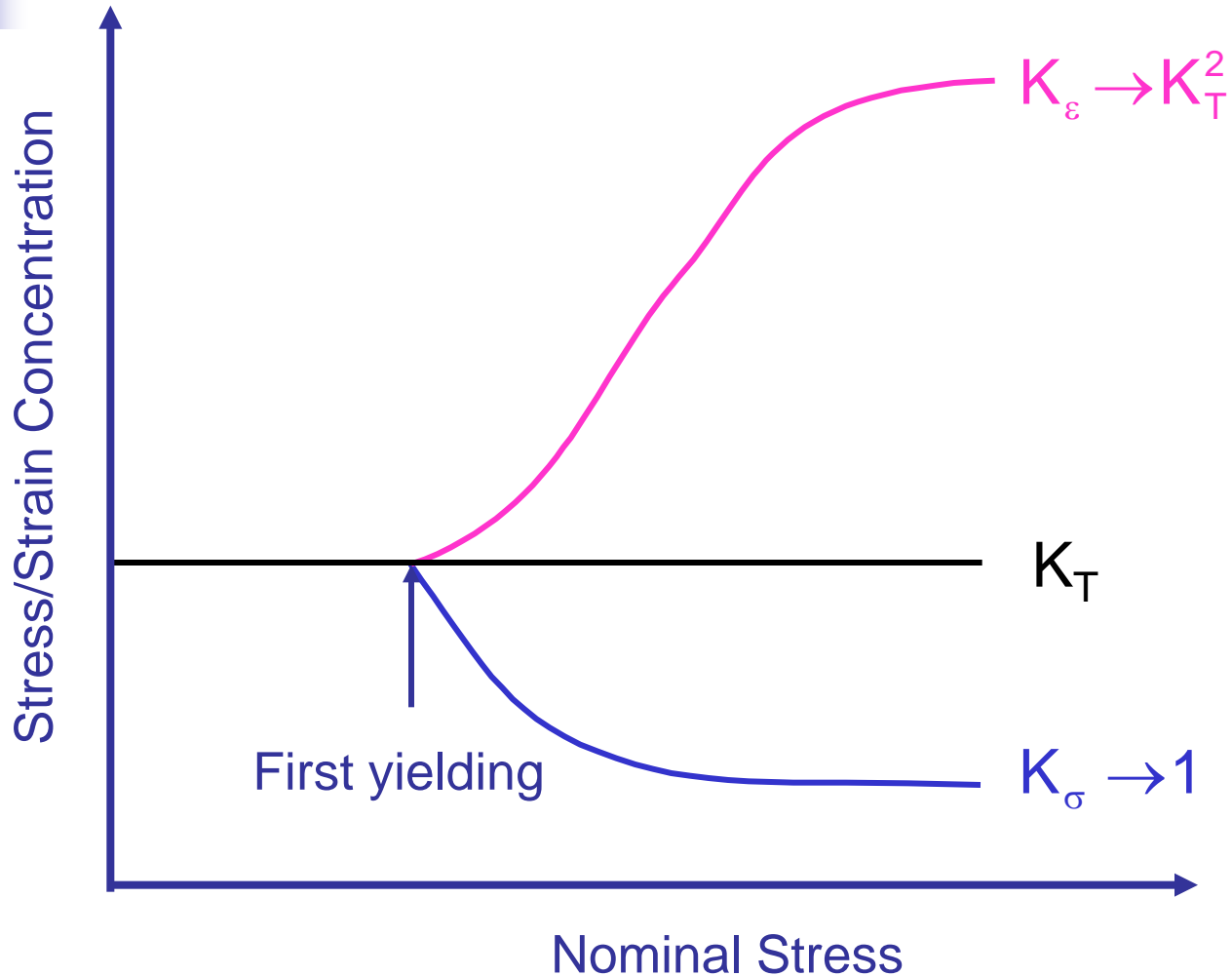
# $K_\sigma$ and $K_\epsilon$



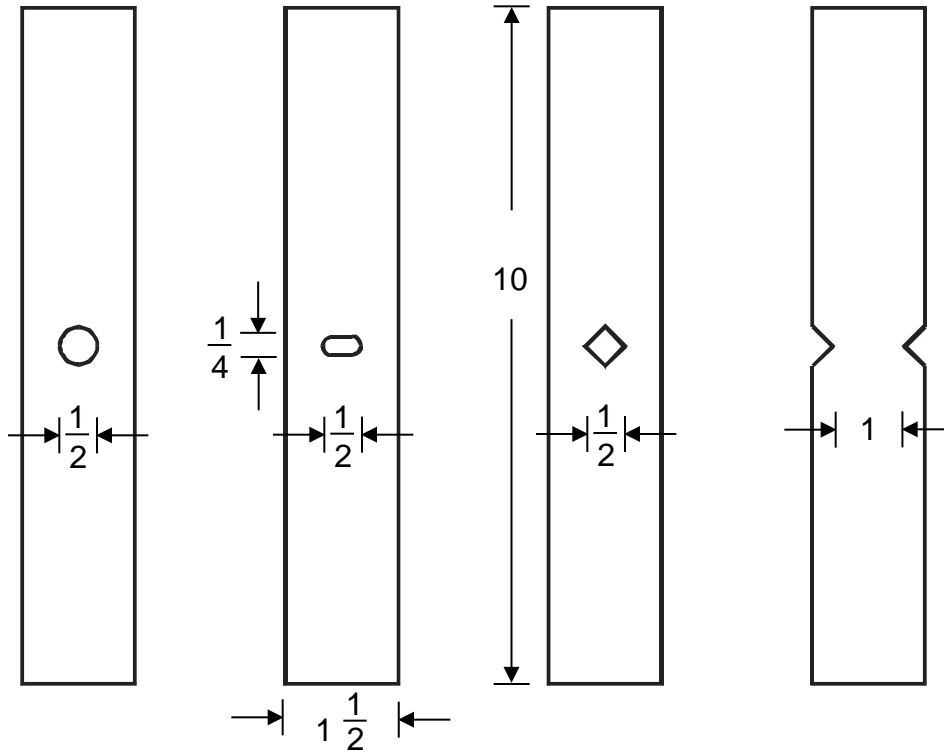
$$K_\sigma = \frac{\sigma}{S}$$

$$K_\epsilon = \frac{\epsilon}{e}$$

# Stress and Strain Concentration

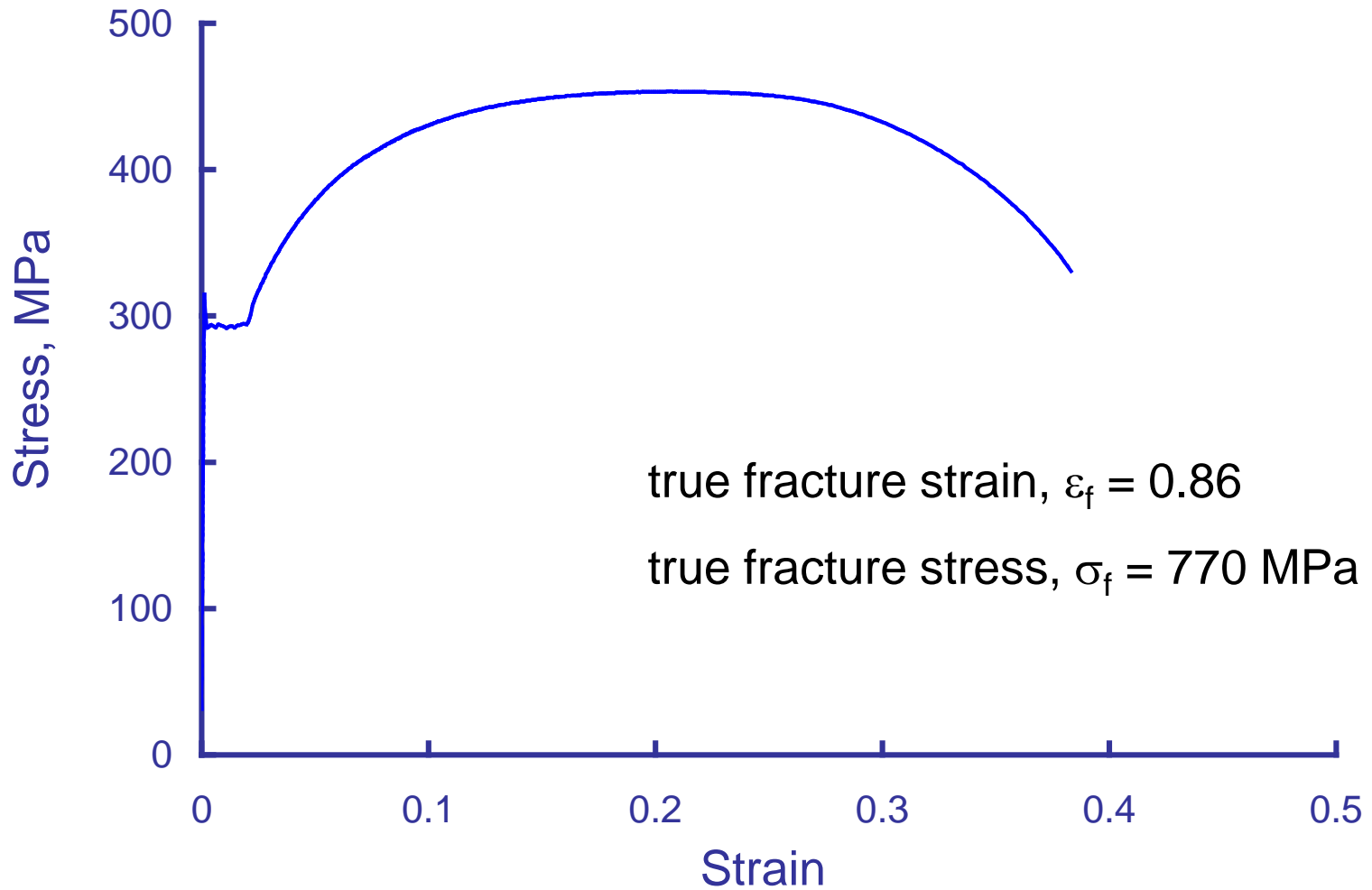


# Notched Plate Experiments

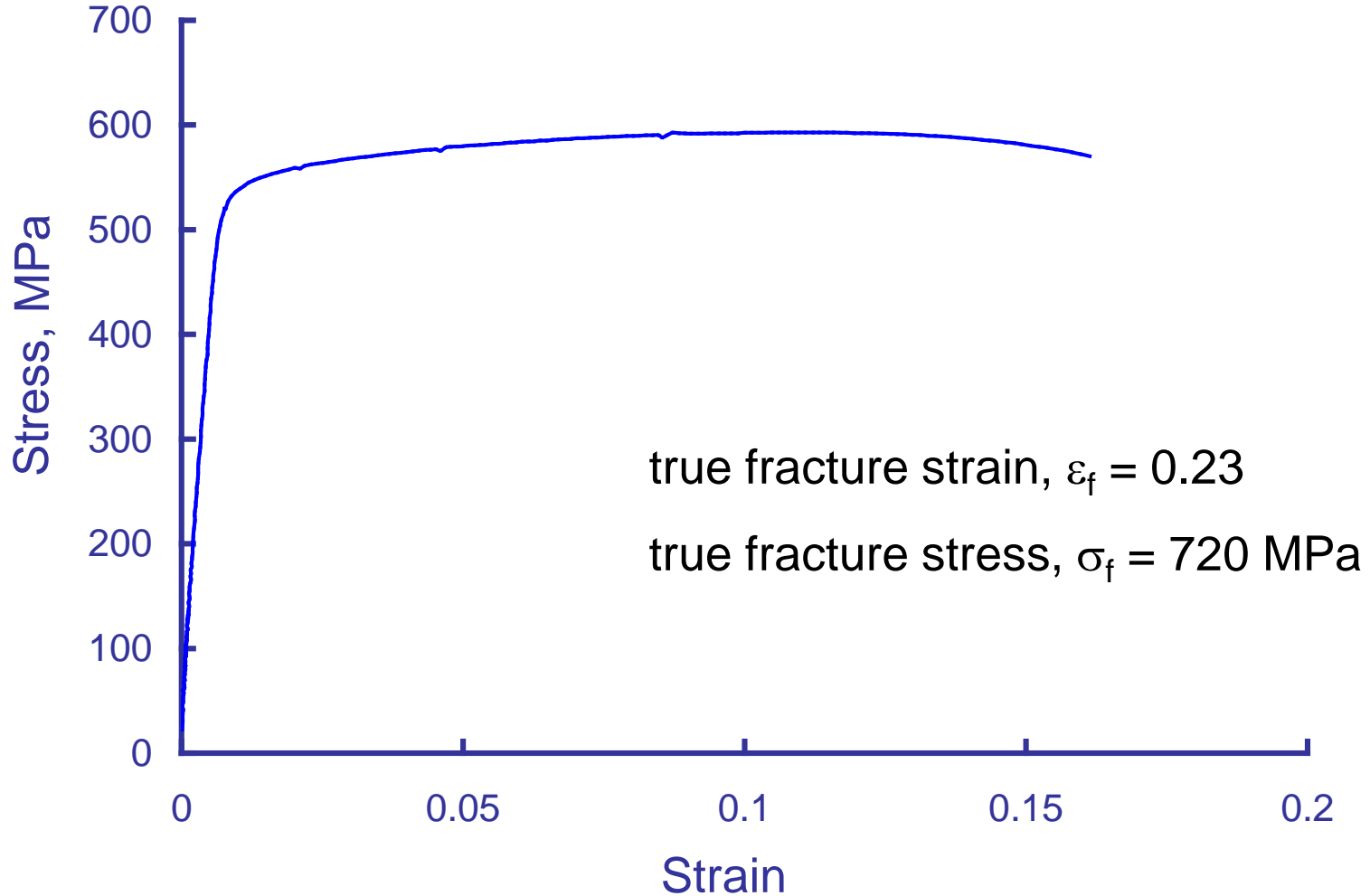


Materials:  
1018 Hot Rolled Steel  
7075-T6 Aluminum  
  
1/4 thick

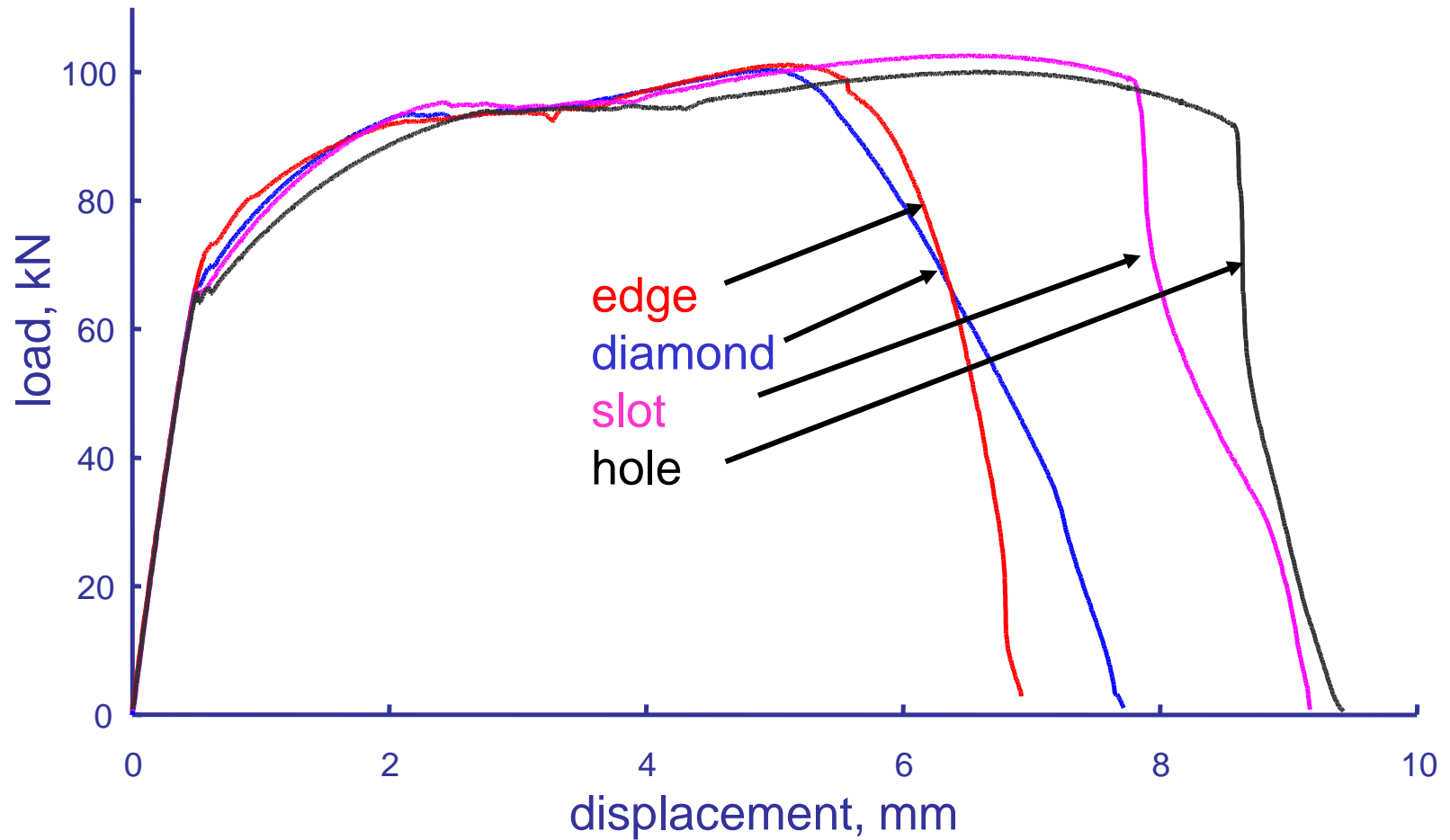
# 1018 Stress-Strain Curve



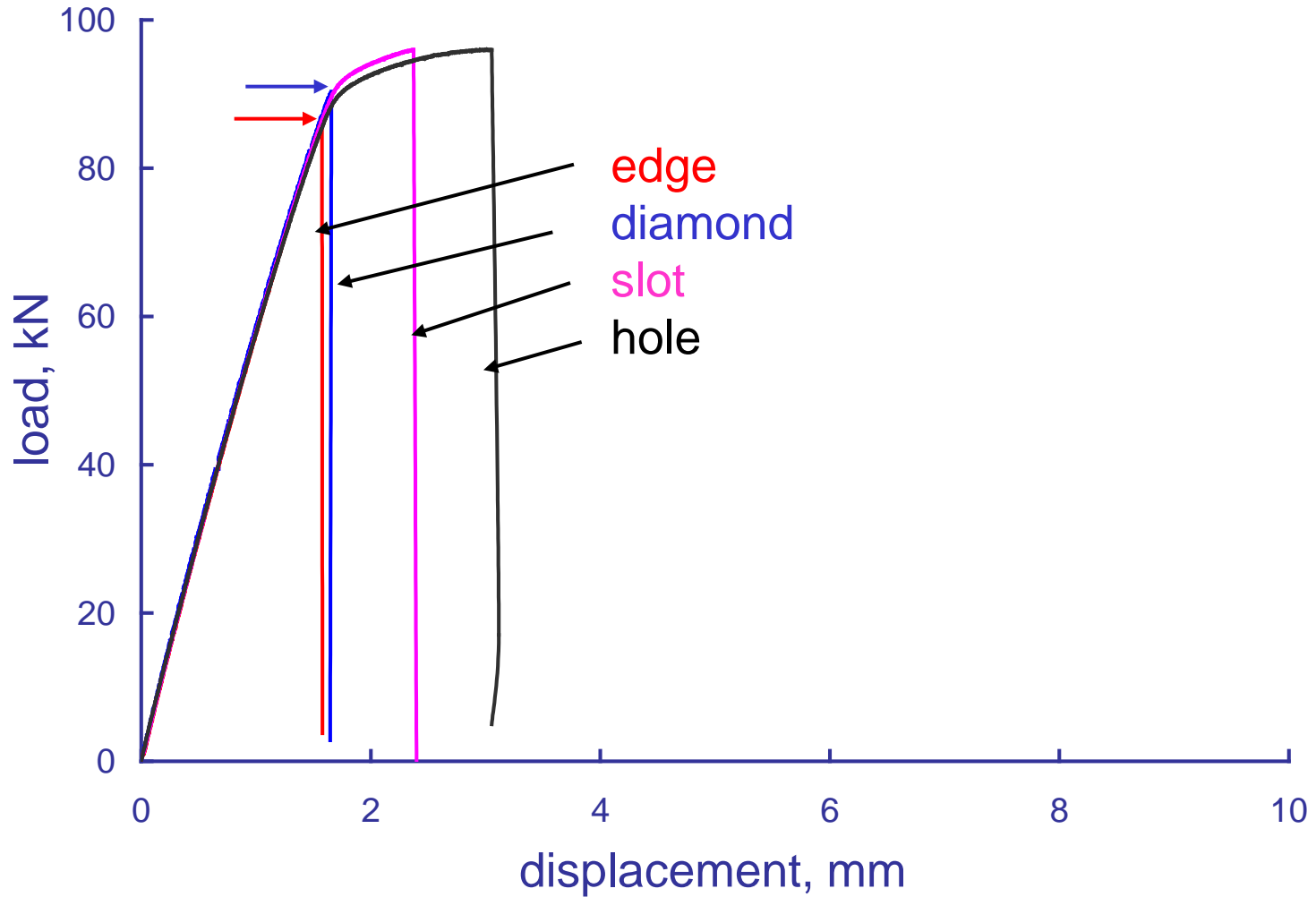
# 7075-T6 Stress-Strain Curve



# 1018 Steel Test Data



# 7075-T6 Test Data



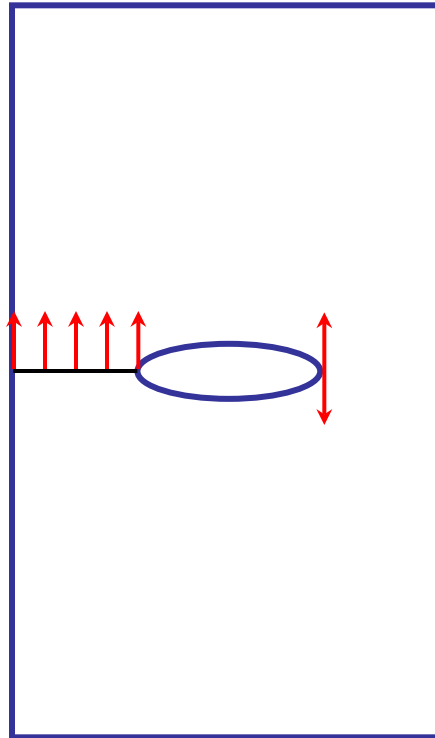
# Failure of a Notched Plate





# Failures from Stress Concentrations

Net section stresses must be below the flow stress



Notch strains must be below the fracture strain

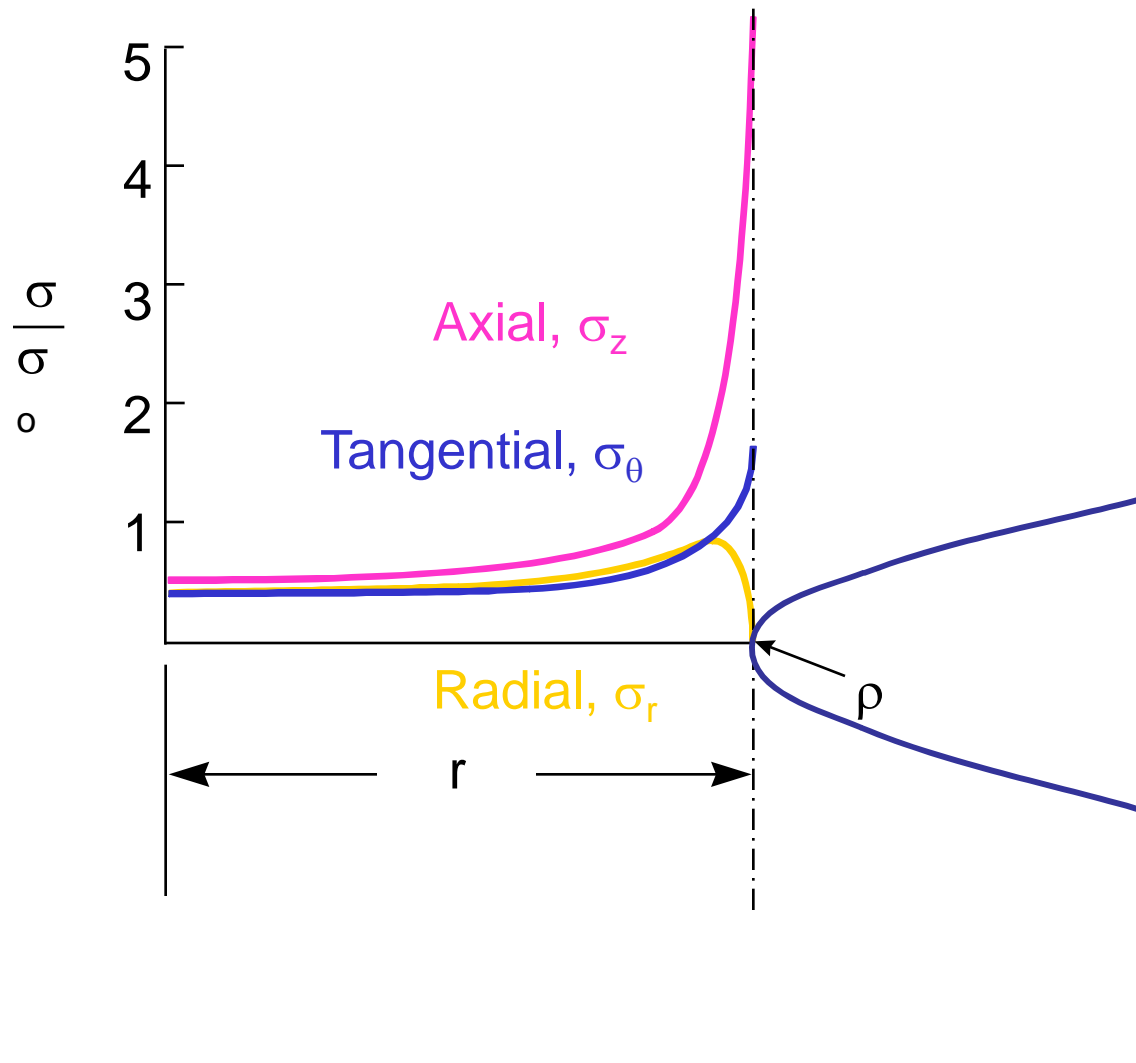


# 2D vs 3D

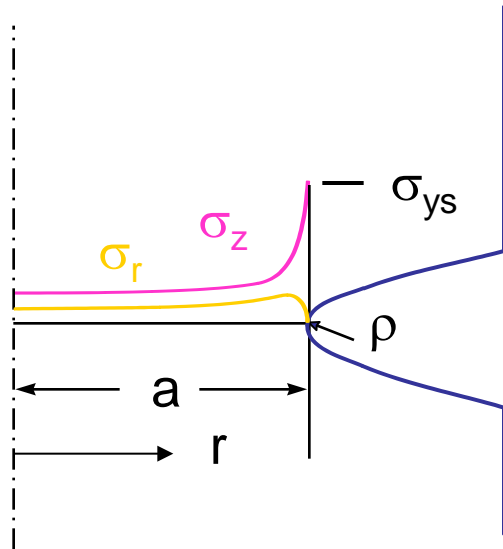
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- Plates and shells
  - 2D stress state
- Solids
  - 3D stress state

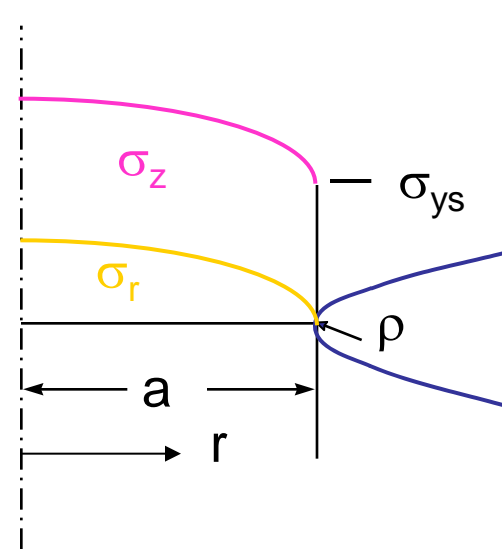
# Stress Concentration in a Bar



# Bridgeman Analysis (1943)



Elastic stress distribution



Plastic stress distribution

$$\tau = \frac{\sigma_z - \sigma_r}{2} = \text{constant}$$



# Stresses

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$$\sigma_z = \sigma_o \left[ 1 + \ln \left( \frac{a^2 + 2a\rho - r^2}{2a\rho} \right) \right]$$

$$P_z = \int_0^a 2\pi r \sigma_z dr$$

$$P_{\max} = \pi a^2 \sigma_{\text{flow}} \left( 1 + \frac{2\rho}{a} \right) \ln \left( 1 + \frac{a}{2\rho} \right)$$

$$P_{\max} = A_{\text{net}} \sigma_{\text{flow}} \text{CF}$$

CF constraint factor



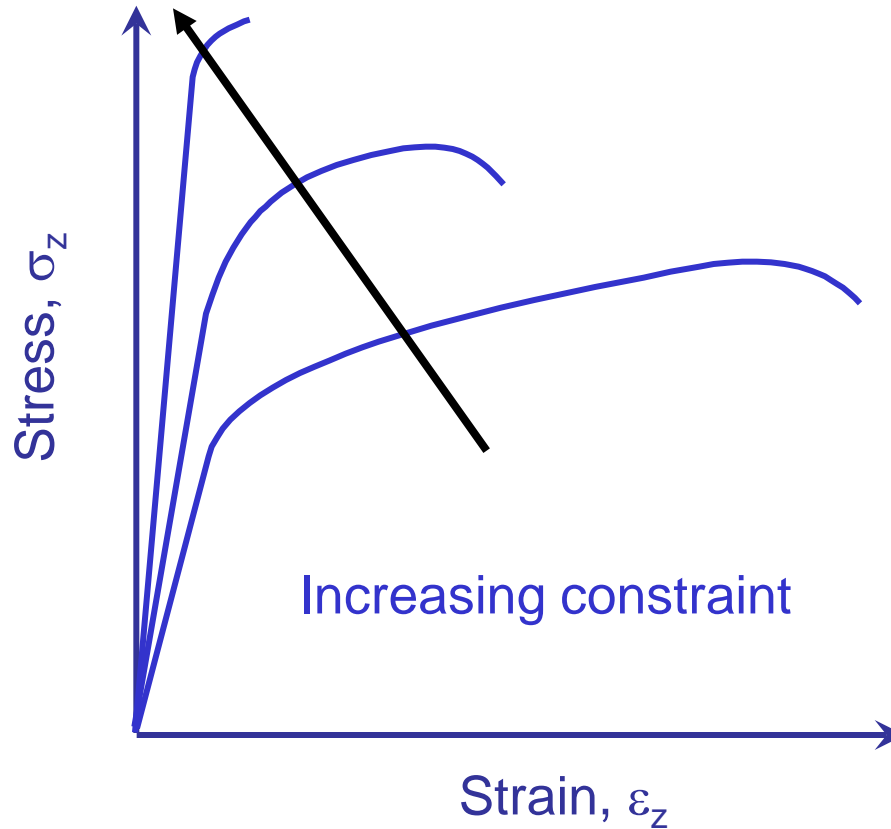
# Constraint Factors

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$a / \rho$	CF
0	1
1	1.21
2	1.38
4	1.64
8	1.73
20	2.63
$\infty$	2.96

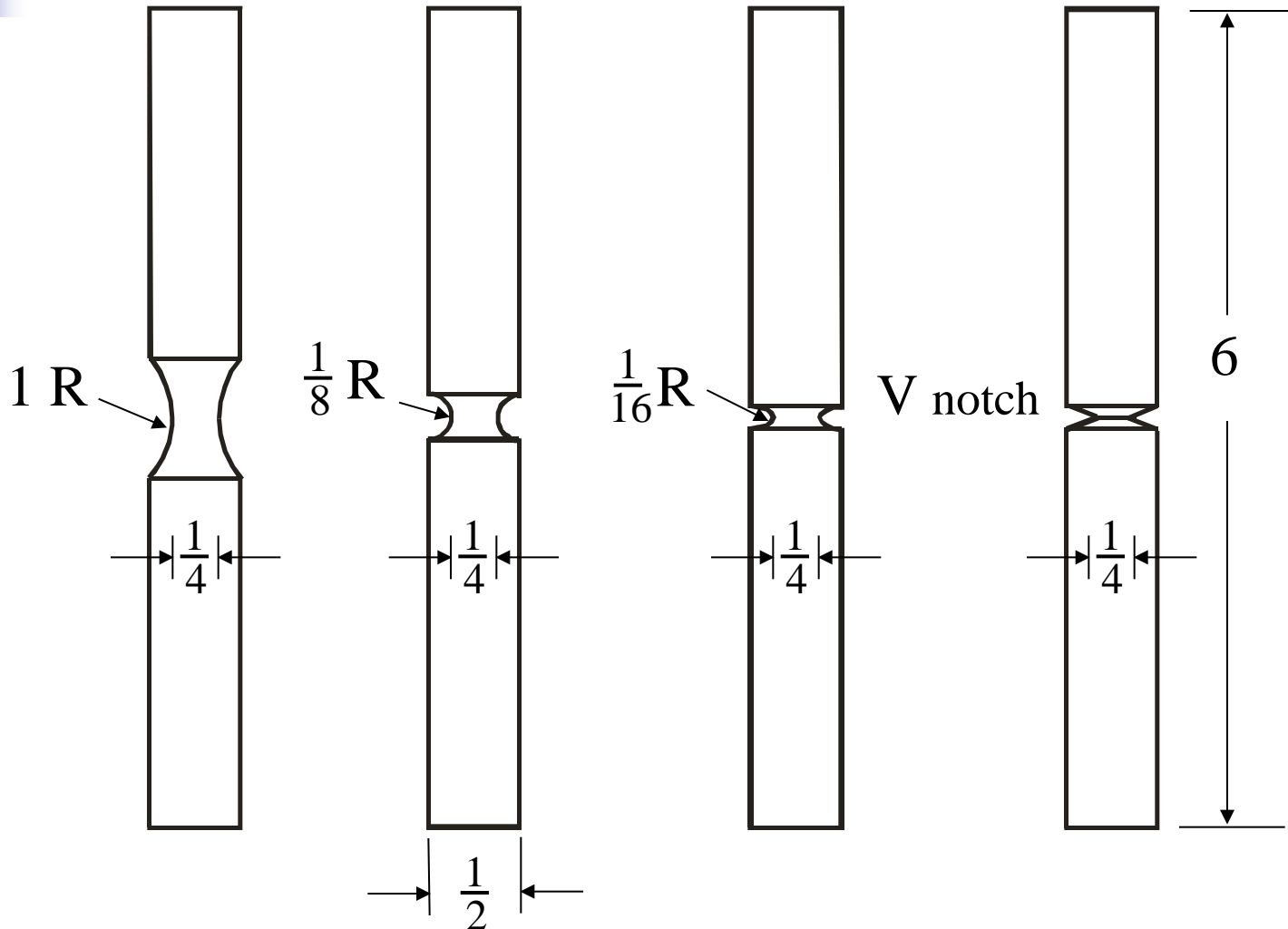
$$P_{\max} = A_{\text{net}} \sigma_{\text{flow}} \text{CF}$$

# Effect of Constraint



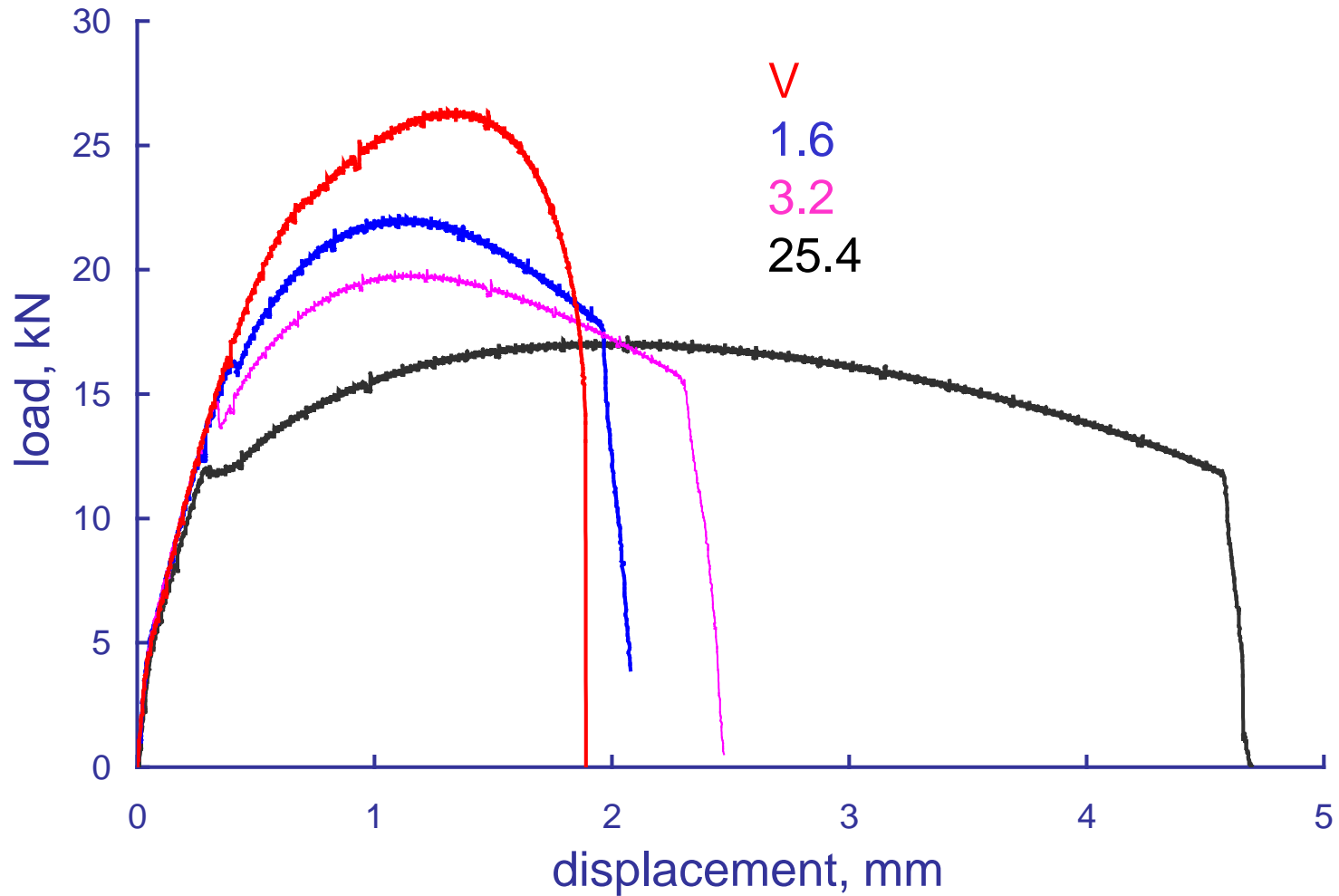
Higher strength and lower ductility

# Notched Bars

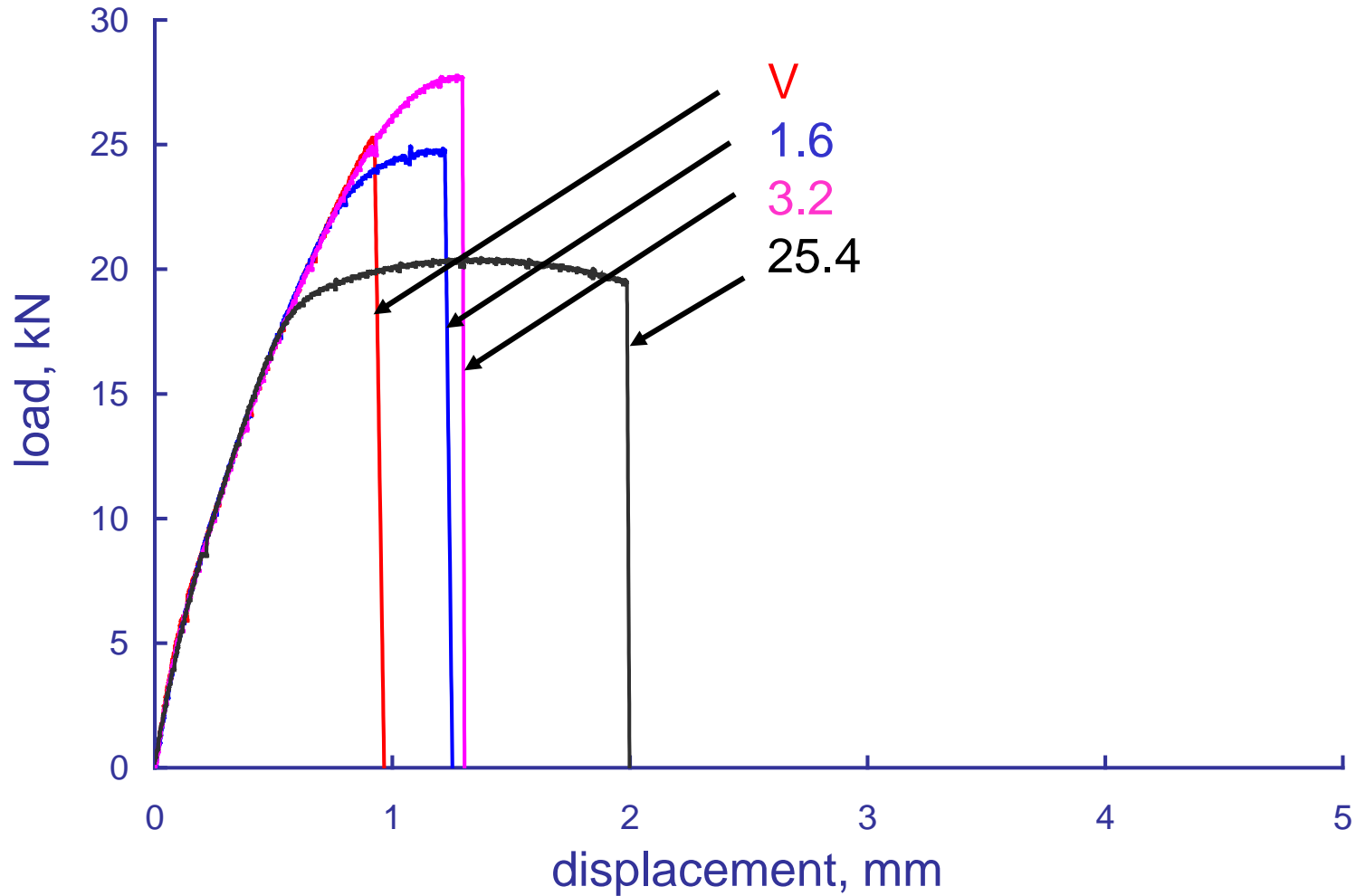




# 1018 Steel Test Data



# 7075-T6 Test Data





# Conclusion

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Net section area, state of stress and material strength control the failure load in a structure only in ductile materials. In brittle materials, cracks will form before the maximum load capacity of the structure is reached.

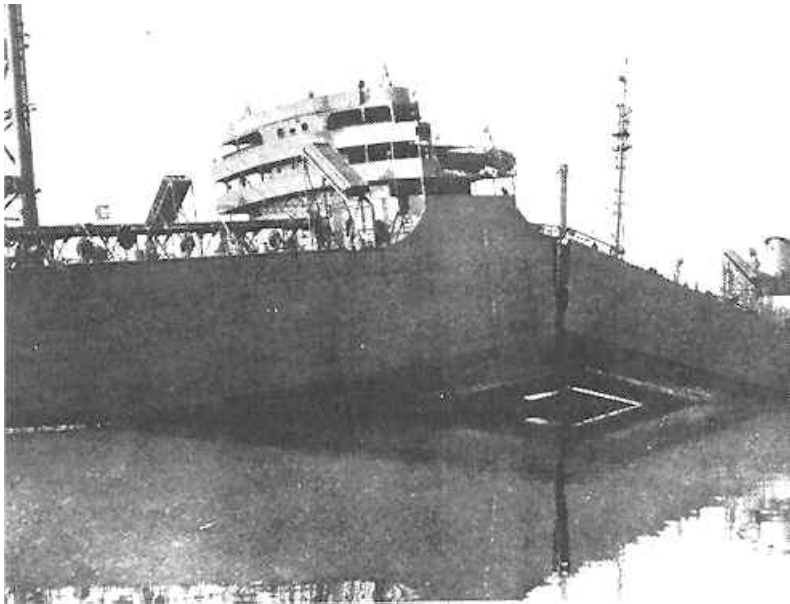


# Static Strength and Fracture

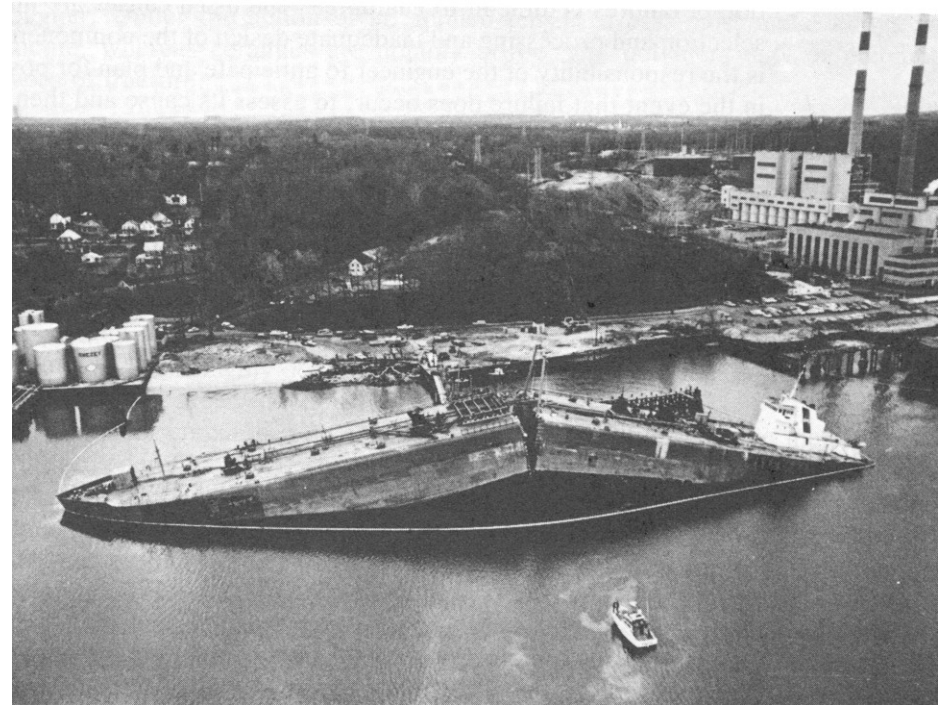
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- Stress Concentration Factors
- **Fracture Mechanics**
- Approximate Stress Intensity Factors
- Ductile vs. Brittle Fracture

# Fractures

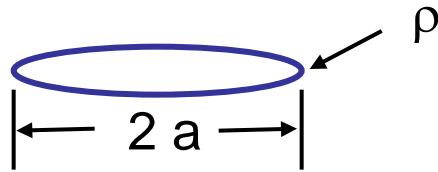


1943



1972

# Stress Concentration



$$K_T = 1 + 2\sqrt{\frac{a}{\rho}}$$

for a crack

$$a \sim 10^{-3}$$

$$\rho \sim 10^{-9}$$

$$K_T \sim 2000$$



# Fracture Mechanics Parameters

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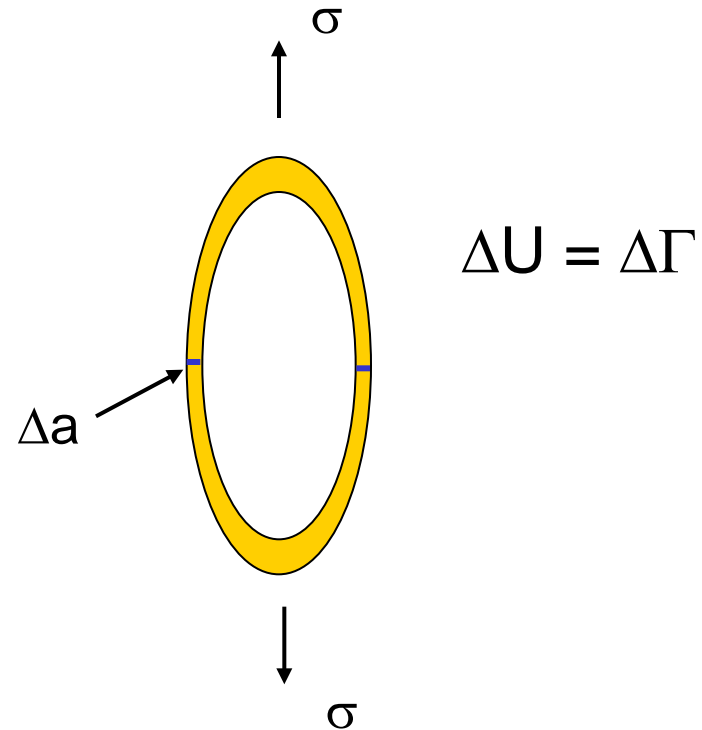
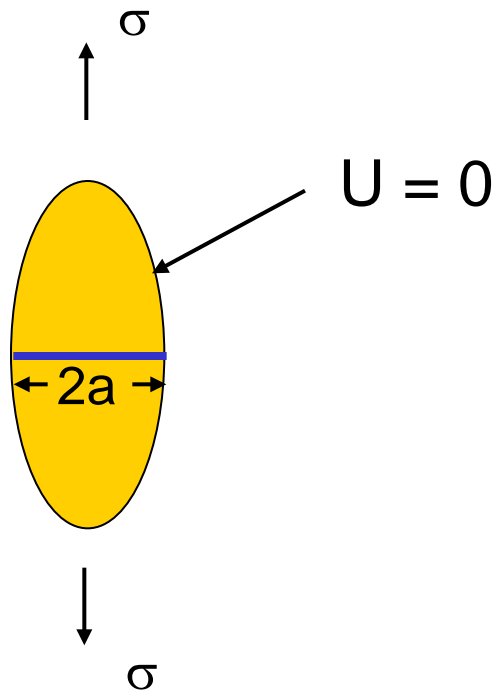
G strain energy release rate

K stress intensity factor

J J-integral

R crack growth resistance

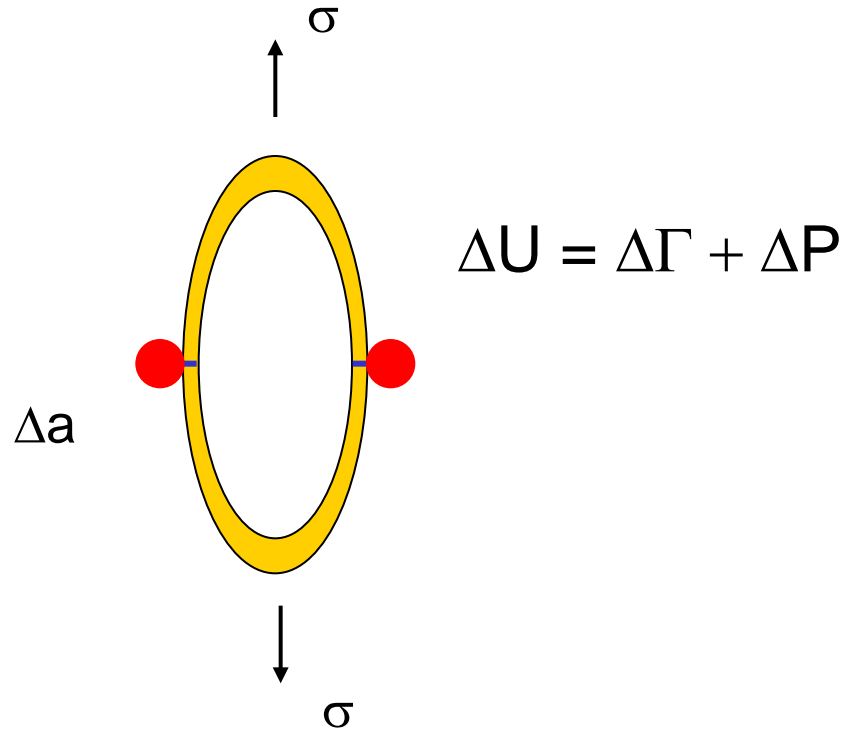
# Strain Energy Release Rate



strain energy  $\Rightarrow$  surface energy

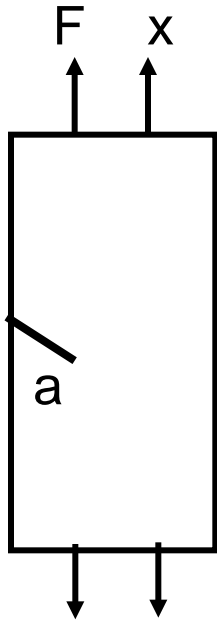


# Plastic Energy Term



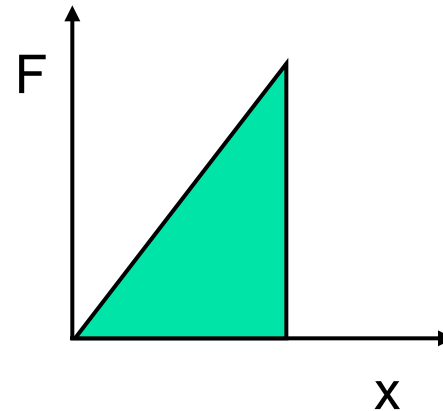
strain energy  $\Rightarrow$  surface energy + plastic energy

# Strain Energy Release Rate, G



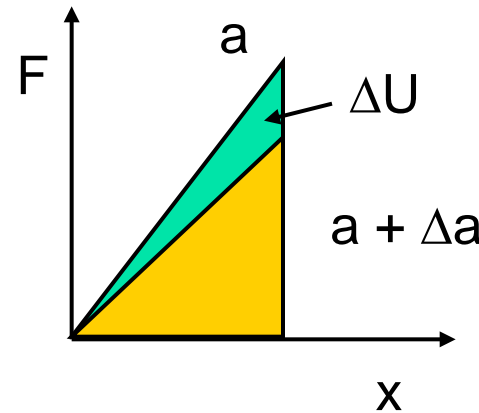
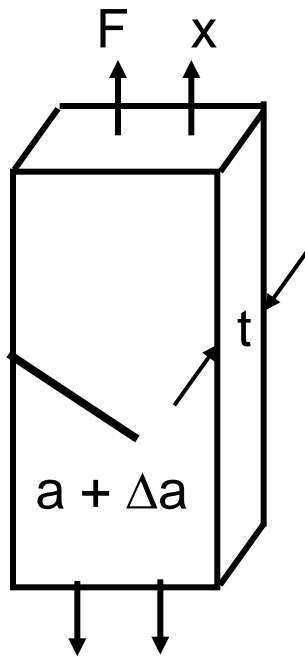
internal strain energy = external work

$$U = \frac{1}{2} F x$$



$$G = \frac{\partial U}{\partial a}$$

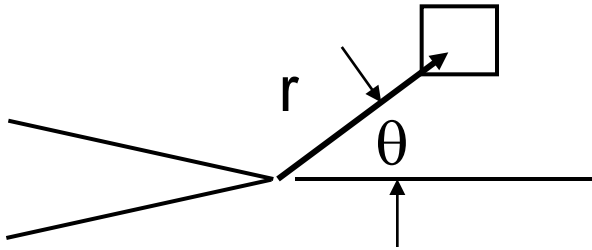
# Strain Energy Release Rate, $G$



$$G = - \frac{1}{t} \frac{\Delta U}{\Delta a}$$

$G$  is the energy per unit crack area needed to extend a crack

# Stress Intensity Factor, K



$$\sigma_x = \frac{K}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left[ 1 - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right]$$

$$\sigma_y = \frac{K}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left[ 1 + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right]$$

$$\tau_{xy} = \frac{K}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \sin \frac{\theta}{2} \cos \frac{3\theta}{2}$$

$$\varepsilon_x = \frac{\partial u}{\partial x}$$

$$\varepsilon_y = \frac{\partial v}{\partial y}$$

$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}$$

$$u = \frac{K}{2G} \sqrt{\frac{r}{2\pi}} \cos \frac{\theta}{2} \left[ \frac{3-\nu}{1+\nu} - 1 + 2 \sin^2 \frac{\theta}{2} \right]$$

$$v = \frac{K}{2G} \sqrt{\frac{r}{2\pi}} \sin \frac{\theta}{2} \left[ \frac{3-\nu}{1+\nu} - 1 + 2 \cos^2 \frac{\theta}{2} \right]$$



# Design Philosophy

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Stress < Strength

$$\sigma < \sigma_y$$

Stress Intensity < Fracture Toughness

$$K < K_{Ic}$$

Two cracks with the same K will have the same behavior

# Fracture Toughness

$$K_{IC} = \sigma \sqrt{\pi a} f\left(\frac{a}{w}\right)$$

↑ fracture toughness

↑ operating stresses

↑ flaw size

↑ flaw shape



# K and G

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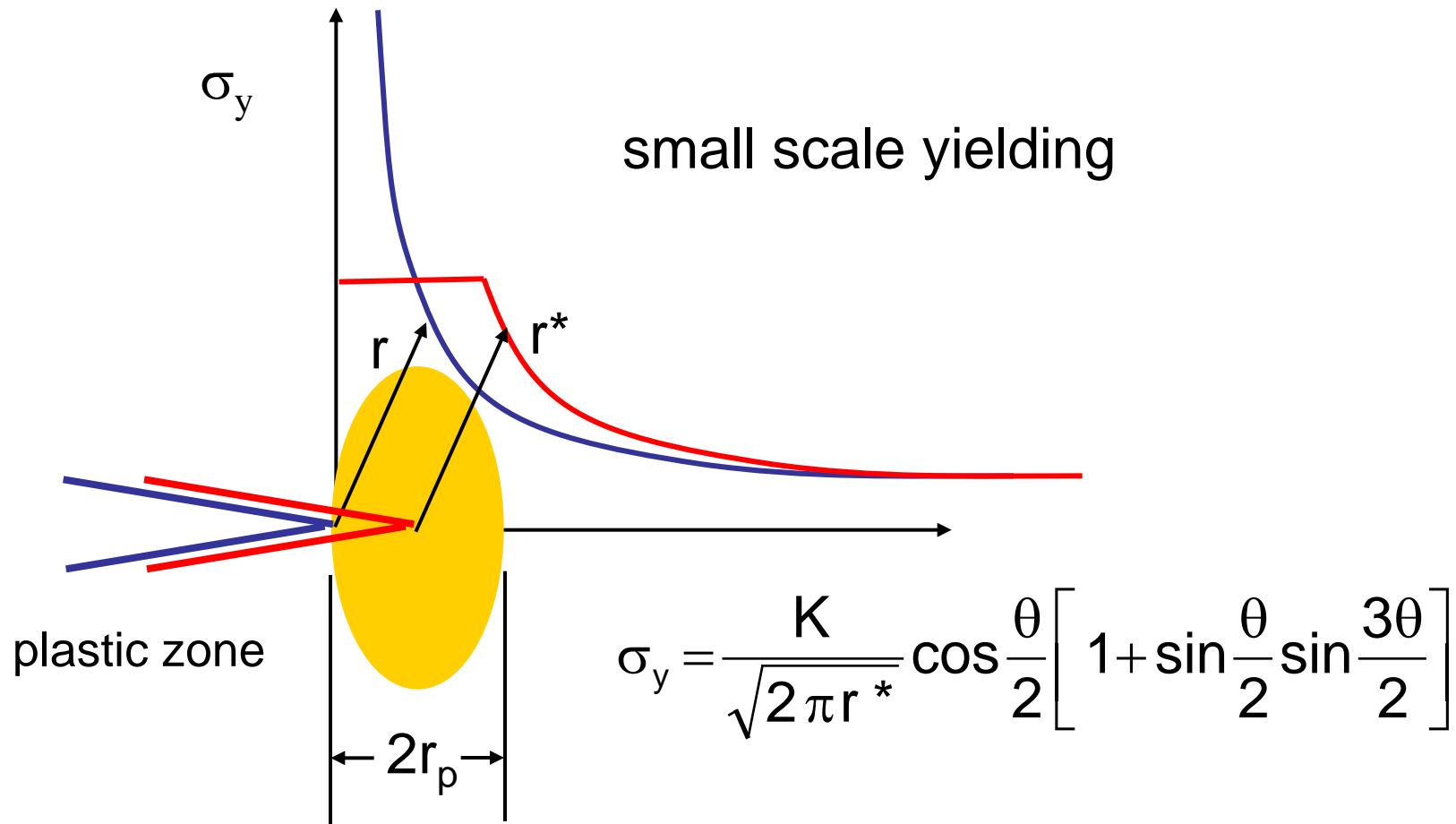
$$G = \frac{K^2}{E}$$

plane stress

$$G = \frac{(1 - \nu^2)K^2}{E}$$

plane strain

# Elastic-Plastic Stress Field







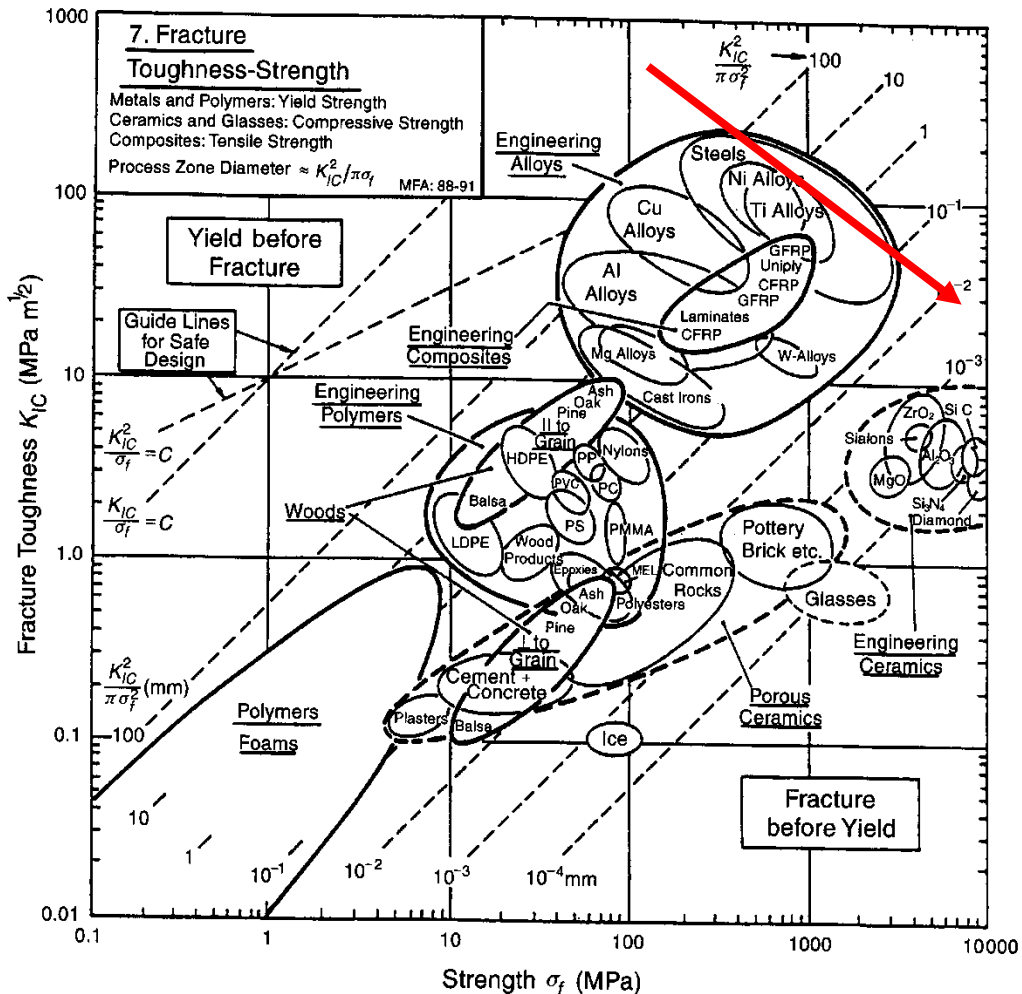
# Critical Crack Sizes

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What would the critical crack size be in a standard tensile test ?

$$a_{\text{critical}} = \frac{1}{\pi} \left( \frac{K_{IC}}{\sigma_f} \right)^2$$

# Fracture Toughness

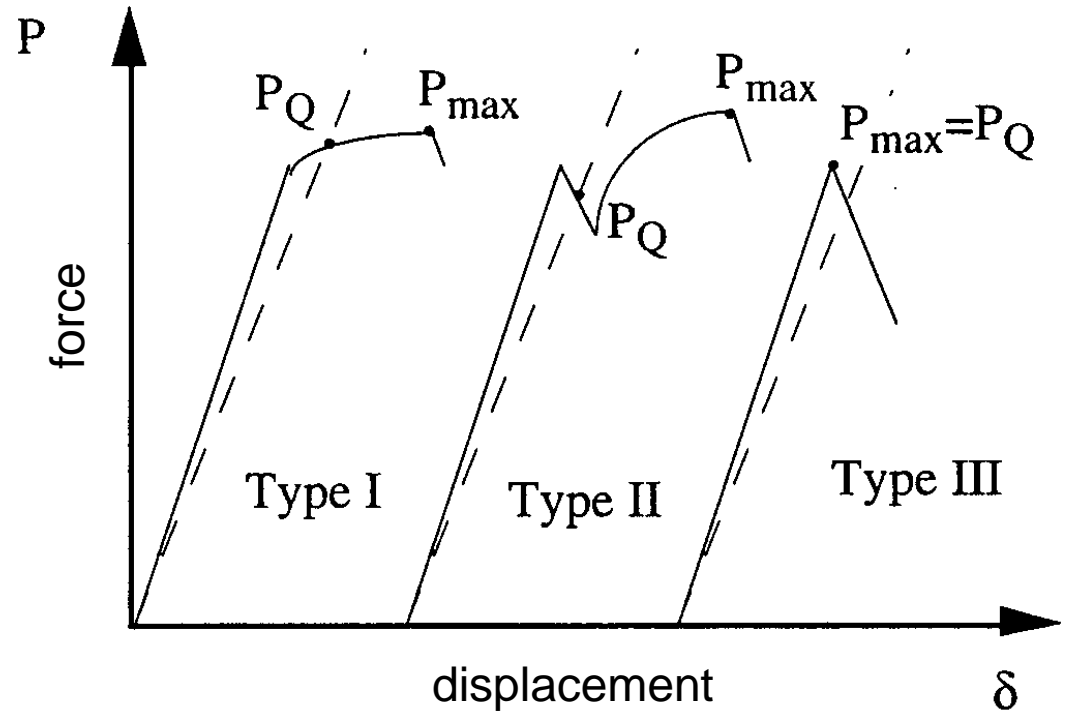


From M F Ashby, Materials Selection in Mechanical Design, 1999, pg 431

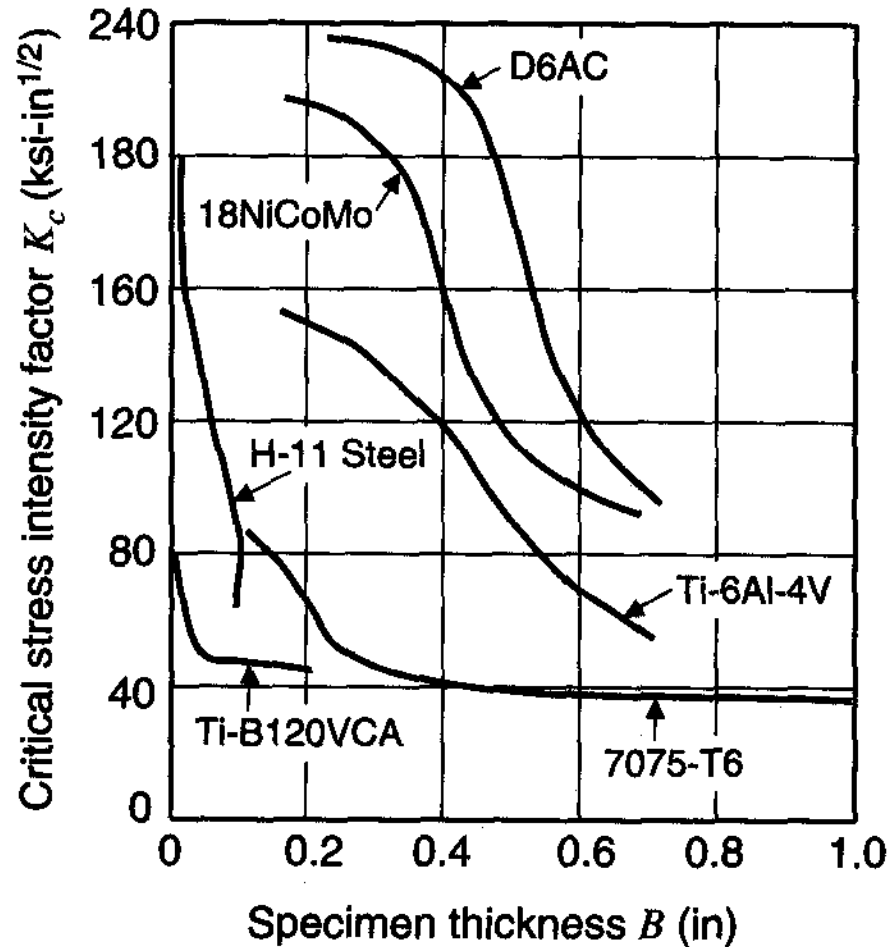
# Measuring Fracture Toughness



[www.enduratec.com](http://www.enduratec.com)

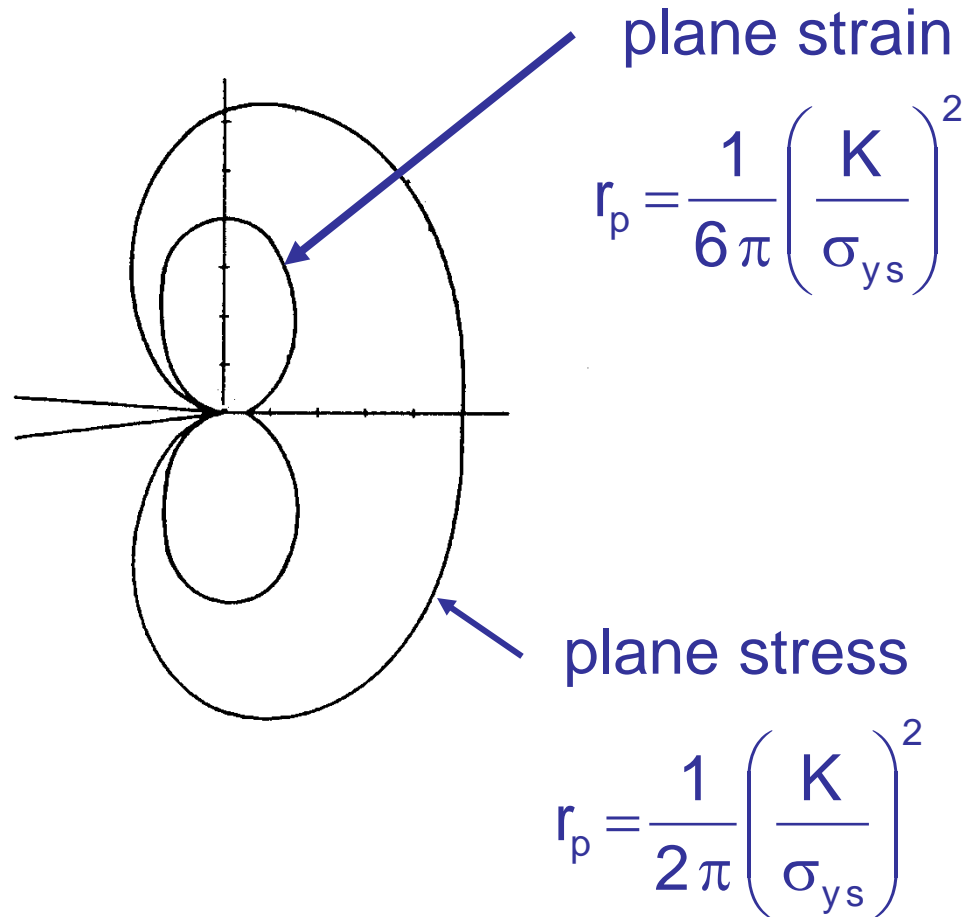


# Thickness Effects

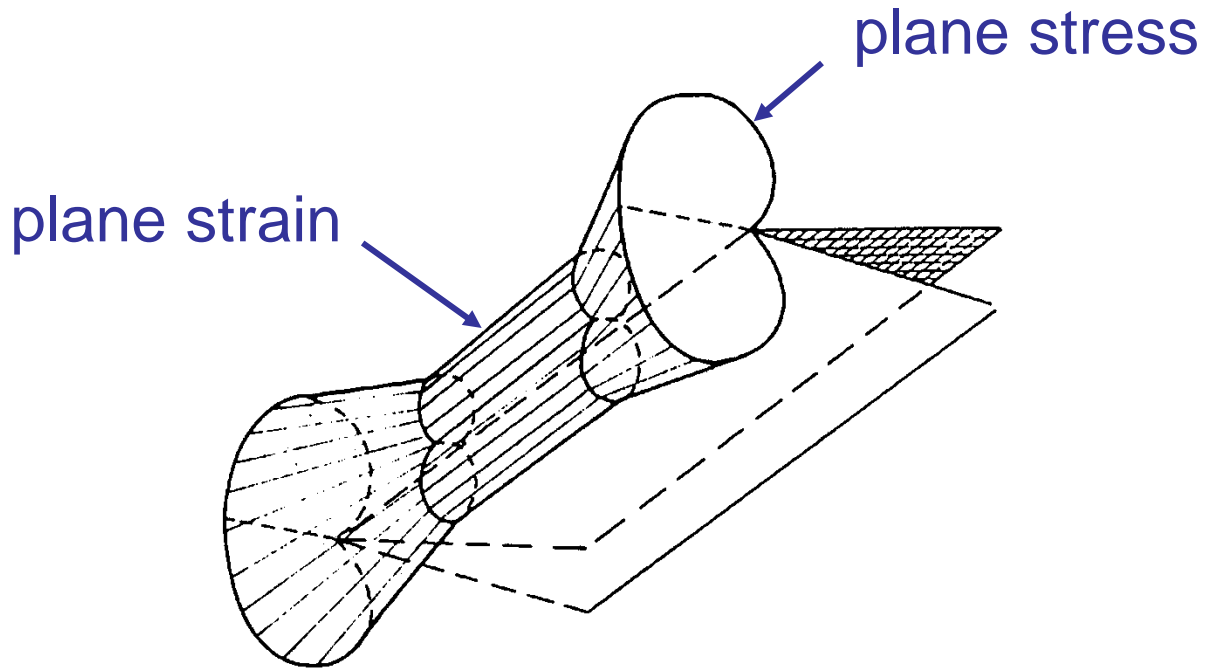


From Wilhem "Fracture Mechanics Guidelines for Aircraft Structural Applications" AFFDL-TR-69-111

# Plastic Zone Size



# 3D Plastic Zone

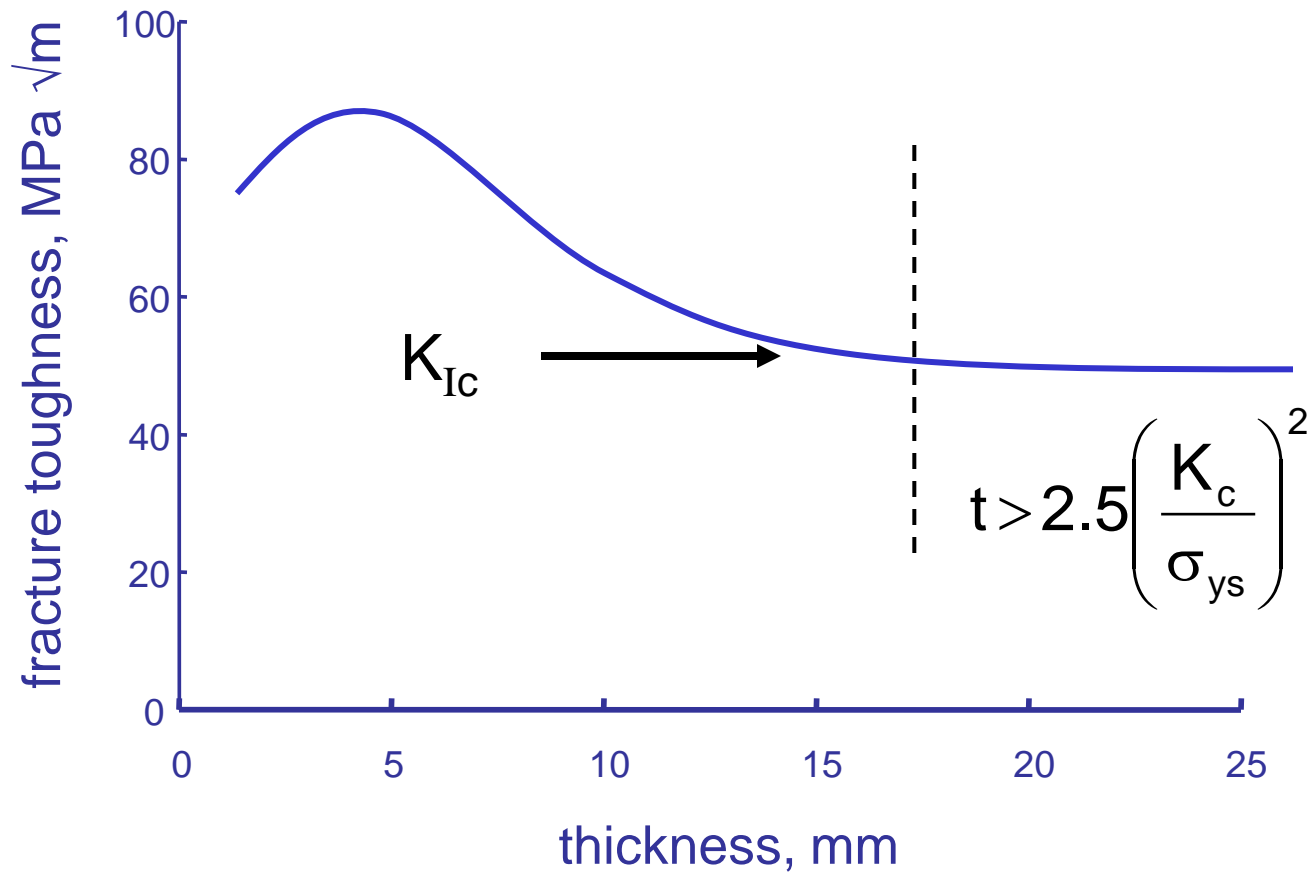




# Fracture Surfaces



# Thickness Requirements







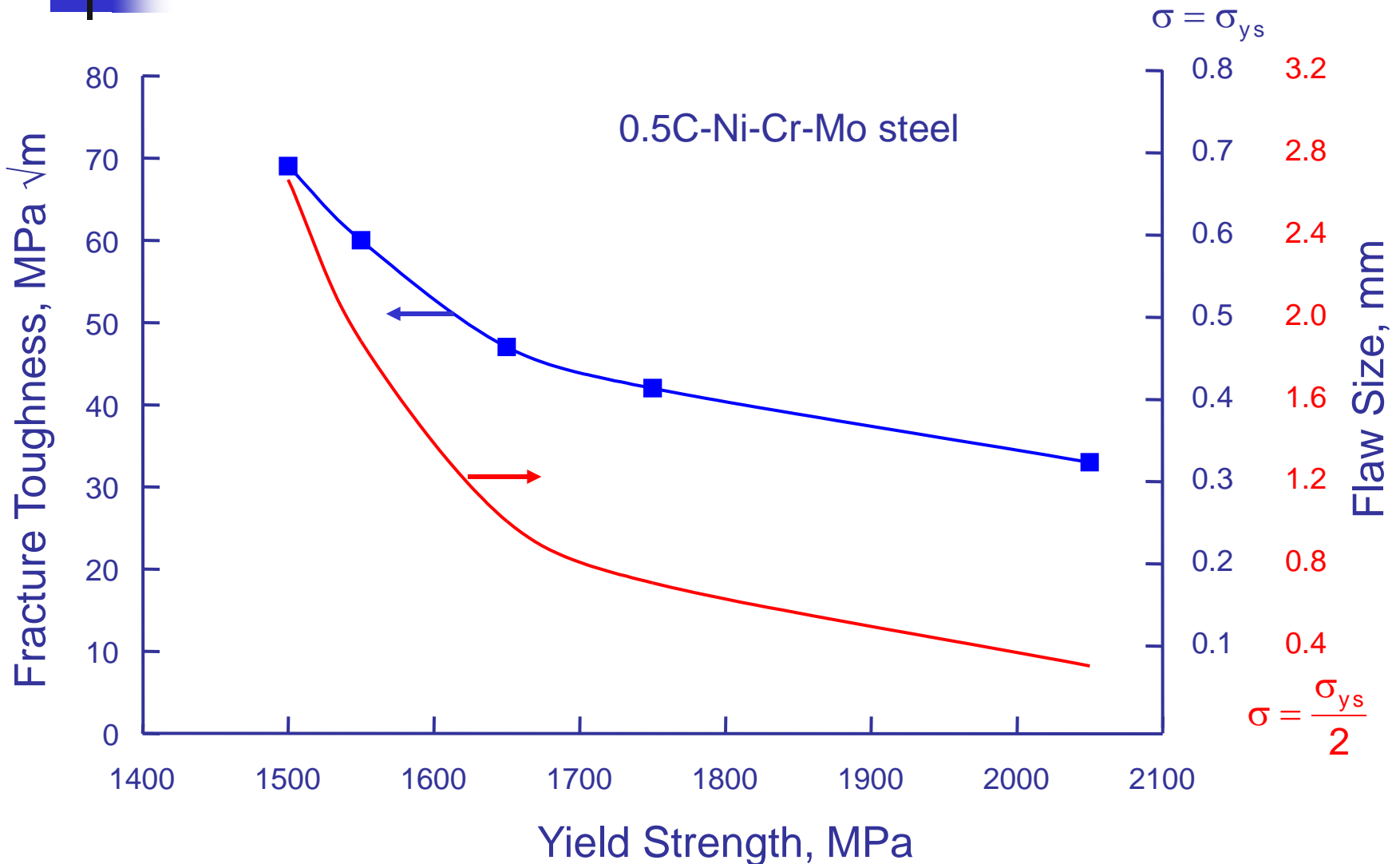
# Size Requirements

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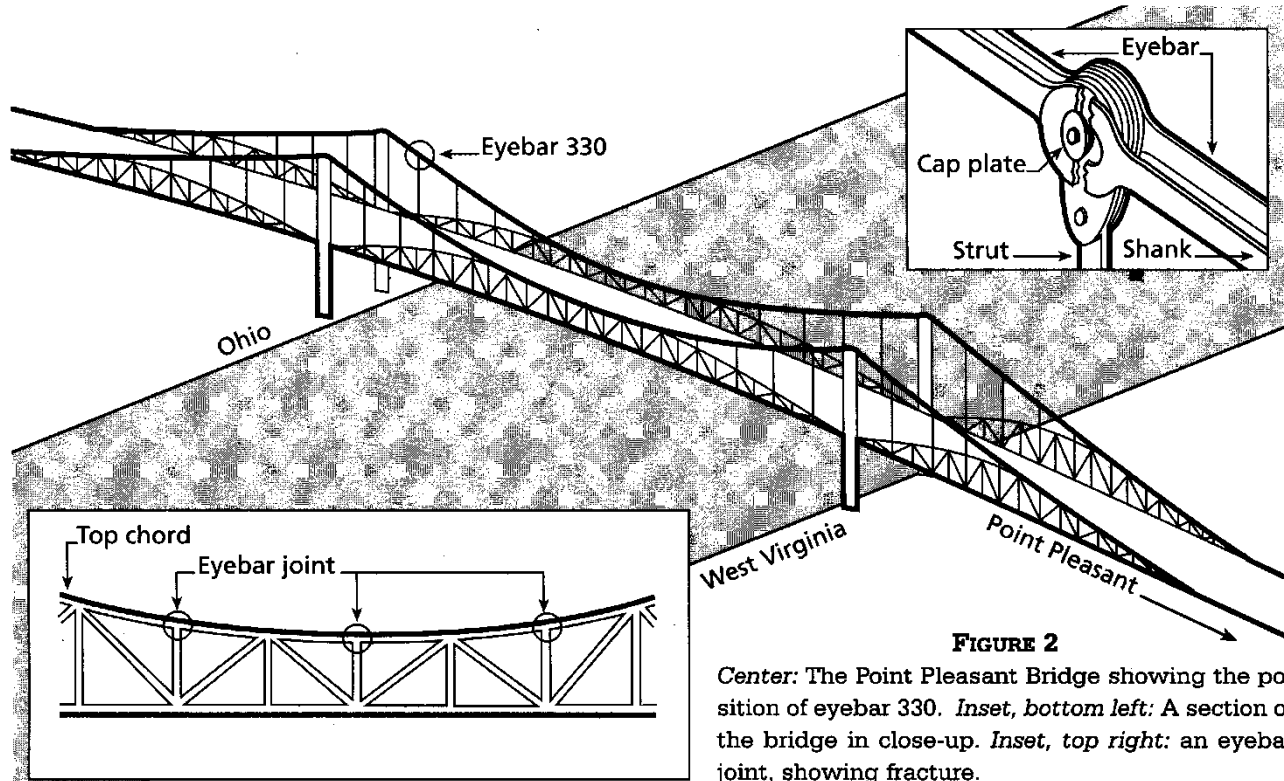
$$t, W - a, a \geq 2.5 \left( \frac{K}{\sigma_{ys}} \right)^2 \approx 50r_p$$

	$\sigma_{ys}$	$K_{Ic}$	t, mm
2024- T3	345	44	40.7
7075 -T6	495	25	6.4
Ti-6Al-4V	910	105	33.3
Ti-6Al-4V	1035	55	7.1
4340	860	99	33.1
4340	1510	60	3.9
17-7 PH	1435	77	7.2
52100	2070	14	0.1

# Strength, Toughness, Flaw Size

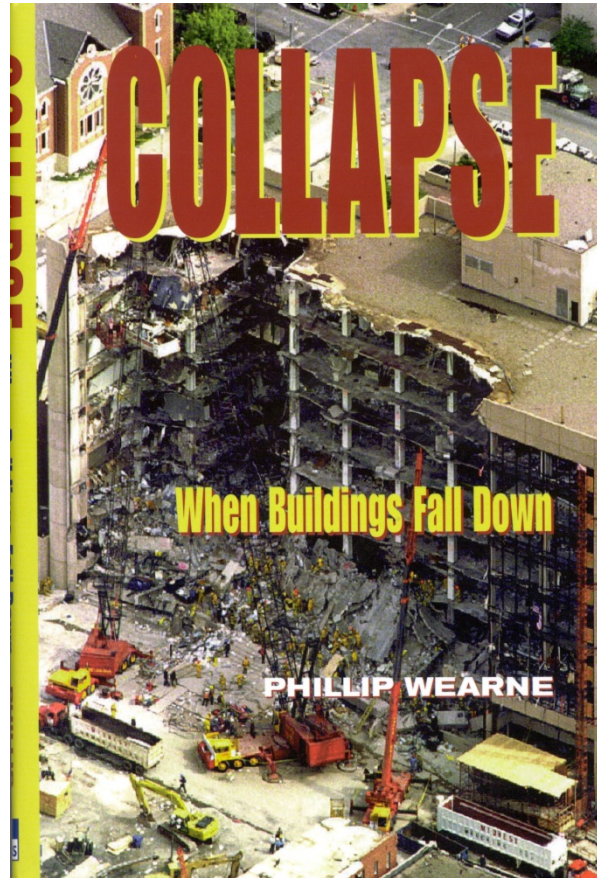


# Silver Bridge

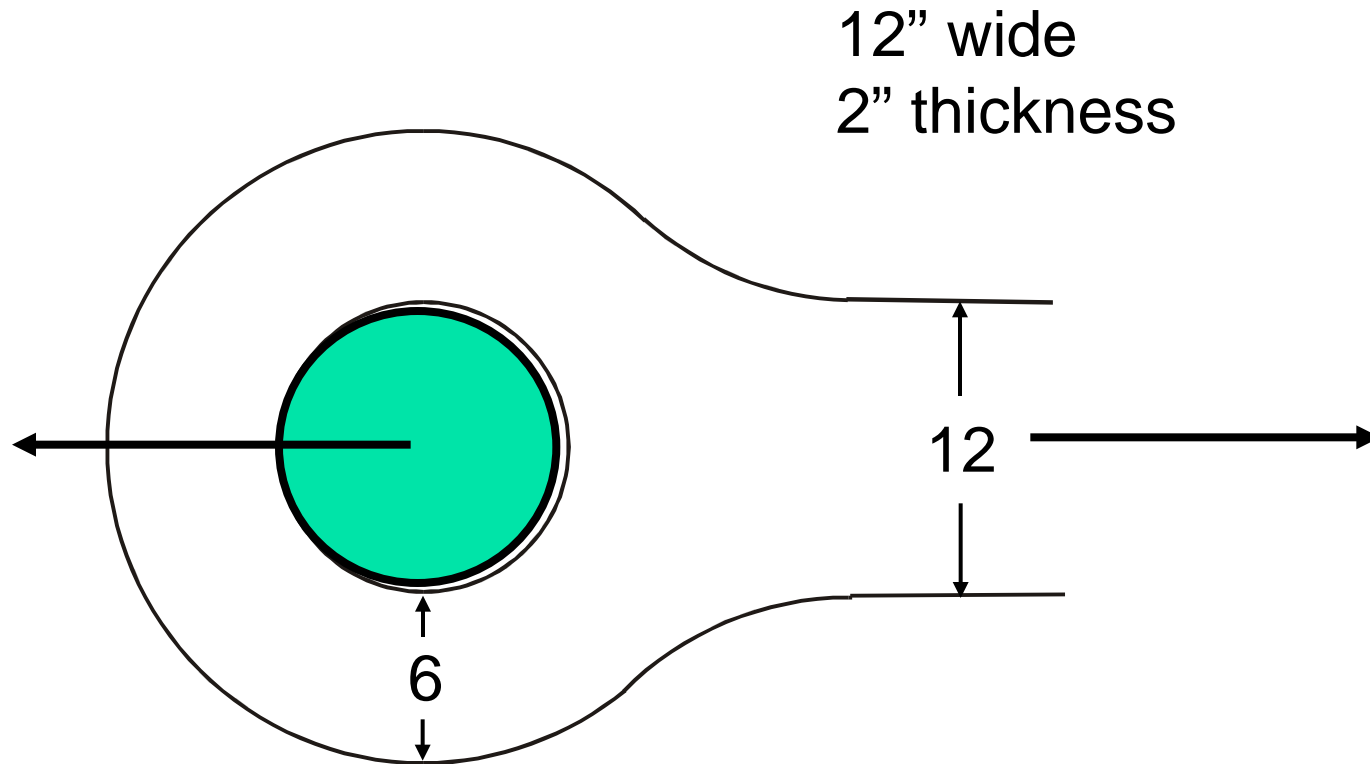


Collapse, Wearne, P. TV Books, NY 1999

# Source



# Eyebar





# Material Properties

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$$\sigma_u = 100 \text{ ksi}$$

$$\sigma_y = 75 \text{ ksi}$$

Working stress 50 ksi

CVN = 2.6 ft-lb at 32° F

CVN = 8.6 ft-lb at 165° F



# Fracture Toughness

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Barsom-Rolfe  $\frac{K_{IC}^2}{E} = 2(CVN)^{\frac{3}{2}}$  (psi – in, ft – lb)

Corten-Sailors  $K_{IC} = 15.5\sqrt{CVN}$  (ft – lb)

Roberts-Newton  $K_{IC} = 9.35CVN^{1.65}$  (ft – lb)

Barsom-Rolfe  $K_{IC} = 15.9\text{ksi}\sqrt{\text{in}}$

Corten-Sailors  $K_{IC} = 25.0\text{ksi}\sqrt{\text{in}}$

Roberts-Newton  $K_{IC} = 45.2\text{ksi}\sqrt{\text{in}}$

Average 28.7



# Critical Crack Size

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Assume a corner crack

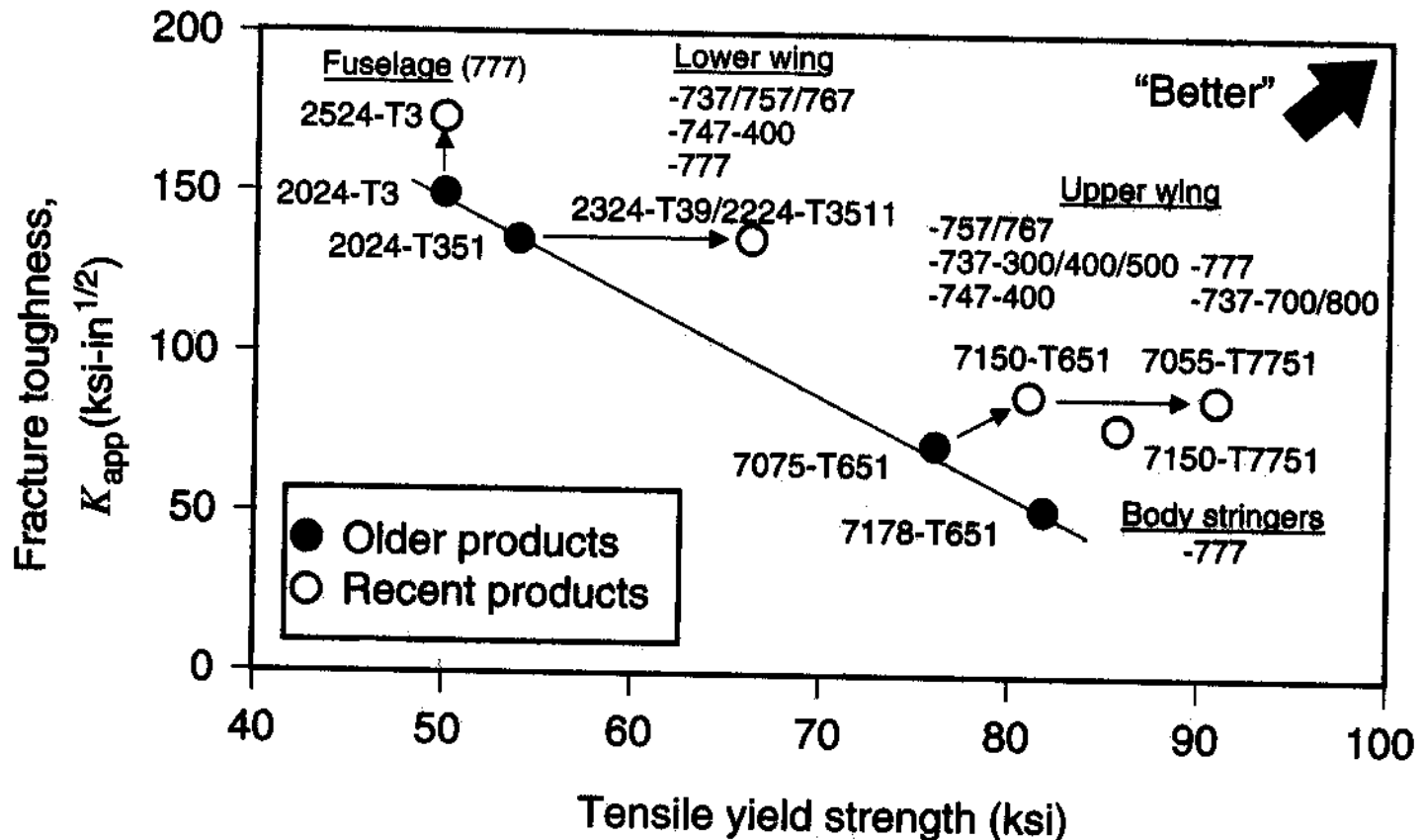
$$K_{IC} = \sigma (1.12)^2 \frac{2}{\pi} \sqrt{\pi a}$$

Let  $\sigma = \sigma_y$        $a \sim 0.073$  inches

Let  $\sigma = 50$        $a \sim 0.163$  inches



# Modern Aircraft Materials



Bucci et. al., "Need for New Materials in Aging Aircraft Structures" Journal of Aircraft, Vol. 37, 2000, 122-129



# Summary

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Both stress and flaw size govern fracture

$$K_{Ic} > \sigma \sqrt{\pi a} f\left(\frac{a}{W}\right)$$



# Static Strength and Fracture

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- Stress Concentration Factors
- Fracture Mechanics
- **Approximate Stress Intensity Factors**
- Ductile vs. Brittle Fracture



# Stress Intensity Factors

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- Analytical
  - Theory of elasticity
- Numerical
  - Finite element
- Experimental
  - Compliance
- Handbook
- Approximate



# Stress Intensity Factors

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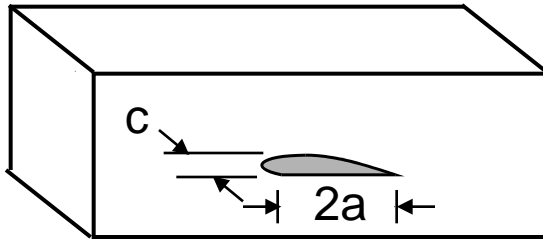
$$K = \frac{M_s M_t}{\Phi} \sigma \sqrt{\pi a}$$

$M_s$  free surface effects

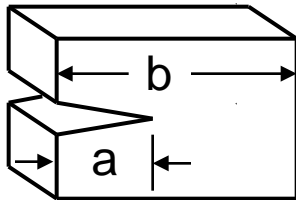
$M_t$  back surface effects

$\Phi$  crack shape effects

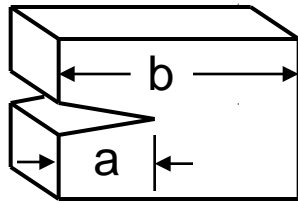
# $M_s$ free surface effects



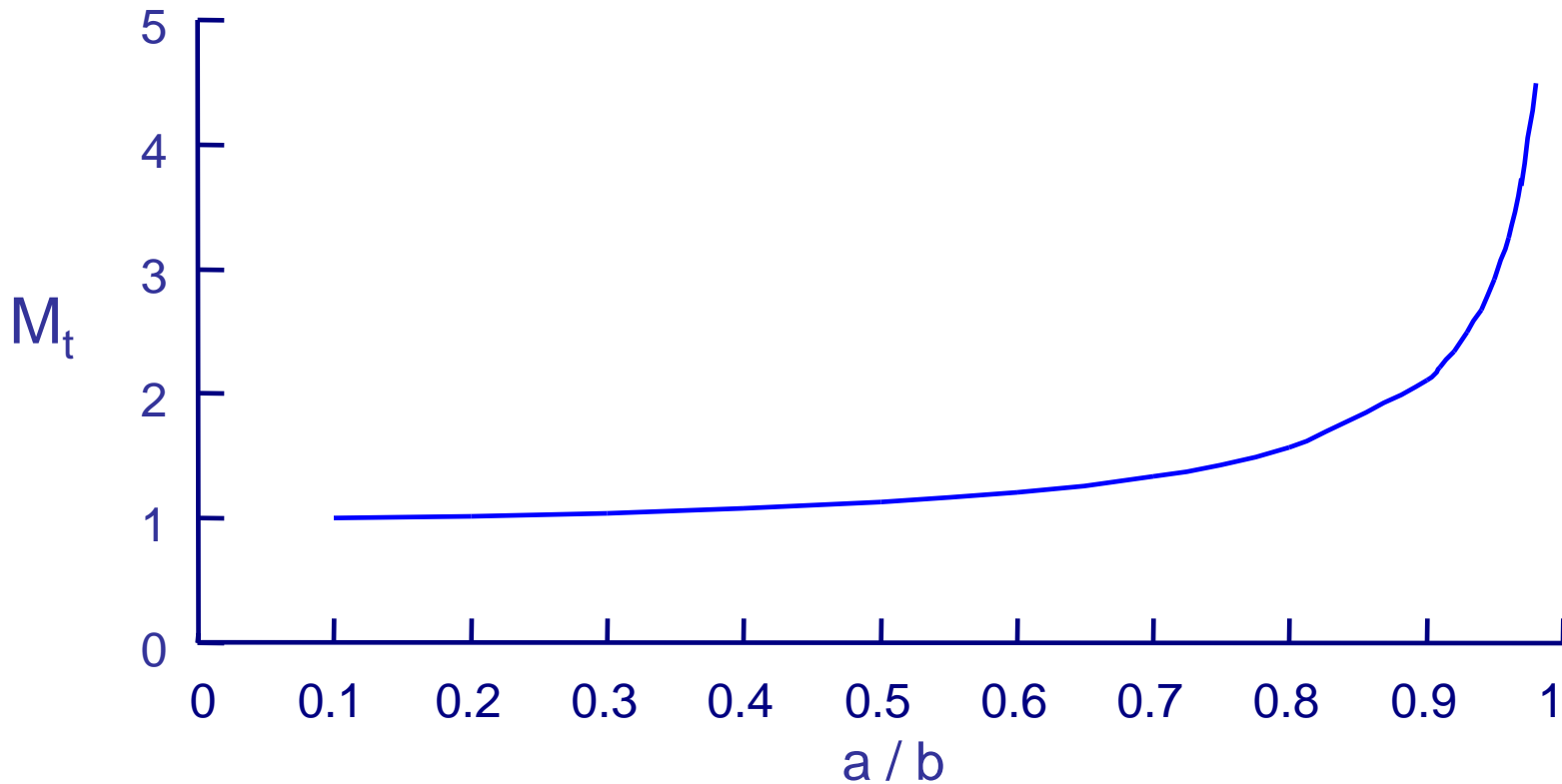
$$M_s = 1.12$$



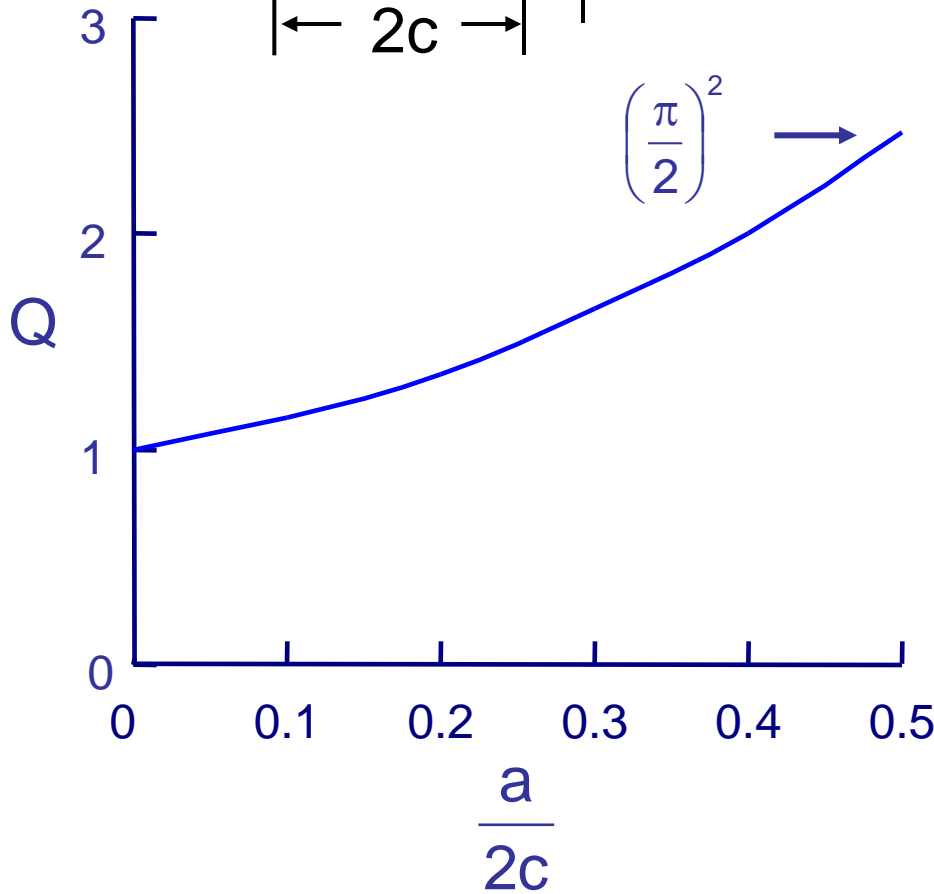
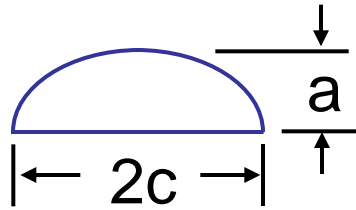
# $M_t$ back surface effects



$$M_t = \sqrt{\frac{2b}{\pi a} \tan \frac{\pi a}{2b}}$$



# $\Phi$ crack shape effects



$$\Phi = \sqrt{Q}$$

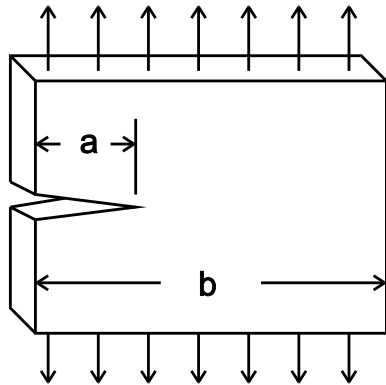
$$\sqrt{Q} = \int_0^{\pi/2} \sqrt{1 - k^2 \sin^2 \beta} \, d\beta$$

$$k^2 = 1 - \left(\frac{a}{c}\right)^2$$

$$Q \cong 1 + 1.464 \left(\frac{a}{c}\right)^{1.65} \quad a \leq c$$

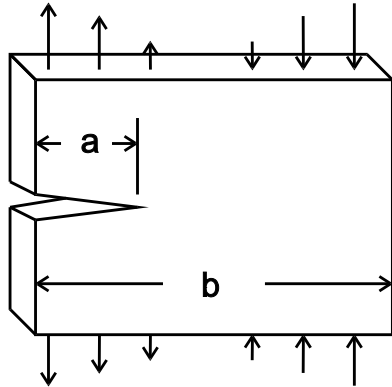


# Edge Cracked Plate in Tension



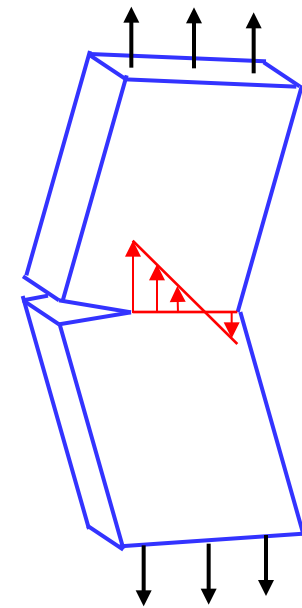
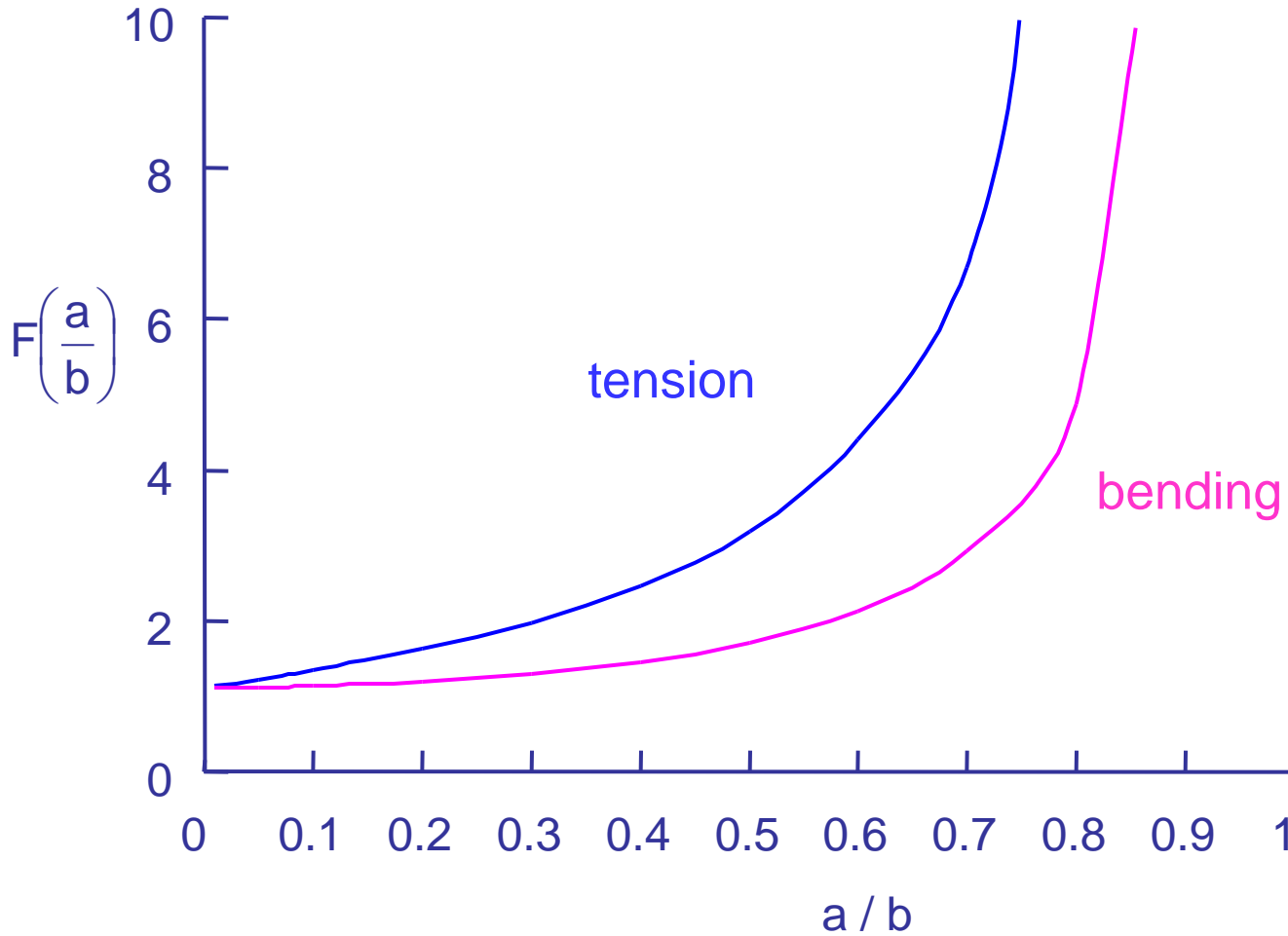
$$F\left(\frac{a}{b}\right) = \sqrt{\frac{2b}{\pi a} \tan \frac{\pi a}{2b}} \left( \frac{0.752 + 2.02 \frac{a}{b} + 0.37 \left(1 - \sin \frac{\pi a}{2b}\right)^3}{\cos \frac{\pi a}{2b}} \right)$$

# Edge Cracked Plate in Bending

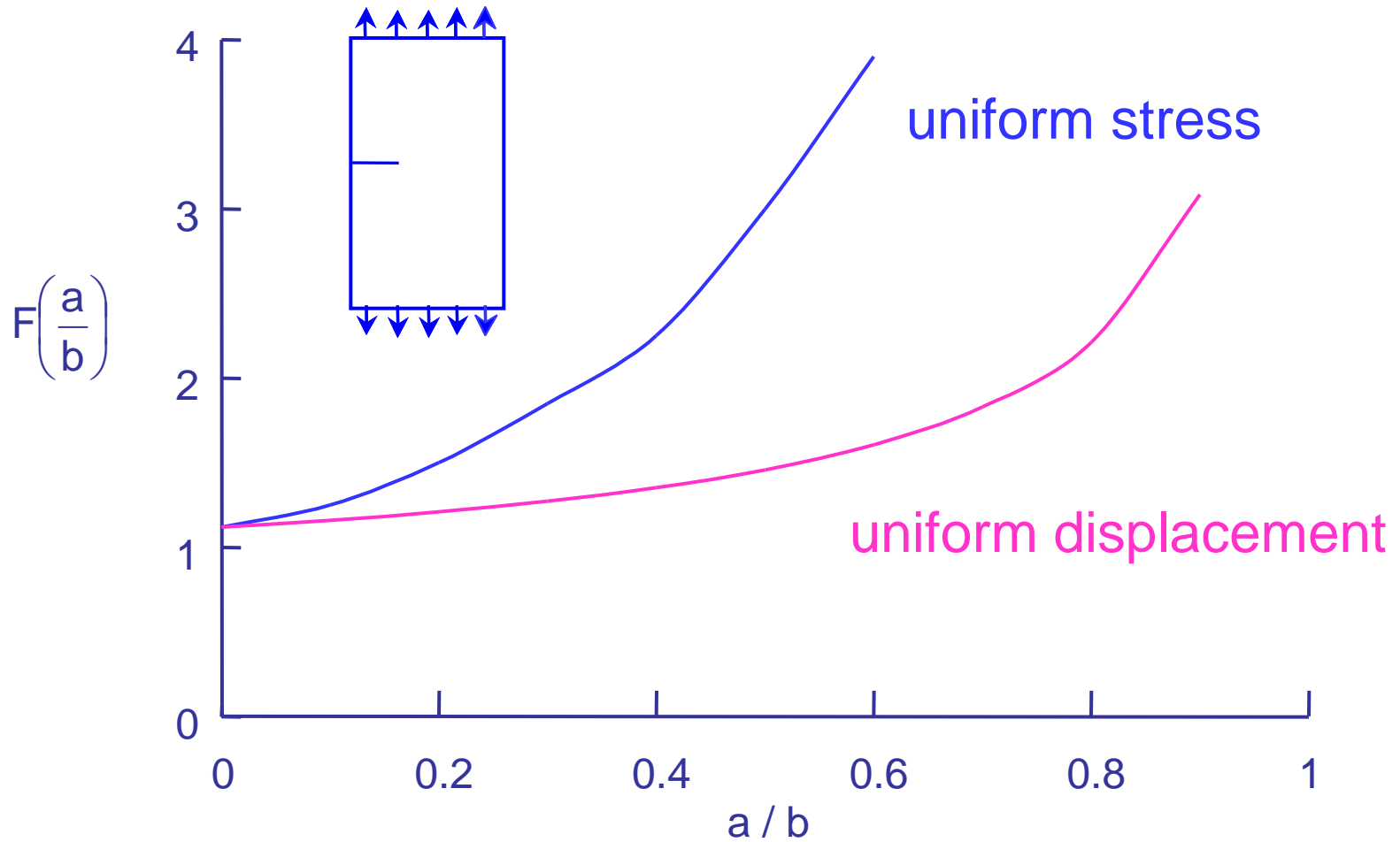


$$F\left(\frac{a}{b}\right) = \sqrt{\frac{2b}{\pi a} \tan \frac{\pi a}{2b}} \left( \frac{0.923 + 0.199 \left(1 - \sin \frac{\pi a}{2b}\right)^4}{\cos \frac{\pi a}{2b}} \right)$$

# Tension and Bending

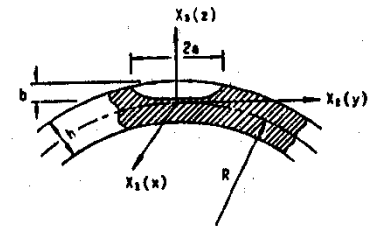
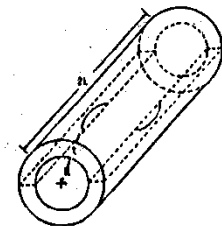
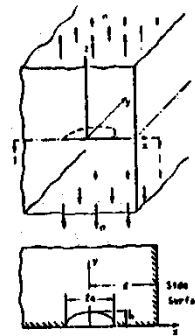


# Boundary Conditions



# Handbook

- 9.32 An embedded elliptical crack near free surface under tension 734
- 9.33 A semi-elliptical crack near corner under tension 742
- 9.34 A semi-elliptic surface crack emanating from the inside of an infinitely thick cylinder subjected to internal pressure 745
- 9.35 A semi-elliptical surface crack in internally pressurized cylinder (the crack faces are pressurized) 748
- 9.36 Internal and external surface cracks in cylindrical vessels 751
- 9.37 Cylindrical shell containing a circumferential or axial part-through crack 759
- 9.38 A pressurized cylindrical shell with a fixed end which contains an axial part-through or through crack 771
- 9.39 Corner crack in a rotating disk 786





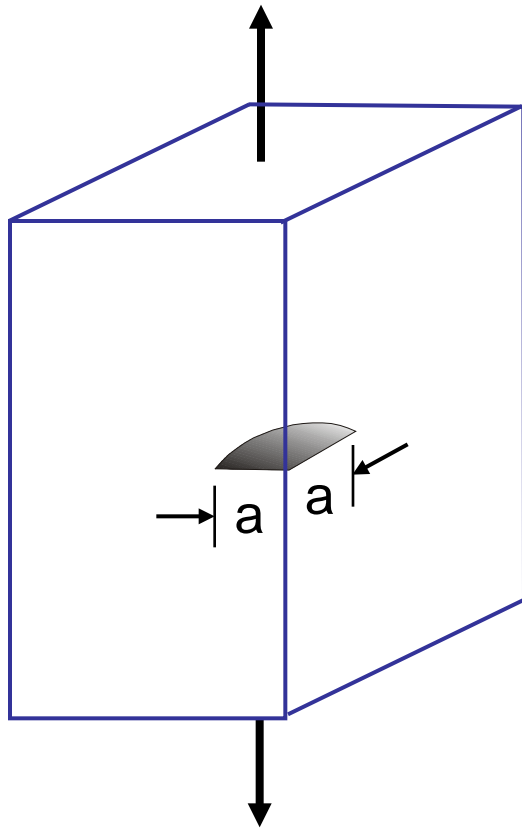
# Handbook

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Stress Intensity Factors Handbook

Y. Murakami Editor, Pergamon Press

# Useful approximations



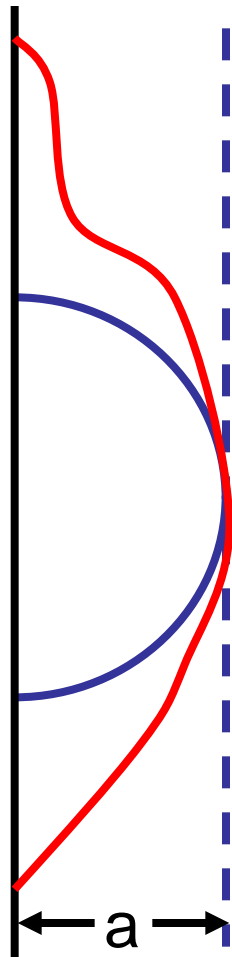
Corner crack

Two free edges

Semicircular shape

$$K = \sigma (1.12)^2 \frac{2}{\pi} \sqrt{\pi a}$$

# Crack Shape



Through crack

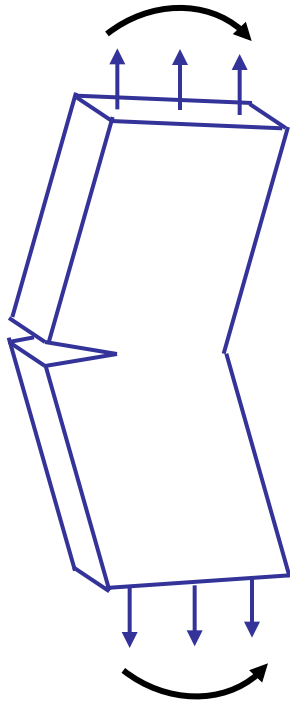
$$K = \sigma 1.12 \sqrt{\pi a}$$

Semielliptical crack

$$K = \sigma 1.12 \frac{2}{\pi} \sqrt{\pi a} = \sigma 0.71 \sqrt{\pi a}$$



# Superposition



tension + bending

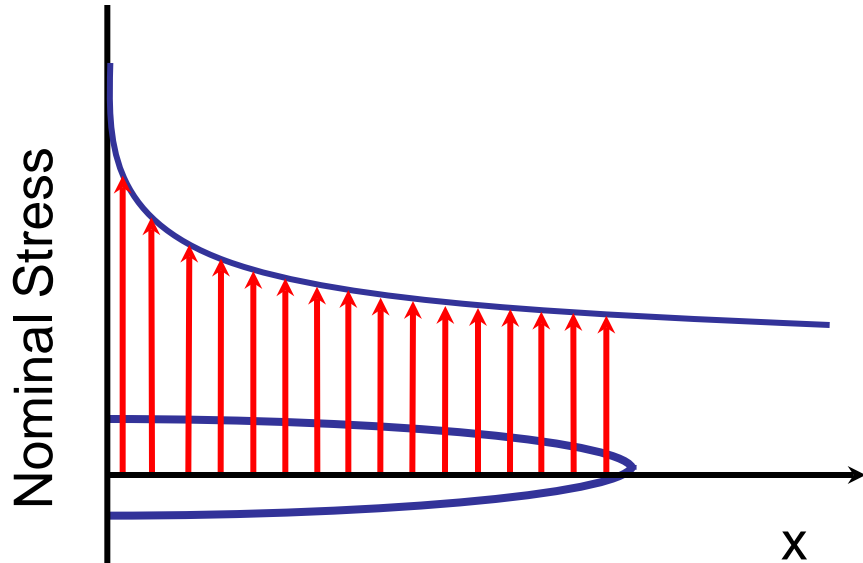
Crack tip stresses:

$$\sigma_{ij} = \frac{K_{\text{tension}}}{\sqrt{2\pi r}} f_{ij}(\theta) + \frac{K_{\text{bending}}}{\sqrt{2\pi r}} f_{ij}(\theta)$$

$$\sigma_{ij} = \frac{K_{\text{tension}} + K_{\text{bending}}}{\sqrt{2\pi r}} f_{ij}(\theta) = \frac{K_{\text{total}}}{\sqrt{2\pi r}} f_{ij}(\theta)$$

$$K_{\text{total}} = K_{\text{tension}} + K_{\text{bending}}$$

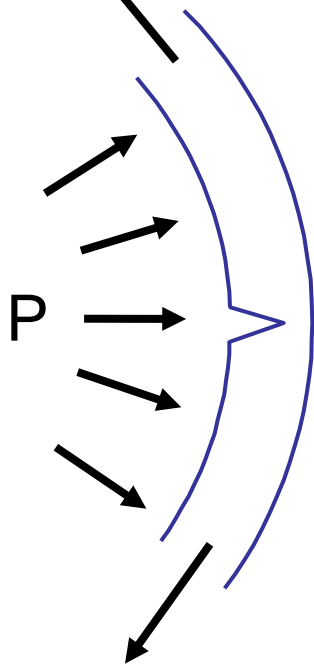
# Stress Gradients



$$K \cong 1.12 \left( \frac{1}{3} \sigma_{\text{average}} + \frac{2}{3} \sigma_{\text{crack tip}} \right) \sqrt{\pi a}$$

# Pressure Vessel

$$\sigma = \frac{Pr}{t}$$

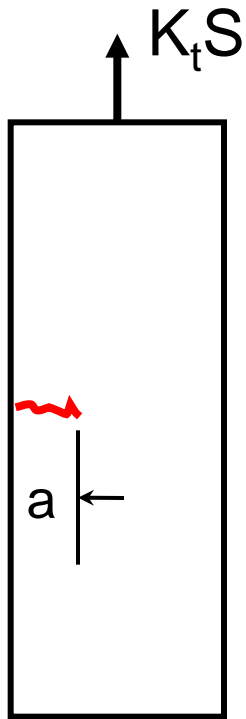
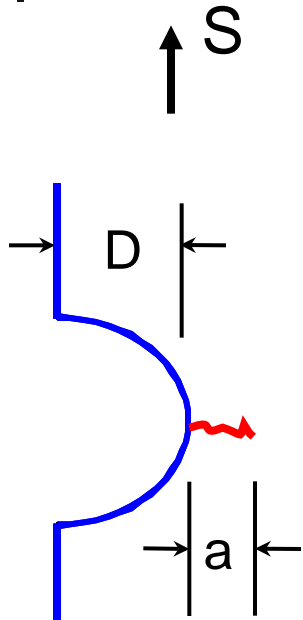


$$\sigma = \frac{Pr}{t}$$

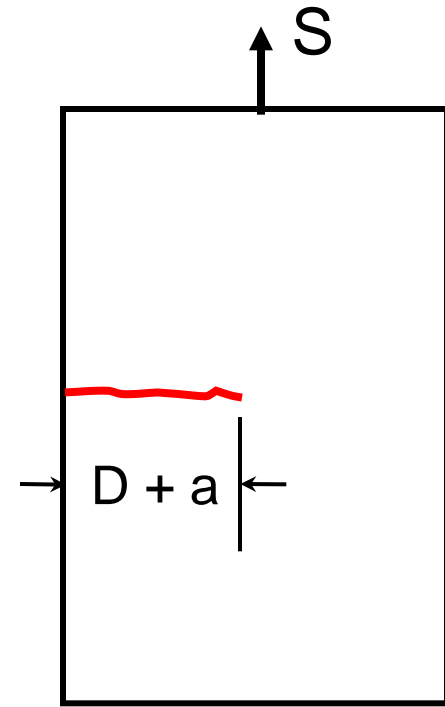
$$K = \frac{Pr}{t} \sqrt{\pi a} + P \sqrt{\pi a}$$

$$K = P \left( \frac{r}{t} + 1 \right) \sqrt{\pi a}$$

# Cracks at Notches

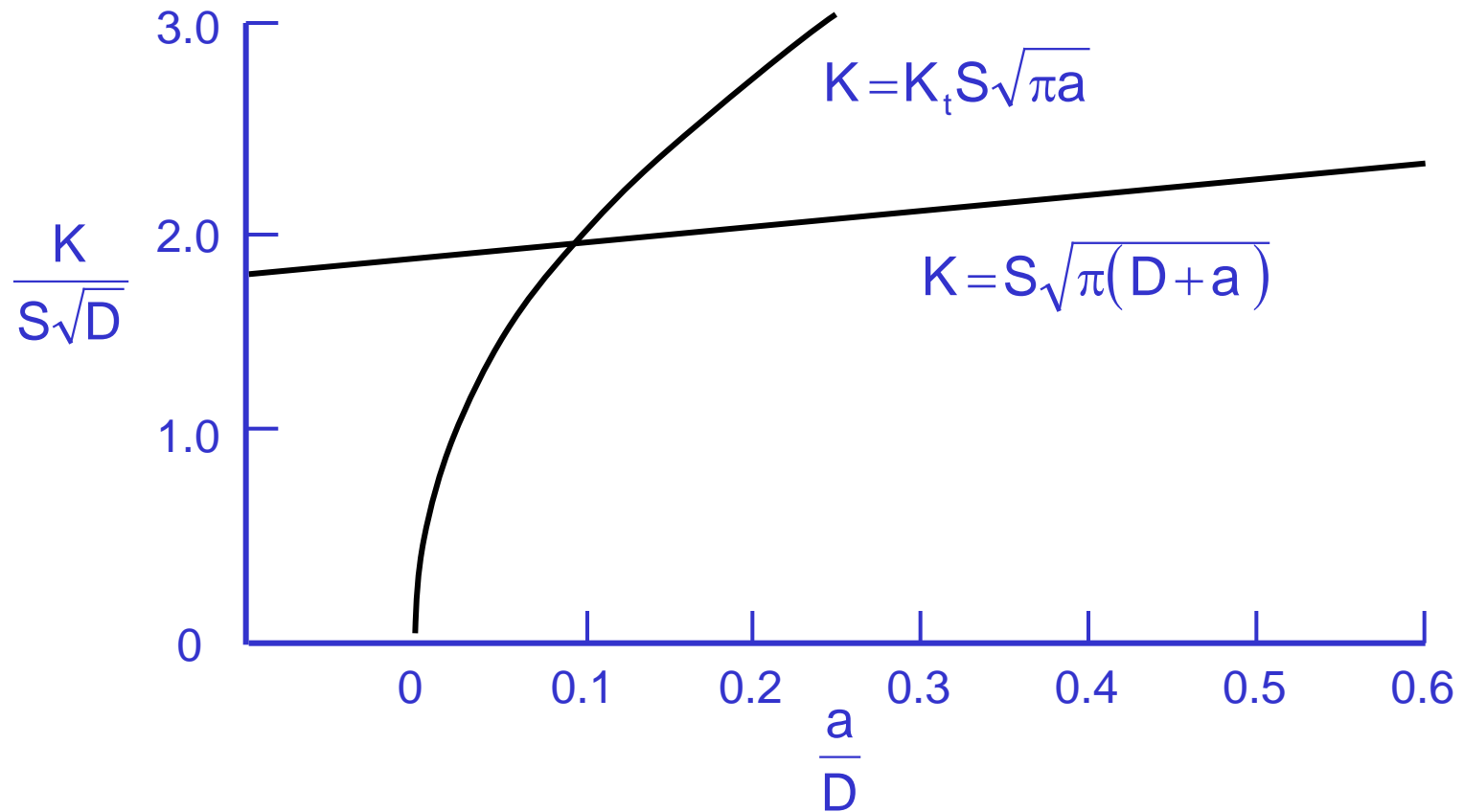


$a \ll D$

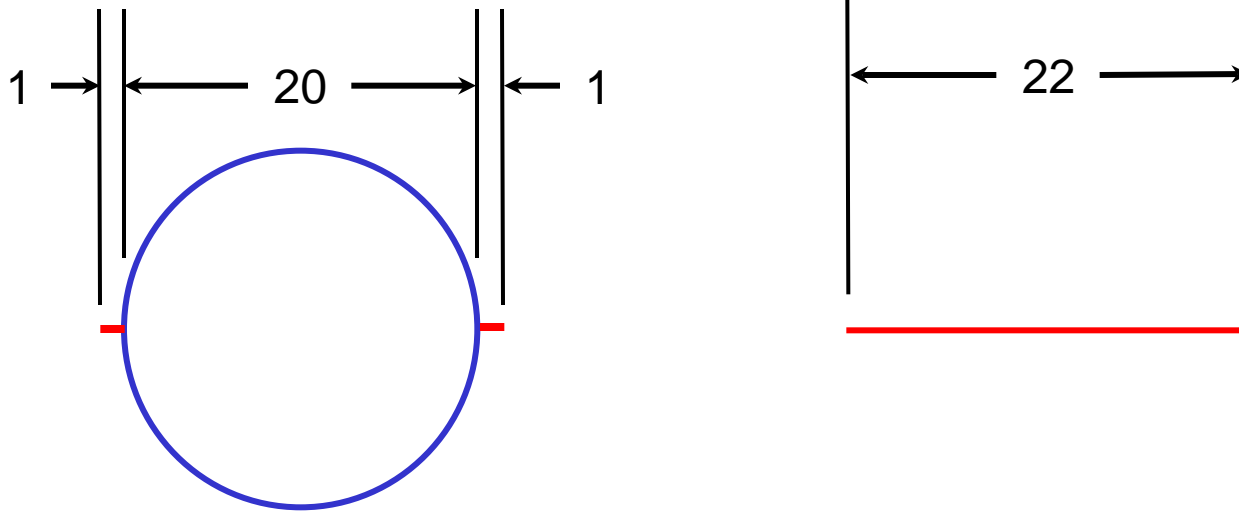


$a \gg D$

# Stress Intensity Factors



# Cracks at Holes



Once a crack reaches 10% of the hole radius, it behaves as if the hole was part of the crack



# Summary

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$$K_{Ic} > \sigma \sqrt{\pi a} f\left(\frac{a}{W}\right)$$

$$\sigma \sim \sigma_{ys} /$$

$$a \sim 0.1 \text{ mm} - 100 \text{ mm}$$

$$f\left(\frac{a}{W}\right) \sim 1 - 2$$



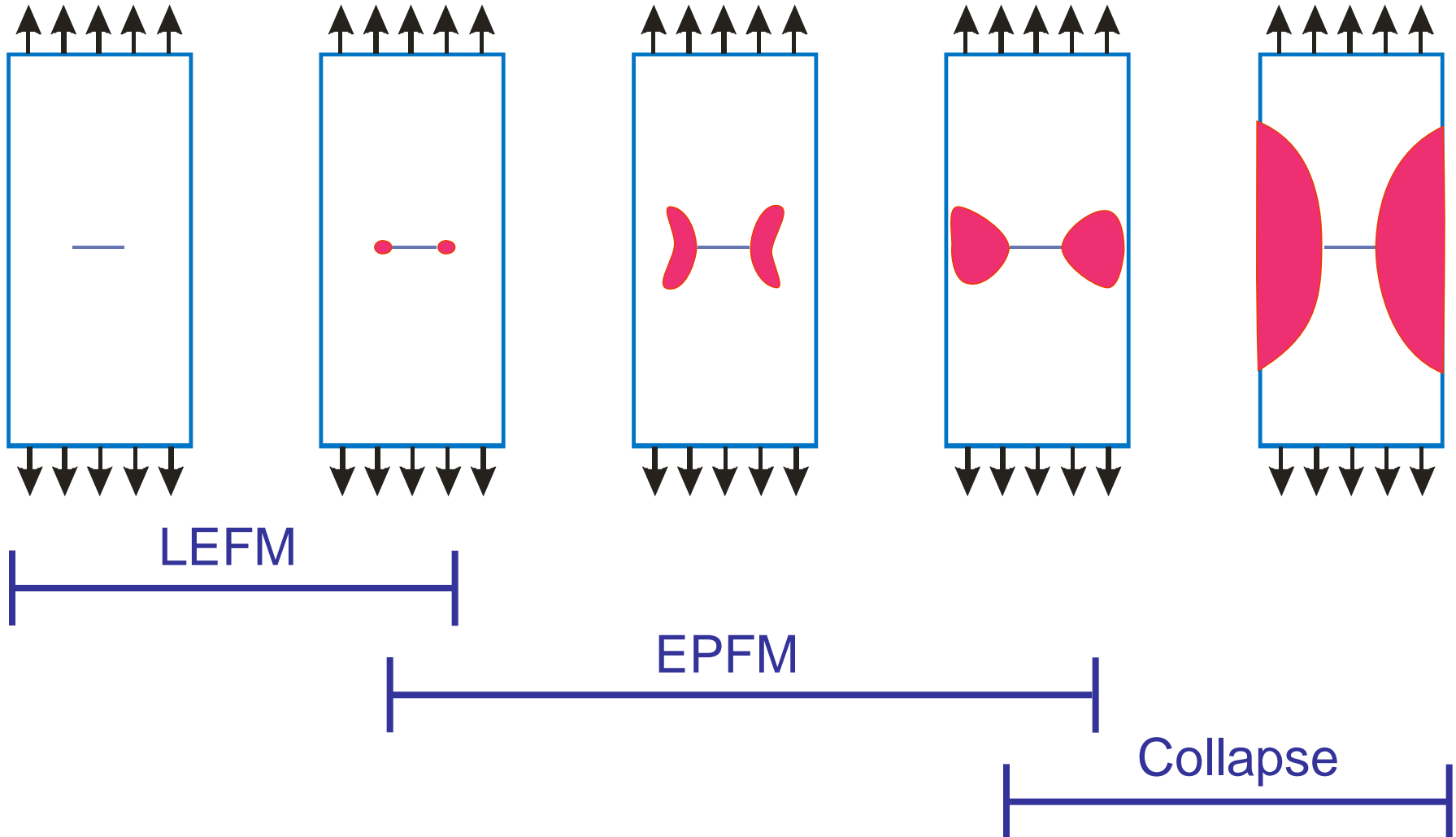
# Static Strength and Fracture

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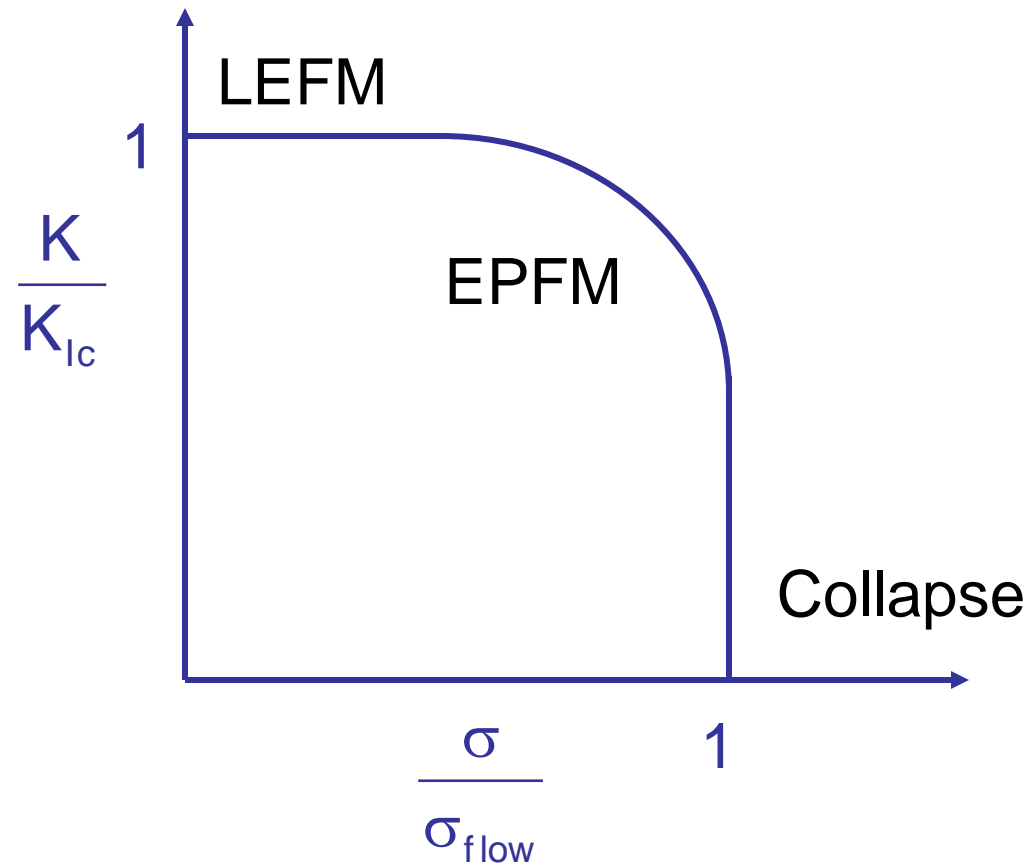
- Stress Concentration Factors
- Fracture Mechanics
- Approximate Stress Intensity Factors
- **Ductile vs. Brittle Fracture**



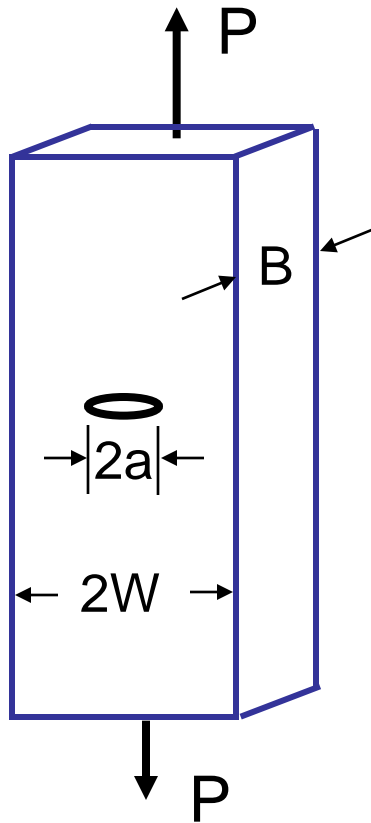
# Fracture Behavior



# Failure Analysis Diagram



# Plastic Collapse

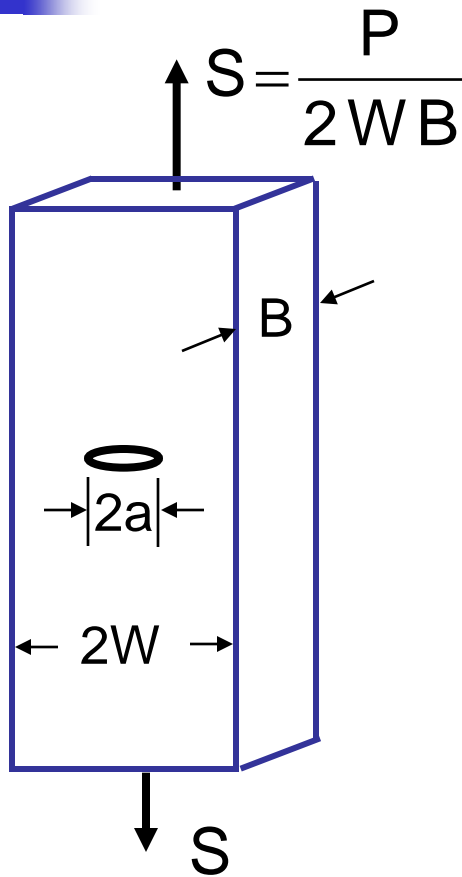


$$\frac{P}{2(W-a)B} < \sigma_{\text{flow}}$$

$$S = \frac{P}{2WB} \quad \text{Nominal stress}$$

$$S = \sigma_{\text{flow}} \left( 1 - \frac{a}{W} \right)$$

# Fracture



$$S \sqrt{\pi a} f\left(\frac{a}{W}\right) = K_{Ic}$$

$$S = \frac{K_{Ic}}{\sqrt{\pi \frac{a}{W}} \sqrt{W} f\left(\frac{a}{W}\right)}$$



# Fracture vs. Collapse

---

Fracture and collapse equally likely

$$\sigma_{\text{flow}} \left( 1 - \frac{a}{W} \right) = \frac{K_{\text{Ic}}}{\sqrt{\pi \frac{a}{W}} \sqrt{W} f\left(\frac{a}{W}\right)}$$

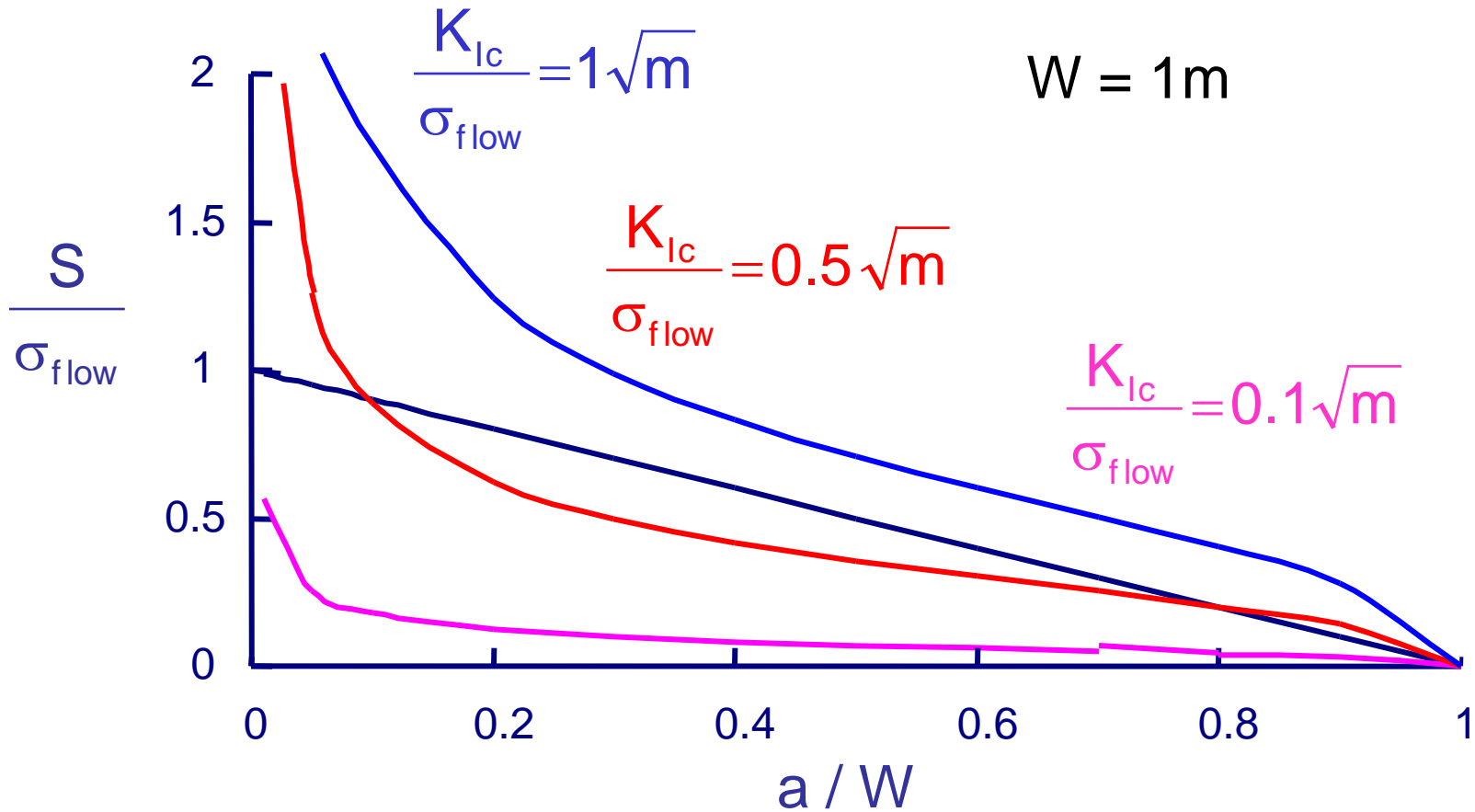
$$\frac{K_{\text{Ic}}}{\sigma_{\text{flow}}} = \left( 1 - \frac{a}{W} \right) \sqrt{\pi \frac{a}{W}} \sqrt{W} f\left(\frac{a}{W}\right)$$



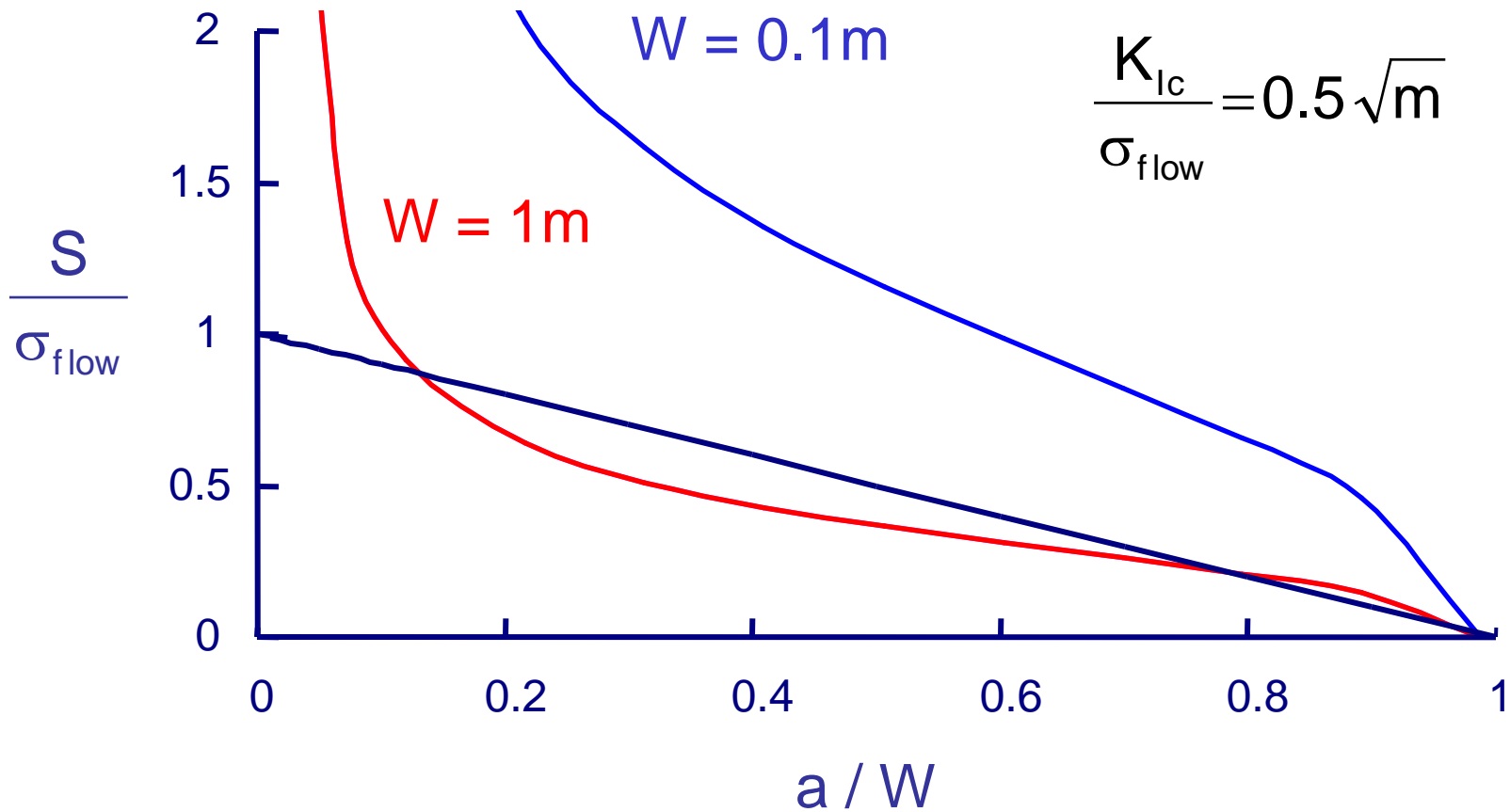
# Material Properties

	$\sigma_{ys}$	$K_{Ic}$	$K_{Ic}/\sigma_{ys}$
1020	250	200	0.800
2024-T3	345	44	0.128
7075-T6	495	25	0.051
Ti-6Al-4V	910	105	0.115
Ti-6Al-4V	1035	55	0.053
4340	860	99	0.115
4340	1510	60	0.040
17-7 PH	1435	77	0.054
52100	2070	14	0.007

# Fracture Diagram



# Fracture Diagram





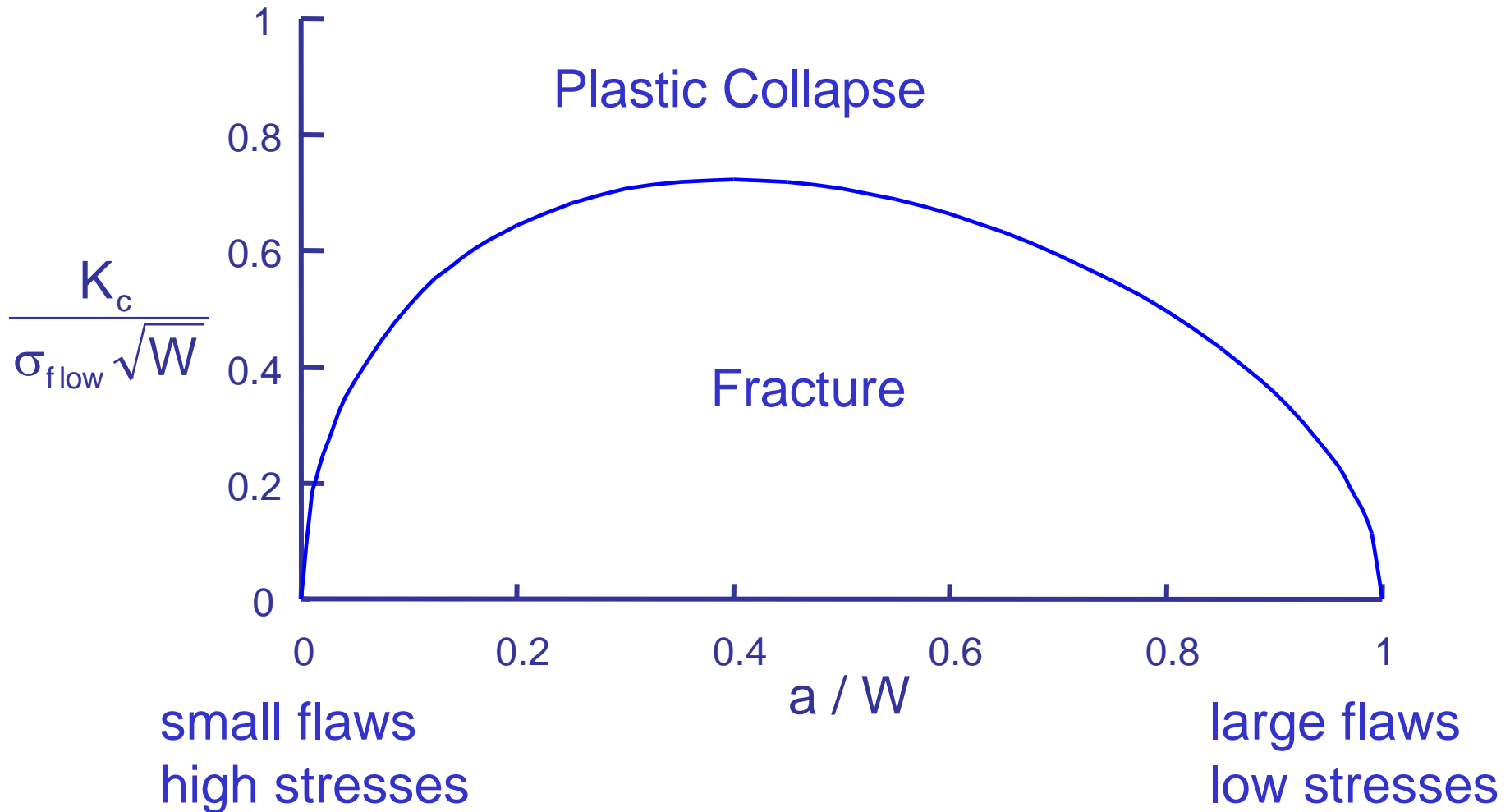


# Fracture vs. Collapse

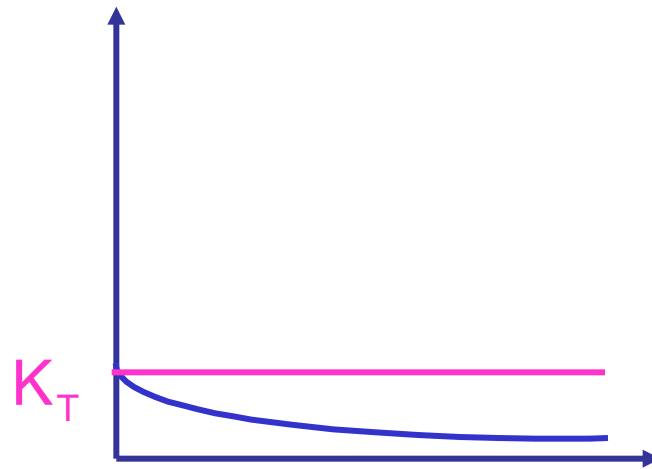
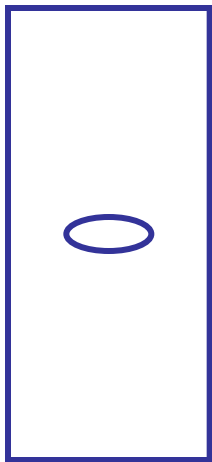
---

$$\frac{K_{Ic}}{\sigma_{flow} \sqrt{W}} = \left(1 - \frac{a}{W}\right) \sqrt{\pi \frac{a}{W}} \sqrt{\frac{2W}{\pi a} \tan \frac{\pi a}{2W}}$$

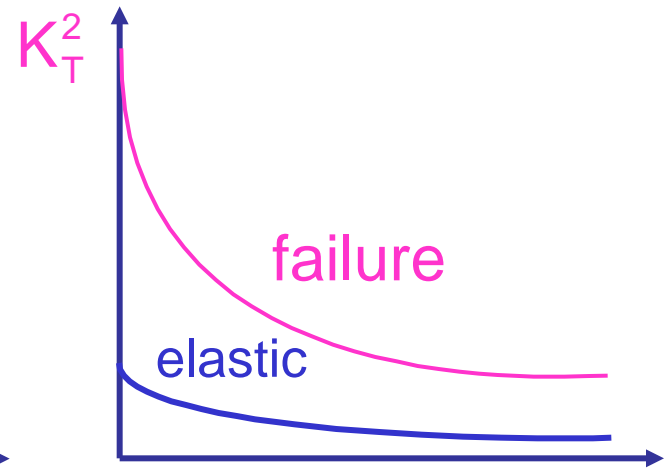
# Fracture Diagram



# Typical Stress Concentration

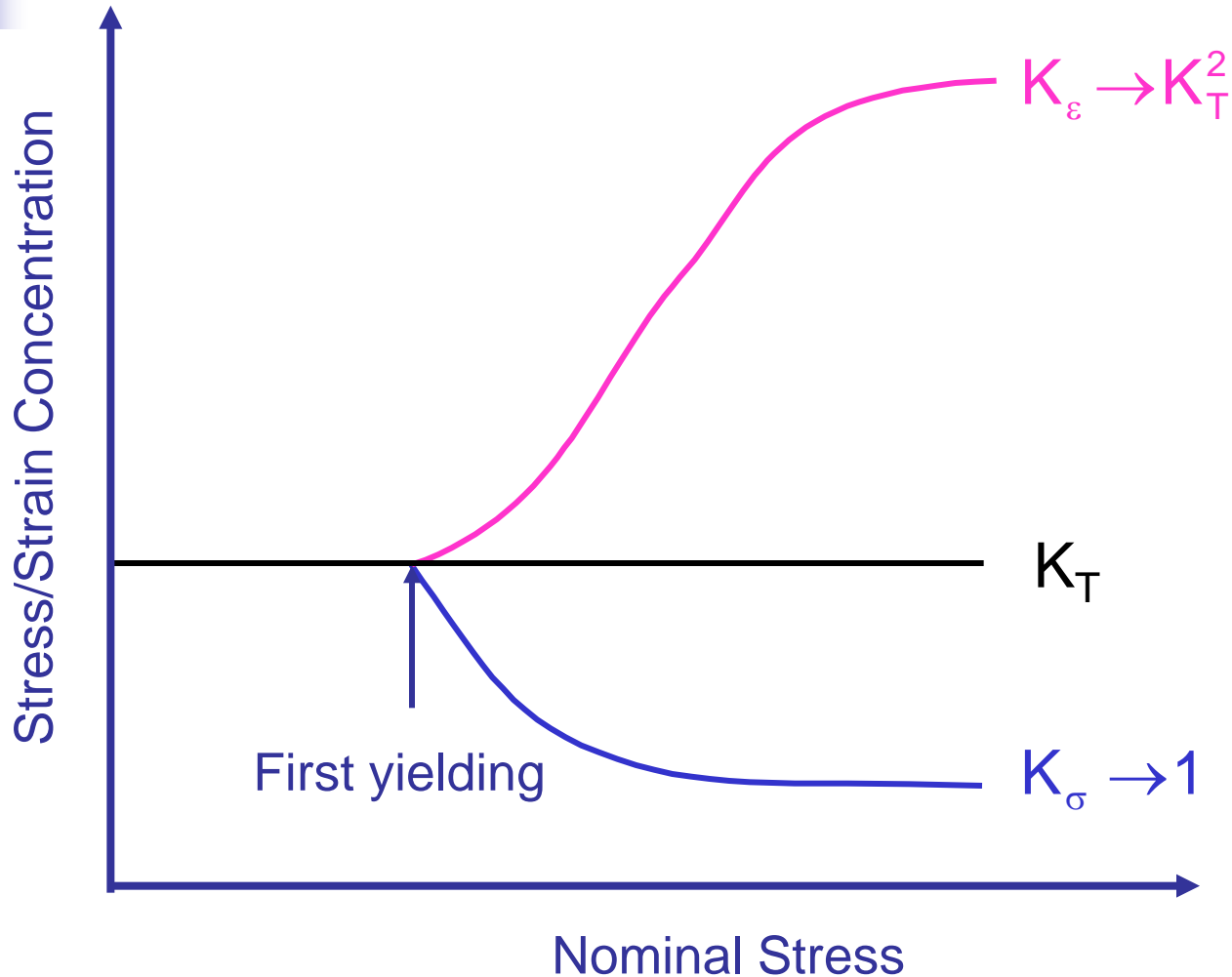


Stress distribution



Strain distribution

# Stress and Strain Concentration



# Failure Diagram – Ductile Material

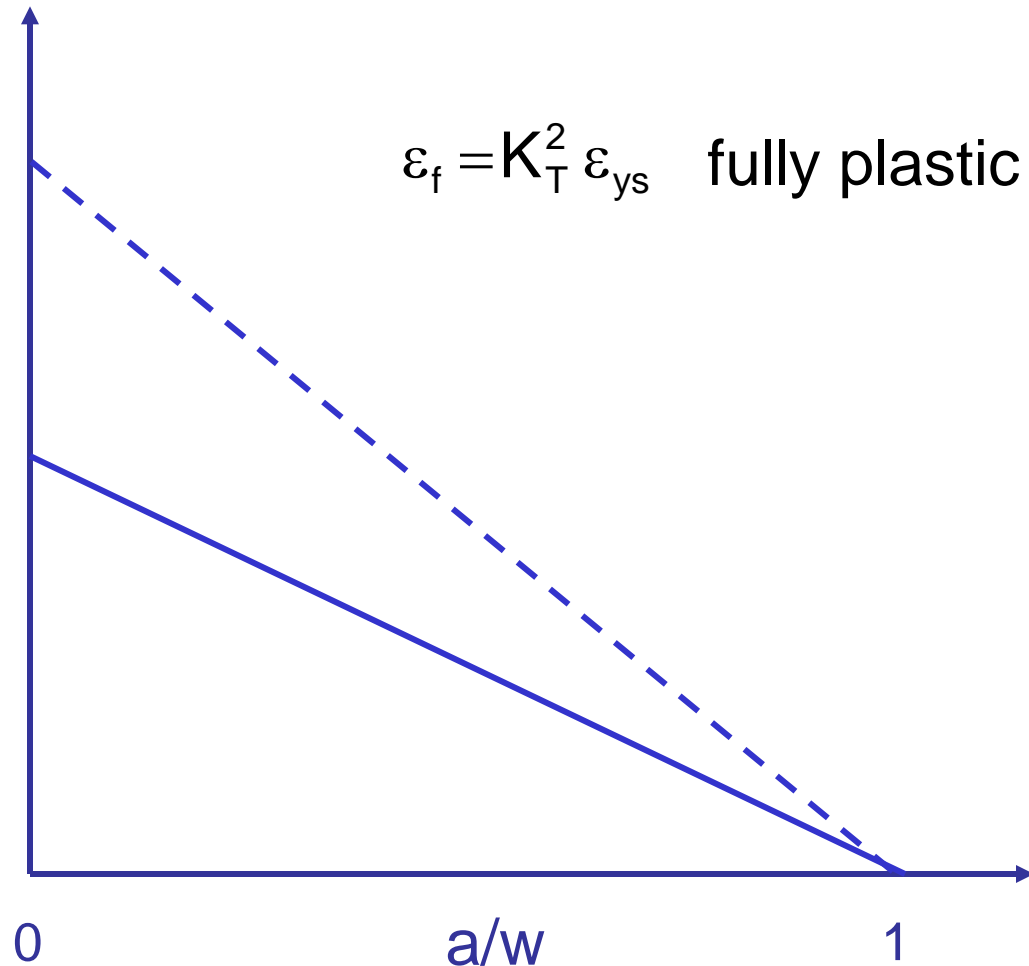
$\epsilon_f$  is large

$$P_{\max} = \frac{A_{\text{net}} \epsilon_f E}{K_T^2}$$

strength limited

$$P_{\max} = \sigma_{\text{flow}} A_{\text{net}}$$

$$\epsilon_f = K_T^2 \epsilon_{ys} \quad \text{fully plastic}$$



Strength limit is reached before cracking at the notch

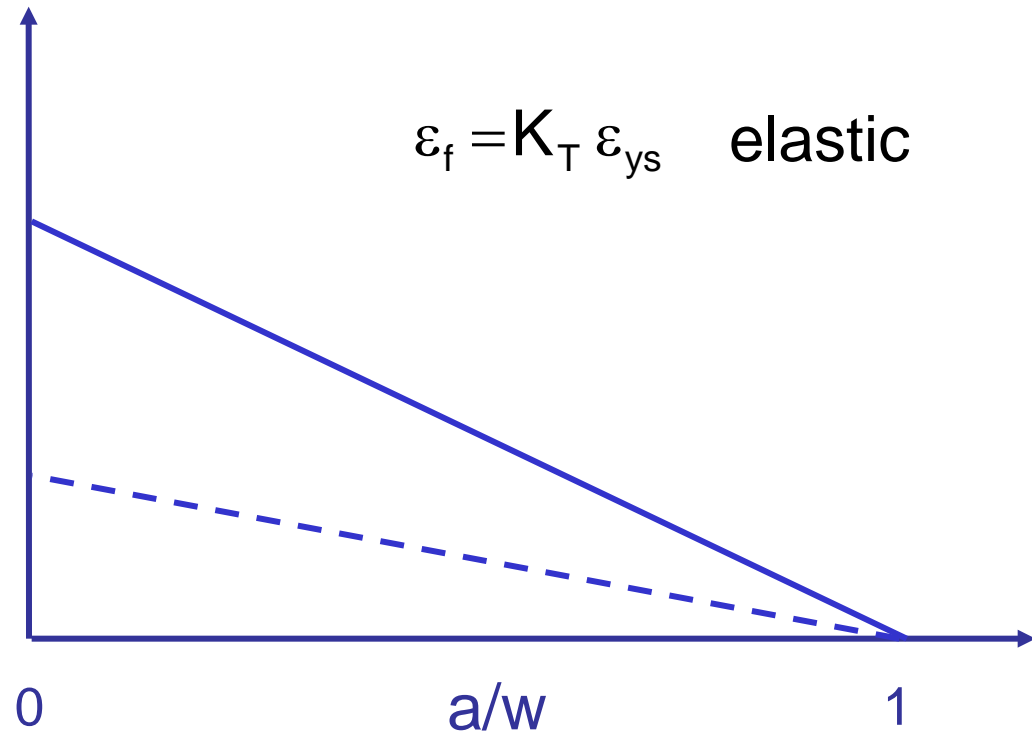
# Failure Diagram – Brittle Material

$$P_{\max} = \sigma_{\text{flow}} A_{\text{net}}$$

$\varepsilon_f$  is small

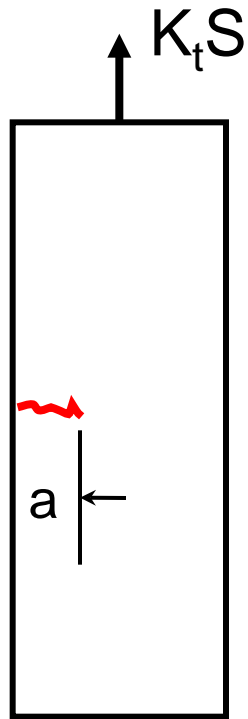
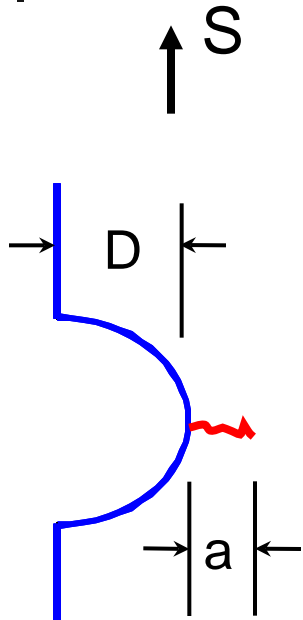
$$P_{\max} = \frac{A_{\text{net}} \varepsilon_f E}{K_T}$$

ductility limited

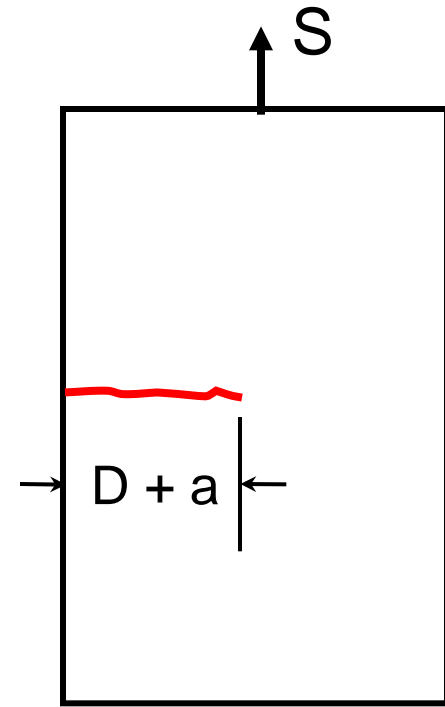


Cracks form at the notch before the limit load is reached

# Cracks at Notches

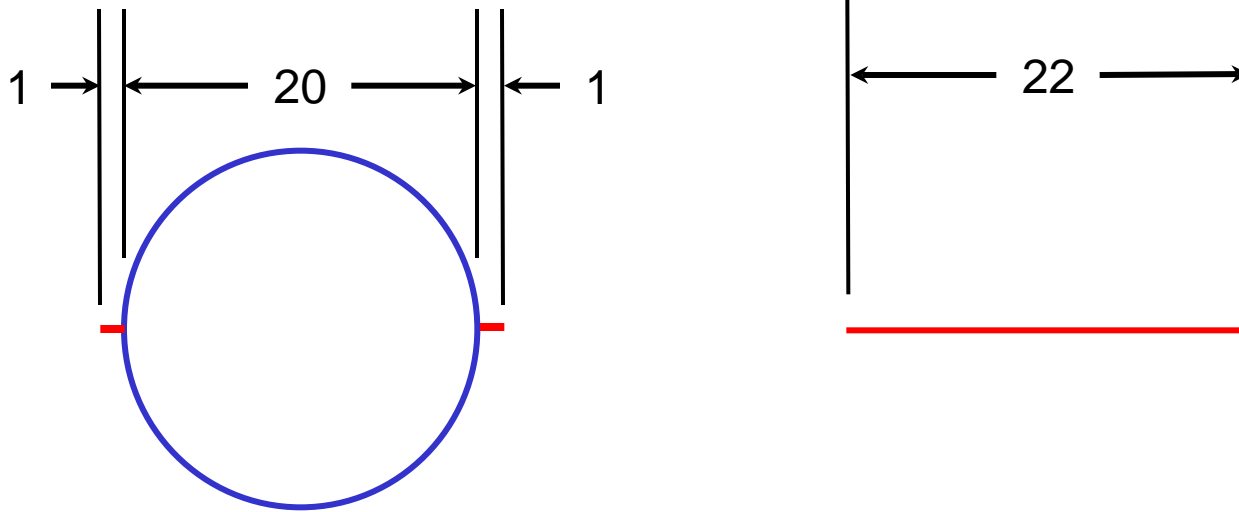


$a \ll D$



$a \gg D$

# Cracks at Holes



Once a crack reaches 10% of the hole radius, it behaves as if the hole was part of the crack





# Notch Fracture

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What ratio of strength to toughness is needed to avoid fracture?

$$K_c = \sigma_{ys} 1.12 K_T \sqrt{\pi 0.1r}$$

For  $K_T = 3$  and  $r = 10$  mm

$$\frac{K_c}{\sigma_{ys}} > 0.18$$

to avoid fracture from the notch



# Material Properties

	$\sigma_{ys}$	$K_{Ic}$	$K_{Ic}/\sigma_{ys}$
1020	250	200	0.800
2024-T3	345	44	0.128
7075-T6	495	25	0.051
Ti-6Al-4V	910	105	0.115
Ti-6Al-4V	1035	55	0.053
4340	860	99	0.115
4340	1510	60	0.040
17-7 PH	1435	77	0.054
52100	2070	14	0.007



# Summary

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- Fracture is a likely failure mode for all higher strength materials
- Fracture is even more likely at stress concentrators

# Fatigue and Fracture

